# ENVIRONMENTAL PROTECTION AGENCY OFFICE OF ENFORCEMENT

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Evaluation Of Particulate Control System

Cherokee Station

Public Service Company Of Colorado

Denver, Colorado

[July 27 - August 31, 1977]

NATIONAL ENFORCEMENT INVESTIGATIONS CENTER
AND

REGION VIII DENVER, COLORADO

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# ENVIRONMENTAL PROTECTION AGENCY Office of Enforcement EPA 330/2-77-023

CHEROKEE STATION
PUBLIC SERVICE COMPANY OF COLORADO
DENVER, COLORADO
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#### I. INTRODUCTION

The Cherokee Station powerplant in suburban North Denver is owned and operated by Public Service Company of Colorado (PSCC). It has four base-loaded coal- and/or gas-fired steam generators with a total electrical generating capacity (gross) of 775 megawatts (MW).

# **BACKGROUND**

The Colorado Air Pollution Control Regulation No. 1.A.1 requires that no person shall emit an air contaminant in excess of 20% opacity. It also allows for a condition called 'upset' which is an unpredictable equipment failure or other malfunction which results in the violation of an emission control regulation, and which is not due to improper or careless operation. An 'upset' is not considered a violation of the Regulation if it is immediately reported to the Air Pollution Control Division of the State of Colorado Department of Health.

Although equipped with extensive air pollution control equipment such as mechanical collectors, electrostatic precipitators (ESP's), and scrubbers, the Cherokee plant has a long history of opacity excursions.\*

The Company attributes these excursions to control equipment operation and maintenance problems and states that every reasonable effort is made on a continuing basis to correct and prevent such problems. However, the opacity excursions have continued.

In June 1977, the Environmental Protection Agency's (EPA's) Region VIII in Denver requested that the National Enforcement Investigations

<sup>\*</sup> This report uses the term 'excursion' to describe an emission in excess of 20% opacity.

Center (NEIC) evaluate the design, operation and maintenance of the Company's air pollution control equipment. Region VIII also notified the Public Service Company of Colorado that NEIC would conduct visible emission observations (VEO's) of the Cherokee Station powerplant (off the property) for 1 1/2 months. In a conference between Company officials and attorneys and EPA representatives, NEIC personnel explained that a process evaluation would be conducted concurrent with the VEO's; the Company thus granted EPA permission to enter the plant.\*

# STUDY OBJECTIVES

The objectives of the NEIC evaluation were as follows:

First, determine the cause of reported emissions of >20% equivalent opacity by evaluating the design, operation, and maintenance of the air pollution control equipment.

Second, determine the accuracy of the in-stack opacity monitors by evaluating the meter capabilities, their location, and the Company procedures in calibrating and maintaining the meters.

Third, conduct visible emission observations (VEO) in conjunction with the above objectives.

Fourth, suggest ways to improve reliability of the particulate control system.

<sup>\*</sup> The Company allowed plant entry at any time under the following conditions: during normal working hours, those entering were to sign the plant's visitor book, contact the senior results engineer or his assistant; on off-duty hours, they were to sign in and then report to the shift supervisor.

The evaluation involved an intensive in-plant observation period in July-August 1977 to collect data on boiler operation, air pollution control equipment, opacity meters, and visible emissions. Additional VEO's were taken in October 1977. (Evaluation data are in Appendix A, and the visible emission data are summarized in Appendix B.) NEIC personnel also observed routine adjustment of the control equipment by the Company and inspected the ESP's and scrubbers when they were removed from service. To further evaluate scrubber operation, design and operating data were compared with literature references.

This report uses the terms availability and reliability to describe operation of the particulate control equipment. Availability, as defined by PSCC is:

Availability = Scrubber hours operation - hours boiler burning 100% gas
Boiler hours operation - hours boiler burning 100% gas

Reliability, as used in the report is defined as the percent of time the boiler is on-line that the particulate control systems are operating and meeting applicable particulate regulations.

#### II. SUMMARY AND CONCLUSIONS

The National Enforcement Investigations Center (NEIC) evaluated in detail the design, operation and maintenance of the Cherokee Station air pollution control equipment during July and August 1977. The evaluation included equipment inspections and in-plant observations of the electrostatic precipitators, scrubbers, and opacity meters, as well as visible emission observations.\* NEIC also suggested ways to improve the reliability of Cherokee's particulate control equipment.

# ELECTROSTATIC PRECIPITATOR EVALUATION

Evaluation of the ESP's was hampered because NEIC was not able to determine if, or to what extent, the gas conditioning agent was being added to the boiler offgases. This made it difficult to interpret the electrical operating parameters, such as the cause of low power input. However, information was obtained on the design, operation and maintenance of the ESP's, along with results of tests run by PSCC between 1965 and 1976.

The ESP's were designed for collection efficiencies of 87 to 94% and the design parameters are comparable to other fly ash ESP's designed in the late 1960's for collection efficiencies of 80 to 90%. A review of the design and operating data indicates these precipitators are undersized if more efficient collection (>90%) of the fly ash is required.

Early test results on the ESP efficiencies indicated that these units are operating at lower than design efficiencies (shown in parentheses), with typical values reported by PSCC as:

<sup>\*</sup> Additional VEO's were made in October 1977, and are included in the VEO summaries contained in Section VI and Appendix B of this report.

Unit 1 - 50% (90%) Unit 3 - 51% (87%) Unit 2 - 89% (94%) Unit 4 - 42% (87%)

These values are five years old, and data on present ESP efficiencies were not available for this study.

Since the outlet loadings from recent tests conducted by the Company are in the same range as the previous efficiency tests, an increase in ESP efficiency is unlikely. Thus, the efficiencies reported above are considered representative of the ESP operations. Because the coal quality and the effectiveness of the gas conditioning system would affect these efficiencies, additional tests would be needed to update these ESP efficiencies.

The electrical operating data indicate that the ESP's are operating at lower power inputs than typical fly ash precipitators, and of the four ESP's at Cherokee, the Unit 1 ESP had the lowest power input. Possibly the Sulfan system on Unit 1 was not operating properly. This was indicated by the Unit 1 ESP controls which appeared to be spark rate limited, causing a reduction in power input. Higher power inputs typically would increase the ESP collection efficiencies.

Although the precipitators are tuned every day during the normal work week, the effectiveness of this procedure could not be determined for two reasons. First, Company representatives indicated that no data is recorded by the electricians during tuning; second, specific values of the normal and abnormal conditions were not identified. In addition, the meter readings on Unit 3 ESP are suspect, since the power efficiencies of the transformer-rectifier sets were greater than "1" in most cases.

The maintenance of the ESP's was adequate. Most of the maintenance on both ESP's and mechanical collectors is done as needed. The maintenance is conducted during Unit outages unless the opacity meters read

>20% as a result of the problem, then a priority is placed on the maintenance for immediate repair. When a wire in an ESP is grounded, a load reduction is scheduled, usually within a week, and the wire is removed.

The internal inspections on the Unit 3 and 4 ESP's did not reveal any severe mechanical problems such as misaligned electrodes, warped plates, corroded internals, etc. The Company personnel conducted the inspection in a knowledgeable and thorough manner.

# SCRUBBER EVALUATION

Wet scrubbers were installed on Units 1, 3 and 4 to supplement particulate control of the existing mechanical collectors and electrostatic precipitators. The scrubbers are Turbulent Contact Absorbers (TCA) designed by Universal Oil Products, Air Correction Division. PSCC incorporated significant improvements in the Unit 4 scrubbers after installing Unit 1 and 3 scrubbers. These included indirect reheat, individual scrubber booster fans for each scrubber section, three recirculating slurry pumps per scrubber section, and weather enclosures. Unit 4 is also operated so that at least one scrubber section is always available as a spare. Units 1 and 3 have scrubber sections with only one recirculating slurry pump, and operate without spare scrubber sections. The effect of these improvements has been that Unit 4 normally meets the opacity regulation and with better reliability than Units 1 and 3.

PSCC is required by Colorado regulations to report air pollution control equipment upsets. Although the availability of previous upset data was limited, some generalizations on scrubber upsets can be made. Scrubber availabilities, i.e. scrubber operating time as a percentage of boiler operating time, from startup date to May 1977 were: Unit 1 (65%), Unit 3 (63%) and Unit 4 (84%). The major areas of malfunction for the Unit 1 and 3 scrubbers, in approximate order of importance are

the in-line reheaters, scrubber internals, recirculating slurry system, and recirculating slurry pumps. Most of the Unit 4 scrubber upsets were caused by the scrubber booster fans. The differences in scrubber availabilities and major problem areas between units are mainly due to the differences in the scrubber system design.

To further document existing scrubber conditions a thorough internal inspection was made of the Unit 3 scrubber during a scrubber outage in August 1977. Numerous problems were observed in the inspection, including: large accumulations of solids in the presaturator, poor mobile ball distribution, numerous plugged recirculating slurry nozzles, and extensive corrosion in the outlet ductwork. Many of these deficiencies would result in reduced particulate removal and create the possibility of exceeding particulate regulations. In addition, it was observed that scrubber instrumentation data taken prior to the inspection may not adequately indicate problems such as solids deposits, ball migration, or pluggage of recirculating slurry nozzles. Therefore, it appears that the scrubbers could be operated at reduced particulate removal before the need for scrubber internal repairs was recognized.

An evaluation of the performance of the scrubbers was made from operating data accumulated in previous tests and during the in-plant inspections. Based on visible emission observations, stack tests, and particulate removal efficiency results, the scrubbers on all units are capable of meeting particulate regulations, however, they do not meet these standards on a continuous basis. This is due in part to the fact that scrubbers are frequently operated at higher-than-design gas velocities, at low pressure drops indicative of gas channeling, and with low liquid flow rates.

Effects of operation outside of design conditions are difficult to quantify but can be discussed in qualitative terms. At high gas velocities, increased liquid entrainment and overall particulate emissions

can be expected. When gas channeling is occurring, the scrubbers' particulate removal efficiency is reduced, which will result in increased particulate emissions. Low liquid flows are especially significant when the only recirculating slurry pump in a single-pump scrubber section is out of service. Continued operation without recirculating slurry drastically reduces particulate removal and can damage scrubber internals. It is expected that the opacity standard is more stringent than the process weight standard and that maintaining consistent compliance with the opacity standard will require operating closer-to-design requirements.

With minor exceptions, the thoroughness of scrubber maintenance appeared acceptable. Company personnel conducted the scrubber inspections in a knowledgeable and thorough manner, and the required repairs appeared to be adequately carried out. However, the frequency of maintenance (for Unit 1 and 3 scrubbers) depends to a major degree on operation of the boilers. Since scrubber instrumentation may not reliably indicate internal scrubber problems, limited maintenance frequency could result in scrubbers operating under conditions where particulate removal is low and standards are not being met. Furthermore, it is expected that the frequency of maintenance required to keep the scrubbers in good operating condition will vary with the type of coal being burned, the operation of the boiler, and the effectiveness of the gas conditioning system. It would be difficult to define an optimum maintenance schedule.

Based on the upset reports and equipment inspections, the following areas account for major operating problems in the scrubbers: breakage and migration of mobile ball contactors; malfunctions with the guillotine dampers, recirculation pumps, reheater sections, recirculation piping and nozzles, scrubber booster fans, and mist eliminators; presaturator buildup; weather related problems; and outlet ductwork. It is difficult to evaluate the exact impact of each of these since many are

interrelated or one can affect another. It is important to note, however, that most of these problems have existed from startup and, so far, the Company's attempts to eliminate them have met with little success. Furthermore, the Company's ongoing efforts will probably have minimal effect on increasing scrubber reliability.

# OPACITY METERS EVALUATION

Daily operation and maintenance for all the opacity meters follow manufacturers' recommendations. However, even with proper operation and maintenance, the Bailey Bolometers on Units 1, 2 and 3 were out of calibration because the span was foreshortened on all three meters. In addition, these meters are not reading exit stack opacities because of the path length difference at the meter and at the stack exit.

The Lear-Siegler meter on Unit 4 was not calibrated by NEIC, but the meter readings compared closely to the stack opacity readings.

The recording and reporting of excursions over the 20% limit of the Colorado regulation is hampered by having opacity meter readings that do not reflect stack exit opacities. Any reliability requirement for the particulate control equipment depends on accurate operation of the opacity meters to read stack exit opacities and detailed reporting requirements and record keeping by PSCC and the control agencies involved.

# VISIBLE EMISSION OBSERVATIONS

The Colorado Air Pollution Control Regulation No. 1.A.1 requires that no person shall emit an air contaminant in excess of 20% opacity. An upset condition is defined as an unpredictable equipment failure or

malfunction which results in the violation of an emission control regulation, and which is not due to improper or careless operation.

Upset conditions are deemed not to be in violation of these regulations if immediately reported to the State Air Pollution Control Division.

VEO's were taken during the July - August evaluation period with additional observations taken in October 1977.

The NEIC procedure used in all observations made in July and August was as follows: A member of the Process Control Branch entered the plant and recorded the operation of the equipment before and after a VEO was taken. A member of the Field Operations Branch would take a VEO following Method 9 from a location outside the plant's property at a scheduled time established in advance with the Process Control member.

Of the 92 VEO's taken for a period of 6 minutes or more, 51 were recorded in excess of 20% average opacity. There were 1,374 individual readings (a reading is taken every 15 seconds during a VEO), taken during the 51 observations of which 1,135 (83%) exceeded 20% opacity, the limit prescribed by the Colorado Air Pollution Control Regulations. Of the readings in excess of 20%, 605 were recorded on the Unit 1 and 2 stack, 334 on the Unit 3 stack and 196 on the Unit 4 stack. Although the fewest observations were made on the Unit 3 stack, most of the readings were in excess of 20% opacity because the scrubber was not operating during the entire period of the evaluation.

Of the 1,135 readings greater than 20% opacity, 727 readings were taken during equipment failures in the particulate control system.

Information reported in PSCC upset reports has shown violations of the 20% opacity standards (as measured by the opacity meters) due to ESP malfunctions. Since these violations occurred when the scrubber was in survice, it must be concluded that the scrubber outlet particulate

loading can exceed standards even when the scrubber is not in an upset condition. Therefore, it is important that the mechanical collector/ESP efficiency be improved and maintained at optimum conditions. Operation of the ESP's at 40 to 60% efficiency is not acceptable.

# RELIABILITY

Based on visible emission observations and Company stack tests, the particulate control system appears to be able to meet SIP particulate regulations. The problem is one of how consistently the particulate control system can meet these regulations. Numerous improvements should be investigated to improve the reliability of the particulate control equipment at Cherokee Station.

The particulate removal performance of the scrubbers will vary with gas velocity, liquid flowrate, inlet particulate loading, gas flow imbalances, etc. For those units with scrubbers, improving the collection efficiency of the ESP's would reduce the inlet grain loading to the scrubbers, thus reducing the solids accumulation, fly ash erosion, and required particulate removal in the scrubbers. Potential areas of improvement in the existing electrostatic precipitators include upgrading the gas conditioning system, upgrading the automatic control system, adding more electrical sections, and adding more collecting plate area. Of immediate concern, the efficiency of the ESP's should be optimized, including upgrading the gas conditioning system and performing ESP efficiency tests. This would indicate if further improvements are necessary, such as upgrading the automatic controls, or adding electrical sections.

There are several potential areas for improving reliability in the scrubber system: adding spare scrubber capacity, replacing direct reheaters with indirect reheaters, adding spare recirculating slurry

pumps, and providing weather enclosures. These changes can be evaluated and adopted with minimal preliminary study. Further improvements can be made to the scrubber packing and mist eliminators; however, these would probably require significant modifications, such as using stationary packing, two-stage mist eliminators, or vertically positioned mist eliminators. These modifications would also require lengthy research and development efforts for successful application. Finally, reliability can be improved by more frequent inspection and repair of the Unit 1 and 3 scrubbers. But, how practical this approach is for a base-loaded plant with a limited natural gas supply is questionable.

An alternative to upgrading the existing ESP's and scrubbers would be to replace them with more efficient ESP's (+99% efficiency) or to replace the scrubbers with baghouses.

The areas of potential improvement of the particulate collection system for each unit at Cherokee are summarized in Table 1. The final means of improving reliability will be specific to each unit and will depend on the reliability at which the particulate control equipment is required to operate. Economics and the use of proper techniques to compare reliability improvement alternatives must be considered.

Table 1

POTENTIAL AREAS FOR IMPROVING PARTICULATE CONTROL SYSTEM RELIABILITY

CHEROKEE STATION

PUBLIC SERVICE COMPANY OF COLORADO

Parameters	Unit 1	Unit 2	Unit 3	Unit 4
ectrostatic Precipitators				
Install SO <sub>3</sub> flow meters	Yes	Yes	Yes	Exist
Add gas conditioning	NA	NA	Yes	NA
Optimize ESP operation	Yes	Yes	Yes	Yes
Upgrade the automatic controls	Yes	Yes	Yes	Yes
Add more electrical sections	Yes	NA	Yes	Yes
Add more collecting plate area	Yes	Yes	Yes	Yes
rubbers (TCA)				
Add spare scrubber capacity	Yes	NA	Yes	Exist
Replace direct reheat with indirect reheat	Yes	NA	Yes	Exist
Add spare recirculating pumps	Yes	NA	Yes	Yes
Provide weather enclosures	Yes	NA	Yes	Exist
Improve mobile bed packing	Yes	NA	Yes	Yes
Improve mist eliminator design	Yes	NA	Yes	Yes
Increase frequency of				
inspections and cleaning	Yes	NA	Yes	Yes
place Scrubber with Baghouse	Yes	Yes <sup>†</sup>	Yes	Yes

<sup>†</sup> Add to existing particulate control system.

#### III. RECOMMENDATIONS

# ADMINISTRATIVE

It is recommended that responsible regulatory agencies convey to PSCC an acceptable "percent reliability," i.e. a required percentage of boiler on-line time that the particulate control equipment is operating and meeting particulate regulations.

Public Service Company of Colorado should then be required to conduct a reliability study. Preferably, the Company should consider having a reliability analysis performed by an outside consultant acceptable to the regulatory agencies. From this study and control alternatives evaluated by PSCC, a control plan should be developed by PSCC for review and approval of the appropriate control agencies.

# TECHNICAL

In order to monitor compliance with the regulations, it is recommended that the opacity meters be operated and maintained to indicate exit stack opacities and that any readings >20% opacity be reported to the control agencies. This will require that the Bailey meter readings must be calibrated and the readings adjusted to relate the opacities to the different path lengths associated with each stack. The Bailey meters should be calibrated using an opacity plate or filter in the 40% opacity range. As an alternative, the meters could be replaced with more up-to-date equipment that can be calibrated to read and automatically record the stack exit opacities.

The monitoring controls for the Sulfan system are limited and should be upgraded, as a minimum, to monitor  $\mathrm{SO}_3$  flow to each ESP. Additionally, a gas conditioning system should be added to Unit 3 and measures should be taken to prevent pluggage in the gas conditioning lines. Further technical recommendations for improving particulate control equipment reliability will depend on the results of the reliability study in which alternatives will be rated according to cost, impact, and degree of expected improvement to the overall system reliability.

## IV. PLANT DESCRIPTION

The Cherokee Station powerplant is owned and operated by the Public Service Company of Colorado, a privately owned utility. It is a base-loaded installation with a normal operating schedule of 24 hr/day, 7 days/week, 52 weeks/year. The plant consists of four primary gas-and/or coal-fired units and two diesel-fired peaking units. Unit 1 began commercial operation in 1957, Unit 2 in 1959, Unit 3 in 1962 and Unit 4 in 1968. A general layout of the plant is shown in Figure 1.

# COAL SUPPLY

The Cherokee Station currently receives coal from two primary sources -- the Energy Reserves Mine in Colorado and the Rosebud Mine in Wyoming. Representative analyses of the coal from these sources is shown as follows:

Source	<u>Heating Value</u> J/kg Btu/lb		Ash Mo	
Energy Reserve Mine	2.58 x 10 <sup>7</sup> 11,100	0.47	9.34	8.77
Rosebud Mine	Not available			

The coal is delivered to the plant by train and stored in 30- to 90-day storage piles. Although less than 0.1% of the coal used at Cherokee Station during 1976 was from the Rosebud Mine, it is anticipated that more Rosebud coal could be used in the future and that at any given time the plant would burn coal from either source.

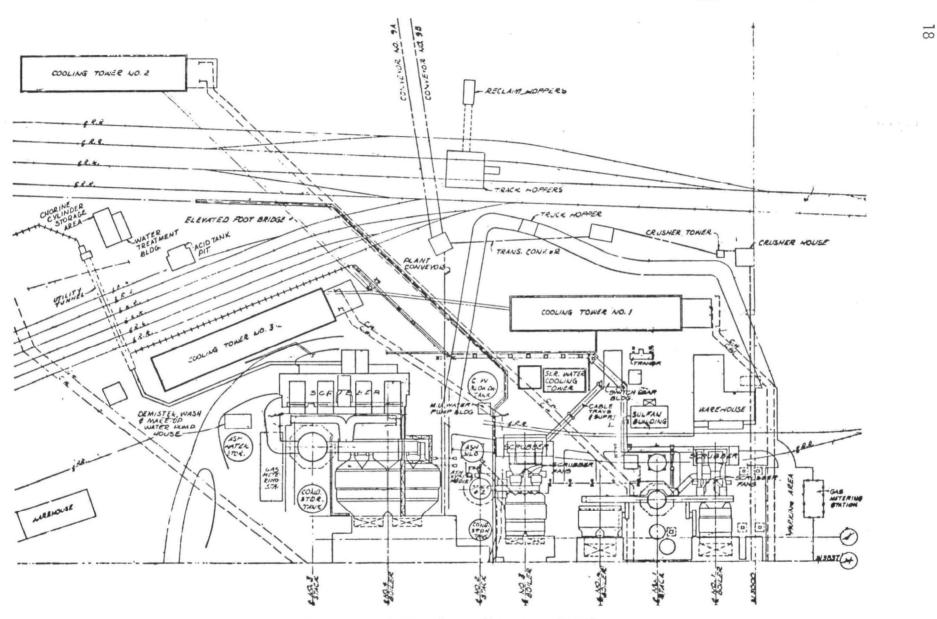


Figure 1. Plot Plan, Cherokee Station Public Service Company of Colorado

Coal is conveyed to each unit by conveyor belts. The coal is transferred first to crusher houses (one for Units 1-3 and one for Unit 4), and then to coal silos which feed the individual pulverizers for each unit. Coal samples are withdrawn from the conveyor belts upstream of the silos. During this investigation, typical coal values were (as measured by weight) 8.86% moisture, 9.73% ash, 0.53% sulfur, and 2.52 x  $10^7$  J/kg (10,840 Btu/lb) heating value.

The coal is routed from the coal silos through counters that weigh the coal fed to the pulverziers. Each Unit is equipped with Riley Stoker Corporation pulverizers that reduce the coal to <50 mesh. From the pulverizer, primary combustion air conveys the coal to the combustion zone of the steam generators. Here it is mixed with secondary air and burned to produce superheated steam which drives the turbine-generators.

# STEAM GENERATING UNITS

The four primary steam generating units are balanced draft, radianttype boilers that burn pulverized coal as the primary fuel. Design parameters for the Units are listed in Table 2.

# PARTICULATE CONTROL SYSTEM

After exiting the air preheater section of the boilers, the flue gas is mixed with a gas conditioning agent,  $SO_3$ . The  $SO_3$  is injected into the gas stream to improve the collection efficiency of the electrostatic precipitators. As indicated in the flow diagram [Figure 2], the gas conditioning system for Unit 3 is not in operation.

The conditioned flue gases are treated by the particulate control system shown in Figure 2. For Units 1, 3, and 4, the system consists of

Table 2

DESIGN PARAMETERS - STEAM GENERATING UNITS
CHEROKEE STATION
PUBLIC SERVICE COMPANY OF COLORADO

Design Data	Unit 1	Unit 2	Unit 3	Unit 4
Initial Year of Operation	1957	1959	1962	1968
Boiler Manufacturer	Babcock and Wilcox	Babcock and Wilcox	Babcock and Wilcox	Babcock and Wilcox
Boiler Type	Balanced draft, radiant type	Balanced draft, radiant type	Balanced draft, radiant type	Balanced draft, radiant type
Steam Capacity, kg(lb)/hr, x10 <sup>3</sup>	387 (852)	387 (852)	517 (1,140)	1,170 (2,587)
Steam Pressure, kg/cm <sup>2</sup> (psig)	109 (1,550)	109 (1,550)	135 (1,925)	180 (2,500)
Steam Temperature, °C (°F)	540 (1,005)	540 (1,005)	540 (1,005)	540 (1,005)
Gross Electrical Generating Capacity,	MW 115	115	170	375
Coal Pulverizer Manufacturer	Riley Stoker Corp.	Riley Stoker Corp.	Riley Stoker Corp.	Riley Stoker Corp.
Number of Mills	4	4	4	5
Design Capacity, kg(lb)/hr, each	16,400 (36,250)	16,400 (36,250)	16,800 (37,000)	30,000 (65,000)

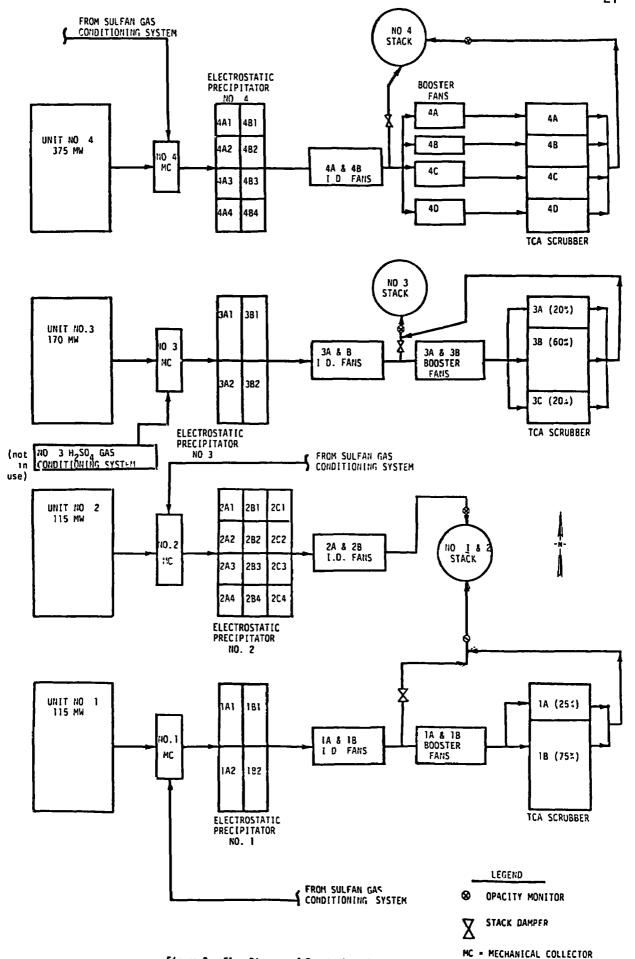


Figure 2. Flow Diagram of Particulate Removal System
Cherokee Station
Public Service Company of Colorado

mechanical collectors, electrostatic precipitators, and a turbulent contact absorber (TCA). For Unit 2, the system consists of a mechanical collector and an upgraded electrostatic precipitator. From the particulate control system, the cleaned flue gases are ducted to stacks for release to the atmosphere. Units 1 and 2 are ducted to a common stack, and Units 3 and 4 are ducted to individual stacks. The scrubbers on Units 1, 3 and 4 can be bypassed so that gas is ducted directly from the respective precipitators to the stack.

#### V. ELECTROSTATIC PRECIPITATORS

All of the electrostatic precipitators (ESP's) are followed by TCA scrubbers except for the Unit 2 ESP which exhausts directly to the common stack for Units 1 and 2. The ESP's installed at Cherokee are cold-side precipitators installed between 1964 and 1968. The ESP's are all preceded by a mechanical collector that precleans the flue gas before entering the precipitators.

The fly ash collected in the ESP hoppers is pneumatically conveyed to ash collecting silos and then hauled off the site by a private contractor for disposal.

ESP operation and control are monitored on the ESP control panels. Metering is provided to monitor primary and secondary voltage, primary and secondary current, and spark rate. The power input to the ESP can be automatically or manually controlled. In the automatic mode, the power input is controlled by saturable core reactors that monitor the spark rate and decrease the applied voltage during excessive sparking or increase the voltage in the absence of sparking. Optimum applied voltage levels occur at a level where sparking is less than 100 sparks per minute.

# DESIGN

# Unit 1 ESP

The electrostatic precipitator manufactured by Western Precipitation Division of Joy Manufacturing Company was installed in 1965. The two-chamber, two-field precipitator is designed to handle a gas flow of

229 m<sup>3</sup>/sec (485,000 ft<sup>3</sup>/min) at 140°C (285°F). The approximate dimensions of the active collection area for the ESP are shown in Figure 3. The ESP has two electrical fields, A and B, each powered by one transformer-rectifier (T-R) set which is energized and controlled from control panels in the ESP control room.

The flue gas enters the ESP through plenum chambers connected to the outlet of the mechanical collectors. The original gas distribution pans at the inlet of each chamber were modified in 1974 by adding a 40% open perforated plate with horizontal and vertical spoilers for proper gas distribution. The collecting surfaces in each field are vertically hung steel plate panels. The panels are hung parallel to each other with the surface of the panel parallel to the gas flow. The gas passage space between the plates is 23 cm (9 in). The discharge elements (stainless steel wire electrodes) are hung in the center of the gas passage from a steel framework that prevents horizontal movement of the wires. The general design parameters for the ESP are given in Table 3.

# Unit 2 ESP

The ESP for Unit 2 was manufactured by Research-Cottrell, Inc. and was installed in 1968. The precipitator was modified in 1976 by splitting the electrical sections in half and adding three transformer-rectifier sets rated at 35 kVa each. These new T-R sets and controls were manufactured by Buell Emission Control, Division of Envirotech Corporation. This modification increased the number of fields from 2 to 3 and increased the precipitator design efficiency from 90 to 94%. The approximate dimensions of the active collection area are shown in Figure 4.

The precipitator has two chambers and three electrical fields, A, B, and C, and was designed to handle  $234 \text{ m}^3/\text{sec}$  (495,000 ft<sup>3</sup>/min) of

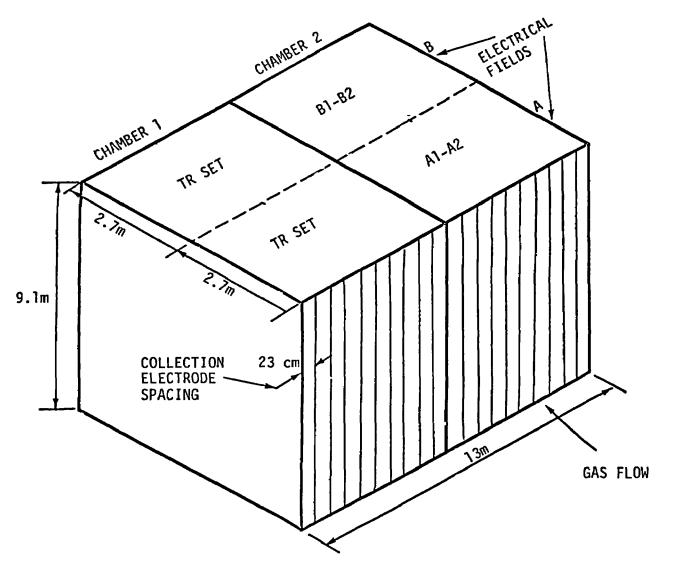


Figure 3. Approximate Dimensions of the Electrostatic Precipitator Active Area Unit 1 - Cherokee Station Public Service Company of Colorado

#### Table 3

# DESIGN DATA FOR ELECTROSTATIC PRECIPITATOR - UNIT 1 CHEROKEE STATION PUBLIC SERVICE COMPANY OF COLORADO

Parameter	Design Value
Manufacturer	Western Precipitation, Division of Joy Manufacturing, Inc.
Date Installed <sup>†</sup>	1965
Design Efficiency	90%
Outlet Loading @ 21°C, l atm (optional)	0.09 gm/m <sup>3</sup> (0.04 gr/ft <sup>3</sup> )
Gas Volume (V)	229 m <sup>3</sup> /sec (485,000 ft <sup>3</sup> /min)
Average Gas Velocity	1.97 m/sec (6.47 ft/sec)
Gas Temperature	140°C (285°F)
Collection Electrode Area (A)	5,418 m <sup>2</sup> (58,320 ft <sup>2</sup> )
Specific Collection Electrode Area (A/V)	24 m <sup>2</sup> /m <sup>3</sup> /sec (120 ft <sup>2</sup> /10 <sup>3</sup> ft <sup>3</sup> /min)
No. of Collector Electrodes	112
Aspect Ratio (L/H)	0.6
Type of Discharge Electrode	Shrouded Wire, 430 SS
Diameter of Discharge Electrode	0.268 cm (0.1055 in)
Type of Rapper and Number	Wire - Eriez Vibrator - 8 Plate - Eriez Rapper - 24
Rapping Frequency	5 sec/20 min
No. of Electrical Fields	2
No. of Chambers	2
No. of Electrical Energizing Sets	2
Transformer Rating	63.6 kVa (each)
Rectifier Wave Form	Full-wave
Rectifier D.C. Voltage	45 kV
Rectifier D.C. Milliamperes	900 mA (each)
Type of Control	Saturable Reactor
H.T. Bus Sections per $10^2 \text{ m}^3/\text{sec } (10^5 \text{ ft}^3/\text{min})$	1.7 (0.8)
Treatment Time	2.9 sec
Migration Velocity	9.45 cm/sec (3.7 in/sec)
Chemical Additive	SO <sub>3</sub> Injection

<sup>†</sup> In 1975-1976, inlet ductwork was modified by adding 40% open perforated plate and horizontal and vertical spoilers. Inlet electrical section A wire discharge electrode was replaced with the Western Mast electrode.

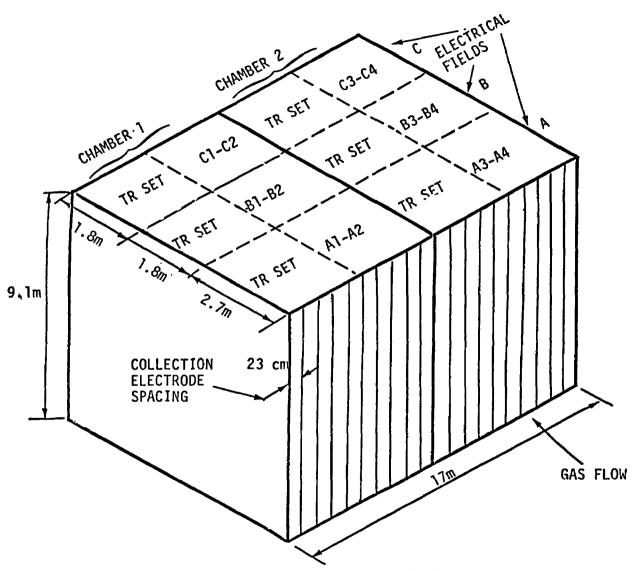


Figure 4. Approximate Dimensions of the Electrostatic Precipitator Active Area Unit 2 - Cherokee Station Public Service Company of Colorado

flue gas at 143°C (290°F). Each inlet section of the two chambers consists of 37 gas passage ducts which are 23 cm (9 in) wide, 9 m (30 ft) high, and 3 m (9 ft) long. The remaining sections are 23 cm (9 in) wide, 9 m (30 ft) high, and 2 m (6 ft) long. The flue gas is precleaned by a mechanical collector, then enters the inlet flue for each chamber. For uniform gas distribution, the inlet duct for each chamber is equipped with turning vanes and perforated distribution plates. The treated flue gases are ducted directly to the stack servicing both Units 1 and 2. Table 4 contains the general design parameters for the Unit 2 ESP.

# Unit 3 ESP

The ESP for Unit 3, manufactured by Western Precipitation Division of Joy Manufacturing Company, was installed in 1964. The two-chamber, two-field precipitator is designed to handle a gas flow of 257 m³/sec (545,000 ft³/min) at 133°C (272°F). The approximate dimensions of the active collection area for the ESP are shown in Figure 5. The Unit 3 ESP is similar to the ESP on Unit 1 with two electrical fields, A and B. One transformer-rectifier set provides the power for each field and is monitored and operated from control panels adjacent to the maintenance offices. Each of the inlet sections consists of 34 gas passage ducts which are 23 cm (9 in) wide, 7.3 m (24 ft) high, and 3 m (9 ft) long; the outlet sections have the same dimensions. The flue gas is precleaned by mechanical collectors, then enters the inlet flue for each chamber which is equipped with a perforated plate of 44% open area for uniform gas distribution. The general design parameters for the Unit 3 ESP are given in Table 5.

# Unit 4 ESP

The Unit 4 ESP, manufactured by Koppers Company, Inc., was installed in 1968. It was designed for a gas flow of 656  $\rm m^3/sec$ 

Table 4

# DESIGN DATA FOR ELECTROSTATIC PRECIPITATOR - UNIT 2 CHEROKEE STATION PUBLIC SERVICE COMPANY OF COLORADO

Parameter	Design Value
Manufacturer	Research-Cottrell, Inc.
Date Installed	1968 <sup>†</sup>
Design Efficiency	94%
Outlet Loading @ 21°C, l atm (optional)	0.07 gm/m <sup>3</sup> (0.03 gr/ft <sup>3</sup> )
Gas Volume (V)	234 m <sup>3</sup> /sec (495,000 ft <sup>3</sup> /min)
Average Gas Velocity	1.51 m/sec (4.95 ft/sec)
Gas Temperature	143°C (290°F)
Collection Electrode Area (A)	8,662 m <sup>2</sup> 93,240 ft <sup>2</sup> )
Specific Collection Electrode Area (A/V)	$37 \text{ m}^2/\text{m}^3/\text{sec}$ (188 ft <sup>2</sup> /10 <sup>3</sup> ft <sup>3</sup> /min)
No. of Collector Electrodes	76 nine ft and 152 six ft
Aspect Ratio (L/H)	0.7
Type of Discharge Electrode	Loop-ring smooth coppered Bessemer
Diameter of Discharge Electrode	0.277 cm (0.109 in)
Type of Rappers and Number	Wire - Syntron Vibrators - 12 Plate - Magnetic Impulse, gravity impact - 32
Rapping Frequency	6 sec/15 min
No. of Electrical Fields	3
No. of Chambers	2
No. of Electrical Energizing Sets	6 <sup>†</sup>
Transformer Rating	3 @ 64 kVa, 3 @ 35 kVa
Rectifier Wave Form	Half-wave
Rectifier D.C. Voltage	45 kV
Rectifier D.C. Milliamperes	3 @ 1,000 mA, 3 @ 550 mA
Type of Control	Saturable Reactor
H.T. Bus Sections per 10 <sup>2</sup> m <sup>3</sup> /sec (10 <sup>5</sup> ft <sup>3</sup> /min)	5.1 (2.4)
Treatment Time	4.24 sec
Migration Velocity	7.7 cm/sec (3.0 in/sec)
Chemical Additive	SO <sub>3</sub> Injection

<sup>†</sup> In 1976 the electrical sections were split in half, making 12 separate sections. Three 35 kVa transformer-rectifier units were added.

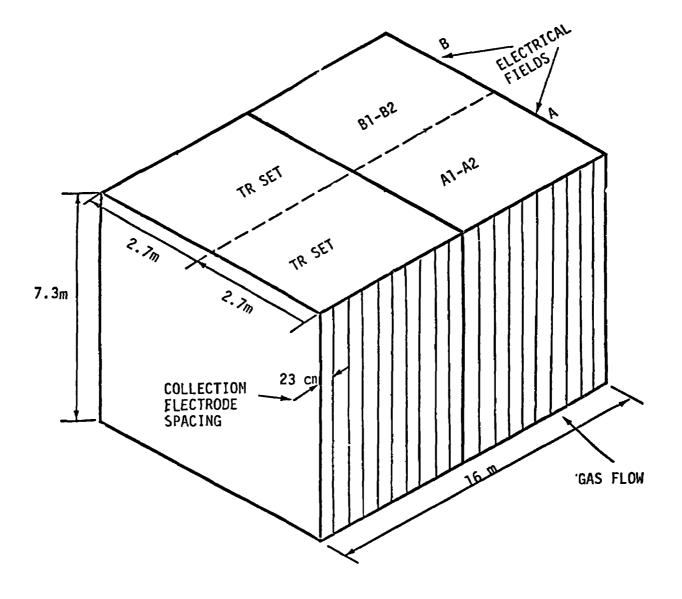


Figure 5. Approximate Dimensions of the Electrostatic Precipitator Active Area
Unit 3 - Cherokee Station
Public Service Company of Colorado

# Table 5

# DESIGN DATA FOR ELECTROSTATIC PRECIPITATOR - UNIT 3 CHEROKEE STATION PUBLIC SERVICE COMPANY OF COLORADO

Parameter	Design Value
Manufacturer	Western Precipitation - Division of Joy Manufacturing, Inc.
Date Installed	1964
Design Efficiency	87%
Outlet Loading @ 21°C, 1 atm (optional)	0.046 g/m³ (0.02 gr/ft³)
Gas Volume (V)	257 m <sup>3</sup> /sec (545,000 ft <sup>3</sup> /min)
Average Gas Velocity	2.3 m/sec (7.44 ft/sec)
Gas Temperature	133°C (272°F)
Collection Electrode Area (A)	5,458 m <sup>2</sup> (58,752 ft <sup>2</sup> )
Specific Collection Electrode Area (A/V)	21 $m^2/m^3/sec$ (108 $ft^2/10^3$ $ft^3/min$ )
No. of Collector Electrodes	134
Aspect Ratio (L/H)	0.75
Type of Discharge Electrode	Shrouded Wire - 430 SS
Diameter of Discharge Electrode	0.268 cm (0.1055 in)
Type of Rappers and Number	Wire - MD 60 Electrical - 8 Plate - MD 60 Electrical - 28
Rapping Frequency	Adustable
No. of Electrical Fields	2
No. of Chambers	2
No. of Electrical Energizing Sets	2
Transformer Rating	57.5 kVa each
Rectifier Wave Form	Full-Wave
Rectifier D.C. Voltage	45 kV
Rectifier D.C. Milliamperes	900 mA each
Type of Control	Saturable Reactor
H.T. Bus Sections per 10 <sup>2</sup> m <sup>3</sup> /sec (10 <sup>5</sup> ft <sup>3</sup> /min)	1.6 (0.7)
Treatment Time	2.4 sec
Migration Velocity	9.6 cm/sec (3.8 in/sec)
Chemical Additive	SO <sub>3</sub> Injection

(1,390,000 ft<sup>3</sup>/min) at 130°C (267°F) and a collection efficiency of 87%. The approximate dimensions of the active collection area for the ESP are shown in Figure 6. The one-chamber, two-field precipitator is powered by four transformer-rectifier sets. After being precleaned, the flue gas enters the inlet sections of the ESP which consist of 78 gas passage ducts 25 cm (10 in) wide, 3 m (9 ft) long, and 9.9 m (32 ft) high; the outlet sections have the same dimensions. The fly ash is handled dry and collected in the #4 Ash Collecting Silo. Table 6 contains the general design parameters for the Unit 4 ESP.

# OPERATING AND MAINTENANCE PRACTICES

The operation and control of the ESP's are monitored on the ESP control panels. Metering is provided to monitor primary and secondary voltage, primary and secondary current and spark rate. The power inputs to the ESP's can be automatically or manually controlled. The ESP's are tuned every day during the normal five-day work week. As part of the adjustment, an electrician checks the meters which monitor current, voltage and spark rate. Both the value of the meter readings and the fluctuation of those readings are observed. If any "abnormal" readings were observed, corrective action or adjustments would be made to the Sulfan system, and/or to the set points for the automatic controls, etc. During discussions with Company representatives, specific values of normal conditions or abnormal conditions were not identified. A Company representative indicated that "by experience" they knew what was "abnormal." No data is recorded by the electricians, however, a strip chart continuously records the precipitator (secondary) current.

Besides the daily checks, the powerplant operator has ground and trip alarms to indicate problems. An electrician is assigned to periodically check the meter readings to minimize the tripping-out of sections.

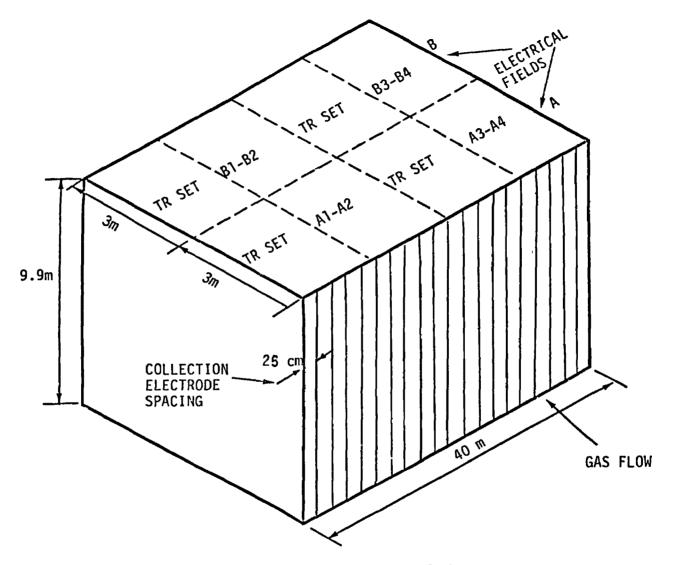


Figure 6. Approximate Dimensions of the Electrostatic Precipitator Active Area Unit 4 - Cherokee Station Public Service Company of Colorado

## Table 6

# DESIGN DATA FOR ELECTROSTATIC PRECIPITATOR - UNIT 4 CHEROKEE STATION PUBLIC SERVICE COMPANY OF COLORADO

Parameter	Design Value	
Manufacturer	Koppers	
Date Installed	1968	
Design Efficiency	87%	
Outlet Loading @ 21°C, l atm (optional)	0.046 g/m³ (0.02 gr./ft³)	
Gas Volume (V)	656 m <sup>3</sup> /sec (1,390,000 ft <sup>3</sup> /min)	
Average Gas Velocity	1.81 m/sec (5.94 ft/sec)	
Gas Temperature	130°C (267°F)	
Collection Electrode Area (A)	17,400 m <sup>3</sup> (187,250 ft <sup>2</sup> )	
Specific Collection Electrode Area (A/V)	26 m <sup>2</sup> /m <sup>3</sup> /sec (135 ft <sup>2</sup> /ft <sup>3</sup> /min)	
No. of Collector Electrodes	640	
Aspect Ratio	0.62	
Type of Discharge Electrode	Round coppered mild steel	
Diameter of Discharge Electrode	0.277 cm (0.109 in)	
Type of Rappers and Number	Wire - Vibrator 24 Plate - Impulse 80	
Rapping Frequency	Adjustable	
No. of Electrical Fields	2	
No. of Chambers	1	
No. of Electrical Energizing Sets	4	
Transformer Rating	96.3 kVa each	
Rectifier Wave Form	Half-wave	
Rectifier D.C. Voltage	45 kV	
Rectifier D.C. Milliamperes	1,500 mA each	
Type of Control	Saturable Reactor	
H.T. Bus Sections per 10 <sup>2</sup> m <sup>3</sup> /sec (10 <sup>5</sup> ft <sup>3</sup> /min)	0.6 (0.3)	
Treatment Time	3.37 sec	
Migration Velocity	7.7 cm/sec (3.0 in/sec)	
Chemical Additive	SO <sub>3</sub> Injection	

Maintenance practices for the ESP's and mechanical collectors are essentially performed on an as-needed basis. When a grounded wire in the ESP is indicated, a load reduction is scheduled, usually within a week, and the wire is removed (note: on Unit 2, if the shorted wire causes an opacity reading of >20% on the opacity meter, immediate action is taken). For ESP problems noted during Unit operation that do not cause opacity of >20%, a Station Service Request (SSR) is originated. An SSR is a request to maintenance that indicates an equipment problem that needs to be checked and repaired during the next shutdown. The mechanical collectors are monitored by reading the differential pressure across the collectors. A high reading would indicate pluggage.

The bulk of the maintenance on both the ESP's and mechanical collectors is conducted during unit outages. For the ESP's, this consists of an inspection and cleaning, if so indicated. Cleaning is accomplished by a combination of vacuuming, scraping and water washing. The mechanicals are inspected during minor and major unit outages. If pluggage has occurred, the collector is cleaned by routing out with poles and/or air lances. Areas of collector found to be excessively worn are also replaced at this time.

## PHYSICAL INSPECTIONS

An internal inspection of the Unit 3 electrostatic precipitator was conducted by EPA and Company personnel on August 14, 1977. The inspection was conducted from the top of the collection plates and from a walkway located just above the hoppers between the first and second fields. The following observations were made.

## Unit 3 ESP

## Discharge Electrodes

Some of the discharge electrodes (approximately 2 to 5%) had a whisker-like buildup of ash over a significant length of the wire. The buildup in some cases reached thicknesses of up to 1 cm. This was not considered a major problem because of the number involved, but would indicate that specific discharge electrode rappers needed to be checked for proper operation. The electrode alignment was good as most electrodes were straight and centered in the gas passages.

#### Collection Plates

The collection plates were also straight, properly spaced and had no signs of warpage. The dust deposits on the plates did not appear excessive (<1 cm). Most of the dust present on the plates could be dislodged with gentle tapping.

## Hoppers

The hoppers were clean and there was no indication of dust buildup. Corrosion of the hopper walls was not observed.

#### Other

The perforated plate, flow distribution device had no significant (>1 cm) solids deposits. The insulators were not closely inspected, but significant (>3 cm) ash buildup was not seen. Company personnel conducted the inspection in a knowledgeable and thorough manner.

## Unit 4 ESP

The internals of the Unit 4 electrostatic precipitator were inspected on August 24, 1977. The inspection was conducted from the top of the collection plates and from a walkway located just above the hoppers between the front half and back half of the ESP. The following observations were made.

## Discharge Electrodes

Some of the discharge electrodes (approximately 2 to 5%) had a whisker-like buildup of ash over a significant length (8 to 10 ft) of the wire. The buildup in some cases reached thicknesses of up to 1 cm. The whisker deposits were easily dislodged by tapping, indicating that the operation of the discharge electrode vibrators could be improved. The electrodes were observed to be straight, hung evenly and centered in the gas passages. There did not appear to be many missing wires.

## Collection Plates

The alignment of the collection plates appeared satisfactory with little or no noticeable warpage. There was one section (approximately  $1\ m^2$ ) of one plate which was warped to the point of touching an adjacent plate. This may have been initiated by corrosion of the plate. Overall, however, corrosion of collector plates did not appear to be a problem. There were very few large (>2 cm) ash buildups on the collector plates but, in general, it was found that more ash was sticking to the plates on the north half than on the south half. In addition, some of the deposits on the north half of the ESP tenaciously clung to the plates and formed very hard deposits. These may have been due to more predominant CaSO<sub>4</sub> formation at these locations caused by reaction of SO<sub>3</sub> with CaO present in the ash.

#### Other

The hoppers were clean with no signs of corrosion or dust buildup that would interfere with precipitator operation. The inlet flow perforated plate showed no signs of dust accumulation. The insulators were not inspected. Company personnel conducting the inspection were knowledgeable and thorough in their procedures.

## SULFAN SYSTEM

The high resistivity ash associated with low-sulfur western coal severely limits the performance of an ESP by limiting the voltage and current at which the ESP operates. The purpose of the Sulfan system is to provide sulfur trioxide ( $\mathrm{SO}_3$ ) for mixing with the powerplant fly ash particles. The  $\mathrm{SO}_3$  conditions the surface of the fly ash particles by increasing the surface conductivity and, thus, reducing the ash resistivity. This improves the collection of the fly ash particles by increasing the current density and voltages at which the ESP can operate.

The Sulfan system presently is installed on Units 1, 2 and 4 and was initially put in operation in June 1971. It consists of a heated tank, piping and manual control valves through which the  $\rm SO_3$  is transferred to each unit, a nozzle distribution system injects  $\rm SO_3$  into the individual gas streams. A diagram of the Sulfan system is shown in Figure 7.

The Sulfan system is designed to provide a maximum of 0.9  $\rm m^3/min$  (32  $\rm ft^3/min$ ) of  $\rm SO_3$ . This should result in a  $\rm SO_3$  concentration in the powerplant offgas of about 20 ppmv, assuming design gas rates inputs to the ESP's and appropriate distribution of  $\rm SO_3$  to each unit. The actual injection rate is controlled by the temperature maintained in the Sulfan tank by an external heat source. The amount of  $\rm SO_3$  used is

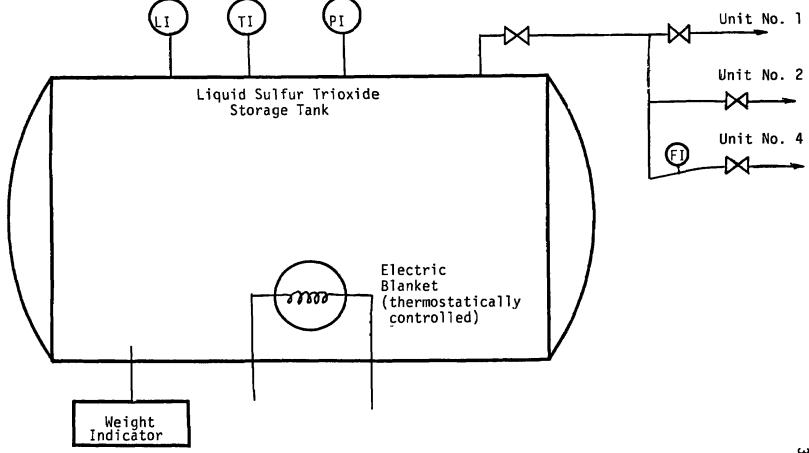


Figure 7: Schematic Drawing of Sulfan Sulfur Trioxide Conditioning System
Cherokee Station
Public Service Company of Colorado

monitored daily by recording the tank weight. There is also a flow meter recording  $S0_3$  flow rate to the Unit 4 ESP but no flow rate indicator exists for the other ESP's.\*

The  $\mathrm{SO}_3$  flow is not normally adjusted unless an ESP problem is indicated such as excessive sparking. When current/voltage inputs to the ESP are low, the first consideration is to further open the valve controlling the  $\mathrm{SO}_3$  to the individual ESP unit. If this does not improve the power input, the spark rate control is adjusted. Additional adjustments are based on specific problems indicated by the ESP instrumentation.

The monitoring controls for the Sulfan system are very limited and the amount of  $\mathrm{SO}_3$  injected to each unit could not be determined. Knowing how much  $\mathrm{SO}_3$  flows to the ESP's is critical to the proper operation of the Sulfan system and the ESP's. Without  $\mathrm{SO}_3$  flow data, the Sulfan system could not be evaluated and the effect of the gas conditioning system on ESP operation was only speculative. No test results were available to determine the effectiveness of the gas conditioning system.

## ESP OPERATING DATA AND EVALUATION

To evaluate the operation of the precipitators, the electrical operating levels of the ESP's were monitored periodically during the entire study period. The electrical parameters monitored were primary and secondary voltage, primary and secondary current and spark rate. From these readings the power levels in and out of the rectifier sets were calculated along with the power efficiencies and current densities for each electrical section. These calculations are contained in

<sup>\*</sup> The Unit 4 flow meter was not in operation during the investigation as a result of pluggage difficulties. In September 1977, the Company installed a heat lamp to remedy this problem.

Appendix C and are summarized in Table 7. Table 8 compares the normal operating conditions to the conditions that were observed during the study period. The normal operating conditions were provided by PSCC. These values are for normal operation at full load and 100% coal. During the period of the study, Unit 1 was operated on 100% coal, Unit 2 on 75% coal, Unit 3 on 25 to 75% coal and Unit 4 on 100% coal.

## Unit 1 ESP

The electrical operating data indicates that the Unit 1 ESP was operating at lower power inputs than the ESP's on the other units. Although the voltages and currents were in the normal operating ranges, the majority of the values were at the lower end of the scale, also indicating low power inputs. The spark rate for the Unit 1 ESP is normally 100 to 150 sparks per minute (spm), but was observed to be greater than 200 spm for the majority of the time. Because of this condition, the automatic controls appeared to be spark rate limited, causing the low power inputs. The high spark rate and low power indicate unstable electrical conditions in the precipitator. One possible explanation could be the failure of the Sulfan system to adequately condition the flue gas. The lines or nozzles could have been plugged, but this could not be determined since metering was not available to monitor the SO<sub>2</sub> flow. Before further evaluation of the power inputs can be made, the Sulfan system needs to be upgraded to monitor flow so that the levels of gas conditioning needed for maximum power input to the precipitator can be determined.

At various times during the study, the secondary current was abnormally low with no spark rate indication. Probable cause could be misadjustment of current and/or voltage limit controls. This condition was usually corrected by the next day and the exact cause of the problem could not be identified.

Table 7

SUMMARY OF OBSERVED ELECTRICAL CONDITIONS FOR

THE ELECTROSTATIC PRECIPITATORS

CHEROKEE STATION

PUBLIC SERVICE COMPANY OF COLORADO

Unit	Primary Power	Secondary Power	Power Efficiency		Current Density		
	(kw)	(kw)	(%)	μΑ/Τζ-	m.A/m²		
Unit 1 A-field B-field	4-12 1-10	1-6 -	14-79 -	1-7 1-8	0.015-0.063 0.007-0.083		
Unit 2 A-field B-field C-field	3-23 3-26 10-25	1-14 1-21 11-21	22-79 30-78 <sub>+</sub> ?1-87	2-15 3-41 11-41	0.023-0.162 0.035-0.437 0.097-0.445		
Unit 3 <sup>††</sup> A-field B-field	No data						
Unit 4 A-field B-field	12-38 12-50	††† 7-20 <sup>†††</sup>	††† 52-69 <sup>†††</sup>	6-15 5-19	0.068-0.161 0.055-0.2		

<sup>†</sup> No secondary voltage data on Unit 2 C1 and C2 field.

tt Secondary power was higher than primary power. Exact cause of problem was not identified.

<sup>†††</sup> No secondary voltage data on Unit 4 A field and B1-B2 fields.

Table 8

COMPARISON OF

NORMAL AND OBSERVED OPERATING CONDITIONS<sup>†</sup>

ELECTROSTATIC PRECIPITATORS

CHEROKEE STATION

PUBLIC SERVICE COMPANY OF COLORADO

Operating Conditions	Unit 1	Unit 2	Unit 3 <sup>††</sup>	Unit 4
Spark Rate (SPM) Normal Observed	100-150 0->500	50-150 0->200	100-150 0-280	250-300 20->200
Primary Voltage (V) Normal Observed	180-200 155-200	50-75 170-370	150-175 50-235	175-200 190-320
Primary Current (A) Normal Observed	45-55 25-68	150-200 10-95	100-125 65-135	60-75 55-150
Secondary Voltage (kV) Normal Observed	34-40 33-41	35-45 20-45	38-40 8-51	30-35 29-36
Secondary Current (mA) Normal Observed	100-125 20-225	500-600 40-550	400-500 280-740	400-500 230-865

<sup>+</sup> Normal operating conditions were provided by PSCC. Precipitator operation varies with unit, load, fuel and sulfan treatment. Values stated are normal operation, full load, 100% coal. Observed readings were taken by NEIC during study period.

Power efficiency =  $\frac{primary\ voltage\ (V)\ X\ primary\ current\ (A)}{secondary\ voltage\ (kV)\ X\ secondary\ current\ (mA)}\ X\ 100\%$ 

<sup>††</sup> Power efficiencies were greater than 1, exact cause of problem was not identified therefore meter readings are suspect.

Under normal conditions, the inlet sections (A sections) are usually operated at lower current densities due to space charge effect and higher particulate concentrations. However, the data taken during the study period [Appendix A] showed that the outlet sections (B sections) operated at much lower current densities than the inlet sections. From this observation, it appears that the outlet sections could be operated at even higher power levels once the spark rate was optimized, thus increasing efficiency. It could also be inferred that the  $\mathrm{SO}_3$  injection is too low, since this would effect current densities. However, this is only speculative since the  $\mathrm{SO}_3$  rates were not monitored. Once the  $\mathrm{SO}_3$  rate can be monitored, the effect of increasing the  $\mathrm{SO}_3$  rate on current densities should be investigated.

## Unit 2 ESP

In 1976, the electrical sections of this precipitator were split in half and new lower-rated T-R sets were added for greater sectional-ization with an estimated increase in efficiency from 90 to 94%. The added sets and controls were manufactured by Buell and were rated lower than the original T-R sets. The new lower rated sets were placed on the northside of the ESP to energize cells 1 and 2 and the original T-R sets were placed on the southside to power cells 3 and 4. Convenience and the ease of installation dictated arranging the T-R sets in this manner. Normally when mixing different rated T-R sets, the best arrangement would be to place the higher rated units at the outlet fields because they can handle higher current densities, thus improving the collection efficiency of the precipitator. Placing the lower rated sets on one side of the ESP would probably lower the overall efficiency since half of the precipitator is operating at lower power inputs and current densities.

The electrical operating data monitored during the study period showed this condition to exist in the precipitator. One side of the precipitator, cells 1 and 2, had very low power inputs, power efficiencies, and current densities when compared to the cells 3 and 4. This condition was consistent in each field (A, B and C) of the precipitator. Although the lower power inputs and current densities were expected, the magnitude of the difference was not expected. A possible explanation of this would be severe flow and particulate loading imbalances in the two sides of the precipitator. In addition, the voltage and current limiting controls should be checked for misadjustment.

It was also noted that the power efficiencies of the T-R sets for cells 1 and 2 had lower calculated power efficiencies than the T-R sets on cells 3 and 4. Assuming the meter readings are accurate, the operation of the T-R sets should also be checked, since the arrangement of the T-R sets should not affect the power efficiencies of the sets.

The power levels, power efficiencies and current densities for cells 3 and 4 are within the normal operating ranges for the precipitator and are higher than the other Cherokee precipitators. The power levels, power efficiencies, and current densities for cells 1 and 2 are lower than the normal operating ranges. As previously discussed, the cause of this problem was not apparent. The current densities for the outlet sections are higher than the inlet sections, even for the low powered cells 1 and 2 as would be expected.

## Unit 3 ESP

Unit 3 boiler was operated on varying amounts of coal for the first half of the study period because the scrubber was down for major maintenance. The entire unit was down for its annual outage (maintenance) during the last half of the study period. Although the

ESP was operating while the boiler was on-line, the data collected could not be evaluated because the power efficiencies of the T-R sets were greater than one in most cases and the exact cause of the problem could not be identified. The meter readings are suspect, therefore, further analysis of the data was not done.

## Unit 4 ESP

Analysis of the electrical operating data indicated that cells 1 and 2 of the precipitator were operating at lower power levels than the 3 and 4 cells. This is the same situation that was found in the Unit 1 ESP, although not as severe. Possible cause of this problem could be imbalances in gas flow and/or particulate loadings. Since this ESP is only a one-chamber precipitator, this condition is not as likely to occur as may be the case with the Unit 1 precipitator, which has two chambers. For the cells 3 and 4, the outlet sections (B3 and B4) had lower current densities than the inlet sections (A3 and A4), which indicates that the B3 and B4 sections could be operated at higher power inputs. For the other side of the precipitator (cells 1 and 2), the current densities were, as expected, higher for the outlet sections. Although secondary currents were high, the actual current densities were relatively low; that would cause a drop in particulate collection efficiency. For example, the ESP on Unit 2 had secondary currents around 500 mA with current densities about 40 uA/ft<sup>2</sup> while the Unit 4 secondary currents were around 700 mA, and the current densities were about 15  $\mu\text{A/ft}^2$ . Because the secondary currents are operating at relatively high levels, the power input cannot be adjusted to significantly improve the current densities. The most probable way of increasing the current density would be to add more T-R sets and increase the sectionalization of the precipitator.

PSCC is experimenting with full-wave and half-wave energization on the Unit 4 ESP to improve its efficiency. PSCC indicated that sections 4AI and 4A2 were on full-wave rectification, but an analysis of the optimum voltage wave form was not available for review.

Use of full-wave energization for the collection of high resistivity dust is not typical because full-wave energization is electrically more unstable than half-wave energization. Half-wave energization produces longer decay periods for the voltage between current pulses, thus allowing ample time for sparks to extinguish. For high resistivity dusts associated with low sulfur coal, which is the case at Cherokee, the optimum voltage wave form is usually with half-wave energization.

There are times when combinations of half-wave and full-wave sets are useful, but usually on precipitators with higher sectionalization, i.e., >3 fields. The usual arrangement is to have the full-wave rectifiers on the outlet sections of the precipitator. The principle involved is that the inlet sections have the highest particle concentration and collection which means relatively high operating voltages but low currents, while the outlet sections have relatively clean gas which means lower voltages and higher currents for these sections.

The use of half-wave rectification for Units 1 and 3 should be investigated by PSCC. Units 2 and 4 are on half-wave rectification.

## COMPARISON OF DESIGN PARAMETERS

The design parameters for the ESP's at the Cherokee Station were compared to the range of values typically found in fly ash precipitators [Table 9]. The values were compiled from precipitators with collection efficiencies from 90 to 99%. The design parameters for the Cherokee

#### Table 9

## COMPARISON OF ELECTROSTATIC PRECIPITATOR DESIGN PARAMETERS CHEROKEE STATION

PUBLIC SERVICE COMPANY OF COLORADO

Parameters	Range of Values for Fly Ash Precipitators	Unit 1	Unit 2	Unit 3	Unit 4
Precipitation Rate					
m/sec ft/sec	0.015-0.18 0.05-0.6	0.09 0.31	0.08 0.25	0.1 0.32	0.08 0.25
Specific Collection Area (SCA)					
m <sup>2</sup> /m <sup>3</sup> /sec	20-157	24	37	21	27
ft <sup>2</sup> /1,000 ft <sup>3</sup> /min	100-800	120	188	108	135
Gas Velocity					
m/sec ft/sec	1.2-2.4 4-8	1.97 6.47	1.51 4.95	2.3 7.44	1.8 5.94
Aspect Ratio (L/H)	0.5-1.5	0.06	0.7	0.75	0.62
Corona Power					
watts/m <sup>3</sup> /sec watts/1,000 ft <sup>3</sup> /min	106-1,060 50-500	6-26 3-12	4-94 2-44	No data -	11-31 5-14
Corona Current Density					
mA/m <sup>2</sup>	0.054- 0.075	0.01- 0.063	0.02- 0.45	No data -	0.055 0.18
μ <b>A</b> /ft <sup>2</sup>	5-70	1-7	2-41	-	5-17
Plate Area/Electrical Set					
m <sup>2</sup>	465-7,430	2,709	3,712 <sup>†</sup> 1,237 <sup>†</sup>	† 2,729	4,349
ft <sup>2</sup>	5,000- 80,000	29,160	19,980 <sup>†</sup> 13,320 <sup>†</sup>	† 29,376	46,812
Degree of Sectionalization					
Bus sections/100 m <sup>3</sup> /sec <sub>3</sub> Bus sections/100,000 ft <sup>3</sup> /min	0.8-8.4 0.4-4.0	1.7 0.8	5.1 2.4	1.6 0.7	1.2 0.6
No. of Fields	4-8	2	3	2	2

t Taken from reference 3. Design efficiencies are from 90 to 99%. The larger number is generally associated with the more efficient precipitators except for gas velocities.

tt Inlet sections

<sup>†††</sup> Outlet sections

precipitators are typical for the fly ash precipitators operating at medium efficiencies (80 to 90%). The specific collection electrode area (SCA) is used to calculate theoretical collection efficiencies in the Deutsch equation. The specific collection electrode area is defined as the collection electrode surface area divided by the gas flow rate. For design purposes the SCA is usually expressed in  $ft^2/1,000$  acfm. This term is used directly in the calculation of collection efficiencies as previously shown in the Deutsch equation. The SCA's for the Cherokee precipitators are at the low range for fly ash precipitators. Higher performance ESP's (99+%) are now being designed with SCA's of up to  $157 \text{ m}^2/\text{m}^3/\text{sec}$  (800  $\text{ft}^2/10^3 \text{ ft}^3/\text{min}$ ). The SCA's for the Cherokee precipitators are much lower and can only be increased by enlarging the precipitator (i.e., increasing the collection electrode area).

The aspect ratio is defined as the ratio of the effective length of the gas passages to the height of the gas passages. As the re-entrained dust is carried forward by the flow of the gas, sufficient gas passage length must be provided to prevent dust from being carried out of the ESP before the dust reaches the dust collection hoppers. If the aspect ratio is too small, dust losses from re-entrainment will increase. Higher performance ESP's usually have aspect ratios greater than 1. As shown in Table 9, all the Cherokee precipitators have aspect ratios less than 1.

Power consumption is another important design parameter that affects the collection efficiency. The ranges of corona power and current density as compared in Table 8 show that the Cherokee precipitators have lower power inputs than typical fly ash precipitators. Precipitators installed in the late 1960's were typically found to be undersized

<sup>\*</sup> Deutsch equation is  $n = 1-e^{-(A/V)w}$  where n is the efficiency, w is the migration velocity, A is the collecting electrode surface area, and V is the gas flow rate. The precipitation rate parameter is considered equivalent to the performance migration velocity for actual operating data.

especially for units collecting high resistivity ash. This is the case for the precipitators installed at Cherokee. The Unit 2 ESP was upgraded in 1976 by adding T-R sets and increasing the electrical sectionalization. This modification is reflected in the power comparisons, where corona power and current density are much larger than the other three precipitators.

Electrical sectionalization and number of fields is important in maintaining the collection efficiency near design. This is especially true when sections are taken out of service for broken corona wires or some other reason. If a precipitator cell only had two fields and one field was out of service, the efficiency of this cell would decrease by as much as 30 to 50%. However, if there were three or four fields and one was taken out of service, then the cell would probably lose less than 20 to 30% in collection efficiency.

Reviewing the design parameters, it appears that the precipitators are undersized for efficient collection (>90%) but are typical for precipitators that were installed in the late 1960's for collection efficiencies of 80 to 90%.

More efficient precipitators have a higher degree of sectionalization and a larger number of fields. The degree of sectionalization and number of fields for three of the Cherokee precipitators are low when compared to typical fly ash precipitators. The recent modification of the Unit 2 ESP increased the degree of sectionalization and number of fields, thus increasing its efficiency.

## ESP EFFICIENCY TEST RESULTS

In response to a request for stack test data, PSCC submitted a summary of test results performed by PSCC on the electrostatic precipitators at Cherokee [Appendix D].

Most of the data were outlet grain loadings only, but there were efficiency tests done on the Units about five years ago. These tests are summarized as follows.

ESP	Tested Efficiency Range %	Typical Efficiency <sup>†</sup> %
Unit 1	24-73	50
Unit 2	76-97	89
Unit 3	48-58	51
Unit 4	28-55	42

<sup>†</sup> Typical efficiencies were reported by PSCC, with the most frequently tested efficiency reported as typical.

These test results indicate that the ESP's are operating well below their design efficiencies. More recent data indicates the same range of outlet grain loadings as when these efficiency tests were done, therefore, these efficiencies are considered typical of the present ESP efficiencies.

## UPSET REPORTING

As part of the upset reporting requirements, PSCC has reported day-to-day malfunctions of the particulate control equipment to the State of Colorado. Prior to May 1975, the Company reported all equipment outages of 24 hours or longer. From May 1975 to April 1977, the Company reported all major outages that occurred when the opacity was greater than 20%. As of May 1977, the Company reports only those malfunctions which prevent all the flue gas from being treated in each scrubber. The data generated in these reports on ESP problems is

<sup>\*</sup> Scrubber data is discussed in the following section.

summarized in Table 10. Because of the limited amount of data on ESP's only the data from July 1975 to May 1977 were summarized. The availabilities are relatively good except for Unit 3 where the major cause of downtime was broken electrode wires. A possible explanation would be the lack of gas conditioning to this Unit. The high resistivity ash, with no gas conditioning, could cause excessive sparking at the electrodes causing the wires to burn out. If this happens, the section would have to be taken out of service until the wire is removed or replaced.

The major upset problems associated with the ESP's are broken electrode wires, but as discussed previously, they are immediately removed or replaced if the opacity exceeds 20%. An important observation was made from reviewing the upset reports. There are times when an ESP section or sections are down, with the scrubber operating normally and the opacity exceeding 20%. It is therefore, very important that the ESP efficiencies are maintained and downtime minimized.

Table 10

SUMMARY OF ESP AVAILABILITY BY UNIT<sup>a</sup>

CHEROKEE STATION

PUBLIC SERVICE COMPANY OF COLORADO

11-24	ES	Р	Major Donostad
		Downtime <sup>c</sup> hrs	Major Reported Downtime Causes <sup>d</sup>
1	91	958	Broken electrode wires - 98% breakage or control circuit failure - 2%
2	e	1,367	Broken wires-37%, unknown 32%, high ash buildup-17% control system failure-6%
3	75	3,299	Broken electrode wires-87% Control system failure-8% Unknown grounds-5%
4	94	648	Low power-48%, broken wire-30% ash buildup-22%

a Data taken from PSCC monthly upset reports for July 1975 to May 1977.

A = Hours boiler operating

B = Hours boiler burning 100% gas

C = Hours ESP had 1 or more sections out of service.

Hours ESP had 1 or more sections out of service as reported by PSCC to the State of Colorado in their monthly upset reports.

d Expressed as percentage of hours ESP was down.

e Data were not available on hours of boiler operation.

b % Availability =  $\frac{A - B - C}{A - B}$  x 100%

#### VI. SCRUBBERS

## DESIGN

Turbulent Contact Absorber (TCA) scrubbers were installed on Cherokee Units 1, 3 and 4 to supplement control of particulate by the electrostatic precipitators and mechanical collectors on those units. The TCA scrubbers were designed by Universal Oil Products (UOP), Air Correction Division. A drawing of the TCA scrubber is shown in Figure 8.

Flue gas exiting from the precipitators enters a scrubber booster fan which discharges into the presaturator section of the scrubber. In this section, scrubber makeup water is sprayed into the gas, reducing the temperature to approximately 52°C (125°F). From the presaturator, the gas enters the scrubber.

Each scrubber consists of three stages of beds packed to a 15 cm (6 in) depth with 5 gm balls (approximately 3 to 4 cm diameter). The mobile packing creates a high gas-side pressure drop across the scrubber and provides liquor-to-gas contact for effective particulate removal. The mobile spheres, when in constant action, also tend to be self-cleaning, thus reducing the potential for plugging. By design, the spheres remain mobile when liquor-to-gas flows balance each other. As a result, gas and liquor rate must be properly controlled within specified limits to insure scrubber effectiveness. Gas velocities in the scrubber should be maintained between 550 and 750 fmp.

The scrubber liquor is pumped to spray headers located above the packing in the top stage. After being contacted with gas flowing up through the scrubber, the scrubber liquor is collected in a hopper at the bottom of the scrubber from where it is recirculated. Slurry is

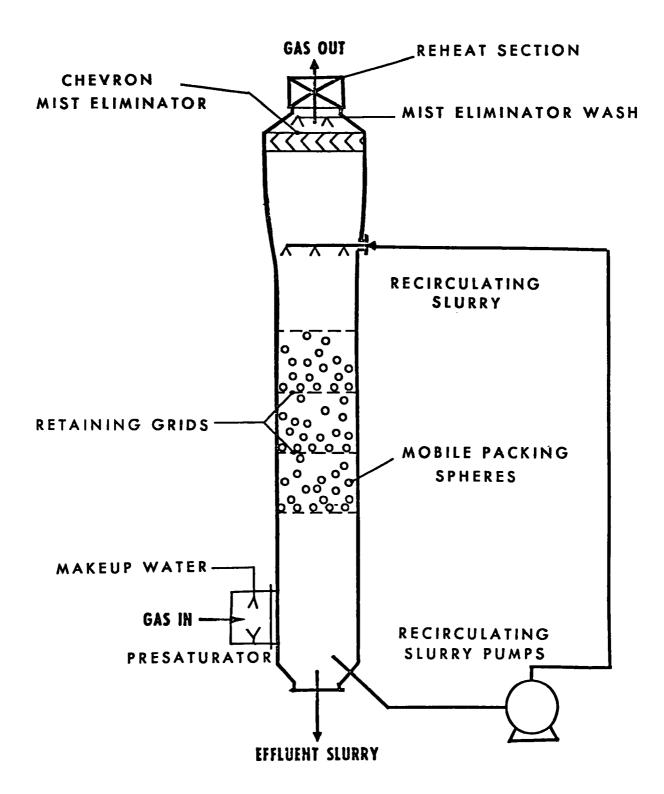


Figure 8. Schematic Drawing of Turbulent Contact Absorber (TCA)
Cherokee Station
Public Service Company of Colorado

purged from the scrubber system to prevent buildup of excessive solids in the recirculated liquor. The slurry from the scrubber is pumped to an ash pond for disposal.

The scrubbed gas passes through mist eliminator and reheater sections prior to being discharged through a stack. The mist eliminator consists of a single section of chevron-type blades and is designed to remove entrained droplets carried over from the scrubber. Deposits which accumulate on the mist eliminator blade are washed off by intermittently operated spray nozzles located above the mist eliminator. After passing through the mist eliminator, the gas is heated by direct in-line steam coils (Units 1 and 3) or by hot air injection (Unit 4). The resulting gas is at a temperature which should prevent corrosion of the stack and duct work and provide plume buoyancy.

The design specifications for the scrubbers on Units 1, 3 and 4 are presented in Table 11.

## Unit\_1

A model 5600 TCA scrubber was installed on Unit 1 in 1973. It has two parallel sections: section 1A handles 25% of the gas flow, and section 1B handles 75% of the flow. The sections receive gas flow from two common parallel scrubber booster fans but are designed to operate independently. Section 1A has one recirculating slurry pump (27,000 1pm or 7,000 gpm) while section 1B has three recirculating slurry pumps (80,000 1pm or 21,000 gpm). The designed particle collection efficiency is 97.5% with three scrubbing stages and a liquid-to-gas ratio (L/G) of 8.1 1/m<sup>3</sup> (60 gal/1,000 ft<sup>3</sup>). Direct stack gas reheat to 93°C (200°F) is provided by in-line steam coils arranged in two tube bundles equipped with soot blowers to remove deposits from tube surfaces.

Table 11

## DESIGN SPECIFICATIONS FOR TCA SCRUBBERS CHEROKEE STATION PUBLIC SERVICE COMPANY OF COLORADO

UOP-ACD 5600	UOP-ACD 6700	UOP-ACD 4200 1974
l-25% flow l-75% flow	1-60% flow 2-20% flow	4-25% flow
3 Variable	3 Variable	variable
880,000 520,000	1,040,000 610,000	2,600,000 1,520,000
0.45-1.05	0.45-1.05	0.45-1.05
30 12	28 11	30 12
97.5%	93%	97%
1.8 0.8	0.69 0.3	1.6 0.7
0.046 0.02	0.046 0.02	0.046 0.02
Not specified	Not specified	Not specifie
Water	Water	Water
106,000 28,000	114,000 30,000	319,000 84,000
8.1 60	7.4 55	8.3 62
3.0	3.0	3.0
Chevron-3 pass 1 Not available	Chevron-3 pass 1 Not available	Chevron-3 pa 1 Not availab
Direct 2 bare/finned 8 93 200	Direct 3 bare/finned 10 85	Indirect 3 bare N/A 79
	5600 1973 1-25% flow 1-75% flow 3 Variable 880,000 520,000 0.45-1.05 30 12 97.5%  1.8 0.8  0.046 0.02 Not specified Water 106,000 28,000  8.1 60  3.0 Chevron-3 pass 1 Not available Direct 2 bare/finned 8 93	5600 1973 1-25% flow 1-75% flow 2-20% flow 3 3 Variable 880,000 1,040,000 0.45-1.05 30 12 11 97.5% 93%  1.8 0.69 0.8 0.046 0.02 0.046 0.02 0.02 Not specified Water 106,000 28,000 114,000 28,000 30,000  8.1 7.4 60 85

<sup>†</sup> Patic of acceptable gas flowrates in scrubber expressed as a function of design flowrate.

tt Pressure drop includes mist eliminator as well as scrubber.

Since its initial operation in 1973, the TCA scrubber on Unit 1 has undergone a number of modifications. These include removing all finned tube steam reheat coils (1975), installing additional soot blowers (1974), replacing the original Fiberglass reinforced plastic (FRP) mist eliminators with stainless steel (1976), and adding turning vanes to the ESP outlet and scrubber inlet ductwork (1974).

Future changes to the scrubber will depend on mist eliminator tests being conducted for the Unit 3 scrubber. A decision will be made on whether to replace all of the mist eliminator sections with a newer design or to expand present reheat capacity.

## Unit\_3

A model 6700 TCA scrubber was installed on Unit 3 in 1975. It has three parallel sections: sections 3A and 3C each handle 20% of the gas, and section 3B handles the remaining 60%. The sections receive gas flow from two common parallel scrubber booster fans but are designed to operate independently. Section 3A and 3C each have one recirculating slurry pump (19,000 lpm or 5,000 gpm) while section 3B has three recirculating slurry pumps (57,000 lpm or 15,000 gpm). Particle collection efficiency is 93% with three scrubbing stages and an L/G of 7.4  $1/m^3$  (55 gal/1,000 ft<sup>3</sup>). Direct stack reheat to 85°C (185°F) is provided by inline steam coils arranged in three tube bundles equipped with soot blowers to remove deposits from tube surfaces.

Since its initial operation in 1972, the TCA scrubber has gone through a number of modifications. These include replacing all finned tube steam reheat coils with plain coils (1975-76), installing additional soot blowers (1973), and replacing the original FRP mist eliminators with stainless steel (1977). In August, 1977, the Company replaced the stainless steel mist eliminators in Sections 3A and 3C with plastic assemblies manufactured by Heil (Heilex Model EB-4) and Munters (Euroform Model 271), respectively. The Company will observe the mist removal

efficiency of these assemblies and, based upon the test outcome, will decide whether to replace all of the mist eliminator sections with one of these newer designs or to expand present scrubber reheat capacity.

## Unit 4

Four model 4200 TCA scrubbers were installed on Unit 4 in 1974. The scrubbers, designated as sections 4A, 4B, 4C, and 4D, were each designed to handle 25% of the gas flow. Each section receives gas flow from an individual scrubber booster fan. There are three recirculating pumps per section designed to provide a total recirculating flow of 80,000 lpm (21,000 gpm). The design particle collection efficiency is 97% with three scrubbing stages and an L/G of  $8.3 \text{ l/m}^3$  (62 gal/l,000 ft<sup>3</sup>). Indirect stack gas reheat to 79%C (175%F) is provided by mixing the scrubbed gas with heated ambient air in a venturi type mixing chamber.

Since its initial operation in 1974, the Unit 4 TCA scrubber has gone through a number of modifications. These include adding a second reheat air fan to each scrubber reheat system (1976) and installing outlet damper purge air systems (1976).

## OPERATION AND MAINTENANCE

The operation of the scrubbers is, for the most part, monitored and controlled from panels located in the respective boiler unit control rooms. The data that is monitored is shown in Table 12. The essential areas of control are the gas flows, recirculating slurry, mist eliminators and reheaters.

Normal scrubber operation requires all scrubber sections in service on Units 1 and 3, and three of four sections in service on Unit 4. When one of the smaller sections of the scrubbers on Units 1 and 3 require repair, the Company treats all the gas flow from that unit in the

## SCRUBBER LOG CHEROKEE STATION PUBLIC SERVICE COMPANY OF COLORADO

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CHEROKEE STATION

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remaining scrubber sections. If the larger scrubber section on Units 1 or 3 is removed from service, part of the gas is bypassed to the stack.

The gas flow to the scrubber is automatically controlled by maintaining the inlet scrubber booster fan pressure to within proper limits (e.g., -1.0 to -2.5 cm W.G.). When the inlet pressure deviates from this range, the fan dampers are automatically adjusted accordingly. No attempt is made to shift scrubber sections in or out of service during increasing or decreasing load conditions. When the booster fan control is unable to keep the fan inlet pressure within the proper range (a situation which exists when there is a high pressure drop across the scrubber) the stack bypass damper is activated and gas flow is bypassed to the stack. Pressure and pressure drop information are monitored and recorded once per shift.

The recorculating slurry flow is manually controlled by maintaining the slurry pump motor amps within a predetermined operating range (e.g., 11-16 amps for recirculating slurry pumps on Unit 4). This is accomplished by adjusting the slurry pump discharge valve. No attempt is made to vary the slurry flow with gas flow or unit load. When the slurry pump motor amps are low (out of the control range), the Company tries to backflush the line. If no improvement is noted, a Station Service Request (SSR) is prepared to initiate proper corrective action. When a slurry pump is taken out of service the affected scrubber section is not removed from service. In the case of scrubber sections 1A, 3A and 3C which have one recirculating slurry pump each, the Company maintains a limited flow of water in the section by running the mist eliminator wash.

The blowdown of recirculating slurry is dictated by the scrubber water balance requirements. During each shift, an operator takes a sample of the slurry and measures settleable solids and pH. If the settleable solids concentration is higher than is to be allowed (i.e. 3.0 weight %), adjustments are made to the fresh water inlet rate and/or an SSR is initiated to check the drawoff lines and other potential trouble areas.

The operation of the mist eliminator wash system varies between units. On Units 1 and 3, a manual wash system has been installed. Once each shift, an operator sequentially opens the mist eliminator wash header valves and each header is left on for two minutes. On Unit 4, an automatic wash system sequentially operates each wash header for a period of three (3) minutes each shift. The timing of the mist eliminator wash system can be adjusted according to changes in operating conditions.

The control of the reheat system is based on maintaining the exit gas temperature within proper limits. The operator adjusts the set point on a pressure control valve which supplies steam to the reheat steam coils. At full load conditions, the control valves are set at maximum design values [21 kg/cm $^2$  (300 psi) for Units 1 and 3; 28-35 kg/cm $^2$  (400-500 psi) for Unit 4]. At low loads, the operator reduces the set point of the pressure control valve accordingly.

Maintenance practices that are reported in effect for the TCA scrubbers can be divided into two categores: daily or routine inspection checks conducted when the scrubbers are in operation; and major and/or minor repairs conducted when the scrubber is taken out of service.

Moreover, the differences between the Unit 4 scrubbers and those on Units 1 and 3 must also be recognized. Since the Unit 4 scrubbers are operated with three of the four equal-size modules in service, it is possible to rotate the operating modules every 3 to 4 months so that frequent internal maintenance can be performed. It is also possible to switch modules when one of the operating modules is performing poorly. This approach is not possible on Units 1 and 3.

Daily maintenance checks are performed on the slurry and recirculation pumps. These are checked for leaky packings, oil level, oil leaks, abnormal noise and vibration. Other pieces of equipment are monitored

by Plant Operations and include data collected from instrument readouts. When instrument values are outside of the appropriate range, Operations personnel initiate a Station Service Request (SSR), which details an equipment or instrument problem that is to be checked and repaired by Maintenance. Then, depending on the urgency of the SSR, immediate action is taken or action is scheduled for the next scrubber outage.

The bulk of the scrubber maintenance work is conducted during major or minor outages. In this case, a major outage is defined as a boiler unit outage exceeding four to six weeks, whereas a minor outage is any other time the scrubber is brought out of service. Besides repairing SSR items, the scrubber is inspected for ball wear and pluggage by solids. The balls are inspected and weighed periodically. If a representative number of balls (100 balls) have lost more than 20% of the weight of an equivalent number of new balls, then the balls are replaced with new balls. If there are a lot of broken balls or balls have migrated, then new balls are added. Solids pluggage is removed with a jackhammer, by manual washing, or through chiseling by hand as required. In general, if anything is found during an inspection that reduces efficiency, i.e., ball migration or missing spray nozzles, repairs are made at that time.

During a major outage, a complete overhaul program is undertaken. The guillotine gates are inspected, shafts are repacked and any item that is not working properly is repaired as time permits. The reheat coils (Units 1 and 3) are cleaned and tested for leaks. The duct work is inspected and cleaned. The recirculation system is inspected for wear, pluggage and failures. The mist eliminators and mist eliminator spray nozzles are inspected and cleaned. The vertical dividers and grid bars are inspected and repaired or replaced as necessary. The presaturator area and hoppers are cleaned and the pump screens are checked and repaired. The presaturator nozzles are inspected and replaced as

needed. The scrubber booster fans are checked and repaired as necessary. The fan bearing oil is changed and new shaft seals are installed. All soot blowers (Units 1 and 3) are inspected and checked for proper operation.

## SCRUBBER INSPECTIONS

## Unit 3

An inspection of the Unit 3 scrubber was conducted on August 5, 1977. The unit had been shut down for a scheduled outage in which a scrubber overhaul was planned. The last previous major scrubber overhaul had occurred in August - September, 1976. The results of the inspection were as follows.

#### Presaturator

The presaturator had large solids deposits in the area of the wet-dry interface. Section C had the largest accumulation of solids forming a layer as much as two meters deep. Sections A and B had solids accumulations of about one meter in depth. There were also solid formations projecting from the top spray nozzles. The solids on both the floor and ceiling of the presaturator formed a very hard deposit. The solid-cone presaturator spray nozzles that were inspected did not appear to be plugged or covered with solids but in some cases solids had accumulated around the nozzle, possibly restricting the spray coverage.

## Scrubber Sumps

The scrubber sumps had accumulated solids at the bottom but the solids level did not appear to reach the spray pump intake lines. All

the pump intake lines have screens. Broken balls had accumulated on most of the screens, but none of the screens were completely plugged.

## Packing Stages

There was evidence throughout the mobile ball sections of ball migration and poor gas flow distribution. This typically appeared as maldistribution of balls and deposition of solids on the bottom of the packing stages. Generally, it appeared that the majority of the gas tended to flow from the presaturator section up the "back side" of the scrubber. (The "back side" of the scrubber is the east side or side opposite the presaturator section.) This could have resulted from the high velocities resulting from restriction in the presaturator section

In section 3A, the ball migration problem was very evident, since it was possible to look up from the eastside of the scrubber sump to the mist eliminator blades because of ball migration. In all three stages, the layer of balls (nominally at 20-30 cm depth) varied from zero thickness for the east one-third of the stage to as much as 1 m near the west end of the scrubber. There did not appear to be any significant solids buildup in the 3A section.

Section 3B showed a less consistent pattern of ball migration. In the first stage (which was made up of twelve wire-grid compartments), two of the compartments on the east side and one compartment on the west side had less than one layer of balls, whereas two middle compartments had 0.5 m and 1 m ball depths. A couple of breaks were noted in the grids separating the far east and middle compartments where ball migration could occur. The second stage had a uniform distribution of balls. The third stage of this section, however, had poor distribution. The south one-third of the stage had less than one layer of balls and the depth of the balls became progressively deeper toward the southwest

corner of the scrubber stage. The only significant solids buildup noted was on the bottom of the first stage. Scaled areas covering about one-fourth and one-sixteenth of the cross-sectional area were seen in the middle of the bottom stage and of the northwest corner of the first grid, respectively.

Section 3C had the most significant solids accumulations. Approximately three-fourths of the bottom of the first stage was scaled over. A large mass (0.5-1 m diameter) of a very hard deposit of accumulated solids and balls was found in the east compartment of this stage. The bottom of the west compartment on the second stage was also scaled over. The third stage did not have significant solid deposits. Ball migration problems were not as evident in this section. The first stage had less balls on the east side of each compartment, varying by as much as 10 to 20 cm (4 to 8 in), while the second and third stages had reasonably good ball distribution.

The physical condition of the ball grids and scrubber liner was reasonably good. The most prevalent ball types found were a solid black rubber ball and a hollow green plastic ball. Less than 5% of the balls appeared to be broken or grossly worn. The support grids were intact with the exception of those noted above. However, it was observed that at the point of contact, overlapping grid wires apparently were "cutting" into one another. In many cases for the first stage, as much as 0.5 to 1 cm wear was observed. This is apparently due to vibration and movement of the strands when the scrubber is in service. The rubber scrubber liner, although found to be in fairly good condition overall, had some bubbles or blisters where the liner had popped off the steel underneath, especially in section 3B stage 1 and section 3C stage 2. There were also a couple of areas in section 3B stage 1 and section 3C stage 1 where pieces of the liner had come off, exposing the underlying metal.

## Recirculating Slurry Pumps and Nozzles

The impellers of the slurry pumps were inspected from the suction side of the pump which had been opened for each pump. The rubber-covered impellers seemed to be intact and in good condition. Inspection was also made of the recirculating nozzles. Several of the nozzles were plugged: 7 of 14 in section 3A, 9 of 42 in section 3B and 7 of 14 in section 3C. The material causing the pluggage was mostly 0.6 to 1.0 cm rubber liner, possibly eroded from the recirculating lines. The orifices of several of the recirculating slurry nozzles were measured and did not show significant wear.

## Mist Eliminator

The mist eliminator area was observed to be in very good condition with the exception of wash nozzle which had broken loose from its header. The chevrons, which were 316 SS, had very little solids deposit and did not show any gross signs of pitting or corroding. The alignment of the mist eliminator blades was also good.

## Reheater Area

At the time of the inspection, only sections 3A and 3C had reheater coils in place. The coils were reasonably clean. There was a thin solids layer (<2 to 4 mm) on most of the coils with significant deposits (1 to 2 cm) found only on the bottom of the tubes in the lowest tube bundles. No pitting of the 316 tubes was observed. There was evidence of severe rusting and corrosion of the ducting which surrounds the coils. In section 3B, in which the reheater coils had been absent, there were several holes in the reheat duct area caused by excessive corrosion.

#### Ductwork and Dampers

The ductwork downstream of the scrubbers, the isolation dampers, and the bypass ductwork were inspected. The downstream ductwork was badly rusted for all three sections and large pieces of the corroded carbon steel ductwork could be easily pulled off by hand. The ductwork was set and in section 3C there was a wet solids accumulation as deep as 10 cm throughout. The inlet guillotine gates were closed and from a limited inspection appeared to be in good condition. The outlet dampers were rusted and corroded. None appeared capable of providing a tight seal and the bottom louver blade in 3C was mired in solids in a half open position (the other blades of the damper were closed). The bypass ductwork was found to have extensive, very hard deposits. The buildups were as much as 1 m deep throughout.

The Company representative who accompanied NEIC personnel on the inspections was knowledgeable of the problems present and how they would be fixed.

A follow-up inspection of the Unit 3 scrubber was conducted on August 24, 1977. The purpose of the inspection was to determine the thoroughness of the scrubber overhaul in light of the problems noted from the previous inspection and to inspect the new mist eliminator assemblies that were to be installed in sections 3A and 3C.

The solids in the presaturator and scrubber sump had been thoroughly removed. The pump intake lines were clear. The recirculating slurry nozzles had been cleaned. The new mist eliminator assemblies were in place and appeared to be properly installed. The reheat ductwork in 3B had had a plate installed to cover the corroded areas. The bypass ductwork area had been thoroughly cleaned out.

The ball sections had not been cleaned and balls had not been redistributed or added, although the Company representative stated that this would be done prior to startup. The areas where the scrubber liner had fallen off were not repatched.

## Unit 4

A very limited inspection of the C section of the Unit 4 scrubber was conducted on August 14, 1977. The boiler unit had been brought down to inspect a leak in the boiler tubes. The only areas open for inspection were the presaturator area and the scrubber sump.

A layer of soft solids approximately 0.5 m deep was present on the presaturator floor. The presaturator nozzles, however, were clear and, in general, the presaturator area was in good shape. A small section (approximately 2 ft square) of rubber liner had come loose from one wall just downstream of the presaturator spray nozzles. The scrubber sump on the north side had its pump intake screen clogged with balls. The other two sumps were filled with water and their intake screens were not visible. The bottom of the first stage of the scrubber had significant solids buildup across approximately 15% of the cross-section. These deposits were noted on the presaturator side of the scrubber.

#### UPSET REPORTING

As discussed in the previous section, Public Service Company of Colorado has reported day-to-day malfunctions of the particulate control equipment. The data generated in these upset reports were used to review scrubber availability and the major sources of scrubber malfunctions.

## Availability

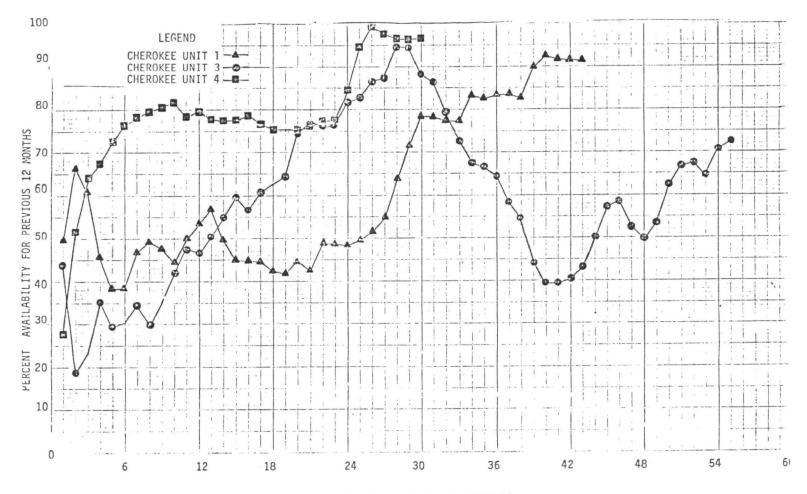
Availability, as reported by PSCC, is defined as:

Availability = Scrubber hours operation - hours boiler burning 100% gas
Boiler hours operation - hours boiler burning 100% gas

A distinction is also made as to how "scrubber hours operation" is defined for each unit. For the Unit 1 and 3 scrubbers, the scrubber is considered to be operating when all sections of the scrubbers are in service. For the Unit 4 scrubber, the scrubber is considered to be operating if 3 of the 4 sections are in service.

Figure 9 shows cumulative 12-month availabilities for the scrubbers at Cherokee Station based on the above definitions. The data include the time period from when the scrubbers were initially put in operation until April 1977, after which time this recording method was discontinued.

Various trends can be identified from Figure 9. All scrubbers appear to go through an initial start-up/shake-down period when scrubber availabilities are low. As the initial problems were solved, availabilities gradually increased until a maximum point (>90% availability) was reached, typically 30 to 40 months after initial startup. Thereafter, the curves appear to take on more individual pattern reflecting the differences between units. The availability curve for Unit 3 began to sharply decrease after reaching the maximum, while the curve for Unit 4 has constantly remained above 90% availability. Unit 1 scrubber availability was not plotted for any significant period after reaching its maximum point but, because of its similarities to Unit 3 (i.e., no spare scrubber sections, limited weather enclosure, direct reheat, etc.), it would be expected to experience a dropoff in availability similar to that of Unit 3.



MONTHS AFTER SCRUBBER STARTUP

Figure 9: Cumulative Twelve Month Scrubber Availabilities, Cherokee Station, Public Service Company of Colorado

Table 13 shows the average availability for each year of scrubber operation. The overall averages for scrubber availabilities are:

Unit 1 65% Unit 3 63% Unit 4 84%

#### Equipment Upset Data

The upset reports provide information on equipment component malfunctions but the reports are not sufficiently comprehensive to allow a definitive scrubber equipment component evaluation to be made. The data do not show causes of failures nor do they allow differentiation between primary and secondary effects, i.e. whether breakdowns were caused by the equipment component itself or were associated with disturbances from other components. Furthermore, a number of months of data were either not available or too imcomplete to be included in this analysis. Finally, it was impossible to properly distribute downtime to equipment when more than one component required repairs during a given outage. As a result of these factors, only a broad definition of scrubber related problems is possible.

Table 14 shows estimates of relative contributions of various scrubber subsystems to scrubber downtime for each unit. The estimates are expressed as a percentage of the reduction in scrubber availability due to major areas for each 12-month period. The estimates are based on scrubber upset reports prepared by PSCC.

The most illuminating observation from Table 14 is that there are significant differences in problems causing outages between the various units. Major problem areas for the Units 1 and 3 scrubbers are the

Table 13

SUMMARY OF PERCENT SCRUBBER AVAILABILITY<sup>†</sup>

BY YEAR OF OPERATION

CHEROKEE STATION

PUBLIC SERVICE COMPANY OF COLORADO

Year	Unit 1 %	Unit 3 %	Unit 4 %
1	52	47	80
2	48	81	83
3	82	64	93*
4	82 90 <sup>††</sup>	49	NA
5	NA	80 <sup>††</sup>	NA
Average**	65	63	84

<sup>+</sup> Availability =  $\frac{Scrubber\ hrs\ operation\ -\ hrs\ boiler\ burning\ 100\%\ gas}{Boiler\ hrs\ operation\ -\ hrs\ boiler\ burning\ 100\%\ gas}$ 

<sup>††</sup> Based on 7 months data.

<sup>\*</sup> Based on 6 months data.

<sup>\*\*</sup> Average is calculated by averaging the availabilities from each year.

Table 14

MAJOR REPORTED PROBLEM AREAS CAUSING SCRUBBER MALFUNCTIONS CHEROKEE STATION
PUBLIC SERVICE COMPANY OF COLORADO

	Unit 1	Unit 3	Unit 4
Year 1	Oct. 1973-Sep. 1974	Oct. 1972-Sep. 1973	Nov. 1974-Oct. 1975 b
	Data not available	Data not available	Booster Fans 92% Isolation Dampers 3% Reheater 3% Other 2%
ear 2	Oct. 1974-Sep. 1975 <sup>c</sup>	(Oct. 1973-Sep. 1974)	Nov. 1975-Oct. 1976
	Reheater 51% d Scrubber, Internals 21% d Booster Fans 21% Recirculating Slurry 3% d Other 6%	Data not avaılable	Booster Fans 90% Reheater 1% Other 9%
ear 3	Oct. 1975-Sep. 1976	(Oct. 1974-Sep. 1975) <sup>c</sup>	Nov. 1976-Oct.1977 <sup>c</sup>
	Recirculating Slurry 58% Scrubber, Internals 33% Reheater 6% Other 3%	Reheater 42% Scrubber, Internals 13% Recirculating Pumps 4% Recirculating Slurry 2% Other 39%	Booster Fans 79% Recirculating Slurry 13% Isolation Dampers 3% Recirculating Pumps 2% Reheater 1% Other 2%
ear 4	Oct. 1976-Sep. 1977 <sup>†</sup>	(Oct. 1975-Sep. 1976)	NA NA
	Recirculating Pumps 67% Scrubber, Internals 27% Recirculating Slurry 4% Booster Fans 1% Other 1%	Reheaters 64% Booster Fans 14% Scrubber, Internals 12% Recirculating Slurry 9% Other 1%	
Year 5	NA	(Oct. 1976-Nov. 1977) <sup>f</sup>	NA
		Recirculating Slurry 65% Reheater 21% Recirculating Pumps 12% Other 2%	

a All data are estimates of scrubber equipment downtime taken from PSCC upset reports expressed as a percent of annual scrubber downtime.

**b** Based on 5 months reported data.

c Based on 6 months reported data.

d "Scrubber, internals" includes the scrubber grids, scrubber liner mobile balls and recirculating slurry nozzles.

<sup>• &</sup>quot;Recurculating slurry" system includes the slurry drawoff, scrubber slurry hopper and recirculating slurry piping.

<sup>1</sup> Based on 7 months reported data.

reheaters, scrubber internals, recirculating slurry system and recirculating slurry pumps. The major problem areas for the Unit 4 scrubbers are the scrubber booster fans.

The major problem areas for the Units 1 and 3 scrubbers in apparent order of importances are: reheaters, scrubber internals, recirculating slurry system and recirculating slurry pumps. The reheaters have resulted from corrosion and pluggage of the in-line steam coils. Typical problems with the scrubber internals include inspection, repair, and replacement of scrubber grids, scrubber liner, and mobile balls. Difficulties in the recirculating slurry system include repairing leaky recirculating slurry piping, unplugging the slurry drawoff line and removing slurry buildups in the scrubber hoppers. The recirculating slurry pump problems appear to mainly be due to bearing, packing and motor difficulties.

The major upset problems encountered in the Unit 4 scrubbers are almost exclusively due to scrubber booster fan malfunctions, and lack of other major problems can be attributed to improved design features. The scrubber booster fans are air foil fans which have been highly subject to erosion from fly ash carried over from the ESP's. Unit 4 is operated with one module as a spare, therefore maintenance to scrubber internals, recirculation pumps, piping, etc. can be routinely scheduled. Furthermore as critical scrubber problems occur and require repair, the affected scrubber section can be replaced with the spare section with relatively short-term opacity excursions. Unit 4 scrubbers also have indirect reheaters which are less subject to corrosion and plugging than are direct in-line heaters. Unit 4 scrubbers are totally enclosed, thus preventing significant freezing problems.

#### EVALUATION OF SCRUBBER PERFORMANCE AND OPERATION

The performance of the TCA scrubber were evaluated using the operating data collected during the study period, and data from previous stack tests, efficiency tests, etc. Prior to this analysis, Meteorological Research, Inc.<sup>4</sup> prepared an analysis of the particulate removal performance of the Cherokee Unit 3 scrubber. Their results and the results of this study are discussed in this section.

Table 15 shows the particulate removal performance data for the scrubbers on each unit based on recent stack tests and published reports. For each unit, a comparison is made between design values and actual values of grain loadings, efficiencies and powerplant load.

The scrubber particulate loadings are important in evaluating compliance with particulate regulations. In reviewing the data, it is noted that, with one exception, the outlet grain loadings show compliance with the process weight regulation requiring particulate emissions to be less than 0.1 lb per MM Btu heat input. The one exception is the average outlet loading of 0.14 g/std m<sup>3</sup> (0.06 gr/SCF) taken from data reported by MRI for November 1974. This loading may have been in excess of 0.1 lb per MM Btu, but could not be determined since the circumstances under which this data was taken could not be evaluated.

Opacity meter data [Table 15] and visible emission observations are also indications of outlet particulate loadings. However, it is significant to note that small particles contribute proportionately more to high opacities than to high particulate loadings. As a result, opacity and outlet particulate loadings are not directly related. The wide variations in opacity data are important because they reflect the wide fluctuations in scrubber operations.

Table 15

ACTUAL AND DESIGN PARTICULATE REMOVAL DATA FOR TCA SCRUBBERS

CHEROKEE STATION

PUBLIC SERVICE COMPANY OF COLORADO a

Unit	Unit Load	No. Oper. Scrubber	Scrubber Inlet Particulate Loading		Part	rubber Ou iculate Lo	oading	Particulate Visible Removal Eff. Emiss. Observ.		
	MW	Sections	g/std m <sup>3</sup>	gr/ft <sup>3</sup>	g/std m <sup>3</sup>	gr/ft <sup>3</sup>	1b/mm Btu	%	% Opacity	
Unit 1										
Design	115	3.	1.80	0.80	0.046	0.02	-	97.5	-	
Actual (PSCC)	94	$\frac{3}{2}$ d	_ g	-	0.076	0.03	0.069	-	-	
Unit 3										
Design	170	3 .	0.69	0.30	0.046	0.02	-	93	-	
Actual (PSCC)	163	2 <b>d</b>	-	-	0.069	0.04	0.069	-	5-60	
Actual (MRI)	160	3	0.87	0.38	0.14	0.06	-	84	-	
Actual (MRI) f	160	3	1.58,	0.69	0.097	0.04	-	94	-	
Unit 4										
Design	375	4.	1.60	0.70	0.046	0.021	-	97	-	
Actual (PSCC)	345	<sup>4</sup> <sub>3</sub> <b>d</b>	_ h	-	0.047	0.02	0.050	-	5-40	

a Data in this table is taken from references, 1, 2 and 4. Actual data taken by Public Service Co. of Colorado is shown as (PSCC) and actual data taken by Meteorological Rearch, Inc. is shown as (MRI).

b Particulate removal efficiencies are not shown for actual PSCC data since inlet and outlet particulate loadings were not taken under similar conditions.

c Visible emission observations are from data taken by EPA-NEIC during July-August, 1977 and represent a wide range of operating conditions. These data are included in Appendices A and B.

d Sections 1A (Unit 1), 3C (Unit 3) and 4D (Unit 4) were not in service during these tests.

e This data is based on tests reported in reference 4 for the dates 11/7/74 - 11/19/74.

<sup>†</sup> This data is based on tests reported in reference 4 for the dates 12/10/74 - 12/12/74.

g Actual PSCC data incomplete.

h Actual PSCC data not reported because recent tests not available.

The data in Table 15 show that the actual scrubber outlet loadings for Units 1 and 3 exceed the design values. To investigate the cause and significance of this observation, it is necessary to consider some of the factors which affect the outlet particle loading, i.e., the scrubber inlet particulate loading and the scrubber's particulate removal efficiency. The outlet particulate loading is related to the inlet loading and scrubber efficiency as follows:

Outlet particle loading = Inlet particle loading x (1 - efficiency)

The inlet loading to the scrubber is dependent on a number of factors including: the coal that is being fired, the boiler operation, the mechanical collector/ESP operation and addition of conditioning agent. With such a variety of factors, it is not unexpected that there are reported differences in inlet particulate loadings between units and between the same unit at different time periods. A more significant observation is that actual scrubber inlet particulate loadings can and do significantly exceed design values. Although the scrubbers have some inherent capability for removing excess particulate, it is not known how large an excess can be handled or for how long. In Section V, information reported in PSCC upset reports has shown violations of the 20% opacity standards (as measured by the opacity meters) due to ESP malfunctions. Since these violations occurred when the scrubber was in service, it must be concluded that the scrubber outlet particulate loading can exceed standards even when the scrubber is not in an upset condition. Therefore, it is important that the mechanical collector/ESP efficiency be improved and maintained at optimum conditions. Operation of the ESP's at 40 to 60% efficiency is not acceptable.

Very limited scrubber efficiency data is available for the scrubber installations at Cherokee. From data developed from other mobile bed

contactors it is expected that particle collection efficiency will be dependent on gas flow, liquid flows, and state of motion of the mobile contactor beds as indicated by pressure drop. In addition, it is important to recognize that nonuniformities such as gas flow imbalances, liquid flow imbalances, solids pluggage, etc. play an important role in determining the particulate removal efficiency of large-scale scrubber installations. These are reviewed in the following discussion.

The only available efficiency data for the Cherokee Station is presented in the MRI study. In their initial set of tests (average particulate removal efficiency of 84%), they found flow and outlet particle loading imbalances between sections of the Unit 3 scrubber. To correct this, the scrubber was shut down, some of the mobile bed packing was redistributed and a clogged reheater was partially cleaned. Efficiency tests conducted subsequent to this shutdown showed improved efficiency (average efficiencies of 94%). MRI also analyzed the scrubber outlet particulate and found a high concentration of soluble components indicating that liquid entrainment was occurring to a significant degree. MRI concluded that the scrubber performance data they obtained reflected specific scrubber conditions and that general scrubber particulate removal efficiency correlations could not be developed from the data.

The scrubber operating data accumulated during the study also relected a wide range of operating conditions. Table 16 shows gas flow, liquid flow and pressure drop data taken during the study. The data was taken from instrumentation located in the boiler control rooms.

The gas flow data are shown in terms of scrubber booster fan motor amps. These data present little basis for analysis since fan motor amps are also a function of pressure drop and fan/motor efficiencies. It does appear, however, from data taken for Unit 4, that the gas flow is

Table 16 SCRUBBER OPERATING DATA CHEROKEE STATION PUBLIC SERVICE COMPANY OF COLORADO

	No. of		Scrubber		Recirculati	ng		Pressu	re Dro	ס
Unit	Observa-	Load	Booster Fan		Slurry Pum	p	System	Mobile	Mist	Reheater
	tions	WM	Amps <sup>†</sup>	1	2 Amps	3	Unit	Beds cm	Elim. H <sub>2</sub> O	
Unit 1	42	84-119	130-150(A) 125-153(B)							
Section A Section B				0-24 21-22	NA <sup>††</sup> 20-28.5	NA 0-24	24-50 24-50	7-23 10-24	2-13 1-7	5-28 2-14
Unit 3	4	110-152	190-220(A) 180-220(B)							
Section A Section B Section C				20-24 25 NO	NA 22 NA	NA 20-21 NA	27-34 27-34 27-34	15-23 14-18 8-10	0-1 3-4 1-2	2-3 NO <sup>†††</sup> 1-3
Unit 4	44	253-360								
Section B Section C Section D			200-250 220-240 225-245	12.5-13 11-13 11-14	15-16 0-14 11.5-14	12-14 12.5-15 12-14	NO 23-58 18-38	10-29 11-38 10-25	2-3 1-5 5-7	NO 1-5 NO

t Fans on Unit 1 and 3 scrubbers provide common flow to the scrubbers.

<sup>††</sup> Not applicable. ††† NO = indicating meter not in operation.

reasonably well distributed between the three scrubbing sectons. No observation can be made for flow distribution between sections in the Unit 1 and Unit 3 scrubbers since these do not have individual fans for each section.

More conclusive data on gas flows are shown in Table 17. These data were taken from recent stack tests performed for the units and are compared against design values. It is noted that in all cases of reduced scrubber section operation the superficial scrubber velocity is 4.8 to 5.8 meters per second (mps) as compared to a design maximum of 40 mps. Furthermore, even under "normal operation" for Units 3 and 4 (4.7 and 4.8 mps superficial velocity, respectively), the design maximum velocity is exceeded. Although this has apparently not affected compliance with the process weight regulation, it can result in improper bed fluidization and high liquid entrainment. Above superficial gas velocities of about 4.0 mps, it has been shown that pressure drops and bed expansion increase to the point where mobile spheres are held up at the top of the retaining grids. Liquid entrainment also increases when gas velocities increase, and can become severe when the velocities are 4 mps and higher. 8

The distribution of gas flow between scrubber sections on Units 1 and 3 could not be determined from operating data accumulated during this study. However, field measurements made by MRI during 1974 do show the magnitude of typical gas flow variations. Figures 10 and 11 show sets of gas velocity profiles taken before and after the Unit 3 scrubber had been cleaned, balls redistributed, and reheater partially cleaned. Theoretically, the profiles should be reasonably close to one another, but as shown in the curves, the average velocities between sections can vary by as much as 3 to 1. This type of variation indicates that even if the bulk scrubber gas velocity (as calculated in Table 17) is within proper design limits, the velocities within each section may be outside the range required for proper operation of the beds. Overall particulate removal may be reduced and liquid entrainment can be significantly increased.

Table 17

DESIGN VS ACTUAL VALUES OF SCRUBBER SUPERFICIAL VELOCITIES

AND LIQUID-TO-GAS RATIOS (L/G)

CHEROKEE STATION

PUBLIC SERVICE COMPANY OF COLORADO

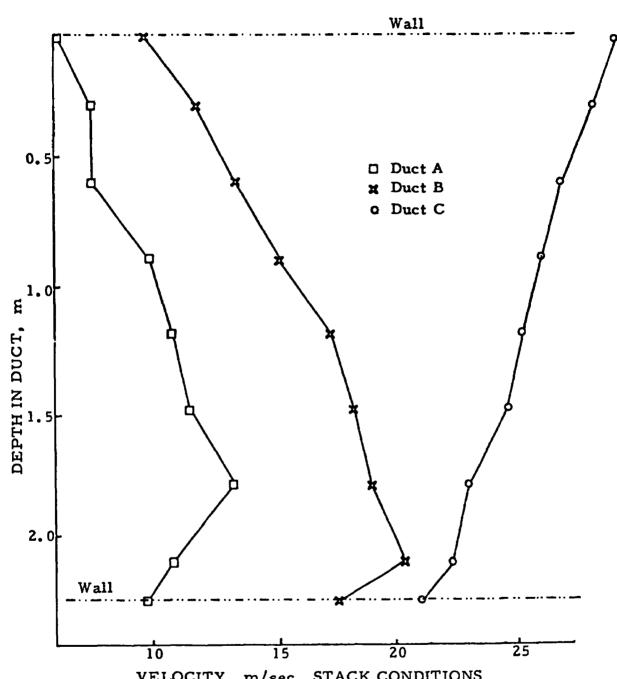
	Actual Co m <sup>3</sup> /hr	onditions ft <sup>3</sup> /min	Scrubber Op Velocity (mps)	eration <sup>††</sup> L/G (1/m <sup>3</sup> )	Reduced Scrubb Velocity (mps)	er Operation L/G (1/m <sup>3</sup> )
Unit 1 Design Actual	730,000 730,000	430,000 430,000	4.0** 4.0**	7.7 7.7**	4.0 5.4	7.7 4.3
Unit 3 Design Actual	850,000 1,000,000	500,000 590,000	4.0 4.7**	7.0 5.9**	4.0 5.8	7.0 3.9
Unit 4 Design Actual	2,200,000 2,100,000		4.0 3.7	7.9 8.6	4.0 4.8**	7.9 4.8**

<sup>†</sup> Data is representative of scrubber conditions (52°C or 125°F) at full load and is taken from references 1, 2 and 4. Design flowrates are design maximums. Actual flowrates are calculated from representative stack test and precipitator outlet data.

tt "Full scrubber operation" assumes all scrubber sections in service.

<sup>\* &</sup>quot;Reduced scrubber operation" assumes one section of the scrubber out of service as follows: 1A in unit 1, 3A or 3C in unit 3, any one section in unit 4.

<sup>\*\*</sup> Indicates normal operation for scrubber unit.



VELOCITY, m/sec, STACK CONDITIONS
Figure 10
Velocity Profiles for Outlet Ducts Before Cleaning
Cherokee Unit 3 Scrubber (11/18/74)
Cherokee Station
Public Service Company of Colorado 4

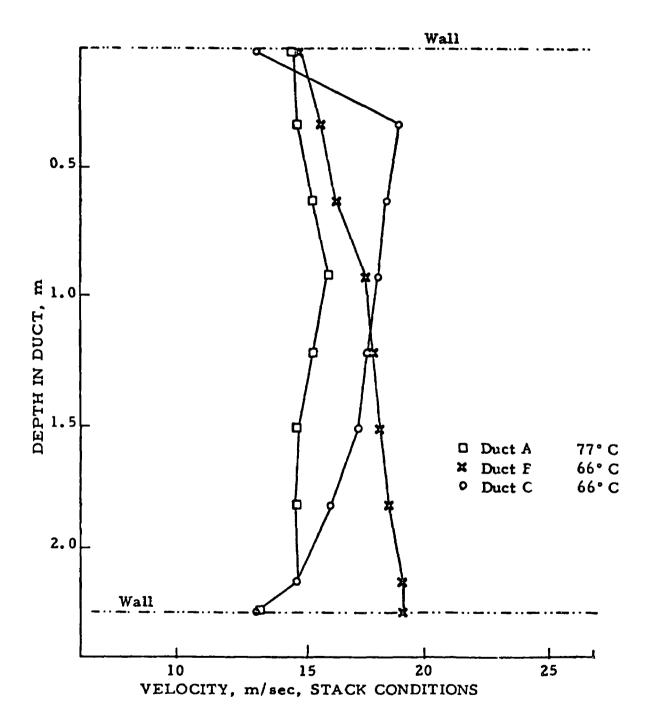


Figure 11. Velocity Profile for Outlet Ducts of Unit 3 Scrubber (12/10/74) After Cleaning Cherokee Station Public Service Company of Colorado<sup>4</sup>

Table 13 also shows variations in liquid-to-gas (L/G) ratios under various full load conditions. Once again, there is a significant departure from design L/G values under "normal" scrubber operations (Units 3 and 4 only) and under "reduced" scrubber operations (all Units). In general, decreasing the L/G ratio (with constant gas velocity and pressure drop) is expected to reduce particulate removal; however, no precise quantitative relationships could be developed from available literature to indicate the expected decrease in particulate removal.

More significant observations of liquid flow rate variation are shown in Table 12. For sixteen of the observations (6 days) on Unit 1, the pump in the single-pump scrubber section (Section 1A) was not in service. Obviously, under these conditions, the particulate removal in that section of the scrubber is very much reduced. Furthermore, prolonged exposure of the scrubber internals to these conditions (where scrubber temperatures approach 90°C (200°F) even with some water introduced continuously through the mist eliminator nozzles, may cause the rubber liner to blister and creep and cause deformation of the plastic sphere.

Minor instances of 1 to 2 day's duration were also observed where one of the three pumps in a scrubbing section was out of service. These cases are not as critical since scrubber internals are not severely affected and overall particulate removal may not be significantly reduced depending on  $\Delta P$  and gas velocity values.

When the current to a given recirculating pump motor is below levels that, by experience, indicate pump or line problems (11 amps on Unit 4 pumps and 20 amps on Units 1 and 3 pumps) an immediate investigation is reportedly made. Typically the problem is one of a plugged suction line and backflushing is initiated. However, as noted in the Unit 3 scrubber inspection, plugging of recirculating spray nozzles may also be occurring and this cannot be detected without an internal

scrubber inspection. Besides reducing scrubber liquid flow, plugged nozzles can cause liquid maldistribution and, if extreme, can lead to improper fluidization of the scrubber bed.

The pressure drop ( $\Delta P$ ) across the mobile bed could be expected to be a primary indicator of the particulate removal performance and of the conditions within the bed such as fluidization, gas channeling, etc. Particulate removal performance as a function of  $\Delta P$  was studied by MRI with a limited amount of data. Table 18 shows the results of that study in which no correlation could be found between particulate removal efficiency and pressure drop. Instead, as noted previously, MRI attributed the variations in particulate removal efficiency to numerous operating factors in existence at the time of their tests.

The pressure drop recorded across the mobile beds should also provide an indication of the conditions within those beds. It was stated by the Company that pressure drops of less than 15 to 20 cm (6 to 8 in) water column (W.C.) at full load are an indication that gas channeling is occurring within the scrubber. At the other extreme, the manufacturer's operating limitations state that the scrubber should not be operated above 30 to 35 cm (12 to 14 in) W.C. due to bed expansion and problems caused by mobile packing held up against the retaining grid. This latter condition can result in flooding within the scrubber. Instances when pressure drops were outside of the lower and upper limits are shown in Table 19.

Interpretation of the pressure drop data in Table 19 is not straight forward. The data indicate, that channeling was occurring in Unit 1 (sections 1A and 1B) and Unit 3 (section 3C), whereas flooding may have been occurring in Unit 4 (section 4A). However, to put the data in proper perspective, it is necessary to compare the Unit 3 pressure drop data with subsequent inspection observations that were made when the scrubber was taken out of service. These inspections indicated that the

Table 18 SCRUBBER COLLECTION EFFICIENCIES 4 UNIT 3 CHEROKEE STATION PUBLIC SERVICE COMPANY OF COLORADO

					Sr.C1	ION A		T	snci	10\ B			SI.CI	10.7 C	
DATE	LOAD	Oa	OUTLLT GAS FLOW ACTUAL	& P SYSTEM	ΔP BCD	ΔP MIST ELIMIN.	EFF.	AP System	ΔP BŁD	ΔP MIST ELIMIN.	EFF.	2 P S) STEM	∆P BED	AP MIST ELIMIN.	EFF.
	mw	Percent	m³/hr	cmli <sub>2</sub> O	cmH₂O	cmH <sub>2</sub> O		cmH <sub>2</sub> O	cmH2O	cnill <sub>2</sub> O		cmll <sub>2</sub> O	cnill <sub>2</sub> O	cmH2O	
11/20	166	3.6	a	41	9.9	0.76	מא	45	25	2.5	84.7	46	24	8.3	ND
11/21	164	3.4	a.	39	9.6	0.76	מא	43	18	1.8	89.9	44	20	5.1	ND
12/10	157	3.4	9.47 × 10 <sup>5</sup>	36	15.2	1.7	96.3	41	20.8	2.5	92.6	41	18, 5	3.8	86. 9
12/11	160	3.0	10.2 × 10 <sup>6</sup>	38	14.7	1.5	96.4	42	22.1	3. 2	93.2	44	22.4	2.5	96.7
12/12	160	2.6	8.78 × 10 <sup>8</sup>	38	14.7	1.8	79.6	44	22.9	2.5	93, 1	46	24, 1	3.8	92.1

<sup>\*</sup> Full velocity traverses were not taken.

\*\*b The control room data were incomplete. Interviews, data from other days and the log book were used to supplement available information.

Table 19 PRESSURE DROP FOR SCRUBBER MOBILE BED SECTIONS DURING FULL LOAD CONDITIONS CHEROKEE STATION PUBLIC SERVICE COMPANY OF COLORADO

Unit	Load	Total No. of Observations	No. of Observations ΔP bed <15 cm W.C.	No. of Observations ΔP bed >35 cm W.C.
Unit 1	≥115 MW			
Section A Section B		22 22	18 <sup>†</sup> 12	0 0
Unit 3	≥145 MW			
Section A Section B Section C		3 3 3	0 0 3	0 0 0
Unit 4	≥350 MW			
Section B Section C Section D		26 26 26 <sup>++</sup>	5 0 2	0 16 1

 $<sup>\</sup>dagger$  Includes 10 observations when 1A1 recirculation pump was out of service.  $\dagger$  Includes 2 observations when section 4D  $\Delta P$  bed instrumentation was out of service.

low pressure drop in section 3C was due to low flow resulting from heavy solids accumulation in both the presaturator and scrubber bed. On the other hand, the inspection revealed that gas flow channeling was existing in other sections (especially section 3A) but was not indicated from pressure drop instrumentation. This may have occurred because the sections were forced to handle higher than design gas flow rates.

It should also be recognized that the type of packing also influences the pressure drop. Studies performed at West Virginia University¹ showed that pressure drop was, in part, dependent on the physical properties of the packing (e.g. shape, weight, size). With the different types of balls being used in the scrubbers and the added problems of ball migration, interpretation of pressure drop measurements is further complicated.

The operating data collected during the scrubber performance evaluation is not conclusive. It is evident that the scrubber sections are typically operated at gas velocities, liquid flowrates and pressure drops outside of design ranges. It is also evident that scrubber instrumentation does not consistently indicate when internal scrubber problems, such as ball migration, gas flow channelling, and solids deposition, are occurring.

#### EVALUATION OF SCRUBBER SYSTEM RELIABILITY

From the previous discussion, it is apparent that even if 100% of the gas is flowing through the TCA scrubber, the scrubber may not be capable of meeting applicable particulate regulations. Scrubber availability is, therefore, not an adequate measure of scrubber performance. Instead, it is necessary to introduce the term "reliability". Reliability, as used in this report, will be defined as: the percent of time the boiler is on-line that the particulate control systems are operating and meeting applicable particulate regulations.

To adequately review reliability in light of the existing Cherokee Station scrubber operation, it is important to consider the individual equipment components which appear to have the largest impact on scrubber reliability. In their May 1975 study, which appears as an appendix to the MRI evaluation, Stearns-Roger, Inc. identified components presenting major maintenance problems for the Unit 3 scrubber. Those problems and problems which appear to contribute significantly to current scrubber reliability deficiencies are shown in Table 20. As can be seen, most of the problems identified in the earlier study are still present. The major reliability problem components are reviewed individually in the following discussion.

#### Wear of Mobile Bed Contactors

Prior to this survey, PSCC had extensively tested balls of varying compositions and designs and indicated that the ball wear problem was their major maintenance item. As the balls were exposed to turbulent conditions in the scrubber, they would wear out, break apart, dimple, etc. The fluidization of the bed was disturbed and balls migrated to cause flow channeling in the scrubber and wear problems in other components of the scrubber system. Obviously, the particulate removal ability of the scrubber was then reduced and the scrubber had to be taken out of service to redistribute balls, replace balls, etc.

PSCC has evaluated balls made of a number of different materials including polyethylene, polypropylene and thermoplastic rubber but has now stated that a polyethylene ball of unique construction provides what they consider to be adequate resistance to wear. (A ball providing "adequate" resistance to wear is expected to have a useful life of about one year.) The ball is a hollow green-colored sphere manufactured by Puget Sound Trading Co. The unique feature of the ball is that it has crimps or indentations which tend to give it greater strength. Reportedly, the indentations also cause the ball to acquire a characteristic

Table 20

# PROBLEM AREAS IDENTIFIED IN SCRUBBER RELIABILITY EVALUATIONS CHEROKEE STATION PUBLIC SERVICE COMPANY OF COLORADO

Major Scrubber Problem Areas Identified in May 1975 Study, Unit 3	Scrubber Reliability Problem Areas  of Major Significance Identified in NEIC Study  Unit 1  Unit 3  Unit 4						
Breakage of mobile bed	Wear of mobile bed	Wear of mobile bed	Wear of mobile bed				
contactors	contactors	contactors	contactors				
Migration of mobile bed contactors	Migration of mobile bed contactors	Migration of mobile bed contactors	Migration of mobile bed contactors				
Guillotine dampers	Isolation dampers	Isolation dampers	Isolation dampers				
Recirculation pumps	Recirculation pumps	Recirculation pumps	-				
Reheater Section	Reheater Section	Reheater Section	Reheater Section				
Rubber lined piping	Recirculation piping and nozzle	Recirculation piping and nozzle	Recirculation piping and nozzle				
Presaturator buildup	Presaturator buildup	Presaturator buildup	-				
Mist eliminators	Mist eliminators	Mist eliminators	Mist eliminators				
Stack damper interlock system							
Recirculation system venturi flow meter							
Scrubber booster fan bearings	Scrubber booster fan	Scrubber booster fan	Scrubber booster fan				
Weather related problems	Weather related problems	Weather related problems	-				
	Outlet ductwork	Outlet ductwork	Outlet ductwork				

spin. This, in turn, results in ball wear in one or two spots rather than at a number of points from which a ball can break into pieces.

The green polyethylene ball is still not ideal and PSCC indicates that they continue to search for an improved design. When the green ball does wear it fills with scrubber slurry and falls to the bottom of the stage. Proper turbulent contact is then difficult to maintain within the scrubber if a significant number of the balls are worn.

No matter what ball is used, operating the scrubber with large flow imbalances is still a significant problem. Certain portions of the scrubber are exposed to high ball wear whereas other areas may see minimum or negligible ball wear. A possible solution to this ball wear problem may be to replace the mobile packing with stationary packing. PSCC does not consider this alternative to be feasible, mainly because they feel that Universal Oil Products will no longer stand behind the scrubbers if such a radical change is made.

The use of an open-type packing has been investigated in tests performed by Southern California Edison at the Mohave Generating Station in 1974 and 1975. A polygrid "egg crate" packing was used consisting of plastic grids 3 cm thick with 5 cm square openings, stacked to a depth of 43 cm in each of three stages. The scrubbing liquid was a limestone slurry. The results of the study indicated that high particulate removal i.e. >90%, could be achieved [Figure 12]. Although a limited number of tests were conducted and problems with scaling were not evaluated, the use of open packing appears to be very promising.

#### Migration of Mobile Bed Contactors

The other major ball problem affecting scrubber operation is ball migration. Balls migrate due to ball wear, ball breakage, and breaks

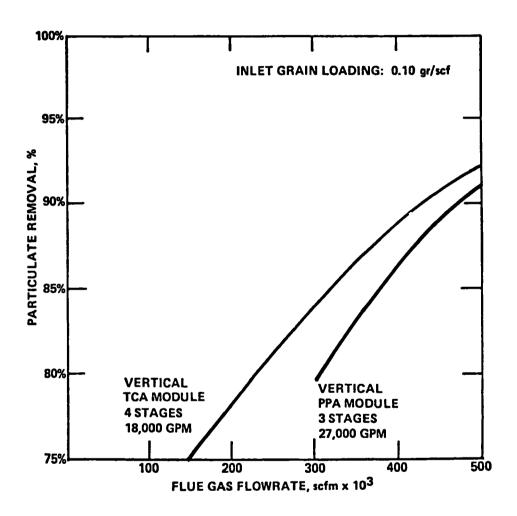


Figure 12. Particulate Removal Tests for a Vertical Scrubber Using Different Types of Packing 9

in the partitions separating ball compartments. As in the case of the ball wear problem described previously, scrubber particulate removal performance decreases and downtime for repair increases.

PSCC has reduced some of the problems brought about by ball migration by placing screens on the suctions to the recirculation pumps. Previously, balls would circulate through the system, cut pump linings, plug nozzles, etc. The problem of migration within the scrubber still remains, however. The migration problem is not always readily determined from pressure drop data as was noted in previous discussions. Obviously, frequent inspections and replacement of worn grids and balls is an important factor in minimizing ball migration between compartments. Another potential solution is to replace the mobile packing with stationary packing.

#### Isolation Dampers

On-going isolation damper problems have plagued the TCA scrubbers since these began operation. Inlet dampers accumulate ash deposits and are exposed to varying gas temperatures and conditions. Outlet dampers accumulate sludge deposits from scrubber carryover and are exposed to varying gas temperatures and conditions depending on scrubber mist eliminator and reheater operation. As a result, the gates and lower blades warp; they are difficult to operate and are hampered by gas leaks into drive trains, couplings, etc.

The best available approach to minimizing the isolation damper problem is to improve the damper operating conditions. At the inlet, this would involve reducing particulate loading by optimizing ESP performance as much as possible. At the scrubber outlet, it would be necessary to minimize flow imbalance and liquid entrainment problems and improve the operation of the reheaters. In addition, it may be

necessary to routinely exercise isolation dampers similar to what is currently being done for the stack bypass dampers.

#### Recirculation Pumps

Although a number of major pump problems had been solved during initial scrubber operations, problems with recirculation pump operation still remain. However, in view of the rugged duty to which these pumps are subjected, e.g. fly ash slurry, almost continuous operation, etc., some problems must be expected. It may not be possible to significantly improve the existing slurry pump operation. Major maintenance and repair areas include pump motors, bearings and packing.

The major problem, as noted previously, is where a scrubber section has only one recirculating pump (sections 1A, 3A, and 3C). When the pump is inoperable, either the scrubber section must be taken out of service or it must be operated with no recirculating slurry. The former condition results in reduced scrubber capacity whereas the latter causes severly limited particulate removal performance and possible exposure of scrubber internals to adverse high temperature conditions. Possible solutions to this problem are to install additional pumps on the existing one-pump sections or to pipe all the recirculating slurry pumps for a given unit to a single manifold which would feed all the scrubber sections of that unit.

#### Reheater Section

There have been numerous problems in the operation of the stack gas reheaters. The direct reheaters on Units 1 and 3 have been subject to pluggage due to carryover from the scrubbers and to corrosion. The reheaters on all three units have been plagued with an inability to provide sufficient reheat of scrubbed gases.

When the direct reheaters on Units 1 and 3 get plugged, the scrubber section is taken out of service for cleaning. The plugging is thought to be caused by water droplets being carried over from the scrubber. When the droplets evaporate, solids which were originally present as dissolved and suspended solids deposit on the in-line coils. Early efforts by PSCC to minimize plugging of reheaters included increasing the number of soot blowers and replacing finned-tube coils with bare-tube coils. Since then PSCC has also attempted to improve mist eliminator performance by installing new mist eliminator designs, but these tests have not yet been evaluated by PSCC. With the gas and liquid flow imbalance problems previously noted, it is questionable whether the new mist eliminator designs will significantly improve the reheater plugging problem. The soundest approach to solving the plugging problem appears to be replacing the direct reheaters with indirect reheaters similar to those now in operation on Unit 4.

Corrosion of the in-line reheater tubes has led to tube failure and resulting scrubber down time for repair. Corrosion is believed to generally occur under the deposits that form on the tubes. 10 Originally the tubes at Cherokee were carbon steel, but after repeated tube failure, PSCC replaced the carbon steel tubes with 316 SS. These have proven to be successful thus far. However, it has been pointed out in other powerplant scrubber applications 10 that 316 SS is highly vulnerable to failure to chloride stress corrosion. A long term solution, as noted above, would be to use indirect reheaters.

The available reheat from the reheater system has been found to be insufficient (less than design) in all three scrubbers. When the reheat is not adequate, condensation occurs in the outlet ductwork and stack, causing corrosion of these components. Also, inadequate reheat results in droplet carryover problems, giving false opacity meter readings.

The cause of inadequate reheat appears to be due to solids build-up on in-line reheater tubes (Units 1 and 3), corrosion of in-line reheater tubes (Units 1 and 3), and presence of liquid entrainment levels (all units). PSCC has reportedly conducted heat balances for the stack gas reheaters. These have shown that much more heat from the steam was used than is necessary for the sensible heat required to provide the stack gas temperatures that are actually measured. Table 21, which shows design and actual observed stack gas exit temperatures, indicates that average stack exit temperatures ranged from 46 to 67°C (115 to 153°F) or about 12 to 40°C (20 to 70°F) less than design values.

Solid buildup on in-line reheater tubes affects reheat by reducing the heat transfer from the tubes to the stack gas while corrosion of inline reheater tubes not only restricts heat transfer but also can cause leaks resulting in loss of steam. Improvement of these problems was discussed previously. The problem of high lquid entrainment requires improvement in the mist eliminator collection efficiency and/or gas flow distribution in the scrubber.

## Recirculation Piping and Nozzles

The recirculating slurry contains fly ash which is composed of very abrasive constituents such as silicon dioxide ( $\mathrm{SiO}_2$ ) and aluminum trioxide ( $\mathrm{Al}_2\mathrm{O}_3$ ). As a result, the rubber lining of the pipes is subject to highly erosive conditions, especially where the slurry impinges directly on the liner. This occurs at pipe bends of Y's and locations where the rubber liner is incorrectly applied and surface liner irregularities are formed. When the liner begins to erode, chunks of rubber are broken away and lodge in recirculating slurry nozzles. As the liner continues to erode at a given location, accelerated wear takes place and an increasingly irregular surface is formed. When the liner has been stripped from the pipe, the underlying metal is also exposed to corresive attack from the low pH slurry.

Table 21 DESIGN AND ACTUAL VALUES<sup>†</sup> OF STACK GAS TEMPERATURES CHEROKEE STATION PUBLIC SERVICE COMPANY OF COLORADO

Parameter	Uni °C	t 1 °F	Uni °C	t 3 °F	Uni °C	it 4 °F
Design	93	200	85	185	79	175
Actual (Average)						
Section A	53 <sup>*</sup>	128	46	115	50	NO <sup>††</sup>
Section B	62	144	65	149	56	132
Section C	NA**	NA	48	118	64	148
Section D	NA	NA	NA	NA	67	153

t Actual values are those noted in July-August 1977 observations from instrumentation reading.

†† NO = Not in operation during July-August observation period.

\* Unit 1, Section 1A values do not include observations made when

\*\* Not applicable.

the recirculation pump was out of service.

Erosion and corrosive attack on the slurry piping will result in reduced scrubber performance and availability. Clogged nozzles will reduce liquid slurry flow rates. Holes in piping will require that scrubber sections be taken down for repair.

The problem of corrosive and erosive attack on piping is impossible to avoid in particulate scrubbers operating on powerplants. Resulting problems can, however, be minimized to some extent by an ongoing inspection system. During shutdowns, nozzles should be inspected for rubber liner pieces. Devices, such as sonic detectors, can be used to measure pipe thicknesses at critical wear points. Nozzle plugging can be minimized to some extent by replacing nozzles with flow diverter cones which essentially have no internal parts to clog.

# Presaturator Buildup

Solids accumulate in the presaturator section in the area around presaturator spray nozzles called the wet-dry interface. In this area, the presaturator surfaces are alternatively exposed to the hot, dusty, gas stream and to the cool, wet, presaturator spray. A solid buildup results, and as the size of the buildup increases, parts of the buildup can break loose, fall into the scrubber hopper and plug the recirculation pump inlet screens. In addition, as noted in the Unit 3 inspection, the presaturator buildup can reach the point where gas flow is restricted and the flow balance is altered not only within the scrubber section but also between scrubber sections. Besides affecting screen plugging and flow balance, presaturator buildup may form hard deposits which require extreme methods for removal, such as using a jackhammer to break up the solids. Damage to the underlying presaturator surface then may result.

PSCC has attempted to minimize the presaturator buildup problem by directing the nozzle sprays so that they point 45° into the scrubber rather than being oriented at 90°, i.e., vertical. This modification has apparently helped to some degree but, based upon the Unit 3 equipment inspections, there is still a need for frequent inspection and cleaning to prevent excessive presaturator deposits from developing. This is especially true for Units 1 and 3 where flow distribution problems are more inherent and where spare modules are not available.

Other modifications to further reduce the solids buildup problem might include reducing the inlet particulate loading and providing a means to constantly wet the wet/dry interface area. Decreasing the inlet particulate loading could be achieved by improving the ESP collection efficiency. Wetting the presaturator area might be accomplished by irrigating the bottom surface from a pipe located just upstream of the wet/dry surface.

#### Mist Eliminators

The mist eliminator installations have presented continuing difficulties in the operation of the Universal Oil Products scrubbers. Problems have arisen in two areas: high pressure drop, and high mist entrainment. The high pressure drop problem was thought to be caused by the initially installed FRP mist eliminators which may have tended to "flutter" when the scrubbers were in service. This problem has reportedly been solved by the substitution of 316 SS mist eliminators.

The problem of high mist entrainment is indicated by the outlet particulate analyses conducted by MRI and by the reheater heat balances conducted by PSCC. Obviously, high carryover not only affects reheater performance but also accounts for decreased scrubber availability due to reheater pluggage from solids carried over with the entrained mist.

Furthermore, mist carryover can cause a high percentage of submicron particulate to be emitted which may not contribute much to the total weight of particulate emissions, but can have a significant adverse impact on the opacity of those emissions.

The problems of high mist carryover can originate from a number of sources. Based upon equipment inspections and discussions held with the Company, mist eliminator blade alignment and mist eliminator plugging are not significant trouble areas. However, sources which may directly or indirectly contribute to high mist carryover are: the heavy mist eliminator inlet loadings, gas flow, liquid entrainment maldistribution, inadequate mist eliminator removal efficiency, and re-entrainment. Unfortunately, very limited droplet loading, mist particle size, and flow distribution measurements have been made for the mist eliminator; however, it must be pointed out that well-developed droplet measuring methods are not presently available. It is apparent, nonetheless, that there are significant gas and liquid flow distribution imbalances to the mist eliminators. This is indicated from velocity measurements, evidence of gas flow channeling within the scrubber, and plugged water nozzles. It is not certain how these imbalances are propagated through the mist eliminator, although the normal  $\Delta P$  across the mist eliminator (typically 2 to 5 cm W.C.) is probably not sufficient to even out significant flow imbalances.

The Company approach to reducing mist carryover is to improve the removal efficiency of the mist eliminators by using a more efficient design. They have installed new mist eliminator designs in section 3A (Heil Model EB4) and section 3C (Munters Model T271). The Universal Oil Products manufactured chevron unit is a 3-pass mist eliminator with a 90° angle between blades and an offset distance between blades of approximately 4 cm. The Munters Model T271 is a chevron type mist eliminator but is composed of trapeze-shaped separating walls with integral liquid drainage channels. The offset distance

between blades is about 5 cm. The Heil Model EB4 is a 4-pass chevron with 4 cm offset between blades. The mist eliminator uses hooks to collect moisture and minimize pressure loss due to turbulence.

Table 22 presents a comparison of the design features of the existing mist eliminators in service. Although the new mist eliminator designs may provide some advantages, it is difficult to reach conclusions from the data presented in the Table. However, there is strong evidence to indicate that more extreme mist eliminator design changes may be required to provide acceptable mist reduction. Potential changes include using a vertical mist eliminator or a two-stage mist eliminator.

The major difficulties which result from the Company mist eliminator program are twofold. First, to properly improve mist eliminator design, the conditions under which the mist eliminator is operating must be fully understood. Questions which must be answered include: How significant is the gas flow distribution problem? What mist carryover loadings, drop sizes, and imbalances will the mist eliminator see? These are difficult questions to answer, but without some insight, possible solutions to the mist eliminator problems become very difficult, lengthy trial-and-error endeavors. Second, in evaluating new mist eliminator designs, it is important to minimize the effect of other variables. If the effect of these variables is not minimized, then a design may be discarded because it was exposed to more severe operating conditions, even though it may be superior to the other designs. This could very easily happen at the Cherokee Unit 3 scrubber, where a number of potential problems affecting mist carryover are known and have been observed to occur.

It is not very likely that modifications other than well-developed design modifications will markedly improve the mist carryover problem. Modifying operating variables such as gas velocity and L/G to improve mist carryover are not plausible. For example, gas velocity and L/G

Table 22

# COMPARISON OF VARIOUS MIST ELIMINATORS INSTALLED IN TCA SCRUBBERS CHEROKEE STATION PUBLIC SERVICE COMPANY OF COLORADO

Mist Eliminator Type	Gas Velocity Range mps	Pressure Drop Range cm W.C.	Minimum Drop Size <u>Collected</u> μm	Separation Efficiency for Min. Drop Size %	Maximum <u>Liquid Load</u> kg/hr-m <sup>2</sup>	Velocity for Reentrain. mps	Reentrain. Drop Size μm
UOP 3-pass stain- less steel chevr	2-4 on <sup>†</sup>	2-3	10	85-95	5% of gas flow by weig	4 ght	100-500
Munters Euroform Model 271 <sup>†</sup>	2-7	2-7	Unknown	Unknown	24.5	7	Unknown
Heil Heilex Model EB-4 <sup>†</sup>	2-7	0.1-0.5	10-20	85+	Unknown	Unknown	Unknown

t Data from product literature.

changes are restricted by the fact that the scrubber must treat all the boiler offgass and must operate at a L/G ratio dictated by particulate removal requirements. Improvements due to revamped maintenance practices are also unlikely. Obviously, the scrubber can be more frequently inspected and overhauled, but it is questionable whether this is a practical procedure for a bas-loaded plant.

Given all these aspects of the mist eliminator problem, it is not expected that the improvements initiated by the Company will have a major impact on upgrading scrubber reliability. Rather, more extreme measures such as reducing upstream gas flow and liquid imbalances, adding an additional mist eliminator stage, or changing the position of the mist eliminator to a horizontal rather than vertical duct may be necessary.

#### Scrubber Booster Fan

Recurring problems with booster fans have been noted in upset reports throughout most periods of scrubber operation. These upsets vary from fan bearing, alignment, and vibration problems caused by build-up of ash on fan blades to more serious problems of erosive wear of the fan blades caused by the highly abrasive nature of the ash. This latter problem is especially critical for Unit 4 since it utilizes air foil type fans (dictated by volumetric flow rate-pressure drop requirements) as opposed to the radial tip fans used on Units 1 and 3. Air foil fans are extremely sensitive to erosion and fan performance rapidly deteriorates under highly erosive conditions. Obviously, when a fan is taken out of service, part of the gas flow must be bypassed to the stack or to a spare module, if available.

These fan-related problems are difficult to avoid in light of the relatively high dust concentrations involved, even with properly operated

fan soot blowers. The most readily apparent solution is then to upgrade the performance of the ESP's, and thereby reduce the concentration of fly ash which the fans must handle. The problem of the fan blade wear on Unit 4 caused by the highly abrasive ash might also be reduced by using harder alloys.

#### Weather-Related Problems

The freezing of lines during cold weather continues to be a potential problem for the Unit 1 and 3 scrubbers but the magnitude of this problem could not be evaluated from upset data or from observations made during the July-August 1977 observation period. In general, freezing can cause leaks in piping, damage valves and cause portions of the slurry and water streams to become inoperative. The particulate removal performance of the scrubber may then be reduced or sections of the scrubber may need to be taken out of service for repair. The Unit 4 scrubber is enclosed and does not have significant freeze problems. On Units 1 and 3, the Company, reportedly, attempts to drain water and slurry lines when the scrubber is taken out of service for long periods. Difficulties are said to typically result during shutdowns when there is not enough time for proper drainage.

#### Outlet Ductwork

The ductwork at the outlet of the scrubbers is unlined carbon steel and is highly vulnerable to corrosive attack. When the scrubber reheaters are not in service or are not operating properly, the ductwork is exposed to gas which is at or below its dewpoint with respect to sulfurous and sulfuric acid. The acid collects on the ductwork surfaces and the metal is attacked. The result is corrosion and rusting of the carbon steel with accompanying loss of structural integrity. Holes form in the ductwork, allowing gas to escape; acid condensation then can occur on nearby structural supports, insulation, etc.

Inspection of the Unit 3 scrubber indicated that extensive corrosion has already occurred in the outlet ductwork. Likewise, although outlet ductwork on Unit 1 and 4 scrubbers wasn't inspected, it is expected that with similar reheater problems, these units will also have severely corroded ducting. At this advanced stage, covering the carbon steel with a protective coating may not be feasible. Therefore, remaining options are to immediately repair ductwork failures as they occur and reduce the amount of time that the scrubber is operated when the reheater is defective. Complete replacement of outlet ductwork sections is not advisable until the reheat problem is solved.

#### EVALUATION OF INSTRUMENTATION

An evaluation of the instrumentation used for measuring smoke density was conducted on July 7 and 20, 1977.

Units No. 1 and 2 exhaust to opposite sides of a single 91 m (300 ft) stack, with a 4.9 m (16 ft) exit diameter. The opacity of Unit No. 1 is measured by a Bailey Dust/density transmitter (bolometer) installed in a 2.1 m (7 ft) wide duct a short distance from the stack. The light source and light detector are on opposite sides of the duct and are joined by a pipe to maintain alignment of the system. Purge air is supplied to both sides of the system to reduce dust accumulation on the lenses. The standard installation, which is indicated to be in place, utilizes a 10 cm (4 in) diameter pipe with a 1.5 m (5 ft) x 3.25 cm (3.25 in) slot across which opacity is measured. The dust path is normal to the plane of the slot. The opacity is registered on a 24-hr circular chart recorder. A clock accumulates the intervals when the opacity exceeds 20%.

Every day the lenses of the transmissometer are cleaned and the recorder charts replaced. All opacity charts are kept at the plant for a one-year period. Unit No. 1 has reheat problems which have reduced the temperature in the duct to about 52°C (125°F). This low temperature reportedly permits ash buildup on the Bailey pipe and reduces the cross-sectional area along the light path. A brush is used to ream the pipe while the unit is in service. During outages the ash buildup is removed by chiseling.

The meter on Unit No. 2 is the same as on Unit No. 1, but is installed across a 2.6 m (8.5 ft) duct. Daily maintenance is the same for each Bailey meter.

Unit No. 3 exhausts to a 91 m (300 ft) tall stack with a 5.9 m (19.5 ft) exit diameter. A Bailey meter, as described above, is installed across a 2.3 m (7.5 ft) duct leading to the stack. A reheat problem exists with Unit No. 3. This has allowed ash buildup similar to that occurring on the piping of the Bailey meter on Unit No. 1.

Unit No. 4 discharges to a 122 m (400 ft) stack with a 6.7 m (22 ft) stack exit diameter. Two Bailey meters are installed on the discharge side of the induced draft (ID) fan. A scrubber downstream of these meters negates use for emission measurements. However, the meters are used for adjusting performance of the unit. A Lear-Siegler RM-4 transmissometer is installed in the duct between the scrubber and the stack. In contrast to the Bailey meters, the RM-4 contains the light source and detector in a single housing on one side of the duct. A pipe is not used to maintain the alignment across the duct. Unlike the older Bailey meters, the RM-4 electronically converts opacity measurements in the duct to read stack exit opacity. The conversion factor is set at the factory prior to installation.

A Leeds and Northrup Speedomax strip chart recorder registers the output of the transmissometer. Charts are replaced when the end of the roll is reached. Plant personnel have found that purge air is effective enough to only require lens cleaning every six months. The filter on the air cleaner must be cleaned every three months.

Operation and maintenance procedures for all meters were found to be acceptable. The location of opacity meters on Units No. 1-3

was also adequate. However, the Lear-Siegler transmissometer is located between two horizontal bends which may create a non-uniform particle distribution.

On July 20, 1977, the Bailey meters on Units No. 1, 2 and 3 were calibrated using a procedure developed at NEIC and standard screens of known opacity (20, 40, 60 and 80%) supplied by the Bailey Meter Company. The Lear-Siegler monitor was not calibrated since that company only supplies an internal standard and NEIC is only now developing a field calibration system for that unit. The procedure permits a check of the linearity and span of the meter while the unit is in operation. A sample calculation is shown in Appendix E.

The test procedure requires that calibrated filters or screens be inserted in the light path to simulate opacity measurable by the transmissometer. The opacity (0) scale is not a linear function but is related to optical density (0D) by the relationship.

$$OD = -\log_{10} (1-0)$$

The optical density is linear and, therefore, is additive while opacity is not. If the duct where the opacity monitor is installed is measuring a background opacity because a unit is in operation, the optical density of screens being inserted is additive to that in the duct. Thus, if a monitor is reading 15% opacity (OD = 0.071) and a 20% opacity (OD = 0.097) screen is inserted, the resulting opacity should read 32% (OD = 0.071 + 0.097 = 0.168) rather than 35% (20% + 15% = 35%).

If the relationship between the meter output and the screen opacity is linear with a 45° slope when plotted in optical density units, then the relationship between meter output and stack opacity is linear [Figure 13]. If, in addition, the meter reads 100% when the light

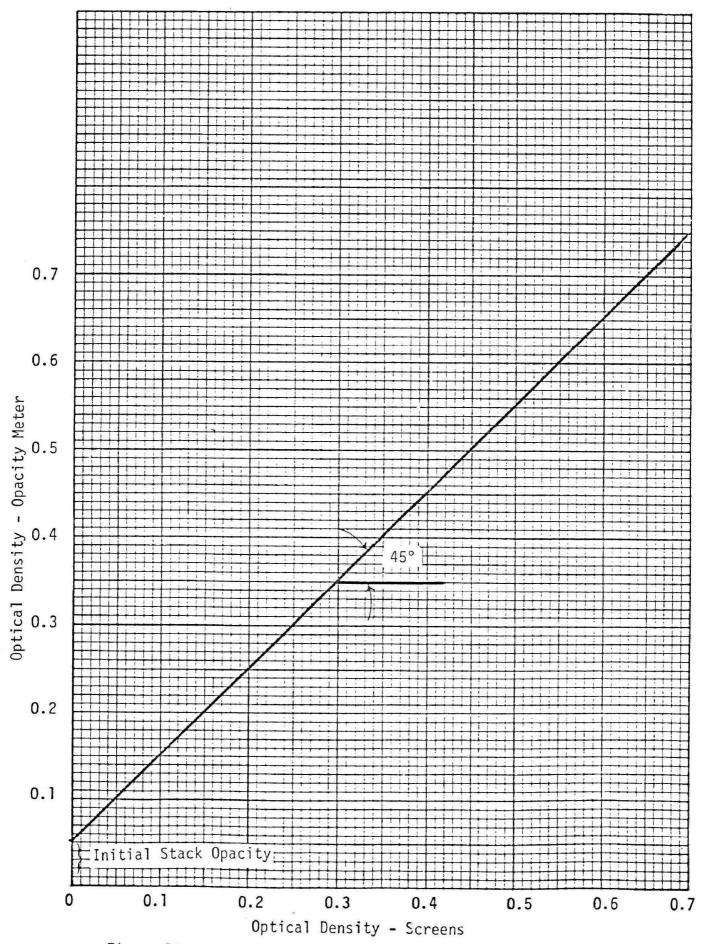


Figure 13. Calibration Curve of Instrument in Calibration Cherokee Station - Public Service Company of Colorado

beam is either completely obscured or extinguished, then the meter can be assumed to be in calibration since a line through the 100% opacity point at 45° slope would also intersect the origin.

When an optical density plot of the meter output vs screen opacity is linear but not at a 45° slope, then the relationship between recorded output and stack opacity is not linear and the meter is out of calibration. In this case the scale is distorted, being elongated if the slope is greater than 45° and shortened when less than 45°. If elongated, the meter will read higher for a given stack opacity, if shortened it will read less. In these cases the meter will still appear to pass through 100% opacity when the light beam is extinguished and, since the zero opacity is not usually measurable during process operations, the transmissometer is thought to be in calibration.

The major problem in calibrating a transmissometer appears to arise from units calibrating near 100% opacity. Figure 14 shows the relationship between opacity, transmittance and optical density. The difference between 0% and 90% opacity is 1 0D unit. The difference between 90% and 99%, or 99% and 99.9% is also 1 0D unit. Thus, calibration procedures causing large changes in optical density result in minor differences in opacity near 100%, but significant variations in the usual range of opacity readings.

While the above calibration procedure is adequate for checking span linearity, it will not determine whether the background opacity reading is a result of smoke in the stack, or is attributable to dust in front of or behind the lenses.

When the smoke density meter on Unit No. 1 was calibrated [Figure 15], the unit was burning 100% natural gas. The recorder was reading -1%, and with the light source extinguished, 99%. The data were shifted

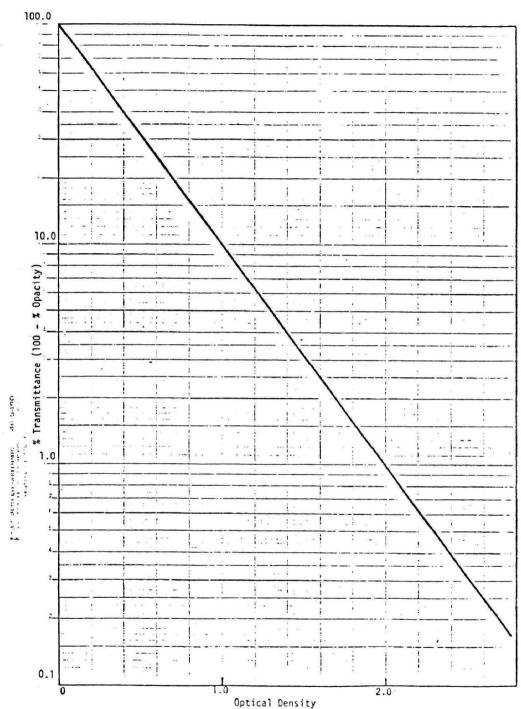
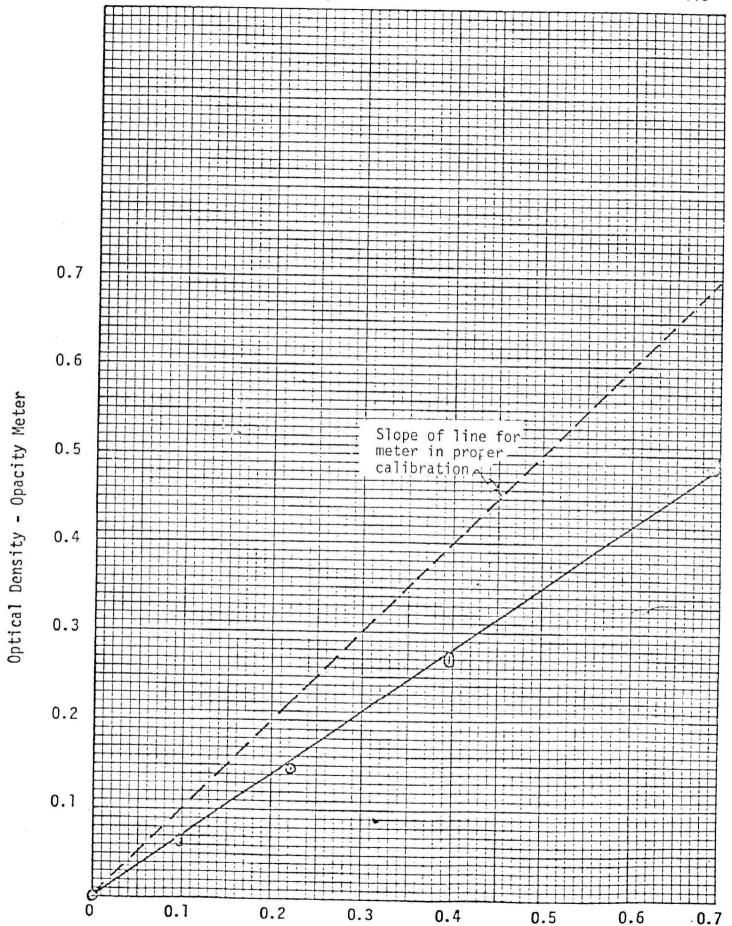


Figure 14. Relationship Between Optical Density,
Transmittance and Opacity
Cherokee Station
Public Serice Company of Colorado



Optical Density - Screens

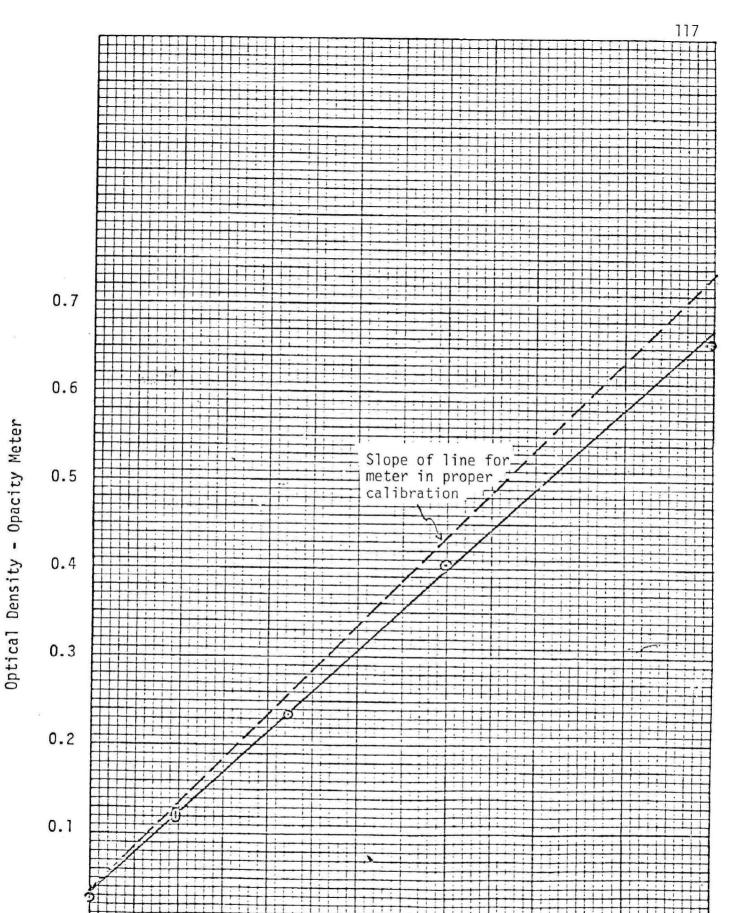
Figure 15 . Calibration of Bailey Smoke Density Meter on Unit No. 1
Cherokee Station - Public Service Company of Colorado

1% upscale before being calculated to account for this offset. Figure 14 indicates that while the linearity is acceptable, the instrument span appears shortened. The shortened span causes reduced output for a given opacity, even though the 0% and 100% points are acceptable. Some of the data points of Figure 14 are indicated as vertical lines where recorder fluctuations (due to opacity variations in the duct) occurred during the calibration procedure.

Unit No. 2 was burning a mixture of 25% natural gas and 75% coal when the smoke density meter was calibrated [Figure 16]. This resulted in a higher background opacity in the duct (6%) as compared to Unit No. 1. When the light source was extinguished, this meter also read 99%. The linearity appears acceptable, however, the span of this unit also is shortened although not to the extent of the meter on Unit No. 1.

If the slope of the line fitting the data is considered an indication of the span, a 45° line (an instrument in proper calibration) would have a value of unity. The opacity monitor on Unit 1 has a slope of 0.70, indicating a span that is 70% of the acceptable value. In a similar fashion, the monitor on Unit No. 2 had a span of 91% of the desired value.

Unit No. 3 was only burning coal when the opacity monitor was calibrated, thus it was registering a slightly higher opacity (9%) than the other two meters. With the light source extinguished, this meter also read 99%. Again, the linearity appeared acceptable but the span was foreshortened to 86% of the expected value [Figure 17]. In all cases the compression of the span will result in the recorded value being less than the measured value in the duct.



Optical Density - Screens
Figure 16. Calibration of Bailey Smoke Density Meter on Unit No. 2
Cherokee Station - Public Service Company of Colorado

0.4

0.5

0.6

.0.7

0.3

0

0.1

0.2

Optical Density - Screens

Figure 17. Calibration of Bailey Smoke Density Meter on Unit No. 3.

Cherokee Station - Public Service Company of Colorado

In addition to the lower opacity reading indicated above, it should be noted that the three meters are only reading smoke density across a 1.5 m (5 ft) path length (the length of the slot in the pipe). Below is a comparison of this length with duct and stack exit diameter for each meter.

	Slot L	ength	Duct	Width	Stack	Diameter
Meter N		ft	m	ft	m	ft
1	1.5	5	2.1	7	4.9	16.0
2	1.5	5	2.6	8.5	4.9	16.0
3	1.5	5	2.3	7.5	4.9	19.5

Since the opacity is a function (logarithmic) of the path length, the meters are only measuring a portion of the opacity when the slot does not extend across the duct. Also, since the ducts are all narrower than the stack exit diameters, the opacity measured at the stack exit would be greater than measured across the duct (all else being equal). The following relationship relates opacities to varying path lengths:

$$\frac{\log_{10} (1-0_1)}{d_1} = \frac{\log_{10} (1-0_2)}{d_2}$$

where  $0_1$  and  $0_2$  are opacities measured across distances  $d_1$  and  $d_2$ . For example, if the meter on Unit No. 3 was reading 10% opacity (across the 1.5 m slot), a meter across the duct (2.3 m) would be expected to read 15%, while 34% opacity would be expected at the stack exit (5.9). These differences are significant and also indicate a case where the meter would be reading below 20%, therefore not requiring a report to State and Federal agencies, while the opacity at the stack would be above the value that requires notification. As indicated earlier, the

Lear-Siegler transmissometer corrects for this difference and reports exit opacity.

In the case of Units No. 1 and 2 which exhaust to the same stack, the observed stack opacity would be a function of the opacities in each duct, but in all cases would be greater than the opacity from a single source. The relationship is given by the equation:

$$\frac{\log (1-0_1) + \log (1-0_2)}{d_1 + d_2} = \frac{\log (1-0_s)}{d_s}$$

where  $0_1$ ,  $0_2$  and  $0_s$  are the opacities recorded on Units 1 and 2, and the opacity of the stack and  $d_1$ ,  $d_2$  and  $d_s$  are the meter path lengths 1.5 m and the stack exit diameter 4.9 m, respectively.

Using the above equation, it is possible to determine the relationship between the two opacity meters that will produce a 20% opacity at the stack as follows:

Opacity Either Unit %	Opacity Other Unit %	Exit Stack Opacity %		
0	13	20		
2	11	20		
4	9	20		
6	7	20		
8	5	20		
10	3	20		
12	ī	20		

The table shows that when either meter is reporting over 13% opacity, the stack exit opacity will be  $\geq$  20% and that even with opacities as low as 7% on each meter this condition can occur.

Even with proper operation and maintenance, the three meters examined were out of calibration in that the span was foreshortened on all three. However, when this problem is corrected, the meters will still not be making the measurement that is desired; i.e., the opacity of the plume at the stack exit. Thus, when the Company reports the incidence of opacity greater than 20%, it will be occurring across the 1.5 m (5 ft) slot and not at the stack exit. On the other hand, as the examples showed, 20% opacity may occur at the stack exit and go unreported because the meter is reading less across the slotted pipe.

The deficiencies in the plant monitoring system can be corrected by the following:

- 1. The three Bailey units should be calibrated using the 40% opacity plate or a filter in that range. This should be done by adding the optical density of the duct opacity to that of the plate or filter to determine a total optical density. When this total is converted to opacity, the value should be set on the meter. Because the meters are presently out of calibration, this may initially require several iterations since the duct opacity will be in error.
- 2. The reporting requirements for Units 1 and 2 should be modified to account for the relationship shown above. This may be done either by installation of electronic circuitry designed to output the relationships between the two instruments to produce a recording of combined stack exit opacity or by use of the above table computing this relationship.
- 3. The reporting requirements for Unit 3 should be modified to account for the relationship between opacity across the 1.5 m (5 ft) slot length and the 5.9 m (19.5 ft) stack exit diameter. From the relationship between opacity and path length, 6% opacity at the bolometer will correspond to 20% at the stack exit (all else being equal).

#### VISIBLE EMISSION OBSERVATIONS

During the study period, visible emission observations (VEO) were randomly made on the three boiler stacks at Cherokee Station. The VEO's were made by eleven different NEIC observers using EPA Method 9. A summary of the observations is given in Table 23. Appendix B contains a listing of the VEO's for the individual stacks at Cherokee. During the study period, 92 VEO's were made and the average opacity exceeded 20% during 51 of those observations [Table 23]. Because the opacity regulations in the State Implementation Plan (SIP) has no time limitation, the individual readings were also summarized for the set of 51 observations. Of 1,374 individual readings, 949 exceeded 20% but were less than 40% opacity.

During the July and August VEO's, an NEIC observer monitored the plant operation and recorded unit load, fuel type, opacity meter readings and control equipment data. This data was recorded before and after each VEO [Appendix A]. Only the process data was recorded during the October VEO's and were normally recorded after the observations were made. It was not possible to correlate the VEO readings with the Bailey opacity meter readings. The readings did confirm the calibration tests results that indicated the stack opacities would be greater than the Bailey Meter readings, due to path length differences. The average stack opacities read by the NEIC observers were greater than Bailey meter readings. Because the Bailey meters were found to be out of calibration, the VEO's were not compared to calculated stack opacities. It is recommended that once the deficiencies in the opacity monitoring system are corrected, that actual VEO's be compared to the calculated stack opacities to ensure that the meters are accurately recording exit stack opacities.

Table 23 SUMMARY OF VISIBLE EMISSION OBSERVATIONS CHEROKEE STATION PUBLIC SERVICE COMPANY OF COLORADO

	No. of 6 min	No. of Times	No	. of Reac	lings
Stack Observations Av		Average Opacity >20%	Total	>20% <40%	>40%
	July	27-August 28, 1977			
nit 1 & 2	34	23 <sup>a</sup>	552	417	4
nit 3	10	8 <b>b</b>	192	113	57
nit 4	27	7	170	110	0
	00	tober 4-18, 1977			
	No. of 9 min Observations				
nit 1 & 2	7	5 <sup><b>ċ</b></sup>	184	72	12
nit 3	7	5 <sup><b>d</b></sup>	168	74	90
nit 4	7	3 <sup><b>e</b></sup>	108	63	23
TOTALS	92	51	1,374	949	186
nit 4	7	3 <sup>e</sup>	108	63	

a The recirculation pump for section A of the scrubber was out of service during 13 of these periods.

e Unit startup during one observation.

b Unit 3 scrubber was off-line during entire period. Fuel was 50% coal and 50% gas. Unit 3 down for scheduled outage on 8/20/77.

<sup>•</sup> Unit 1 reheater plugged. Scrubber being bypassed three of these periods.

d Recirculation pump out of service one time and booster fan out with 50% bypass during other four observations.

#### VIII. PARTICULATE CONTROL SYSTEM RELIABILITY IMPROVEMENT

Before any discussion of alternatives to improve system reliability is presented, there are a number of related topics that should be reviewed. Some consideration must be given to the ultimate reliability goal to be attained. Thought must be given to how various options for improving reliability are to be evaluated. It must also be recognized that economics and redundancy will have a large impact on reliability considerations.

A determination of required system reliability is of primary importance. There are a number of ways of expressing system reliability but normally it is done on the basis of percent boiler on-line time. An "acceptable" percent reliability will vary and depend on, among other things, the specific application, and revelant SIP regulations as interpreted by the administering agencies. Ninety percent reliability is considered to be achievable for powerplant flue gas desulfurization processes and is also acknowledged to be within the limits of particulate removal technology by most particulate control equipment manufacturers. The State of Indiana requires 95% reliability for meeting their particulate regulations.

A method is also needed to gauge how various modifications will affect reliability. If the necessary reliability component data is available, it is possible that reliability analysis techniques pioneered in the nuclear industry could be applied to particulate control systems. For example, when reasonable estimates of mean time to failure and mean repair times of critical equipment can be made, then a fault tree analysis can be conducted and quantitative comparisons can be obtained. Otherwise, reliability analysis must be left to qualitative engineering judgments which are often subject to extensive debate and disagreement.

Economics will be an important part of comparing control plan alternatives. Any system can be upgraded to provide 99.9%+ reliability. The cost of that system, however, may be prohibitive. Although economics were not evaluated in this study, such effects must be considered in any further analyses.

Redundancy will be a key factor in achieving consistent operation of any particulate control system. Equipment used in near continuous service and exposed to dusty environments, variable temperatures and corrosive conditions will eventually break down. Therefore, to achieve reliable operation under such circumstances, it is necessary to provide spares for critical equipment components. The problem is in determining which are the critical components. Some of the areas of improvement are discussed below.

### SCRUBBER IMPROVEMENTS

There are numerous areas for improvement suggested from the scrubber evaluation. In this section, only those changes which are considered to have a significant impact on scrubber reliability are presented. These include: adding spare scrubber capacity, replacing the direct reheat systems with indirect reheat, adding spare recirculating pumps, providing for more frequent inspection and cleaning of those scrubbers that don't have spares, improving the mist eliminator design, improving the mobile packing design, and providing an enclosure for all scrubber sections.

The addition of spare scrubber modules would have a very significant effect on reliability. Provision for spare modules would allow for a scheduled maintenance program in which modules would be routinely taken offline for cleaning and repair. A spare module would also allow for switching modules on- and offline when emergency repairs were required.

On those units with direct reheaters, improvement in scrubber availability would be realized if the direct reheaters were replaced with indirect reheaters. The in-line tubes are subject to plugging and corrosion which is highly dependent on upstream scrubber and mist eliminator conditions. The ability of indirect reheaters to stay in service is much less dependent on such conditions. As an added advantage, indirect reheat air fans can be used to provide fresh air to scrubber sections during maintenance, reducing the need for tight isolation damper shut offs.

Adding spare recirculation pumps is considered very important in improving operation of single-pump scrubber sections. When a recirculating pump is out of service in a single-pump section, the section must be taken off line or operated under very reduced capability. Adding spare recirculation pumps to a scrubbing section which has three operating pumps is less critical.

Increased frequency of inspection and repair must be considered as a potential alternative for improving reliability on Units 1 and 3. PSCC's current maintenance practices and thoroughness in performing maintenance does not appear to be improper. The problem occurs when scrubber instrumentation does not always indicate when scrubber internal problems are occurring. Detection of such problems then requires frequent visual inspections. Unfortunately, the practicality of frequent visual inspections on a base-loaded plant is questionable. Furthermore, the required frequency at which inspections must be made is affected by the quality of the coal fired, the operation of the plant, the operation of the ESP's, etc. An optimal inspection frequency will be different under different conditions.

PSCC is currently attempting to improve the operation of the scrubber system by improving the mist eliminator design. The success of this effort will depend on how scrubber operations affect new design

considerations, and how the mist eliminator tests are being conducted. Any program of this type must be considered a research effort and will require time for adequate tests to be run and evaluated. It is probable that the Company will have to resort to major mist eliminator modifications, such as installing two horizontal mist eliminator stages or a single vertical mist eliminator stage, to markedly reduce mist entrainment.

The improvement of the scrubber operation is very significantly affected by the type of packing used in the scrubbers, and PSCC has expended considerable effort in this direction. They have not tested stationary packings; however, and in view of success of stationary packings observed in other related applications, this appears to be an area which should be thoroughly investigated. As in the case of improving mist eliminator design, a research and development effort is required and additional time will be needed for proper evaluation.

Enclosing the scrubbers on Units 1 and 3 would reduce down time due to freezing lines. This has proven effective on the Unit 4 scrubbers.

## **ESP IMPROVEMENTS**

The evaluation of the precipitators was hampered by not knowning the flow of gas conditioning agent to the boiler off-gases. The first area of improvement should be to add flow monitoring devices to monitor the  $\mathrm{SO}_3$  flow to each of the units (including Unit 3), then an evaluation program must be undertaken to determine the effectiveness of the gas conditioning on ESP efficiency. Once this is done, the operation of the ESP may be inproved to the point of meeting design efficiencies. Major modifications would need to be undertaken to significantly improve the collection efficiency of the ESP's and thus reduce the particulate loadings to the scrubbers and to the stack in the case of Unit 2. These

would include adding more electrical sections, increasing collection plate area, and upgrading the automatic control systems. Adding more electrical sections would increase the power input to the ESP's and provide for higher corona power and current densities. This would also provide for a more efficient and reliable precipitator since a smaller portion of the precipitator would have to be taken out of service when broken wires are changed. Enlarging the precipitators by increasing the plate area would probably be the most expensive way of increasing the efficiencies of the precipitators, since this would essentially be the same as adding a new precipitator to the existing system. The existing automatic controls are of the saturable core reactor type and are typically slow in responding to voltage changes. This is especially critical if excessive sparking occurs and the controls do not respond fast enough to prevent corona wires from burning out. This was not observed during the study but may be a problem if higher power inputs are wanted.

The previously discussed improvements were based on evaluations made on the existing particulate control equipment. Other alternatives not evaluated in this report, include replacing the existing equipment with high efficiency (+99%) precipitators or replacing the scrubbers with baghouses. These options should also be considered when evaluating a program for improving reliability.

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# APPENDIX A

PROCESS DATA SHEETS
CHEROKEE STATION
PUBLIC SERVICE COMPANY OF COLORADO
July - August, 1977

Date:	7/27/72	7/27/27	7/24/17	7/2-/77	1/30/2	7/30/17	8/4/z)	9/1/27_	
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'lant									ļ
Mw. Gross	116	106	115	116	46	102	1/82	117	
XS 02 (%)	2.4	2.5	3.6	3.6	1.6	2.8	3.2	4.5	
Steam Flow (1bs/hr)	4.44.105	\$4x165	23:10-	91.05	9.5,103	8 4x105		JOXIC	
Steam Pressure (psig)	1430	1420	1430	1430	1120	<u></u>		1410	
Steam Temperature (°F)	970	• .	340	980		4/40	1002	1000	
Opacity, <del>Sypass</del> (%)	<u> </u>		0		4	_5	6		
Opacity, Scrubber (%)	Gas	Gas	CAS	کهی	Coal	Cogl	Cogl	CUAT	
FUE!	Gus	UQJ				,	}		
Section A1-A2; AC Voltage (v)	180	186	180	180	180	/8/2	/8c	180	
AC Current (a)	50	55	SÙ	50	60	75	35	<u> </u>	
DC Voltage (Kv)	3.3	33	32	33	3.3	33	3.5	34	
DC Current (ma)	€-180	180	160	160	170	120	40	40_	
Spark Rate (spm)			_0_	0	>500	460	330_	300	
Cont 03 00 40 W 13	•								
Section B1-B2; AC Voltage (v)	160	<u> </u>	155	<u> </u>	160		165_	165_	
AC Current (a)	_!_50	- 55	-22	<u>. 53 </u>	, CO	027	35	35	
DC Voltage (Kv) DC Current (ma)		0u+ 00+	210	210	220		Out.	110	
Spark Rate (spm)	<u>, <del>0</del>25</u> 0	230 O	φ <u>ι</u> Ο	5		>500	330	240	
Spark race (Spar)			<b></b> !	i :					
Scrubber		د.ا	Schr	Salar		,			
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Fan A Outlet Pressure (in. H <sub>2</sub> 0)	_Ozento		Opentu	oceation	185	_16	19.5	19	
Fan B Outlet Pressure_(in. H2O)					18.2.	16	79.5	/9	
Fan A_Amps					150	110	720	150	
Fan_B_Amps					150	/\$7	! _	J55 _	
Stack Damp A, Pos. (% Opn)					_0_	0	0_	0	
Pumps; Recirc. Pump Al (amps)	;		,	!			1		
Recirc. Pump B1 (amps)	<del></del>			· 	23	٤٤		225	<b>'</b>
Recirc. Pump B2 (amps)			•		2.2	2१		24.5	-
Recirc. Pump B3 (amps)		<del></del>			23	23	53.2	23	
Towns on the contract of the c	<u> </u>			· · · · · · · · · · · · · · · · · · ·			<u></u>	2:5	
Reheater: Steam Flow (M 1bs)	<u> </u>			I	100	· 1247	10st	0:5	
Steam Pressure (psig)			·	<del>                                     </del>	1		<del></del>		
Continue A. Dunnah Hatar Flori ()			•		مر ا	!			
Section A: Presat Water Flow (gpm)	أ		L	<del> </del>	54	-65	64	.4	<del></del>
Outlet Gas Temp. (°F) Bed Diff. (in. H <sub>2</sub> O)	<u></u>		·	<del> </del>	142	140	140	0	ļ
	'			<del></del>	9.0	<u> </u>	8.0	8.6	· 
					14.5	3:3	_5	4.25	
Gas Outlet Flow (in. H20)				<u> </u>	307	363	3.5	-3-3-	
1100		·	•		3.0	2.5	<u> </u>	2.4	
ection B; Presat Water Flow (gpm)	1				21.	224	-15-	20	
Outlet Gas Temp. (°F)	~ . ~~~		·		-24v - 150	320	<u> 215</u>	38	
Bed Diff. (in. H <sub>2</sub> O)					9 = .5	150	120	150	
Demister Diff. (in, H2O)					24.5	12.3	95 355	9.5 2.1-5	
RH_Diff. (in. H <sub>2</sub> 0)					3.01		3,0	30	
Gas Outlet Flow (in. H <sub>2</sub> O)					0.2		0.6	Ö	
omments Nat'l Gas Flow (sofu)									

Spark rate meters not in operation

# exercised the stack damper demister wash on

					<b>2/*</b> 1 2
	066	-1.1	~/./·	alsta	8/4/17
Date:	8/3/77	8/3/71	8/4/17	2/4/17	
Time:	1650	1915	1000	1150	BI-116 1=-
Plant			BASE	1 -	
Mw, Gross	112	109	LOAD 	113	1/3 //
XS 02 %	2.8	2.8	2.8	3.0	1.6 _ 2
Steam Flow (lbs/hr) x105	8.9	7.6	9.2	9.2	9,4 9.2
Steam Pressure (psig)	1420	1420	1420 .	1420	436 /4.3
Steam Temperature (°F)	1020	1000	1000	99C	15
Opacity, <del>Dynass</del> (%)	10	. 15	10		-100 % Gal-
Opacity, Scrubbon (%) FUEL	100% COAL	. IUU'A CUAL	100% CUAL	.100%.COAL	0 0
NAT. GAS FLOW (SCFH)	O	0	0	0	
ESP Section A1-A2; AC Voltage (v)	160	190-10-	190±10	200±10	190-10 3
AC Current (a)	50±10	50±10	<i>50</i> 120 -	. 50120 -	45.=20.145
DC Voltage (Kv)	35	37t2	38t 2	38±2	3752 59
DC Current (ma)	JIO ± 10	140 £ 20	140£20	140±20_	100, ion
Spark Rate (spm)	450±20	200 ± 20	400±40	- 380±20	380=10 86
make a supple following out to the participation to the same of th	<del></del> -	450			120 11
Section 81-B2; AC Voltage (v)	160	170	165	175_	. 175 /82
AC Current (a)	35 t 10	35±10	40±/0	. 35±10	· 30=5 25="
DC Voltage (Kv)	*	. 8	2	2	Out 1
	12040	100120	100±20	100 = 20	300-5 140-1
Spark Rate (spm)	370 ESI	1 <b>350</b> ± 50	.29u.± 40	-290±30 -	300-5 - 140-1
Scrubber .		. 250			
Fans; Fan Inlet Pressure (in. H <sub>2</sub> 0)	-0.3	-0.25	-0.25	-0.25	-0.3  -
Fan A Qutlet Pressure (in. H <sub>2</sub> 0)	16.5	9.5	. 15	14	16 1-13
Fan B Outlet Pressure (in. H2O)	16.5	9.5	15	14	16 15.5
Fan A_Amps	145	130	145	145	150 1
Fan B Amps	150	125	145	145	150 15
Stack Damp A, Pos. (% Opn)	0	.0	0	0	٠٥
	-				
Pumps; Recirc. Pump Al (amps)	23	24	23	23	~ 1
Recirc. Pump B1 (amps) Recirc. Pump B2 (amps)	21	21	21	21	22 1
Recirc. Pump B3 (amps)	23	23	23	23	23,5 2
" " " " " " " " " " " " " " " " " " "	24.	24	24	24	23 5 . 23.5
Reheater; Steam Flow (M lbs)	OUT	OUT	OUT	04T	Out ic.
Steam Pressure (psig)	-	~	-	~	
	, ,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,			_	
Section A; Presat Water Flow (gpm)	64 <b>X</b>	56	64	64	64 64
Outlet Gas Temp, (°F)	140	150	175	175	120 1
Bed Diff. (in. H2O)	5	4	5.6	<u>5,0</u>	5.2. 3
Demister Diff. (in. H <sub>2</sub> O) RH Diff. (in. H <sub>2</sub> O)	4.5	3 .,	. <u>3.</u> 4	3,8	4.8 4
Gas Outlet Flow (in. H <sub>2</sub> O)	3,5	'2'		<del>3</del>	3.6 3.7
das outlet 110w (111. 1120)			. 100 .	_ 1.6	- 1.8   1^
' Section 8; Presat Water Flow (gpm)	2 20	220	2,20	220	
Outlet Gas Temp. (°F)	150	. 155	155	155	720 . 2
	77.5	46	62.2	6±.2	160 155
Demister Diff. (in. H2O)	. 1.5 <u>*</u> .5	1:0±.3.	1.5.± -3	1.5±.2	7
RH Diff. (in. H <sub>2</sub> 0)	3 .	2		3	3
_Gas'Outlet Flow"(in. H <sub>2</sub> O)	0.4	0.2	0.5	0.4	63 10.3
Comments 8/3/71 TABLE MOLLITAIN	1/ ^	<b>a</b> le 30	TING UNITS	٠	<del></del>
·			COMPY CONTIN	, _	
DISPATCH CONTR	L CENTER	FOR ARGA.			

UBSERVER - R. TOA

	<del></del>		1247	0.60/20	<del></del>	1	
Date:	8/3/72	8/8/77	84/17	8/9/27			
Time:	0000	1036	1351	1454		-{	<del></del>
Plant	<u> </u>					<del></del>	
H. Croce	1,			רוו	!	) i	İ
Mw. Gross	<del>  <u>                                   </u></del>	118	117	2.3	<del>- i</del>	<del> </del>	<del></del>
XS 02 Steam Flow (1bs/hr)	1 2.5	1.8	2,2	9.4		<del></del>	
Steam Pressure (psig)	1420	14=0	1420	1420			
Steam Temperature (°F)	1015		1015	1015	1		
Opacity, Bypass (%)	7 7	7	9	8	1	i	
Opceity, Scrubber (*) Fuel	coal	Coal	coal	coal			
Nat'l Go's FICW (SCHA)	+ = = =				1		1
ESP	.	į		'			i
Section Al-A2; AC Voltage (v)	180	175	185		1		
AC Current (a)	40-15		· Ao tiu	40=10 1	<u> </u>		
DC Voltage (Kv)	37=2	37 t 2	137=2	135			
DC Current (ma)	80	90	80	180		_!	
Spark Rate (spm)	430=30		460-20	44c=301			
		·	1		•	ı	į
Section B1-B2; AC Voltage (v)	155	. <i>15</i> 5	11.0	155 .	<u> </u>		
AC Current (a)		35=10	35-0	35.0	!		
DC Voltage (Kv)	<b>ウルヤ</b>	O.+	OUT	Ou+!			
DC Current (ma)	90		' 90	100	<u>i</u>		<u>,                                     </u>
Spark Rate (spm)	200-20	170540	2407-10	220-20	1		
	1	į	1	!!	- 1	1 :	
Scrubber	i	į		} !		1	,
<u>Fans: Fan Inlet Pressure (in. H2O)</u>	<u> </u>	<u>' -,7                                   </u>		! - 7		<u> </u>	<del></del>
Fan A Outlet Pressure (in. H <sub>2</sub> 0)	15.5	<u>' '\S.5 </u>	14.5	14.3			· · · · · · · · · · · · · · · · · · ·
Fan B Outlet Pressure (in. H20)	ــــــــــــــــــــــــــــــــــــــ	15.5	_/4.5	<del></del>	·		
Fan A Amps	150¢	150	145	145			
Fan_B_Amps	1550	155	150	150		!	
Stack Damp A, Pos. (% Opn)	<u></u>	<u> </u>	. 0	0			
Dumpe Decine Dump Al ()				! 1	ı	i	
Pumps; Recirc. Pump Al (amps)	23	22.5	<u> </u>	23		<u> </u>	<u> </u>
Recirc. Pump B1 (amps) Recirc. Pump B2 (amps)	<u> </u>	21	21.5	1 21.5			
Recirc. Pump B3 (amps)	28.2		ت بعد ـــ	28.5			
RECITC. Pump 83 (amps)		<u> </u>	$\nu$	<u> </u>	·	<u>:</u>	
Reheater; Steam Flow (M lbs)	out	Out	100+	LOUF	1	1	•
Steam Pressure (psig)		. <u> </u>	_ <u>~~~</u>	1 001	<del></del>	<del>-                                    </del>	
Steam Fressure (psig)		<u>-</u>	<del> </del>	<del>                                     </del>		<del></del> -	
Section A: Presat Water Flow (gpm)	67	68	68.	اجورز	i	,	t t
Outlet Gas Temp. (°F)		1.68	: 115	115	<del></del>	<del></del>	
Bed Diff. (in. H <sub>2</sub> O)					;	<del></del>	<del></del>
Demister Diff. (in. H <sub>2</sub> 0)	-5.U 3.8 = 2		4.0	4.2	<u>-</u>	<del></del>	
RH Diff. (in. H <sub>2</sub> O)	3.8			3.8	<del></del>		
Gas Outlet Flow (in. H <sub>2</sub> 0)	0.62	3 & 0.4	3.0	3.8		· · · · · · · · · · · · · · · · · · ·	
	<u></u>	. <i>U.</i> T	-3-()	3-6	<del></del>	<del></del>	<del></del>
Section B; Presat Water Flow (gpm)	225	! 225	. 245	225	:	Í	
Outlet Gas Temp. (°F)	155					<del></del>	
Bed Diff. (in H <sub>2</sub> 0)		<u>_/</u>	120	170			·
Demister Diff. (in. H2O)	1.8 5.3	15:3	15±.3	15±3		· <del></del>	
RH_Diff. (in. H <sub>2</sub> 0)	3.2	123 - 13	3.2	15±3 3.2	<del>- : :</del>		<del></del>
Gas Outlet Flow (in. H <sub>2</sub> O)		0.4			<del>- :</del>	<del></del>	<del></del>
	0.6	. 0.4	. 0	, 0.3	I	ı	

Comments

#1, 300 F.H #1 400 F.H

	10/4/	10/. / 11	10/1	19/100
Date:	8/6/77	86/71	8/10/77	8/14/77
	1310	1445	2/30	2330
Plant				
Mw,_Gross	102	96	86	84
XS 02 %	3.5	94.0	4.4	5
Steam Flow (lbs/hr) x/03	800	710	660	640
Steam Pressure (psig)	1420	1400	142c	1420
Steam Temperature (°F)	990	980	950 1	970
Opacity, Bypass (%)		19	10 :	10
Opacity, Scrubber (%) FUEL	100% COAL	100 % Care	100% COAL	100% Col
opening, on their (a) pare	10000	1	1	
ESP			200 - 1 BU - 10'	200=10
Section Al-A2; AC Voltage (v)	200	180T 10	-10°	1200-10
AC Current (a)	25	30±20	30:20	35-20
	0 400	37	140 E3 :	140±3
DC Current (ma)	100	120 ± 20	130 tab	1620±30
Spark Rate (spm)	150±50	450150	300±20	250±50
Section B1-B2; AC Voltage (v)	170	170	170±10	170±10
AC Current (a)	25	25±10	30 1 20	30 ± 10
DC Voltage (Kv)	2 out		12-	3
DC Current (ma)	60	60 ± 20	100 T 20	100 = 20
Spark Rate (spm)	60=10	150200	400120	3/0 ± 20
Fan A Outlet Pressure (in. H <sub>2</sub> 0) Fan B Outlet Pressure (in. H <sub>2</sub> 0) Fan A Amps	10.5	12.5 140	10.5	10
Fan_B_Amps	/30	140	130	130
Stack Damp A, Pos. (% Opn)	0		0	10
Pumps; Recirc. Pump Al (amps)	' 33۔	<u>' 23  </u>	23	, 3 ي
Recirc. Pump B1 (amps)	22	22.		
Recirc. Pump B2 (amps)	23	23	2/	121
Recirc. Pump B3 (amps)	24	24	25	25
	1		) ,	(
Reheater; Steam Flow (M lbs)	OUT	OUT	CUT	1 047
Steam Pressure (psig)				
Section_A; Presat Water Flow (gpm)	68.	671	68	68
Outlet Gas Temp. (°F)	120	125	125	125
Bed Diff. (in. H <sub>2</sub> 0)	4.0	4.0	3.8	3.7
Demister Diff. (in. H <sub>2</sub> O)	30	3,4	2.8	
RH Diff. (in. H <sub>2</sub> 0)	2.5	3.0	2.5	2.8
Gas Outlet Flow (in. H <sub>2</sub> 0)	6.0	3,5	OUT	047
	'		•	
Section B; Presat Water Flow (gpm)	22.5	23.5	26	23
Section B; Presat Water Flow (gpm) Outlet Gas Temp. (°F)	160	155	26	23 //5
Section B; Presat Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H <sub>2</sub> 0)	52.2	/55 5, <b>6</b> T, 2	26 115 4.5±.2	/15 4,6 t.2
Section B; Presat Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H <sub>2</sub> 0) Demister Diff. (in. H <sub>2</sub> 0)	5 t . 2 1 t . 2	/55 5, 6 T,2 /, 5 = , 2	115 4.81.2 1.02.4	/15 4,6 t.2
Section B; Presat Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H <sub>2</sub> 0)	52.2	/55 5, <b>6</b> T, 2	115 4.5 I. Z	115

8/6/77 - Units 182 are being regulated Comments

not bee look. 8/10/17 BAME AS ABOVE

Date:	8/12/27	8/12/27	8/13/77	8/13/77
Time:	1330	1500	0920	1055
Plant				
Mw, Gross	116	116	108	115
XS 02	5%	3%	370	2,5
Steam Flow (1007hr) x 7/1	40.150	940	830	910
Steam Pressure (psig)	1420	1420	1420	1920
Steam Temperature (°F)	1000	1000	8	10
Opacity, Bypass (%)	75% COIL	75% COAL	100% COAL	100% COAL
Opacity, Scrubber (%) FUEL  NAT. 945 SCFH X/03		280	1	700.0041
ESP SCEH XIO	280	1 i		
Section Al-A2; AC Voltage (v)	200±10	190±10	190-10	190 ± 10
AC Current (a)	25±10	30±10	25=10	30 <u> </u>
DC Voltage (Ky)	42±2	40£4	40 I 2	140±4
DC Current (ma)	80-120	80=20	170±10	180 T20
Spark Rate (spm)	220±30	200±20	200±50	200140
			m.+	MOTO.
Section B1-B2; AC Voltage (v)	170±10	170±10	Mot 5	17015
AC Current (a)	30=10.	30±10	30 I 10	30±10
DC Voltage (Kv)	70=10 KIT	68±20		60±20
DC Current (ma)			70=10	
Spark Rate (spm)	150 = 20	150±20	150±20	200±50
Scrubber	1	;	1	į
Fans: Fan_Inlet Pressure (in. H <sub>2</sub> 0)	-0,55	-0.6	-0.3	-0.6
Fan A Outlet Pressure (in. H <sub>2</sub> 0)	17	16.5	14,0	15.5
Fan B Outlet Pressure (in. H2O)		16.5	14.0	15.5
Fan A_Amps	150	150	140	150
Fan B Amps	150	150	135	150
Stack Damp A, Pos. (% Opn)	0			0
_Pumps; Recirc. Pump Al (amps)	<u>: 33                                   </u>	<b>23</b>	22	23
Recirc. Pump B1 (amps)	21	21	2/	21
Recirc. Pump B2 (amps)	1 21	21	21	21
Recirc. Pump B3 (amps)	2.5	<i>25</i>	25	25
Debeston, Steen Flor, /H 11-1			:	
Reheater; Steam Flow (M 1bs)	04T	1 64T	OUT	041
Steam Pressure (psig)	<del></del>	<del></del>		1
Section_A; Presat Water Flow (gpm)	.57	68	68	68
Outlet Gas Temp. (°F)	115	120	110	115
Bed Diff. (in. H <sub>2</sub> 0)		4.8	4.2	4.4
Demister Diff. (in. H <sub>2</sub> 0)	4,8 4,4 4,5		3.8	4.6
RH Diff. (in. H <sub>2</sub> 0)	4.5	4,4 4.5	3,5	4,4
Gas Outlet Flow (in. H <sub>2</sub> 0)	7,6	8,2	0	1.5
·		<del>-</del>	<del></del>	,
Section B; Presat Water Flow (gpm)	24	23 '	23	23
Outlet Gas Temp. (°F)	حيد ا	725	120	120
Bed Diff. (in. H <sub>2</sub> 0)	6	(2±,5	5 = . 2	6±.5
Demister Diff. (in. H20)	1.5 = 1.0	1.67.5	1,3± -4	1.5±.3
RH Diff. (in. H <sub>2</sub> 0)	4.	4	3.5	4
Gas Outlet Flow (in. H <sub>2</sub> 0)	1.0	0.4	0.5	0.8

Comments

Date:	8/15/17	8/15/77	8/22/11	8/22/17
Time:	0935	1110	0920	1100
Plant				
	1		4.42	
Mw, Gross	116	116	119	116
XS 02	2.5%	4%	5.5%	BENG CHLIELATED
Steam Flow (lbs/hr) x0	940	940	945	945
Steam Pressure (psig)	1420	1420	1430	14.20
Steam Temperature (°F)	1000	/000	1010	1000
Opacity, Bypass (%)	15%	15%	23%	VOO? COAL
-Opacity, Scrubber (%) FUEL	1/CUPS COAL	100% COAA	1/07)-7-(OAL	VOOTECORE
NAT GAS SCEH	!	-	-	į.
ESP	180±10	170-20	180±2	0 1 180520
Section Al-A2; AC Voltage (v)	35±10	35=10	1 30530	· 35=10
AC Current (a)	35-10	38 = 3	36 1	4: 38=2
DC Voltage (Kv) 38±2	300-30	80 I 20	150	159 B 130 I 20
DC Current (ma)	80 = 20	300 I 30	25015	0 : 230540
Spark Rate (spm)	290 I 30	300 - 30	<u> </u>	30270
CAl 01 00 40 U-14 / /	16.0	170-10	160	160
Section B1-B2; AC Voltage (v)  AC Current (a)	160		5	1015
	35±10	35±10'		70-5
DC_Voltage (Kv)			30	
DC Current (ma) 70-10		70±10		! 30
Spark Rate (spm)	200 ± 40	200 \$ 401	0	
Countle	ı			,
Scrubber	-0.3	-0.3!-	1 2	i-0,3
Fans; Fan Inlet Pressure (in. H20)		<del></del>	1-0,3	
Fan A Outlet Pressure (in. H <sub>2</sub> O)	.16.5			- 1/2
Fan B Outlet Pressure (in. H2O)	14.5	150		
Fan A Amps	150		150	145
Fan B Amps	150	150	15 <u>u</u>	145
Stack Damp A, Pos. (% Cpn)			0	
Bumps Dosine Summ Al (same)		22	- 3r	!
Pumps; Recirc. Pump A1 (amps)	23:	23 1	DFF *	off
Recirc. Pump B1 (amps)	2/		2/	
Recirc. Pump B2 (amps) Recirc. Pump B3 (amps)	<u> </u>	21	2/	
Recirc. Pump B3 (amps)	25	25	25	25
Reheater; Steam Flow (M lbs)	OUT	OUT	NUT	į
Steam Pressure (psig)	<u> </u>		547	
Steam Fressure (bard)				
Section A; Presat Water Flow (gpm)	68	. 68	67	: 67
Outlet Gas Temp. (°F)	120	120		220
Bed Diff. (in. H20)	4,6	4.8	2.5	2.8
Demister Diff. (in. H <sub>2</sub> 0)	4.3			
RH Diff. (in. H <sub>2</sub> 0)		4.5 3.5	9.5	0.9 8.5
Gas Outlet Flow (in. H <sub>2</sub> 0)	<u> </u>	0.4	10	10
	<i>2.0</i>	<u> </u>		
Section B; Presat Water Flow (gpm)	23	23	22.5	23
Outlet Gas Temp. (°F)		- <del> </del>	15.5	
Bed Diff. (in. H <sub>2</sub> 0)	120 5.5.1.5	120 5.5!.5	150 5±.5	/50
	211 + -	2 11 6		4.5±0.5
	110 60 D	2.01.5	2.01.5	1.5 to.5
RH Diff (in Han)	Z : =			1/ /-
RH Diff. (in. H <sub>2</sub> 0) Gas Outlet Flow (in. H <sub>2</sub> 0)	2.0 7.5 5.5 0.5	<u> </u>	0.4	4,5 0.5

Comments & MOTOR ON RECIRC PUMP AT BURNED OUT. WILL TRY AND REPEACE AND KEEP

ON-LINE UNLESS TEMP GETS TOO HIGH (OUTLET TEMP).

	8/23/21	\$/23/77	\$/24/17 8/24/11	824/17 822/7
ate:	0925	1030	0850 1115	1250 1445
ime:				100%. CUAL 100%
Plant FUEL	100% COAL	JOUP, COLL	180% CUAL 100%	1001. COAC 100%
Mw. Gross	10	116	108.116	116 116
XS 02	3.5%	OUT	3.5% 3.5%	3.0% 3%
Steam Flow (1bs/hr) x/03	850	940	850 940	945 950
Steam Pressure (psig)	1430	140	1420 1420	1420 1420
Steam Temperature (°F)	1010	1600	1000 990	990 1000
	1776	1770	17% 22%	20% 22%
Opacity, Byrnss (%)				
Opecity, Scrubber (%) NAT GAS FLOW				
SP (SUF)	*/			l late
	180T10	180 IO	1180 TIO 180	180 180
Section A1-A2; AC Voltage (v)	40±10_	40510	417 E 20 50 510:	45=10 45=10
AC Current (a)	38 E 2	36T2	3752 35±2	36±2 36±2
DC Voltage (Kv)	2320	100 ± 20	140 TXO 130 = RO	140 ± 20 /30 ± 2
DC Current (ma) //o=	20 30 70	450 = 50	450:50 450:50	450±50 45UE
Spark Rate (spm)	380±40	430 - 30	430-30 12-32	
Coction D1 D2. AC Voltage (u)	11-12	160±10	160 160=10	160=10 160
Section B1-B2; AC Voltage (v)	160	70-10	20=5 25=5	20±5 20±5
AC Current (a)		CAT	OUT OUT	OUT OUT
DC_Voltage (Kv) ***	2 OUT		50 = 10   50 = 10	50=10 50=10
DC Current (ma)	20	20	740±50 250550	270=30 250±4
Spark Rate (spm)	120±50	120550	440±30 630-30	210-30 216-9
ath.		'	1 1	
Scrubber		(1.2	10.25 -0.25	0.35 0.44
ans; Fan Inlet Pressure (in. H <sub>2</sub> 0)	- <u>0.3</u>	-0.3		-0.25 -0.25
Fan A Outlet Pressure (in. H <sub>2</sub> 0)		14.5	13.2 15.5	1515_
Fan B Outlet Pressure (in. $H_2^{-}$ 0)	14	14.5	13.5 15.5	1515
Fan A_Amps	145	145	140 150	150 145
Fan B Amps		145	140 150	150 145
Stack Damp A, Pos. (% Opn)			0 0	0 0
	* OFF	OFF	OFF OFF	UFF
Pumps; Recirc. Pump Al (amps)	007	1217	०च्म व्या	DET OF
Recirc. Pump BI (amps)	22	در	21 22	22 22
Recirc. Pump B2 (amps)	21	21	2/ 31	21 21
Recirc. Pump B3 (amps)	25	25	25 25	25 24
			, ,	
eheater; Steam Flow (M 1bs)	out	1 our	DUT OUT	out our
Steam Pressure (psig)				
ection A; Presat Water Flow (gpm)	58	. 58	59 59	58 58
Outlet Gas Temp. (°F)	135	160	150 100	160 195
Bed Diff. (in. H <sub>2</sub> 0)	3.0		13,2 3,2	3,2 3.2
Demister Diff. (in. H <sub>2</sub> O)	0.75	3,0	07.09	0.9 0.9
RH Diff. (in. H <sub>2</sub> 0)	7,5	8.0	7.5 8.5	8.0 8.0
Gas Outlet Flow (in. H20)	2,5	2.4	3.0 3.8	3,0 6.0
PR		—— <del>———————————————————————————————————</del>		
ection B; Presat Water Flow (gpm)	231	. 23	22 22	71 20 5
Outlet Gas Temp. (°F)				22 22.5
Bed Diff. (in. H <sub>2</sub> 0)	4,5 = 3	150	150 150	150 150
Demister Diff /in Uon		4.55.3	4.5 4.5 1.3	5.0 5.02
Demister Diff. (in. H2O) RH Diff. (in. H2O)	1.07.3	1.4t.3	1.4:02 155,3	1,SI. 3 1, SI.
Gas Outlet Flow (in. H <sub>2</sub> 0)	4.0	4.5	4,5 5,0	4.7 4.7
MASS COLT LOT ACTION 118 M (1)		1 0 2	0.2.0	מ אנ
ads outlet Flow (III. n20)	0.4	0.3	0, 2, 0	.25 _ 0

\*\* SIGN ON METER INDICATING OUT OF SERVICE

Date:	8/25/11	8/25/77				
Time:	0620					
Plant						
7			<del></del>	<del> </del>		
Mw, Gross	108	114	Ì	ļ ,		
XS 02	13%	4%	<del></del>			- t
	820 1000	930		<del> </del>		
Steam Flow (lbs/hr) x/0 <sup>2</sup> Steam Pressure (psig)	1420	1420		1		
Steam Temperature (°F)	990		<del></del>	1		
Opacity, Bypass (%)	17%				1	
Opacity, Scrubber (%) FUEL		· /OC/s COAL	<del>i</del>	1		
NAT. GAS FLOW SCF4	70078 20712	- 100 13 (BAL		<del> </del>		
ESP		ļ	1 .		1 :	,
	180±20	170±10	i	į i		
Section Al-A2; AC Voltage (v).	40-20	40±10		1		
AC Current (a)	35±5	33-5				
DC Voltage (Kv)					1	1
DC Current (ma)	140 40	HO120		<del></del>	:	
Spark Rate (spm)	350150	450I50		<u> </u>	<del></del>	
Continu D1 D2. AC Halters (a)	DUIJO	170	-		1	•
Section B1-B2; AC Voltage (v)			<del></del>	<del> </del>		
AC Current (a)		10	<del></del>	<del>!                                    </del>	<del></del>	
DC Voltage (Kv)	out	OUT		<del></del>		
DC Current (ma)	20	50	<del></del>	<del> </del>		<del></del> -
Spark Rate (spm)	50550	100-400		<del>`</del>		<del></del> -
•	! ;	i		1	·	•
Scrubber	· · ·	11		1		:
Fans: Fan_Inlet Pressure (in. H <sub>2</sub> 0)	-0.5	-0.6		<u> </u>		
Fan A Outlet Pressure (in. H <sub>2</sub> 0)	14	15		'	<del></del>	
Fan B Outlet Pressure_(in. H20)			<del></del>			
Fan A_Amps	145	150		•		
Fan_B_Amps	145	150				
Stack Damp A, Pos. (% Opn)	0	0			i	
	1	!	•			
Pumps; Recirc. Pump Al (amps)	OFF	OFF		<u>i                                     </u>		,
Recirc. Pump 81 (amps)	21	21	1	1	i	
Recirc. Pump B2 (amps)	121	12/				
Recirc. Pump B3 (amps)	25	21				
		_	1	!		
Reheater: Steam Flow (M lbs)			<u>'</u>	<u>'</u>		
Steam Pressure (psig)		<del></del>		<u> </u>		
	-,-	م.بـ	1	1 .	İ	
<u>Section A; Presat Water Flow (gpm)</u>	55	<u> </u>		<u> </u>		
Outlet Gas Temp. (°F)		115	<u> </u>	<del>!</del>		
Bed Diff. (in. H2O)	3.4	3,4	<u> </u>	····		<u> </u>
Demister_Diff. (in. H <sub>2</sub> 0)	0.4	1,0				
RH Diff. (in. H <sub>2</sub> 0)	7.7	8.0				
Gas Outlet Flow (in. H-0)						
PR		<u> </u>				
Section B; Presat Water Flow (gpm)	21.5	21		•		
Outlet Gas Temp. (°F)	150	150				······································
Bed Diff. (in. H <sub>2</sub> O)	5.0	4.8				
Demister Diff. (in. H20)	15.3	1.3±4		,	<del></del>	
RH_Diff. (in. H <sub>2</sub> 0)	4.5	50		i		
Gas Outlet Flow (in. H <sub>2</sub> O)	• •			:	!	<del></del> -
	. 0	0.6				<u></u>

Comments

Date:	0/12	19	8/24	8/3	3.26.	8/20	8/28	8/28
Time:	1012	1112	1724	1838	1404	1500	0923	1035
Plant	1012		1-1-2-1-					
	<del></del>	ļ	<del> </del>			<del> </del>	<del> </del>	<del></del>
M. Cross	80	100	11.7	116	11:03	' כוו	103	108
Mw,_Gross						2.6	4.6	3.6
XS 02	OUT	0.+	Out20	1.6		950	760	e40
Steam Flow (1bs/hr) x103	<u> </u>	910	950_	950		7,20	1410	1920
Steam Pressure (psig)	1380	1400	1+20	1220	73.5		980	970
Steam Temperature (°F)	170	990	עבשע	990	12		122	27
Opacity, Bypass (%)	<u>: 11                                  </u>	10	1 30	18		20	<del></del>	
Opacity, Scrubber (%) Fuel	coal	COO	للوصن	_لم_	Cai	COAL	Coal	COal
	1	1			I	ŀ	1	
ESP			[	i .	j I	i		
Section Al-A2; AC Voltage (v)	1 200	200	1625	<u>מנו י</u>	1 1-2	165	125	1752
AC Current (a)	40 =10	40=15	45=5	45 -10	49 3	-2210	40-10	_40 <u>+</u> 5
DC Voltage (Kv)	40	40	34		977		13253	5 ± 2 E
DC Current (ma)	40	80	80	80	රට	30	60	60
Spark Rate (spm)		270-20		>500	12-17		100-20	420-30
Shark vare (Shiii)	<u> </u>	<u> </u>					1	
Section DI D2. AC Voltage /ul		14.5		. ,, .	145	1 , 5	165	165
Section B1-B2; AC Voltage (v)	<u>180</u> -			-466-				
AC Current (a)	_ عائدتـــ	<u> 35 -10</u>	<u>_&lt;2</u> ×_	<25	<u> </u>	1 -7, -	25	25 Ov t
DC Voltage (Kv)		20°	<del></del>		<u> </u>			
DC Current (ma)	720	120	710 + 210	<u>. 50</u>	• •••		1-60	60
Spark Rate (spm)	31.0	430	210220	1277		<del></del>	<u>380±50</u>	_3&: <u>20</u>
				ļ	•	į	<u> </u>	
Scrubber				į	!	•		
Fans; Fan Inlet Pressure (in. H <sub>2</sub> 0)	- 3	4	-4	<u>' - 5</u>	1 7.7	- 7	`~. ( <u>.</u>	7
Fan A Outlet Pressure (in. H <sub>2</sub> 0)	43	15	13.5	150	17.0	120	:13.6	
Fan B Outlet Pressure (in. H <sub>2</sub> 0)		15	15.5	15.0	7- ()		13.0	-40
Fan A_Amps		145	145	1 <u>85</u>	1/45	, <u>-</u> -0	135	_145
Fan B Amps	•			110			Ţ	
Stack Damp A, Pos. (% Opn)	135	<u> </u>	450		15		132	145
Stack Dalip A, POS. (& Upn)	<u> </u>			o	<u> </u>	<u> </u>	<u> </u>	0
O				•	ł	•	ı	
Pumps; Recirc. Pump Al (amps)	23	, 53	' 0	<u>'                                    </u>	<u> </u>		0	6 ,
Recirc. Pump B1 (amps)	21	. 7!	215	! 2:	1215	2::	131	
Recirc. Pump B2 (amps)	21	21	21	21		مستخطعة الماسية r>الماسية الماسية  -21		
Recirc. Pump B3 (amps)	25	1 25	25	25	21	33,	20.5	
			<del></del>				<u> </u>	<del></del>
Reheater: Steam Flow (M lbs)	· out	Out	Out	1001	Our	<u> </u>	Out	out
Steam Pressure (psig)						<del></del>	~~~~~	
The state of the s	· , · · · · · · · · · · · · · · · ·			·		<del></del>	<del></del>	
Section A; Presat Water Flow (gpm)		60	55'	52	27		52	57
	6.20 .	<u></u>						
Outlet Gas Temp. (°F)		ــتعنـــ			1/65	120	28	
Bed Diff. (in. H20)	3.9 -	.,	3.0	3.0	12.2	3,?	4.4	<u>4,8                                     </u>
Demister Diff. (in. H <sub>2</sub> 0)	2.2	14)	0.9	0.9	<u>ي: الم</u>	0.7	0.6	0,6
PH Diff. (in. H <sub>2</sub> D)	4,0	5.0	8.5	8.3	15,42	4.0	6.5	7.6.
Gas Outlet Flow (in. H20)	2.7	3 ()	0,4	0	3.0	3.0	Ü	<b>ፈ</b>
Section B; Presat Water Flow (gpm)	225	100	· 318	235	715	215	22.5	~~~
Outlet Gas Temp. (°F)	-	145					سجبيب	220
Bed Diff. (in. H <sub>2</sub> 0)	450		720	- TZO	<u> 150 </u>	150	<u> 45</u>	
Demister Diff. (in. H2O)	416	4.5	_50_	5.0	6.3	<del>~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~</del>	4.4	5.?
PH Diff /:- U O/	<u></u>	13:4			1.07 /		0.8	
RH Diff. (in. H <sub>2</sub> 0)		<del>- 4'8</del>	<u>.48</u>	4.9	1515	2/2	4.5	3.0
Gas Outlet Flow (in. H <sub>2</sub> O)	0	, 0	Ö	<u>ن</u> (	ပ	. 0	' 0	6.2 _

Comments

<sup>\*</sup> Demister wash water is on

late:	7/27/17	7/27/27	1/20/77	7/20/1-	7/30/27	7/30/2	8/1/27	8/1/77	
ime:	/335	1-41	0218	112/16	1926	1942	1723	1924	_
lant	1333	1246	<u> 10.11.50</u>	1/2/66	1866	1335	1/53	122	
Tant									
Mw. Gross	113	IDL	иo	12L	94	88	114	94	
XS 02 (%)	4.3	3,5	4.0	3.8	5.6	3.)	4.8	5/2	
Steam Flow (1bs/hr)	9.3 KIC 5	8, 2 × 10,0	8 26	49x105	2.5×0°	4.9415	9.3410	7.2×65	
Steam Pressure (psig)	1910	1440	1440	1440	1400	1/120	1120	1920	
Steam Temperature (°F)	990	970	490	980	480	990	495	980	_
Opacity, Bypass (%)	15	15	14	20	15	12	12	12	
Evel	754/20	756 /56			154/2517	756/56	756,656	755/253 -	
SP		,	7.	,-,	,	<b>'</b>	′	, ,	
Section A1-A2; AC Voltage (v)	200+	Z00"	10.00	130	30.45		170		_
AC Current (a)	200	- <del>200</del>	180	175	325	755	120	- 475	
DC Voltage (Kv)				23	36/37	36/37	22/25	24/26	•
DC Current (ma)	1 24	25	23						
Spark Rate (spm)	50	40	12/18-	35 7200/20	192/200	195/200	3:/25	35/25	
Spark Rate (Spin)	!	<i>-</i> 07/35 \	12/10-	704720	0/5-	0/5	7200/98	7260/100	
	₹40 +	300	2,0240	-	4.		i		
Section A3-A4; AC Voltage (v)		200	12.1	300	285	3/0	350	230	
AC_Current (a)	15B-1-	42	24.58		12/41	60	50	45	
DC Voltage (Kv)	443-3	<b>24</b> 48	2142	42	12/9.]	15/44	40/32	37/36	
DC Current (ma)	<b>∙</b> ⊖3∞	a <del>o</del> 3cc	260	300	310	300	200	15,0	
Spark Rate (spm)		0/0	0	0	C	Ó	٥	0	
	i							1	
Section B1-B2; AC Voltage (v)	250++	290	210	175	Tr. Arec	Ticken	260	<b>370</b>	
AC Current (a)	15	39	24	20	برن	Dut	10	ن د	
DC_Voltage_(Kv)	33	<b>₩</b> 22	24	27			37/35	38/35	
DC Current (ma)	40	130	60	50			10/35	50/45	
Spark Rate (spm)	1 70	5/0	0/3	0/0			0/0		_
Shark under Ashmy		, , ,	,	/-				6/0	
Section B3-B4: AC Voltage (v)	20.00	20.43	280	455		3/0	- 350°	370	
AC Current (a)	290	540		290	290		36.2	230	
	- 93	58	93	93	93	93	35	<u>65</u>	
DC Voltage (Kv)	3_8	4230	37	35		40/40	Q +3755	31/31	
DC Current (ma)	9550	835	550	530	590	550	-	370	
Spark rate (spm)	! -	0	0	O	0	0	مهر	٥	
	200	204	•	200	22.0	2012	1 3 2 2		
Section Cl-C2: AC Voltage (v)	290	296	290	255	320	305	320	312	
AC Current (a)	40	93	40	34	56	<u>50</u>	36	34	
DC_Voltage (Kv)	<u> </u>	38	GUT	CUt	700	607	OLT	00-	
DC Current (ma)	140	0	130	110	200 /m	300/95	100/92	135/120	
Spark rate (spm)		0	0/0	0/0	0/0	0/0	0/0	0/0	
	<del></del>						<del>i</del>		
Section C3-C4: AC Voltage (v)	255	250	245	250	255	360	240	295	
AC Current (a)	92	92	91	ç <sub>2</sub>	92	92	<i>8</i> 3	73	
DC Voltage (Kv)	938	38	36	37	38/38	39 5 km -	1/1/41	<del>11/41</del>	
DC Current (ma)	55000 A	6.32	570	530	20	310	520	536	_
Spark Rate (spm)	938 500 100	<b>43</b>	10	73	10	10	130 230	10	
		<del></del> -			<del> </del>		<del> </del>	<del></del>	
Section_D1-D2: AC Voltage (v)	1.			1	1 1			} }	
AC Current (a)									
De Voltage (Kv)					·	<del></del>	<del></del>		
DC Current (ma)					i	<del></del>	<del></del>	<del>`</del>	
Spark Rate (spm)							∮~		•
Shell inne (shell )			·		ļ		<del> </del>		
Section 03-D4; AC Voltage (V)	į l				l i		İ '		
AC Current (a)				•			]		
DC Voltage (Kv)	1								
DC Curkent (ma)									
Spark Rate (spm)				<del></del>			† <del></del>	51%05_	_

Comments
+ 450 v fivetvarions

44 -100 V

# ESP travole tot alarm, execused etack damper

			*		•	
Date:	-	1 0/3	18/3 <sup>-7-</sup> 1	8/4	18/4	8/1 4
Time:		1705	1930	BASE LOAD	1	1427
Plant	··	ASO COAL	75% CORL	55 2 COAL	\$5% COIL	75% Ox 70%
Fuet	25%6A			<i>I</i>	, , , , , , , , ,	
Mw, Gross		109	. 62	110	1/2	
XS 02 Steam Flow (lbs/hr)	x 10 3	3,5 870	5	4	3-5	3.8
Steam Pressure (psig)		1440	1420	14-20	900	880 840
Steam Temperature (°F)		1000	1000	1000	1000	1940 149¢
Opacity, Bypass (%)	., 5	16	12	11	10	12 3
	EH_XIO	2.6.26	0.75		· ···· /•7	1.7
ESP Section Al-A2; AC Voltage	e (v) -	150-400		340±50		303 5-33 3
AC Current	t (a)	250±20 ?	350	28 t 7	340	280 ± 70 °
DC Voltage	e (KV) 24-40/	240-464/	40/40	42/42 (DEFLE	c 1 lug) 42/42	35/35 0
DC Current		40 60/60T20	190/190	90/90	90/90	60/60 60/
Spark Rate	e (spm)	90/10	10/8	30/10	38/8	_110/23 1
Section A=44; AC_Voltage	e (v)	310	320	340	340	340 °
AC Current		? <b>200</b>		47	42	32
DC Voltage	e (Kv)•	40/36	38/34	39/36	40/36	40/26-42/3
DC Curren	t (ma) 85	654 <del>263</del> 6	160/140	150/130	130/110	80/20.
Spark Rate	e (spm) '	do	0/0	0/0	0/0	0/0 '
CI-C2 Section 81-82: AC Voltage	e (v)	: 310	310	330	350	330 350
AC Curren	t (a)	35	50	56	58	
DC Voltag		out	out	out.	OUT	ا يستان
DC Curren		120/120	190/180	170/160	210/210	.140/136 L
Spark Rat	e (spm)	90	0/0	olo	0/0	0/0 %
A3-14 Section <b>A3-14</b> ;_AC Voltag		240±10	, •	250 t 20	270-10	
AC Curren	it (a)	50	. 260	55 ± 5	55±5	
DC Voltag	يو (Kv)	1/388	39/38	39/38	42/40	50 <sup>2</sup>
DC Curren	nt (ma)	200	300	220	250	170 175
BB-84 Spark rat	e (spm)	i <b>o</b> .	. 0	0	0	0 -
Section C3522: AC Voltage	ים (ע)	250±30	290	250 ± 30	230±30	4
AC Curren		70-10	92	70 = 10	75±10	230:30
DC Voltag	je (Kv)	30	38/39	.31/31	31/31	32/32
DC Curren		300= 20	<u>&gt;&gt;</u>	350	370	330 (
Spark rat	te (spm)	0	0	; <b>o</b>	6	\ _Q
Section E3-C4:_AC Voltage	ne (v)	280 '	265	270	270	270-10
AC Currer	nt (a)	94	93	93	93	92
DC. Voltag	ge (Kv)	42042	39/39	40/40	4/4	42/42
DC Currer		550	540	540	540	540 436
Spark Rai	te (spm)	10 '	10	10	0	10 10
Section D1-D2 AC Voltage	na dví	•1			,	
AC Curren	ge /(v) nt (a)			:		
DC Voltag	ge (Kv)	1		· ·		
OG Corre	nt (ma)					
Spark Ra	te (spm)			·		
Section D3-D4, AC Volta	~ /v) ·	• •	- •	•		
AC Curre	gar (v) nt (a)	'.	•		•	i
DC Volta		: ! _	<del>-</del> 			
DC/Curre	rt (ma)					
Spark Ra	te (spm)				;	
Comments + C-CO		· · · · · · · · · · · · · · · · · · ·	, I		1	'

Comments # ESP METERING VERY STEADY,
OBSERVER R. IDA

Date:	18/2/22	8/01/27	8/4/77	8/2/27						
Time:	1000	1041	1356	1503						
	7000	1041	7 2.3 10	1505	-					
Plant	1 2 24 (n 24)	200C (2011	75-/256	201600			··· ·· - ·			
Mw_ Gross	136/256	156	109	156/250	1					ĺ
			3.4	1025						
XS 02 Steam Flow (1bs/hr) x 103	3-5	3.2		06.11						
Steam Pressure (psig)	130	260	915	800						
Steam Temperature (°F)	1 1040	1/40	990 990	485	<del></del>					
Opacity, Bypass (%)	1 12	13	13							
Matil Ger (SCER) XICE	2.0	ر ج	 સ	7.7						İ
	ì					1				
ESP Section Al-A2; AC Voltage (v)	286~10	330,200	20.440	200						
AC Current (a)	20.54	25=4								
DC Voltage (Kv)		25/25								
DC Current (ma)	35/38	10/60	112/12							
Spark Rate (spm)	35/10		707110	35/5		<u>i</u>				
•	33710	7975	13/10	33/3					<b> </b>	<del></del>
Section A3-A4; AC Voltage (v)	300	270	300	295	L	·				
AC Current (a)	30	20	.38	34	,					
DC Voltage (Kv)	37/33	39/34	37/3/5	38/23						
DC Current (ma)		90/20	120/10		· · · · · ·					i —
Spark Rate (spm)		0/0	3/0	0/0	<del></del>	i	•			
C1-C2	0/0	3/6	020	9/0	<del> </del>	<del></del>				<del></del>
Section B1-B2: AC Voltage (v)	335	330	340	320_		1				l
AC Current (a)	57	423	>-7	17		Ţ				
DC_Voltage_(Kv)	c). <del>j-</del>	Out.	71.0	Out						
DC Current (ma)		15/60		205/195				,		
Spark Rate (spm)	; 0/0	0/0	0/0	0/0						
<u>-</u>	1 40	-70	1-70	/	<b> </b>			·	ļ	<u> </u>
Section B3-B4: AC Voltage (v)	220	200	220	310		1		!		:
AC Current (a)	50	48	50	50						
C Voltage (Kv)	35/34	<del></del>	34/33	35/14	1	i				
DC Current (ma)	170	150	170	180		i		<del></del>		:
Spark rate (spm)	1 6	130	5	0	<del></del>	i	·		<del> </del>	i
\$2.174	<del></del>			ļ. ——				<u> </u>		<del> </del>
Section Cl-£2: AC Voltage (v)	>50=18	220510	2020	225710	! {			1	l	!
AC Current (a)	170510		70:5	70510					!	
DC_Voltage_(Kv)	1 32/32			31/31				!	1	
DC Current (ma)		350550	400	34c					<u> </u>	<del></del>
Spark rate (spm)	330	1 3	, 3 <sub>C</sub>	0						<del></del>
	<del></del>		<u> </u>	<u> </u>	<b></b>			<b> </b>	}	
Section C3-C4: AC Voltage (v)	210-20	210±20	200-10	210	!					<u> </u>
AC Current (a)		75±0		70					1	
DC Voltage (Kv)		133/30		1				1	[	
DC Current (ma)	260)	30-40	3:50 520	1 170	;		<del></del>			
Spark Rate (spm)	10	10	10	10			· <del></del>	]		
<del></del>	+	<del></del>	<del> </del>	<del></del>	<del>  -</del>			-	<del>                                     </del>	<del>                                     </del>
Section_DI_D2: AC Voltage (v)	i	1	l	1	<u>                                     </u>			<u> </u>	<u> </u>	L
AC Current (a)				I				1	1	
DC Voltage (Kv)	~ ·	:	1	1						
DC Current (ma)									<u> </u>	i
Spark Rate (spm)		,					-			· ·
		1	<del> </del>	<del></del>	<b> </b>	{		<del> </del>	<del> </del>	<del> </del>
Section-D3-D4; AC Voltage (V)	1	j		<u>!</u>		l				ļ
AC Current (a)	1			•		l		L	<u>i</u>	
DC Volkage (Kv)	J			1	<u> </u>			<u> </u>	<b>!</b>	<b> </b>
DC Corrent (ma)				<u> </u>	LL	l		L	ļ	
Spark Rate (spm)				I				1	1	1
	<u> </u>	\		1	L			ــــــــــــــــــــــــــــــــــــــ	<b>!</b>	<u> </u>

Comments

		/-/					-/-			· <del></del> -
Date:		8/2/77	8/6/11		8/10/11	'} <u></u>	8/10/7	2		<u> </u>
Time:	<u>-</u>	1315	1445		2/45		2330		-}	J
Plant 3 (MILLS)	85% CO	AL	85/2 COA	- 8	59.60	<u> </u>	85% Co.	4		<del> </del>
Me. Gross		60	9/		70	1	72	}		1 .
XS 02	70	6.5	3,4		5.0	1	5.8	7		1
Steam Flow (lbs/hr)	103	450	700		580	(401AL)	540		-1	
Steam Pressure (psig) Steam Temperature (°F)		1420	1440		420		1420			<del></del>
Opacity, Bypass (%)		1000	990	<del></del>	950	<del> </del>	940		<del></del>	+
N. G. PLAN SCFH	× 103	140	160		60		160		<u> </u>	í
ESP										i
Section Al-A2; AC Voltage	(v) *	360	340 0		35		330			<del></del>
AC Current DC Voltage	(a)	60	34 D		60		60		<u> </u>	
DC Current		41/41	120/120		0/37	<del> </del>	200/21		<del>-</del>	<del></del>
Spark Rate						<del> </del> -	2/5	<del></del>	-}	<del> </del> -
BI-87		0/5	25/20		/5	<del> </del>		<del>. </del> -	-{	<del>- </del>
Section A3=A4; AC Voltage	.{v} <u></u> _ <u>k</u> _	300	330	<i>a</i>	80	<u> </u>	280	<u> </u>		<u> </u>
AC_Current		47	48		47	<del> </del>	4.7		-1	<del> </del>
DC Voltage DC Current		36/32	40/36		4/26	.	32/26		-{	<del></del>
Spark Rate		160/135	160/140		<u>0/135</u>	<del> </del>	160/13			<del></del>
CI-CZ		0/0	10/0		10	<del> </del>	do	<del></del>		<del></del>
Section BES2: AC Voltage		3/6	330		250	<u> </u>	280			<u> </u>
AC Current		57	57		56	<del> </del>	57	<del>:</del>		<del></del>
DC Voltage_ DC Current		260/200	OYT		OUT V/145	<del> </del>	out		- <del> </del>	<del> </del>
Snark Pato	(spm)	0/0	210/200			<del> </del>	200/19	<del>`}</del>	<u> </u>	· <del> </del>
As-A4 Spark Rate			0/0		0/0	ļ	0/0	· <del></del>	<del> </del>	<u> </u>
Section B3-R4: AC Voltage	(v)	280	270±20	350 3			250I2			
AC Current		60	155 ES	<u> </u>	<u> 53</u>		SSI			4
DC Voltage DC Current		41	40		40/	7——	35/3	<del>/</del>		!
Snark rate		300	270		00	<del> </del> -	300	<del></del>	+	<del> </del>
	(op)		0		0	ļ	0	<del>-}</del>	_}	ļ
Section_C1=EZ:_AC_Voltage	(v)	280	1300±10		0 I 30		230 T			1
AC Current	- (a)	92	90±10		0=10		80 T		<u> </u>	
DC_Voltage DC_Current	(ma)	3k 540	39-35   500±50		8/38 CC IS		36/36		<del></del>	<del> </del>
Spark rate	Spm)	0	0	ح. ا	0	<del>]</del>	450±	5. <u>0</u>	<del></del>	<del> </del>
C3-84		<del>  </del>	+			1		<del></del>	-}	<del> </del>
Section_CZ=C4: AC_Voltage	_(v)	250	260		20*	<b></b>	260	-		<u> </u>
AC Current	-\a)	93	92		92	ļ	92	- <del> </del> -	· I	ļ
DC Voltage DC Current	-\ma\	37 530	38 530		40	<del> </del>	38/38	<del> </del>	<del></del>	<del> </del>
Spark Rate		10	10		10	<del> </del>	540	- <del> </del>		
	/		1 1		, 0	<del> </del>	10	<del> </del>	<del></del>	<del> </del> -
Section_D1-D2: AC Voltage	(y)		11			<u> </u>		<u></u> .	<u> </u>	1
AC\ Current	(a) _								i	
1 / NC WAITE	/\\\\		<del> </del>			<b>]</b>		<del> </del>	1	
DC Woltage	'ima'					<del> </del>		- <del>-</del>		
DC Current	(spm)		<del> </del>	i		1	ĺ	1	i	ı
DC Current Spark Rate	(spm)							<del> </del>	· <del> </del>	<del></del>
DC Current Spark Rate	(spm)									
DC Current Spark Rate Spark Rate AC Voltage AC Current	(spm)									
DC Current Spark Rate Spark Rate AC Current AC Current DC Voltage	(spm) (v) (a) (Kv)									
DC Current Spark Rate Spark Rate AC Corrent DC Voltage DC Current	(spm) (v) (a) (Kv) (ma)			•						
DC Current Spark Rate Spark Rate AC Current AC Current DC Voltage	(spm) (v) (a) (Kv) (ma)									

Comments & ALL READINGS STEADY

D = DEFLECTING

Date:	8/12/17	8/12/17	8/13/11	8/13/71
[ime:	7340	1510		1100
Plant FUEL	75% COAL	75% CGAL	75% COAL	757. COIL
Mw. Gross	103	102	92	106
XS 02	4%	590	400	12
Steam Flow (lbs/hr) x/0	810	810	720	850
Steam Pressure (psig) Steam Temperature (°F)	1420	1920	1410	1420
Onacity, Bypass (%) 75	7373001K	- 1000 	5%	230
NAT. GAS FLOW SCFH	230	220	720	230
ESP			i i	
Section Al-A2; AC Voltage (v) AC Current (a)	340	340 54	250 = 100 20 = 7	180 - 30
DC Voltage (KV)	39/39	39/38	26 t 6 /30 110	2414/2514
DC Current (ma)	190/190	160/160	50±30/50±30	35±10/35±10
Spark Rate (spm)	5/5	12/10	* 150/15	7200/50
B1-B2				
Section A3=74: AC Voltage (v)AC Current (a)	320	320 55	310	290 * leflection
DC Voltage (Kv)	37/32	37/34	37/34	38/34 meters
DC Current (ma)	20/180	210/170	140/110	50/45 IN SECTIO
Spark Rate (spm)	0/0	0/0	0/0:	:0/0
C/-CZ	1 ' .	7	1 /	350
Section B1-B2: AC Voltage (v) AC Current (a)	300 56	<del>700</del>	320 58	55
TOP / Bairom DC Voltage (Kv)	OUT	047	OUT	OUT
CI/Cz DC Current (ma)	200/190	1200/190	210/200	180/180
Spark Rate (spm)	10/0	0/0	0/0	0/0
P3-114	90 E 10 i		190±10	
Section 93=04: AC Voltage (v) AC Current (a)	50°E 5	190±10 50±5	5015	180±10 45±5
DC Voltage (Kv)	31/30	29/28	3//3/	29/29
DC Current (ma)	180 = 20	180±30	150	180±26
Spark rate (spm)	0	0	0	0
B3-84/3park rate (3pm)	200:20	210 ± 20	220=10	200 T20
Section Class: AC Voltage (v)  AC Current (a)	70 ± 10	75=10	70±10	180±10
DC Voltage (KV)	28/29	27/27	30/301	25/25!
DC Current (ma)	375 tao	400	350140	1450±50
Spark rate (spm)	0	10	0	0
20 24 20 14 14 20 (11)	200 £ 20	201±10	200 T 20	2007/0
Section C3-C4: AC Voltage (v) ' AC Current (a)	75+10	75=10	70 \$10	801/0
DC Voltage (Kv)		31/30		30/30
DC Current (ma)	31/31 28:0±20	400 E 20	33/33 350±50	36/30 140 140
Spark Rate (spm)	10	10	10 .	
0 00 00 00 Holes (m)				
Section D1-D2; AC Voltage (v)  AC Current (z)				
DC Voltage (Kv)	<del></del>			<del></del>
DC Current (ma)				
Spark Rate (spm)	1	!		
Santian D2 D4: 45 Valence (v)				
Section_D3-D4: AC_Voltage (v) AC_Current (a)		<del></del>		
DC Voltage XKv)		'		
DC_Current (ma)				
Spark Rate (spin)			}	1 1 1

\* NO READING, CALIDRATING CHART
\*\*\* SPARK RATE WP FOR PREVIOUS

Date:	8/15/77	8/11/17	8/22/17	8/22/11	
Time:	0940	1115	0430	1100	
Plant FuEL	75% COAL	75% COA4	75% COAL	757, CUAL	
Mw. Gross	110	115	0415	//3	
XS 02	13%	740	4%	3,5%	1
Steam Flow (lbs/hr) x/112	870	920	930	930	
_Steam_Pressure (psig)	1420	1420	1420	1420	
Steam Temperature (°F)	13%	1000	1010	990	
Opacity, Bypass (%)  NAT GAS FLOW SCF		290 390	320	201/0	
ESP MAT GAS FLOW SCF	,				
Section A1-A2; AC Voltage (v)	300	300	325	342	
AC Current (a)	58	1 58	57	340	
DC Voltage (Kv)	34/34	34/34	36/37	39/39	
DC Current (ma)	195/300	190/200	190/190	200/200	!
B/-62 Spark Rate (spm)	0/5	0/5	0/5	0/5	
Section A3=A4; AC Voltage (v)	250	250	21.0	250	1
AC_Current (a)	58	52	50	50	<del>i</del>
DC Voltage (Kv)	28/24	30/25	30/24	30/26	
DC Current (ma)	160/140	170/160	165/140	1/60/140	
Spark Rate (spm	010	0/0	0/0	10/0	
C/-C2		<del></del>			<u> </u>
Section Bar B2: AC Voltage (v)_	* 240   56	240 40	240	240	
AC Current (a) DC Voltage (Kv)	OUT	OUT	CUT	56 04T	<del>i</del> -
DC Current (ma)		195/40	195/190	195/190	
Spark Rate (spm	) '/ i			2/01	
A3-A4	<u> </u>	0/0	0/0		
Section B3-B4: AC Voltage (v)	230!	235	320	325	
AC Current (a)	<u>57</u>	59	58	59	j
DC Voltage (Kv)	34/34	35/34	145/45	48/46	
DC Current (ma) Spark rate (spm	, ————————————————————————————————————	300	3'00	300	
B3-84 Spark rate (spiii	) 0	0			
Section CI=CZ: AC Voltage (v)	250	1250	270	250	
AC Current (a)	91	91	91	91	
DC_Voltage_(Kv)	32/33	33/33	35/36	36/36	
DC Current (ma)	540	540	530	520	
Spark rate (spm	"   0	0			
Section C3-C4; AC Voltage (v)	230	230	220	220	
AC Current (a)	92	42		91	
DC Voltage (Kv)	34/34	34/34 520	91 32/32 420	9/ 32/3 L 520	
DC Current (ma)	520	530	420	:5,20	
Spark Rate (spr	<sup>n)</sup> /o	in	10	10	
5					
	<del></del>		<del> </del>	<del></del>	
DC Voltage (Kv	·				
DC Current (ma	)				
Spark Rate (spi	n) ;				
Section_D3-D4; AC Voltage (v)					
AC Current (a)					
DC Volkage (Kv	) . <u> </u>				
DC Current (ma	)i				<del></del>
Spark Rate (sp	m/ : }	1 1	1 1	1 1 1	1

)ate:	18/23/71	8/25/17	8/24/17	8/24/11	8/29/71	8/24/77
[1me:	10930	1035	0855	1115	1255	1450
Plant FUEL	759, CM4	75-9,	75 % COH	_ 75%. COAL	75% COAL	75% 6
Mw. Gross	104	113	82	115	115	115
XS 02	5-5%	5,5%	4.5%	4.0	4%	3.5%
Steam Flow (lbs/hr) x//2	840	850	650	940	940	940
Steam Pressure (PS19)	1420	1420	1420	1420	1420	1420
Steam Temperature (°F)	990	10:00	15%	1000	15%	1000
Opacity, Bypass (%) NAT GAS FLOW (SCFH)XI	3 13%	15% 3CD	110	15% 310	3/6	13/0
SP	: 1					
Section A1-A2; AC Voltage (v)	340	360	350	340	340	1350
AC Current (a)	56	58	562	58	340 57	58
DC Voltage (Kv)	39/39	41/41	36/37	37/39	40/40	40/40
DC Current (ma)	190/200	200/200	180/190	200/200	200/200	200/20
Spark Rate (spm	.0/5	0/5	0/5	0/51	0/5	0/5
Section A3-A4; AC Voltage (v)	290	290	280	290	300	300
AC Current (a)	58	1 56	58	58	58	58
DC Voltage (Kv)	34/28	134/28	33/28	34/30	34/30	35/30
DC Current (ma)	200/175	190/160	200/170	200/170	190/160	200/17
Spark Rate (spm	0/0	10/0	0/0	0/0	6/0	1 0/0
Ct-Cl-Cl-Cl-Cl-Cl-Cl-Cl-Cl-Cl-Cl-Cl-Cl-Cl-	· / · · ]	250	240	240	250	250
Section B1=B2: AC Voltage (v)	240	56	56	56	56	- 22
DC Voltage (Kv)	OUT	CUT	047	OUT	OUT	OUT
		200/190	190/190	190/190	190/190	190/185
DC Current (ma) A3 -A4 Spark Rate (spm	)	1 /		0/0	do	/
A3_H4		<u> </u>	0/0			0/0
Section B3=B4; AC Voltage (v)	320	220 I 40	290	270=20	2505201	250 T
AC Current (a)	5,7	50.55	58	55±5	50±5	50±
DC Voltage (Kv)		34/34	142/40	35/3.7	38/37	38/3
DC Current (ma)	250F20	150510	300	230	200	1,200
63-54 Spark rate (spi	0			. (2		
Section Cl=E2: AC Voltage (v)	295	230 I 40	290	25U=40	230±5U	1220±
AC Current (a)	9.2	70±20	91	80±30	70±101	70 I/
DC_Voltage_(Kv	38/38	3.2/3.1	38/38	32/33	30/30	30/30
DC Current (ma)		350 E \$0	540	400=50	310 = 30	1300 52
Spark rate (spr	" 0		0	0	101	9
Section C3-C4: AC_Voltage_(v)	230	250	230	240	250	250
AC Current (a)	91	92	9/	91	192	92
DC Voltage (Kv		37/37		35/35	36/36	37/3
DC Current (ma	520	37/37 530	570	520	520	530
Spark Rate (spi	n) , /O	10	10	10	10	10
> (v)						
Section_D1-D2; AC Voltage (v) AC Current (a)		<del> </del>	<del> </del>	<del></del>		
DC Voltage (Kv	/ ,i	<del></del>	<del></del>	<del>  </del>	<del> </del>	
DC Current (ma	<u> </u>	<del></del>	<del></del>	<del> </del>		
Spark Rate (sp	m)					
	<del></del>	<del></del>	<del> </del>	<del> </del>	<del></del>	
$\_$ Section_D3-D4; AS Voltage (v)	. <del></del>			<del>}</del>		
AC Surrent (a)	<del></del>	<del></del>	<del></del>	<del>  </del>		<del></del>
DC Voltage (Kv	<u> </u>	<del></del>		<del>  </del>		
DC_Current (ma Spark Rate (sp	m)	<del></del>	<u> </u>			<del></del>
Act back vard lak	···,	. i		1 1	1 1	ì

Date:		8/25/11				
Time:	0630	0530				
Plant FUEL	75% CORL	75% cons				
Mw. Gross	1 98	112				
XS 02	9.5	4.5	<del></del>		<del>  </del>	_
Steam Flow (1bs/hr)	940	940			<del> </del>	
Steam Pressure (psig)	14.20	14.20			<del>  </del>	<del></del>
Steam Temperature (°F)	980	1000			<del> </del>	<del></del>
Opacity, Bypass (%)	13%	300				
NAT GAS FUW ( SCFH)	240	300			<b> </b>	
ESP				<u> </u>		
Section A1-A2: AC Voltage (v)	≠ 370	* 360			ļ	
AC Current (a)		1 54	<del>~~ ~~~</del> }~~	<del></del>	<del></del>	<del></del>
DC_Voltage (Kv)		× 40/40			<b></b>	<del></del>
DC Current (ma)	200/210	* 183/180			<del> </del>	
Spark Rate (Spi	3/5	4/5				
Section A3-A4; AC Voltage (v)	300	700		ł	1	1
AC Current (a)	60	58				
DC Voltage (Kv)		34/29				1
DC Current (ma)	210/180	190/160				
Spark Rate (spr	n) 0/a	0/0		i	1	1
				— <del> </del>		
Section B1-B2: AC Voltage (v)	240	250			<del> </del>	<del></del>
AC Current (a)	<u></u>	150		<del>-</del>	<del> </del>	<del></del>
DC_Voltage_(Kv		OUT		<del></del>	<del>  </del>	<del></del>
DC Current (ma	200/190	200/190			<del></del>	<del></del>
Spark Rate (spi	0/0	0/0			<u>!</u>	
Section B3-B4; AC Voltage (v)	320	270 ± 30			1	!
AC Current (a)	59	S.2± 3			1	
DC Voltage (Kv		42/41	<u> </u>		:	
DC Current (ma	300	220±20			i	i
Spark rate (sp	m) &	10				
		250 t 50	<del></del>		<u> </u>	
Section_Cl=C2: AC_Voltage (v)	300		<del></del>		<del> </del>	<del> </del>
AC Current (a)	94	75-1-20		<u> </u>	<u> </u>	
DC_Voltage_(Ky		36/36			<del>  </del>	<del></del> -
DC Current (ma	550	400 ESU			<del>:</del>	
Spark rate (sp	m) 0	0				
Section C3-C4: AC Voltage (v)	235	240			<u> </u>	
AC Current (a)	92	92			<u> </u>	
DC Voltage (Kv	35/35	36/36				
DC Current (ma	540	520				
Spark Rate (sp	m) /0	10		· · · · · · · · · · · · · · · · · · ·		
Section_D1-D2: AC Voltage (v)	\- <i>/</i>				<del>  </del>	
AC Current (a)	·				!	_
DC Voltage (K)	?					_
DC Current (ma	<u>''</u> \	<del></del>			<del> </del>	
Spark Rate (sp	om)					
Section_D3-D4; AC_Voltage (v	) [	!				
AC Current (a	)					
DC Voltage (K	v)					
DC Current (m	a)					
Spark Rate (s	pm)			1	1	
2 Ann 1 Ann 12	' ' (   <b> </b>	l L		t	1 1	1

Comments & DEFLECTING

Date:	
Plant	5.0
My Gross	
My Gross	
Steam Flow (lbs/hr)   x 103   750   400   930	
Steam Flow (1bs/hr)   x 103   950   40   930   930   10   550   720   120	
Steam Pressure (psig)	
Steam Temperature (?F)	
Opacity,   Opacity,	0.
Seption Al-A2; AC Voltage (v)   300   310   370   175   415   415   250 250   290	0.
Section Al-A2; AC Voltage (v)   300   310   370   175   915   2502x0   290	0.
Section A1-A2; AC Voltage (V)  AC Current (a)  BC Current (ma)  Control (a)  AC Current (ma)  Control (ma)  Contro	0.
AC Current (a) 42	0.
DC Voltage (KV)   35/36   22-1	0.
DC Current (ma)   ROPER   ROS/195   200/40   PS 32   TOO   100/11	5.
Spark Rate (spm)   O/S   O/S   S/S   S/S   S/S   O/S	5
Section A3=A4; AC Voltage (V) 230 273 360 300 70 370 290 270  AC Current (a) 43 75 360 300 70 370 290 371 32/22  DC Voltage (KV) 29/21 29/21 38/30 25 30 32 33/21 32/22  BC Current (ma) 550/50 125/22 260/10 265/25 35/17 27 7 190/50 135/25  Section B1=B2; AC Voltage (V) 240 240 250 250 250 10 260 200 260  AC Current (a) 51 55 56 56 10 57 57 57  DC Voltage (KV) 29/21 09 09 09 09 07 0 00 00 00 00 00 00 00 00 00 00 00 0	5.
Section A3=A4; AC Voltage (V) 230 277 380 300 70 320 280 270  AC Current (a) 43 75 35 59 77 52 42 32 3347 32/22  DC Current (ma) 50/30 85/32 260/70 265/73 25/72 25/73 32/42  Section B1=B2; AC Voltage (V) 240 240 250 250 250 250 260 200 260  AC Current (a) 51 75 55 56 76 77 57 57 57 57 57 57 57 57 57 57 57 57	
AC Current (a) 29/22 23/27 32/20 25/20 25/20 23/27 32/20 25/	0
DC Voltage (KV)   29/2   24/2   25/2   28/3   35/2   33/7   33/2   33/	0
DC Current (ma)	0
Spark Rate (spm)   O/O   C/O   O/O   O/O   O/O   O/O   O/O	
Section B1=B2: AC Voltage (v)	
Section 81-82: AC Voltage (V)	1
AC Current (a) 51 55 56 57 57 57 57 57 DC Voltage (Kv) 201 201 201 201 201 201 201 201 201 201	_]
DC Current (ma) 175/90 Pr 100 M3/80 175/90 75 Pr 200/95 Spark Rate (spm) 0/0 0/0 0/0 0/0 0/0 0/0 0/0 0/0 0/0 0/	
DC Current (ma)   175/10  PC   100   190/185   175/190	_ ! !
Spark Rate (spm)	s l
Section B3-B4; AC Voltage (V)  AC Current (a)  AC Current (a)  AC Current (a)  AC Current (a)  AC Current (ma)  AC Current (a)  AC Current (a)  AC Current (a)  AC Current (a)  AC Current (ma)	
Section B3-B4; AC Voltage (v) 265 2135 220 220 220 220 220 220 220 220 220 22	<del></del>
AC Current (a) 59 52 48 48 50 50 50 56 56  DC Voltage (Kv) 40/30 41/30 53/33 35/34 27 37 32 41/39 91/40  DC Current (ma) 300 700 160 160 120 20 310 310 310  Spark rate (spm) 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
DC Voltage (KV) 40/30 41/30 53/33 35/34 37 37 38 39 39 37 37 38/38 38/38 37 37 38/38 38/38 37 37 38/38 38/38 37 37 38/38	
DC Current (ma)   300   700   160   160   150   310	
Section 61-62: AC Voltage (v) 250 230 20 230 -30 -10 270 2	ı
Section 61=62: AC Voltage (v) 250 250 230 230 230 230 230 230 230 230 230 23	
AC Current (a) 91 1 70 10 70 10 10 10 10 10 10 10 10 10 10 10 10 10	<del></del>
DC Voltage (KV) 33/33 23/2 30/21 - 3 38/38 38/38 58/3  DC Current (ma) 526 520 260 300-20 302 7 5 50 550 550 550 550 550 550 550 550	<u> </u>
DC Current (ma) . 520   520   260   300   20   372   250   5	
Spark rate (spm) 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	81
Section 63=64; AC Voltage (v) 215 215 260 260 16. 235 235 AC Current (a) 92 21 93 93 23 23 92 92 93 00 Voltage (kv) 32/31 22 238/38 38/38 2 39 37 37 35/35 35/35	
Section 63-64; AC Voltage (v) 215 215 215 260 260 162 235 235 235 AC Current (a) 92 21 93 93 21 21 92 93 21 21 21 21 21 21 21 21 21 21 21 21 21	
AC Current (a) 92 27 93 93 37 92 93 93 00 Voltage (Kv) 32/31 77 7 38/38 38/38 3 37 37 35/35 35/35	1
DC Voltage (KV) 32/31 77 77 38/38 38/38 27 39 37 37 38/38 38/38	
DC Cumpont (mg)	
UC current (ma) 520 530 530 530 530 530 530	
Spark Rate (spm) 10 10 10 10 10 10	
Carrie and the Marketon (v)	1
Section Dt-D2. AC Voltage (v)  AC Current (a)	
DC Voltage (Kv)	<del></del>
DC Current (ma)	
Spark Rate (spm)	
SPECIAL CONTRACT CONT	
Section D3 D41-A6-Yoltage (*)	
AC Current (a)	
DC Voltage (Kv)	
DC Current (ma)	
Spark-Rate (spm)	
Sporter made Committee Com	

<sup>\*</sup> Soot Blowing in progress; to be completed in ~ 30 min.

						<del></del>			<del>,</del>	
Date:	7/32/72	7/27/22	7/24/27	7/24/17	7/20/17	7/30/77	8/1/27	7/1/27	<b>1</b>	<del> </del>
Time:	13.55	1533	0735	1057	1849	. 1951 .	1836	1635		ļ
Plant				•		į	1			i
	•	,	1							,
Mw, Gross	153	110	145	145	132.	132.	134	136		
XS 0 <sub>2</sub>	. Z.L.	3.6	3-6	4.0	2.6	2.5	2.3	1.3.£.		
Steam Flow (lbs/hr)	W70xi	834	115040	1.15+10	1,01,00	1.042106	106,106	1.21.40	¢'	•
Steam Pressure (psig)	1800	7800	1800	1900	1800	1800	الأورثور	7900	:	
Steam Temperature (°F)	1000	220 -	105()	(A 22.5	Imago-	1000	100.0	1000		
Opacity, B <del>ypass</del> (%)		130	13	1050	200.0	3	1 700.0.	1 7000		
Opacity, Scrubber (%)	1	-18	!~	- 12	. سيمر	j- <b>~</b> - ,		<u> </u>	,	
- Sau Gas Flow Gray hat -3 c = H				: <del>-</del> -	1.5140	المن كامرا <i>م</i>	1.7210	. 1.7ء ۾		
		: ,		•	·	1	i	İ		
_ESP Ba-b-2 _Section_A <del>1-A2</del> ;_AC_Voltage_(v)		120	٠. ٠	170	100	1,4	1,50	1	· · · · · ·	
AC Current (a)	.170 .		1.70	. 170.		/65	170	-+70-	•	
DC Voltage (KV)	- 5 <del>0</del> 5-		10.7	10 7	<i></i>		107	107-		<del></del>
	<u> 40</u> _	_10	40	_4	40	40	+41	10		<del></del>
DC Current (ma)			-6 <u>6</u> 0	- ଜ୍ୟୁ	330.	590 .	-600-	-600		
Spark Rate (spm)	0	0	0	0		0	, 0	. 0		
A1- A2	<del></del>		~~							
Section B1-B2; AC Voltage (v)	50	<u> </u>	<u>50</u>	<u> </u>	_50_	_50_	_<50_	—হত		<del></del>
AC Current (a)	<u>-9</u> υ	<del>-10</del> -	-90	9-3.	<u>42</u>	-92-	ସୁ≨ି	95		_i
DC Voltage (Kv)	·8	8	_8		ع	<u> </u>	<u> </u>	<u>:8</u>		<del></del> -
DC Current (ma)	510	<u> </u>	<u>-₹</u> 0	510	<u> 510</u>	1 250	<u> </u>			
Spark Rate (spm)	0	<i>5</i> .	Q	O	, 0	, 0	0			
					<u> </u>	1		<u> </u>		<u> </u>
_Scrubber					-Serubly	scr.R	-25.62	- Schr	•	:
Fans; Fan A Inlet Pressure (in. H20)	6	. 0	Ο.	· O	Not it	L. Not is.	. Nec.in	. Not in	·	
Fan B Inlet Pressure (in. H2O)		<u>- 8 </u>	-1.5	6	_حصن دراً	Service	Service	<u>servi</u>	L	. !
Fan A Outlet Pressure (in. H2O)		105	135	13.5		1				1
Fan B Outlet Pressure (in. H2O)	<del>7</del> 1	10:5	135				·	' 		
Fan A Amps	230	190	215	220				!	•	j
Fan B Amps	210.		315	220		• • • • • • • • • • • • • • • • • • • •		-;		
Stack Damper A, Pos. (% Opn)		G	0	0						
Stack Damper B, Pos. (% Opn)	<u>،</u>	<u>v</u>	7	. 0		<del></del>		1		
and the second s					· <del></del>					
Pumps; Recirc. Pump Al (Amps)	20	<u>. 23</u>	23	. 24	1	,				1
Recirc. Pump B1 (Amps)	25	25	25		<del></del>		· · · · · · · · · · · · · · · · · · ·			
Recirc. Pump B2 (Amps)				25			<del></del>		****	
Recirc. Pump B3 (Amps)	22	20	22	- 22	<del></del>	<del></del>			<del></del>	
Recirc. Pump C1 (Amps)	20 04-4		71	20 , Dain				<del></del>		
RECIFC. Pullp CI (Allips)	GOTA.	7 0000.4	0	Dan				-	}	1
Pahantana Chana Elay (M. lhc/hm)	Out	۱ +ره	7. E	101	ī		i		† —	1
Reheater; Steam Flow (M lbs/hr)	OUF.	- 05.	<u> </u>	<del> </del>	<del>i</del>	<u> </u>	<del></del>	<del></del>		+
Steam Temp. (°F)	<del></del>	<del></del>	<u> </u>	<del>- 1</del>	1	<del></del>	- <del> </del>		_i	-
Steam Pressure (psig)				• 1	1	1	1	<u> </u>	1	
A A B		. 3-		4-	1	-		:	!	
_Section A: Presat. Water Flow (gpm)		72	73_	167	<del> </del>	<del>-</del>				
Outlet Gas Temp. (°F)	120	105	125_		<del></del>		· i	·		- }·
Bed Diff. (in. H2O)	<u> </u>		8.3	:- B-9-			4			
Demister Diff. (in. H2O)		<u> </u>	_0	<u>, o</u>	<u>'</u>	<del></del>	<del> </del>			-
: RH Diff. (in. H20)	_1,0	ــــــــــــــــــــــــــــــــــــــ	07	10,7	<del></del>				<del></del> -	
Gas Outlet Flow (in. H2O)	Ov†	107	004	Ou+	-		1		;	ł
				<del></del> -	- <del></del>	<u></u>	- !	;		
Section B: Presat. Water Flow (gpm)	210	<u>טוג</u>	210	194	<u> </u>			<u></u> .	· · · · · · · · · · · · · · · · · · ·	
Outlet Gas Temp. (°F)	120	130	115	115	L		_1	1		_i
Bed Diff. (in. H <sub>2</sub> 0)	6.4	5.4	6.4	7.0				1		1
Demister Diff. (in. H2O)	1.2	101	1.6	1.5			1			
RH Diff. (in. H <sub>2</sub> 0)		- RHOUT	D)1 0.1	1.5	1	1		1.	!	
Gas Outlet Flow (in. H2O)	- 7/74/007	+ 0 0+	OUT	: 03+	<del></del>	1	1	$\top$		1
222 222 1 100 1 100 100			<u> </u>	<del></del>	+		<del>-i</del>		<del>.</del>	-+
Section C; Presat. Water Flow (gpm)	. <b> ?</b> o_	70	72	- 62	;		•	1	!	- [
Outlet Gas Temp (°F)		120			!	<del></del>	<del>                                     </del>	+		-+
Bed Diff, (in, H20)	130	3.4	34_	110	<del> </del>	<del>                                     </del>	<del>- </del>	+	<del>-i</del> -	<del></del>
ped pitt. (in. nyo)	<del></del> _			3.2	<del> </del>	<del>-i</del>	-+	<del></del>	<del></del>	
	_ /									
Demister Diff. (in. H2O)	_0.6	<del>, 0,4</del>	0.8	0.8	<del></del>		<del></del>	<del></del>		
	0.6		0.8	0.92	-		<del> </del>	<b> </b>	<del></del>	
Demister Diff. (in. H2O)										

## · Comments

Heil Mountains Phogramy exact operations

<sup>+</sup> No steam flow

<sup>\*</sup> Instructions were if Unit 1 on coal fix unit 3 on ges

Date: Time: Plant	' 8/3 1730	8/3 1950	8/4 1045 BASE LOAD	184	8/5 8/1 1459 151
Mw, Gross (CHAQT) XS 02 Steam Flow (lbs/hr) Steam Pressure (psig) Steam Temperature (°F) Opacity, Bypass (%) Opacity, Scrubber (%)  NATULAL GAS FLOW (SCFH) ×10 ESP	58 500 1700 930 369, NG 800	135 1.5 1075 1860 950 35% NG	135 tag 1.2 1050 1800 975 0 NG 1550	132 1.2 103 <b>0</b> 1800 1020 No	13 1 13 1000 10 1000 1000 1000 1000 100
Section Al-A2; AC Voltage (v) AC Current (a) DC Voltage (Kv) DC Current (ma) Spark Rate (spm)	. 50 100 8 520 0	50 90 8 510	50 95 8 57.0 0	50 95 8 520 0	50 C: 95 F 8 F 520 570
Section B1-B2; AC Voltage (v) AC Current (a) DC Voltage (Kv) DC Current (ma) Spark Rate (spm)	170 110 41 <b>6</b> 1 600 •0	170 110 40 <b>#</b> 600 0	170 110 40 600	170 112 40 600	170 1 107 108 41 34 600 6
Scrubber Fans; Fan A Inlet Pressure (in. H <sub>2</sub> 0) Fan B Inlet Pressure (in. H <sub>2</sub> 0) Fan A Outlet Pressure (in. H <sub>2</sub> 0) Fan B Outlet Pressure (in. H <sub>2</sub> 0) Fan A Amps Fan B Amps Stack Damper A, Pos. (% Opn) Stack Damper B, Pos. (% Opn)	NOT IN SERVICE	NOT IN SERVICE	NOT IN SERVICE	NIT IN SERVICE	MGT MOT
Pumps: Recirc. Pump Al (Amps) Recirc. Pump Bl (Amps) Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump Cl (Amps)					
Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)		1			
Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H2O) Demister Diff. (in. H2O) RH Diff. (in. H2O) Cas Outlet Flow (in. H2O)			-		
Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H2O) Demister Diff. (in. H2O) RH Diff. (in. H2O) Gas Outlet Flow (in. H2O)			-	-	
Section C; Presat. Water Flow (gpm) Outlet Gas Temp (°F) Bed Diff. (in. H2O) Demister Diff. (in. H2O) RH Diff. (in. H2O) RH Diff. (in. H2O) Gas Outlet Flow (in. H2O)					
Comments 8/3/77 - 1750 OPERATING OBSERVER R. IDA	.! ! on ! BFW	pump 2	000 - 2 Bfm	Pumps of	ERATING

Date:	8/0/	2/9/22	8/4/27	8/2/2			7	γ		
Time:		1051	1411							
Plant	1015	1101	<del>  /**                                  </del>	1314	<del>  </del>	<del></del>		1		
· · · · · · · · · · · · · · · · · · ·	<del> </del>		<del> </del>							
Mw, Gross	/37	135	137	135						
XS 0 <sub>2</sub>	1.7	1.9	2.3	2.2						<u> </u>
Steam Flow (lbs/hr) × 103	1020	1050	1050	1050	· ·					
Steam Pressure (psig)	1800	1800		1800	<u> </u>					<del></del>
Steam Temperature (°F)	1000		975	475	<del></del>		<del></del> -i			
Opacity, Bypass (%)	Gas	643		736/300	<del>   </del>					<del></del>
Opacity, Scrubbar (2) Fice Narvat -Gas From (Scrip) x103	166	160			!					
ESP	166	•••	1115	1118		ł	ļ	1		
_Section Al-A2; AC Voltage (v)	190	190	230	220					•	
AC_Current (a)	48	62	45	- 65	!					
- DC Voltage (Kv)	3-	35	36	38						
DC Current (ma)	310	3.0	280	280						
Spark Rate (spm)	0	: 0	1 0	0	i i	1		1	l	
Continue DI DO. AC Voltage (v)				1.3-						
Section B1-B2; AC Voltage (v) AC Current (a)	170	110	170	170	<del>! •</del>				····································	<del></del>
DC_Voltage (Kv)		41	120 1 A1	<del>: 42</del>	<del>i                                    </del>	- i				
DC Current (ma)		630	1 620	1 620	<del>  </del>					
Spark Rate (spm)	· O	0	10	10	1					
	<del>-</del>	<del> </del>	-	<del>:</del>	<del> </del>	<del></del>				<del></del>
Scrubber	·	!	<u>:</u>	ļ					 	<u> </u>
Fans; Fan A Inlet Pressure (in. H <sub>2</sub> 0)	Not In	Ligh In	Not In	Not In	!	!				
Fan B Inlet Pressure (in. H20)	عصاررو	Socia	Security	Some	ļ					ļ
Fan A Outlet Pressure (in. H2O) Fan B Outlet Pressure (in. H2O)		<del>: /</del>	<del>:</del>	<del>/</del>	<del> </del>					<del> </del> -
Fan A Amps		<del>!                                     </del>	<del>:/-</del>	<del></del>	<del></del>				<u>.                                    </u>	
Fan B Amps		<del>;                                    </del>	<del></del>	<del>:                                    </del>	<del> </del>			ļ		<del> </del>
Stack Damper A, Pos. (% Opn)	$\overline{}$	+ \	<del>-                                    </del>	<del>:-                                    </del>	<del> </del>			! !	!	<del> </del>
Stack Damper B, Pos. (% Opn)	·	$\overline{}$	<del>. \</del>	1			<del> </del>		!	
	<del></del>	<del>; \</del>	+ \-	<del>'</del>	<del>                                     </del>		<del> </del>		<del> </del>	<del> </del>
_Pumps: Recirc. Pump Al (Amps)		! )	$\perp \lambda$	$\perp$	<u> </u>			<u> </u>		<u> </u>
Recirc. Pump B1 (Amps)		<u> </u>	<u> </u>	1	<u> </u>	·	<u> </u>	<u> </u>	<u> </u>	<u> </u>
Recirc. Pump B2 (Amps)	<i></i>	<del>;                                    </del>	1	<del>`</del>	<u> </u>	<u> </u>		<del>ļ</del>	· 	<del>!</del>
Recirc. Pump B3 (Amps) Recirc. Pump C1 (Amps)	<del>'</del>	<del>                                     </del>	<del>: /</del>	<del></del>	ļ	:	<u> </u>			<del> </del>
RECITC. Pump CI (Amps)		<u> </u>	<u> </u>	1			L	L		<u></u>
Reheater; Steam Flow (M lbs/hr)		17	1 / •	1 /	i					
Steam Temp. (°F)	1		1/	1	l	ĺ			İ	
Steam Pressure (psig)		T	1		l l		i			
	<del></del>	+ ;	<del>                                     </del>	<del>- / -</del>	<del> </del>	<del>                                     </del>	<del> </del>	<del> </del>	<del></del> -	<del> </del>
Section A: Presat. Water Flow (gpm)		+		<del>!                                    </del>	<del> </del>	ļ	ļ	<del> </del>	<u> </u>	<del> </del>
Outlet Gas Temp. (°F)	<del>}-</del>	<del>- \-</del>	<del>. \ -</del>	+	<del> </del>	<del>!</del>	<del> </del>	<del> </del>	·	<del> </del>
Bed Diff. (in. H20)  Demister Diff. (in. H20)	<del></del>	<del>                                     </del>	<del> </del>	<del>                                     </del>	<del> </del>	<del> </del>	<del> </del>	<del>                                     </del>	<del></del>	1
RH Diff. (in. H20)	<del></del>	+	<del>  \                                   </del>	+ \	<del>i                                    </del>	1	<del>                                     </del>	<del>!</del>	<del></del>	<del> </del>
Gas Outlet Flow (in. H2O)	<del></del>	1	1	1		1		1	i	
, 555 556 5 100 (100 120)		<u> </u>	+	+		!	<del> </del>	<del> </del>	<b> </b>	<del> </del>
Section B; Presat. Water Flow (gpm)	/	<u>'                                    </u>	11	\		<u> </u>		<u> </u>	<del>!</del>	<u> </u>
Outlet Gas Temp. (°F)		· 7	1 ]	1	1			<u> </u>	<u> </u>	<del></del>
Bed_Diff. (in. H20)		! /	<del>/-</del>	<del> </del>	<b>/</b>	<u> </u>	1	<del> </del>	<u> </u>	<u> </u>
	<del></del>	<del></del>	<del> /</del>	<del></del>	<del> </del>	<del>!</del>	<del> </del>	<del> </del>	<del>!</del>	
RH_Diff. (in. H20)		- <del>  </del>	+ /	<del> /</del>	<u>'                                    </u>	<del> </del>	<del> </del>	<del> </del>	1	1
Gas Outlet Flow (in. H <sub>2</sub> 0)			1/	1_/	<u> </u>	<u> </u>		<b></b>	<u> </u>	
Section C; Presat. Water Flow (gpm)	\		$+$ $I^{-}$	1 /	1	1			]	1
Outlet Gas Temp (°F)		<del></del>	<del>                                     </del>	<del>                                     </del>	<del> </del>	<del>                                     </del>	<del> </del>	1	1	1
Bed_Diff, (in. H2O)	<del></del>	7-1	1	1 1	<del> </del>	<del> </del>	<del> </del>	1 -	<del> </del>	1
Demister Diff. (in. H2O)		Ì					I	1	<u> </u>	
RH Diff. (in. H <sub>2</sub> O)	Ī					ļ			<u> </u>	
RH Diff. (in. H2O)		<del></del>	+	+	1	ļ	<u> </u>	+	<del></del>	<b></b>
Gas Outlet Flow (in. H2O)	1 0	l d	🕏	4	}		}	J	İ	1
	<del>_</del>	- <del></del>					<del></del>			

\* Had to put on some coal so that unit could be uncre easily country

' Comments

Date: .	8/6/77	5/6/11	8/10/11 12205	8/15/11	
Time:	1335	1450	12205	2345	
Plant					
	0:	22/	120	96	
Mw, Gross	DówN	DOWN	2.5		<del>!</del>
XS_0 <sub>2</sub>				4.5	-j <del></del>
Steam Flow (1bs/hr) 105 105 105 105 105 105 105 105 105 105		<del></del>	1050	7/0	<del></del>
Steam Temperature (°F)			1000	1800	<del></del>
Opacity, B <del>ypass (%)</del>	8	<del></del>	16	14	
Opacity, Scrubber (%) FUEL			150% COAL	50% cool	!
NAT. GAS FLEW SLF	H X103	<del></del>	720	540	<del></del>
ESP					·
Section_Al-A2;_AC_Voltage (v)	1		: 240 !	235	· · ·
AC Current (a)			130=5	£5/30	
DC Voltage (Kv)			40	40 720	
DC Current (ma)			700		
Spark Rate (spm)	·	<u></u>	0	5	i
Section B1-B2; AC Voltage (v)	1	;	220	220	· •
AC Current (a)			138	/27	1
DC Voltage (Kv)		······································	46	46	
DC Current (ma)			700	700	
Spark Rate (spm)	:	1 ,	0	. 0	
	·	<del></del>		<del></del>	!
_Scrubber	, <del>i</del>		DOWN	DOW_N ,	; ; <u>.</u>
_Faṇs; Fan A Inlet Pressure (in. H <sub>2</sub> 0 Fan B Inlet Pressure (in. H <sub>2</sub> 0	\	·			
Fan A Outlet Pressure (in. H20					
Fan B Outlet Pressure (in. H2	0)	<del></del>		<del></del>	
Fan A Amps	·	<del></del>	<del></del>	<del></del>	
Fan B Amps		<del>i</del> -	<del></del>		
Stack Damper A, Pos. (% Opn)					
Stack Damper B, Pos. (% Opn)					<del></del>
	<del></del>	<del></del>	<del></del>	<del>-</del>	
_ Pumps: Recirc. Pump Al (Amps)					<u> </u>
Recirc. Pump B1 (Amps) Recirc. Pump B2 (Amps)					<b></b>
Recirc. Pump B3 (Amps)		<del></del>	<del></del>	<del></del>	<del>+</del>
Recirc. Pump C1 (Amps)	<del>-</del>			<del>- i  </del>	<del> </del>
	<u> </u>	· · · · · · · · · · · · · · · · · · ·			.4 ———
Reheater: Steam Flow (M lbs/hr)		1 1			<u> </u>
Steam Temp. (°F)		! !			<del></del>
Steam Pressure (psig)			1		
		1 1			1
Section_A; Presat. Water Flow (gpm)		· · · · · · · · · · · · · · · · · · ·	<del></del>	<del></del>	<del>'</del>
Outlet Gas Temp. (°F)Bed Diff. (in. H2O)		<del>i-</del> -		<del></del>	<del></del>
				<del></del>	
					7
Gas Outlet Flow (in. H20)	)				i
	·	<del></del>			
		, ,	1 1 1	i 1 1	,
Section B; Presat. Water Flow (gpm)	!				
Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F)					
Section B; Presat. Water Flow (gpm)Outlet Gas Temp. (°F)Bed Diff. (in. H20)					
Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20)					
Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20)					
Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20)					
Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Gas Outlet Flow (in. H20	)				
Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Gas Outlet Flow (in. H20  Section C; Presat. Water Flow (gpm)	)				
Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Gas Outlet Flow (in. H20) Section C; Presat. Water Flow (gpm) Outlet Gas Temp (°F)	)				
Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Gas Outlet Flow (in. H20  Section C; Presat. Water Flow (gpm) Outlet Gas Temp (°F) Bed Diff. (in. H20)	)				
Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Gas Outlet Flow (in. H20) Section C; Presat. Water Flow (gpm) Outlet Gas Temp (°F) Bed Diff. (in. H20) Demister Diff. (in. H20)	)				
Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Gas Outlet Flow (in. H20  Section C; Presat. Water Flow (gpm) Outlet Gas Temp (°F) Bed Diff. (in. H20)	)				

DC Voltage (Kv)   39   38   39   39   39   39   39   39	131 2.5% 1090 1750 980 980 980 980 980 980 980 98
Mw., Gross	131 2.5%, 1040 1750 980 980 20 20 20 20 20 20 20 20 20 2
MM   Gross	2.5%   1090   1750   980
XS   Q	2.5%   1090   1750   980
XS 02	2.5%   1090   1750   980
Steam Flow (lbs/hr)   7000	1090 7, 20 980 7, 20 740 10 240\$10 125 125 36 38 720-20 700-20 60 ± 20
Steam Pressure (psig)	7, 20 980 9, 20 0AL = 50% COAL 740 125 125 36 38 720-20 700 = 20 200 = 20
Steam Temperature   SF	980 980 20 20 20 20 20 20 20 20 20 2
Opacity, Scrubber (1)   Fuel   Solical   So	2509.coal 740 10 240\$10 125 125 36 38 720-20 700 = 20 200 = 20
Space	2509.coal 740 10 240\$10 125 125 36 38 720-20 700 = 20 200 = 20
Section Al-A2; AC Voltage (v)	740 240\$10 25 /25 36 38 720-20 700 20
Section Al-A2; AC Voltage (v)	10 240 10 125 125 36 38 120 20 700 20 200 20
AC Current (a)	125 125 36 38 720±20 700±20 30±30 60±20
DC Voltage (KV)   37   38   39   38   39   30   DC Current (wa)   680   700   370	36 38 720 20 700 ± 20 300 ± 20
DC Current (wa)   GVO   70D   370   Spark Rate (spm)   O   150 2100   D   150 2	720-20 700 ± 20 200-30 60 ± 20
Spark Rate (spm)	60±20
Section B1-B2; AC Voltage (v)	
AC Current (a)	126
AC Current (a)	. 200 :
DC Voltage (Kv)   57   52   50     DC Current (ma)   730   740   740     Spark Rate (spm)   O   O   O     Spark Rate (spm)   O   O   O     Scrubber	135
DC Current (ma)   730   740   740   740   Spark Rate (spm)   O   O   O   O   O   O   O   O   O	
Spark Rate (spm)  Scrubber Fans; Fan A Inlet Pressure (in. H <sub>2</sub> 0) Fan B Inlet Pressure (in. H <sub>2</sub> 0) Fan A Outlet Pressure (in. H <sub>2</sub> 0) Fan A Outlet Pressure (in. H <sub>2</sub> 0) Fan B Amps Fan B Amps Stack Damper A, Pos. (% Opn) Stack Damper B, Pos. (% Opn) Stack Damper B, Pos. (% Opn)  Pumps; Recirc. Pump A1 (Amps) Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump B1 (Amps) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H <sub>2</sub> 0) Demister Diff. (in. H <sub>2</sub> 0) RH Diff. (in. H <sub>2</sub> 0)	
Scrubber Fans; Fan A Inlet Pressure (in. H <sub>2</sub> 0) Fan B Inlet Pressure (in. H <sub>2</sub> 0) Fan A Outlet Pressure (in. H <sub>2</sub> 0) Fan B Outlet Pressure (in. H <sub>2</sub> 0) Fan B Amps Fan B Amps Fan B Amps Stack Damper A, Pos. (% Opn) Stack Damper B, Pos. (% Opn)  Pumps; Recirc. Pump Al (Amps) Recirc. Pump Bl (Amps) Recirc. Pump Bl (Amps) Recirc. Pump Bl (Amps) Recirc. Pump Bl (Amps) Recirc. Pump Cl (Amps)  Recirc. Pump Cl (Amps)  Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (qpm) Outlet Gas Temp. (°F) Bed Diff. (in. H <sub>2</sub> 0) Demister Diff. (in. H <sub>2</sub> 0) RH Diff. (in. H <sub>2</sub> 0)	0
Fans; Fan A Inlet Pressure (in. H <sub>2</sub> 0) Fan B Inlet Pressure (in. H <sub>2</sub> 0) Fan A Outlet Pressure (in. H <sub>2</sub> 0) Fan B Outlet Pressure (in. H <sub>2</sub> 0) Fan A Amps Fan B Amps Stack Damper A, Pos. (% Opn) Stack Damper B, Pos. (% Opn)  Pumps; Recirc. Pump A1 (Amps) Recirc. Pump B1 (Amps) Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump C1 (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H <sub>2</sub> 0) Demister Diff. (in. H <sub>2</sub> 0) RH Diff. (in. H <sub>2</sub> 0)	
Fan B Inlet Pressure (in. H20) Fan A Outlet Pressure (in. H20) Fan B Outlet Pressure (in. H20) Fan B Outlet Pressure (in. H20) Fan A Amps Fan B Amps Stack Damper A, Pos. (% Opn) Stack Damper B, Pos. (% Opn)  Pumps; Recirc. Pump Al (Amps) Recirc. Pump Bl (Amps) Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump C1 (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20)	our
Fan A Outlet Pressure (in. H <sub>2</sub> O) Fan B Outlet Pressure (in. H <sub>2</sub> O) Fan A Amps Fan B Amps Stack Damper A, Pos. (% Opn) Stack Damper B, Pos. (% Opn)  Pumps; Recirc. Pump Al (Amps) Recirc. Pump Bl (Amps) Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump C1 (Amps)  Recirc. Pump C1 (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H <sub>2</sub> O) Demister Diff. (in. H <sub>2</sub> O) RH Diff. (in. H <sub>2</sub> O)	
Fan B Outlet Pressure (in. H2O) Fan A Amps Fan B Amps Stack Damper A, Pos. (% Opn) Stack Damper B, Pos. (% Opn)  Pumps; Recirc. Pump Al (Amps) Recirc. Pump Bl (Amps) Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump C1 (Amps)  Recirc. Pump C1 (Amps)  Rebeater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H2O) Demister Diff. (in. H2O) RH Diff. (in. H2O)	
Fan A Amps Fan B Amps Stack Damper A, Pos. (% Opn) Stack Damper B, Pos. (% Opn)  Pumps; Recirc. Pump Al (Amps) Recirc. Pump Bl (Amps) Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump C1 (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20)	
Fan B Amps Stack Damper A, Pos. (% Opn) Stack Damper B, Pos. (% Opn)  Pumps; Recirc. Pump Al (Amps) Recirc. Pump Bl (Amps) Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump C1 (Amps)  Recirc. Pump C1 (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (qpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20)	
Stack Damper A, Pos. (% Opn)  Stack Damper B, Pos. (% Opn)  Pumps; Recirc. Pump Al (Amps) Recirc. Pump Bl (Amps) Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20)	
Stack Damper B, Pos. (% Opn)  Pumps; Recirc. Pump Al (Amps) Recirc. Pump Bl (Amps) Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20)	<del></del>
Pumps; Recirc. Pump A1 (Amps) Recirc. Pump B1 (Amps) Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump C1 (Amps)  Recirc. Pump C1 (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20)	
Recirc. Pump B1 (Amps) Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump C1 (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20)	'
Recirc. Pump B1 (Amps) Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump C1 (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20)	
Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump C1 (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20)	
Recirc. Pump B3 (Amps) Recirc. Pump C1 (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20)	
Recirc. Pump C1 (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20)	
Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20)	
Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20)	<del></del>
Steam Pressure (psig)  Section A; Presat. Water Flow (gpm)  Outlet Gas Temp. (°F)  Bed Diff. (in. H20)  Demister Diff. (in. H20)  RH Diff. (in. H20)	
Section A; Presat. Water Flow (gpm)  Outlet Gas Temp. (°F)  Bed Diff. (in. H20)  Demister Diff. (in. H20)  RH Diff. (in. H20)	
Outlet Gas Temp. (°F)  Bed Diff. (in. H20)  Demister Diff. (in. H20)  RH Diff. (in. H20)	
Outlet Gas Temp. (°F)  Bed Diff. (in. H20)  Demister Diff. (in. H20)  RH Diff. (in. H20)	
Bed Diff. (in. H20)  Demister Diff. (in. H20)  RH Diff. (in. H20)	
Demister Diff. (in. H20)	
RH Diff. (in. H20)	
Gas Outlet Flow (in. H2O)	
, , , , , , , , , , , , , , , , , , , ,	
Section B; Presat. Water Flow (gpm)	
Outlet Gas Temp. (°F)	
Bed Diff. (in. H20)	
Demister Diff. (in. H2O)	
RH Diff. (in. H20)	
Gas Outlet Flow (in. H2O)	
Seeding Co. Burnet, Hotor Story (1994)	
_Section C; Presat. Water Flow (gpm)	
Outlet Gas Temp (°F)  Bed Diff. (in. H20)	
Demister Diff. (in. H2O)	
RH Diff. (in. H <sub>2</sub> 0)	
RH Diff. (in. H <sub>2</sub> 0)	
Gas Outlet Flow (in. H2O)	

Date: .	8/15/77	18/15/11	8/22/17	8/22/11	
Time:	0950	1120	10950	1115	
Plant					i
	173	172	84.5	OUT	
M, Gross	132	133	OUT		<del></del>
XS 02 Steam Flow (1bs/hr)	1050	1050	<del></del> -/	<del>   </del>	
Steam Pressure (psig)	1780	1780		<del></del>	<del></del>
Steam Temperature (°F)	990	990	<del></del>		<del></del>
Opacity, Bypass (%)	990 227,	990			
Opacity, Scrubber (%) FUEL	509 CUAL	50% COAL			:
WAT GAS SCEN TO 3	750	150	<del></del>	<del></del>	
ESP	230±10	1 1	-  -		<del></del>
Section A1-A2; AC Voltage (v) AC Current (a)	125± 10	230±10 125±10	<del></del>		<u>-</u>
DC Voltage (KV)	36	36	<del></del>	<del></del>	
DC Current (ma)	700	720 20	<del></del>		<del></del>
Spark Rate (spm)	180120	350 £ 50			· <del></del>
	<del>`</del>		<del></del>	<del></del>	<del></del>
Section_B1-B2; AC_Voltage (v)	230	230			
AC Current (a)	130	135	<del></del>		· · · · · · · · · · · · · · · · · · ·
DC Voltage (Kv)	50 500	52			<del></del>
DC Current (ma) Spark Rate (spm)	730	740	<del>-     -   -   -   -   -   -   -   -   -</del>	<del></del>	
Shall use (shii)	O'				
Scrubber	OUT:	OUT			
Fans: Fan A Inlet Pressure (in. H20)					
Fan B Inlet Pressure (in. H2O)					
Fan A Outlet Pressure (in. H20	:: <i>\</i>	<del></del>	<del>  </del>		
Fan B Outlet Pressure (in. H2C	"···				
Fan B Amps	· - <del></del>	<del></del> }		·	
Stack Damper A. Pos. (% Opn)				i- <del> </del>	
Stack Damper B, Pos. (% Opn)					
	<del></del> :	<del></del>	<del></del>	<del></del>	
Pumps: Recirc. Pump Al (Amps)	·		i i		i
Recirc. Pump B1 (Amps) Recirc. Pump B2 (Amps)		<del></del>	<del></del>		
Recirc. Pump B3 (Amps)	·	<del></del>		<del></del>	<del></del>
Recirc. Pump C1 (Amps)			<del></del>	<del>- i 1- i -</del>	<del></del>
<del></del>					
Reheater; Steam Flow (M lbs/hr)		1 1			
Steam Temp. (°F)		<del></del>			
Steam Pressure (psig)					
Section A; Presat. Water Flow (gpm)			•		!
Outlet Gas Temp. (°F)					
Bed Diff. (in. H2O)				<u> </u>	
Demister Diff. (in. H <sub>2</sub> 0)					
RH Diff. (in. H20)					
Gas Outlet Flow (in. H2G)		ii			
Section B; Presat. Water Flow (gpm)					
Outlet Gas Temp. (°F)	<del></del>	<del></del>		<del></del>	_;
Bed_Diff. (in. H2O)					
Demister Diff. (in. H2O)					
RH Diff. (in. H <sub>2</sub> 0)	<del></del>				
Gas Outlet Flow (in. H2O)	' <u>_</u>				
Section C; Presat. Water Flow (gpm)				1	
Outlet Gas Temp (°F)	· }	<del></del>	<del></del>	<del></del>	
Bed Diff. (in. H20)		<del></del>	<del>-   -   -   -   -   -   -   -   -   -  </del>		<del></del>
Demister Diff. (in. H2O)					
RH Diff. (in. H <sub>2</sub> 0)					
RH Diff. (in. H20)	·				
Gas Outlet Flow (in. H2O)	, , , , ,				

Date:	8/23/11	8/23/11	8/24/17	×/24/11	18/24/11	
Time: Plant	0445	1045	09/01	1255	1500	
M, Gross	OUT	GUT	OUT	OUT	out	
XS 02 Steam Flow (lbs/hr) Steam Pressure (psig)	-1-					
Steam Temperature (°F) Opacity, Bypass (%) Opacity, Scrubber (%)						
ESP Section_A1-A2;_AC Voltage (v) AC_Current (a) DC Voltage (Kv)						
DC Current (ma) Spark Rate (spm)		,				
Section_B1-B2; AC_Voltage (v)						
Spark Rate (spm) Scrubber			:			
Fans; Fan A Inlet Pressure (in. H <sub>2</sub> 0) Fan B Inlet Pressure (in. H <sub>2</sub> 0) Fan A Outlet Pressure (in. H <sub>2</sub> 0) Fan B Outlet Pressure (in. H <sub>2</sub> 0) Fan A Amps Fan B Amps Stack Damper A, Pos. (% Opn) Stack Damper B, Pos. (% Opn)		•				
Pumps: Recirc. Pump A1 (Amps) Recirc. Pump B1 (Amps) Recirc. Pump B2 (Amps) Recirc. Pump B3 (Amps) Recirc. Pump C1 (Amps)						
Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)			1			
Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Gas Outlet Flow (in. H20)						
Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed_Diff. (in. H20) Demister Diff. (in. H20) RH_Diff. (in. H20) Gas Outlet Flow (in. H20)						
Section C; Presat. Water Flow (gpm) Outlet Gas Temp (°F) Bed Diff. (in. H20)					.	

•					
Date: .	8/25/11	8/25/77			
Time:	0640	0845			
Plant					
H. Case	OUT	out			
Mw. GrossXS 0 <sub>2</sub>	1047	1			
Steam Flow (1bs/hr)	<del></del>	<del></del>	<del></del>		
Steam Pressure (psig)			<del></del>		<del></del> -
Steam Temperature (°F)					<del></del>
Opacity, Bypass (%)					
Opacity, Scrubber (%)			! _ \		
ren .	1 1	1			
ESP Section_A1-A2;_AC_Voltage (v)					<del></del>
AC Current (a)	<del></del> -	·	<del></del>		
- DC Voltage (Kv)		;		Į	-
DC Current (ma)		1			
Spark Rate (spm)	1.	!	1	į	1
Continu D1 D2. AC Valence (u)	<del>-                                    </del>	<del></del>	<u>-</u>	1	
Section B1-B2; AC Voltage (v) AC Current (a)		····			<del></del>
DC Voltage (Kv)		<del></del>	<del>-                                    </del>	<del></del>	+
DC Current (ma)			<del></del>		<del></del>
Spark Rate (spm)	-1				
	<del>/</del>	<del></del>	<del></del>		<del>-                                    </del>
Scrubber	_ <del>   _ i _</del>		<del></del>		_
Fans; Fan A Inlet Pressure (in. H <sub>2</sub> 0) Fan B Inlet Pressure (in. H <sub>2</sub> 0)	/÷,	<del></del>		<u> </u>	<del>_</del>
Fan A Outlet Pressure (in. H20)	· - <del> </del>	<del></del>	<del></del>	<del></del>	
Fan B Outlet Pressure (in. H20)				<del></del>	
Fan A Amps		· · · · · · · · · · · · · · · · · · ·			
Fan B Amps					
Stack Damper A, Pos. (% Opn)				1	
Stack Damper B, Pos. (% Opn)	<u> </u>				
Pumps: Recirc. Pump Al (Amps)	] [	! ]		i j	Ì
Recirc. Pump 81 (Amps)	<del>-</del>	<del></del>	<del></del>	<del></del>	
Recirc. Pump B2 (Amps)	<u> </u>			i	
Recirc. Pump B3 (Amps)					
Recirc. Pump C1 (Amps)	1 1			ì	Ì
Reheater; Steam Flow (M lbs/hr)				<del></del>	
Steam Temp. (°F)	<del></del>	<del></del>	<del>-ii</del>	i	
Steam Pressure (psig)	<del>  </del>	<u>i</u>	<del>-   -    </del>		
	<del></del>	<del></del>	<del></del>	·	<del></del>
Section A; Presat. Water Flow (gpm)	<del></del>				
Outlet Gas Temp. (°F)			<del></del>		
Bed Diff. (in. H2O)  Demister Diff. (in. H2O)		<del></del>		<del></del>	<del></del> -
RH Diff. (in. H20)			<del></del>		
Gas Outlet Flow (in. H20)				1	
		<del></del>			
Section B; Presat. Water Flow (gpm)_	i				
Outlet Gas Temp. (°F)					
Bed Diff. (in. H20)  Demister Diff. (in. H20)	<del></del>	<del></del>	<del></del>		
RH Diff. (in. H20)	· <del></del>	<del></del>	<del></del>	<del></del>	<del>- </del> -
Gas Outlet Flow (in. H <sub>2</sub> O)			<del></del>		
	<del></del>	<del></del>	<del></del>	<del></del>	
Section C; Presat. Water Flow (gpm)				·   _   _	
Outlet Gas Temp (°F)	<del>-\-</del>				
Bed Diff. (in. H20)		<del></del>			
Demister Diff. (in. H20)		<del></del>	<del></del>		
RH Diff. (in. H2O) RH Diff. (in. H2O)	<del></del>	<del></del>	<del>-   </del>	<del></del>	
Gas Outlet Flow (in. H20)		<del></del>	<del></del>	····	
and ancies tion fills usal	i i	1 \	1 1 1		

Date: .	8/19/17	8/2/2		7			
Time:	1030	1:22		1	Ī		
Plant	1 7 2 2 2		<del></del>				1
Fiel	800/308-	30C/SOC	<del></del>			<del>- </del>	<del></del>
Mw, Gross	140	141				<u>.l</u>	
XS 02	2-1	3 0					•
Steam Flow (lbs/hr) 3/03	1/730	1050	<del>-</del>	· · · · · ·		<del></del>	
Steam Pressure (nsig)		1800		<del>-;</del>		<del> </del>	
Steam Flow (lbs/hr) Steam Pressure (psig) Steam Temperature (°F)	1200	920		<del>-;</del>		- <del></del> -	<del></del>
Opacity, Bypass (%)	- 640-	750		·· <del>·</del>	<del></del>		<del></del>
Opacity, Scrubber (%) vari		<del></del>				<del></del>	
abacity accommendate ( 345	7.4	7.5	<u> </u>	<u> </u>			
ren .	-	1	ı	1	1	_	
_ESP		'- <u>-</u>		-			<del></del> '
Section A1-A2; AC Voltage (v)	<u> 240 </u>	330				_l	
AC_Current (a)	D:O	115		<del></del>		<del></del>	
DC Voltage (Kv)	37	37	<del> </del>	<u> </u>		<del></del>	
DC Current (ma)	610	620	<u> </u>	<del></del>		· ;	
Spark Rate (spm)	50	180 '	•		1	į	
		<del></del>	<del>-</del>			<del></del>	
Section B1-B2; AC Voltage (v)	. 230	230	<u> </u>	! . !	<u> </u>	<u>;                                    </u>	
AC Current (a)		132					
DC Voltage (Kv)	50	50					
DC Current (ma)	740	7/30			1		
Spark Rate (spm)	0	5	i		!		· · · · · · · · · · · · · · · · · · ·
,	<del></del>	<del>!                                    </del>	•	<del></del>	<del></del>	<del></del>	
_Scrubber	Out of	Dut C	ļ	i		1	i .
Fans; Fan A Inlet Pressure (in. H20)		_ ناماند	· · · · · ·	<del></del>		1:	
Fan B Inlet Pressure (in. H20)	200 4/15	اعاشتان	·			1	<del></del>
Fan A Outlet Pressure (in. H2O)	. <i>[</i>	;	<del></del>	<del>-i</del>	·	<del></del>	
Fan B Outlet Pressure (in. H20)	/	<del></del>			<del></del>	<del></del>	
Fan A Amer	- (	·	· <del></del>			_}	
Fan A Amps				.	i	<u> </u>	<u> </u>
Fan B Amps			<del></del>	_ <del></del>			<del></del>
Stack Damper A. Pos. (7 Opn)	\	. <del> </del>					
Stack Damper B, Pos. (% Opn)	\	! \ !		i :		1	1
		1 \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \	ı	<u> </u>	1	<del></del>	<del></del> -
Pumps: Recirc. Pump Al (Amps)		1				<u> </u>	
Recirc. Pump 81 (Amps)							
Recirc. Pump B2 (Amps)		<u>'</u>		<u> </u>			
Recirc. Pump B3 (Amps)		, ]			1	1 1	
				1	1	,	
Recirc. Pump Cl (Amps)		1 . !				!!!	1
Recirc. Pump Cl (Amps)		<u>  :                                   </u>			<del></del>		
Recirc. Pump Cl (Amps)			1			_	
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr)	7	,				<del>-  </del>	
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr)  Steam Temp. (°F)	7	<del></del>					
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr)		/					
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr)  Steam Temp. (°F)  Steam Pressure (psig)		<del></del>					
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm)		<del></del>	1				
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F)		<del></del>	1				
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20)		<del></del>	1				
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20)		<del></del>					
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20)  RH Diff. (in. H20)		<del></del>					
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20)		<del></del>					
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Gas Outlet Flow (in. H20)		<del></del>					
Recirc. Pump C1 (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Gas Outlet Flow (in. H20)  Section B; Presat. Water Flow (gpm)							
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Gas Outlet Flow (in. H20)  Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F)		<del></del>					
Recirc. Pump C1 (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Gas Outlet Flow (in. H20)  Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20)							
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) Gas Outlet Flow (in. H20)  Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20)  Demister Diff. (in. H20)							
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) Gas Outlet Flow (in. H20)  Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20)  RH Diff. (in. H20) RH Diff. (in. H20)							
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) Gas Outlet Flow (in. H20)  Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20)  Demister Diff. (in. H20) Demister Diff. (in. H20)							
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) Gas Outlet Flow (in. H20)  Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20)  RH Diff. (in. H20) RH Diff. (in. H20)							
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) Gas Outlet Flow (in. H20)  Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Outlet Flow (in. H20) Gas Outlet Flow (in. H20) Gas Outlet Flow (in. H20)							
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) Gas Outlet Flow (in. H20)  Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) QRH Diff. (in. H20) Gas Outlet Flow (in. H20) Section C; Presat. Water Flow (gpm)							
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) Gas Outlet Flow (in. H20)  Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Outlet Flow (in. H20) Section C; Presat. Water Flow (gpm) Outlet Gas Temp (°F)							
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) Gas Outlet Flow (in. H20)  Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Outlet Flow (in. H20) Section C; Presat. Water Flow (gpm) Outlet Gas Temp (°F) Bed Diff. (in. H20)							
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) Gas Outlet Flow (in. H20)  Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Gas Outlet Flow (in. H20) Section C; Presat. Water Flow (gpm) Outlet Gas Temp (°F) Bed Diff. (in. H20) Demister Diff. (in. H20)  Section C; Presat. Water Flow (gpm) Outlet Gas Temp (°F) Bed Diff. (in. H20) Demister Diff. (in. H20)							
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) Gas Outlet Flow (in. H20)  Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Gas Outlet Flow (in. H20)  Section C; Presat. Water Flow (gpm) Outlet Gas Temp (°F) Bed Diff. (in. H20)  Section C; Presat. Water Flow (gpm) Outlet Gas Temp (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) RH Diff. (in. H20)							
Recirc. Pump Cl (Amps)  Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F) Steam Pressure (psig)  Section A; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) Gas Outlet Flow (in. H20)  Section B; Presat. Water Flow (gpm) Outlet Gas Temp. (°F) Bed Diff. (in. H20) Demister Diff. (in. H20) RH Diff. (in. H20) Gas Outlet Flow (in. H20) Section C; Presat. Water Flow (gpm) Outlet Gas Temp (°F) Bed Diff. (in. H20) Demister Diff. (in. H20)  Section C; Presat. Water Flow (gpm) Outlet Gas Temp (°F) Bed Diff. (in. H20) Demister Diff. (in. H20)							

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Jate:	'	7/21/17	7/37/77	7/24/27	7/29/27	1/20/22	3/30/17	8/1/27	2/4/27_		
ime:		1406			1104			1740	1838		
lant											
Mw, Gross		357	355	360	355	3.15	341	306	323		
XS 0 <sub>2</sub>		4.0	3.8	4:2,	4.3_,	3.ė,	_3.Z.,	4.0	4.0		
_Steam_Flow_(lbs/hr)			24×106		2340				عبداوها		
<u>Steam Pressure (psig)</u> <u>Steam Temperature (°F)</u>	<del></del>	2400 980 i	2350		980	3350	7300	1950	990		
Opacity_Bypass_(%)		17-160			20/16	35/10		27/20	50/20	<del></del>	
Opacity, Scrubber (%)	——————————————————————————————————————	4/20,9			77.18	31	37/20	26	20/2/	<del></del> +	
	<del></del> t	-/	40.51	1.*		-51			-39	1	
ESP	, [				İ		370		]		
Section Al: AC Voltage (	<u>'</u>	270	270_	280	LULT	200		230	570		
AC Current (a		140		-442	THE	430	80	130	152		
DC Voltage (k	na) ;-	-Gyt.	_ <u>Qut</u>		DISCOLL	DJt.	Out-	024	100		
Spark Rate (s		15	_680_ 3<	<u>. 700</u> _ ! 35	DATA	87	25	680	97		
		<del></del>	-5	-00	<del> </del>	<del></del>	1 53	<del>-2</del>	4/		
Section A2: AC Voltage (	v)	Same	Some	Same		Same	GUNE	Samo	Same	<u> </u>	
AC_Current (a	a) :				1	<u> </u>	<u> </u>				
DC Voltage (		·		<u> </u>	<u> </u>	<u>:</u>	1				
DC Current (r		- į			!		<u> </u>	ļ	ļ ————		
Spark Rate (	spm);			<del></del>	<del> </del>	<del> </del>	<del> </del>	<del> </del>	·		
Section A3; AC Voltage (	v)	250	252		1	32-	225	200_	200	-	
AC Current (	a)	110	. <b>25</b> 0	250.   120		220	275	200	80		•
DC Voltage (		- 510 C:		120 - 0 u E	<del> </del>	80	Qui.		Cut :	<del></del> †-	
DC Current (1	ma)	-510 ·	520		<u></u>	430	- 410	400	390		
Spark Rate (	spm) i	OUT	Out	†uo	1	Out	Tout	Out	001 :		
	, 1			<del> </del>	1	<del>                                     </del>		<del> </del>			- ~
Section A4; AC Voltage (		SCME	SOMO	SNE	<del> </del>	Source	Samo	-S-Me	20WE	<del>-</del>	
AC Current (				<del> </del>		}	<del></del>	<del> </del>	·		
DC Voltage ( DC Current (			<del></del> -	<del> </del>	<del> </del>	<del> </del>	+	1	+	<del></del>	
Spark Rate (				<del>i</del>	<del>                                     </del>	<del>                                     </del>		<del> </del>	<del> </del>	<del></del>	
	- <del> </del>			<del> </del>	<del> </del>	1	<del> </del>	+	<del>                                     </del>	<del>+</del>	
Section Bl: AC Voltage (	v)	300	3,0	320	<u> </u>	300	300	290	280		
AC Current (	a) _	150	150	145	ļ	145-	150	150_	130	<u> </u>	
DC Voltage (		00+	OUT	40	<b></b>	<u> </u>	100+	00+	00		
DC Current (		800	800	1 740_	<b></b>	<del>- 740</del>	7:0_	710	6/0	4.	
Spark Rate (	spm)	>200	7200	185	1	>200	3500	185	185 :		
Section B2; AC Voltage (	v)	Se	Same	C. 40		1 54.24	8	-	Same '	ı	
_Section_B2; AC Voitage ( AC Current (	نام ا		,	1	<del></del>	عسد	Same	T 3300.	- Ormic		
DC Voltage (			<del></del>	·}	<del>†</del> -	<del></del>	<del> </del>	<del> </del>	<del></del>	<del>i</del>	
DC Current (			<del> </del> -	<del> </del>	<del> </del>			<b>—</b>	·	<del></del>	
Spark Rate (		· ·		1	1	<del> </del>	1	1		<u>-</u>	
		<del></del>	<del> </del>	<del> </del>			1.	+			
_Section_B3:_AC_Voltage (		250	250	250	<del></del>	230	240	233	220	<del></del>	
AC_Current (		100.	100	100	<del> </del>	25	65	<u> 80</u>	75		
DC Voltage ( DC Current (		31	30	30	<del></del>	30	31	350	. 30	<del> </del>	
Spark Rate (	Spm)	480 30	500	1480	<del></del>	27Ú	780	350.	1330		
Jean nace (	·- ··· / 	·	<del> </del>	<del>  3</del> -	<del> </del>	-1-		<del></del>			-··- ·
Section B4; AC Voltage (	(v)	3400	Same	STINE		Saurc	Same	SARE	Zame	1	
AC Current	(a)	,			1				لسل		
DC Voltage (				1				- I			
DC Current			i	<u> </u>	+		<del></del>	<del></del>	!		
Spark Rate	r c DM i			I	•		!				

	·	<del></del>	<del></del>	,	<del></del>	<del></del>	r	<del></del>	
Scrubber	<u> </u>			<u></u>					L
Fans; ID_Discharge Pressure (in. H2O)		1-3	-,5	一. 不	- 5	-4	-5	3	<u> </u>
Fan & Outlet Pressure (in. H2O)	OUT	Out	OUT	0.15	out	20 t	مريدے	シット	
Fan C Outlet Pressure (in. H2O)	13.5	<u> </u>	15.5	15.5	14 _	13	9	13.5	
Fan D Outlet Pressure (in. H2O)	13.5	13.5	μ	·4	L.S.	11.5	8	<u> </u>	
Fan_B Amps	230	230	230_	235	205	210	230	310	
Fan C Amps	230	230	230	230	22.0	220	225	372	
Fan 5_Amps	340	240	230	230	_310_	230	245	235	<del></del>
Stack_Damper A, Pos. (% Opn)	٥	<u> </u>	0	8-	<u> </u>	10	-용	્ર	·
Stack Damper B, Pos. (% Opn)	0	0	0	1 0	ට	0	0	0	1
Pumps: Recirc. Pump B( (amps)	13_	' 12.5	12	! /2	.2	125	12.5	13.5	l.
Recirc. Pump 82 (amps)	15	155	15	15	15	15.5	16	16	Li
Recirc. Pump A3 (amps)	14	13,5	14	14	14	14:	. 14	! 13'5	
Recirc. Pump 🕝 (amps)		12	12	12	11	_11	11	//	
Recirc. Pump C2 (amps)	12	: 12	12	12	. //	-11	- 11	111-	
Recirc. Pump (3 (amps)	13	13	13	1.73	3	13.5	13.5	13.5	
Recirc. Pump 🔼 (amps)	14	14	13	12		14	11.5.	-//-5	•
Recirc. Pump 🕟 (amps)	14	14	14	14	_/4_	13.5	11.5	–	, <b></b>
Recirc. Pump DS (amps)	. 14	14	14	/4	. 14	14	1/2	-12	ļ
Reheater: Steam Flow (M lbs/hr) Steam Temp. (°F)	55	. 55	55	55	'45	45	45	45	
Steam Pressure (psig)	1500	1500	7650	1670	1480	1460	1480	/48c	
_Section β; Presat. Water Flow (gpm)_	182	185	185	10-	185	185	185	.185	
Outlet_Gas_Temp. (°F)	145	145	135	135	140	140	140	1 140	!
Bed Diff. (in. H <sub>2</sub> O)	7.2	5-4	4.2	6.7	5.4	3.2.	5.0	5-0.	
Demister Diff. (in. H <sub>2</sub> 0)	.85	0.25	0.9	0,9	0.62	0.20	0.7	0.7	
RH Diff. (in. H <sub>2</sub> 0)	٥	ا_ ت	370		D	ن	C.	U.	:
Gas Outlet Flow (in. Hoo)	300	. 210	370	390	300	300	410	. 410	
_ Section_ c ; Presat Water Flow (gpm)	(80 _	183_	(8:5	izo	175	i75	180	_175_	
Outlet Gas Temp. (°F)	155	155	150	155	150	1 150	150	150	
Bed Diff. (in. H <sub>2</sub> 0)	14.6	14.6	10,6	10.8	8.7	2.0.	.5.6.	8.5.	
Demister Diff. (in. H <sub>2</sub> 0)	2.0	0.3	0.3	0.3	2,0	0.3	0.3		l
RH Diff. (in. H <sub>2</sub> 0)	950	450	3 0.3	0.2	0.7	109	24.		
Sas Outlet Flow (in Hoo)	450	450	450	450	320	320	340	330	I
Section D: Presat. Water Flow (qpm)	186	186	185	185	195	185	185	185	
Outlet Gas Temp. (°F)	160	160	150	160	150	150	160	140	
Bed Diff. (in. H20)	9.2	9-7	12.3	72	60	70	4.2	6.0	<u> </u>
Demister Diff. (in. H20)	2.6	2.5	1.9	Vin E	2.2	2.4	2.0	1 2.1	
RH Diff. (in. H2O)	د :	_ c2	B	Tex	Ü	5	٥	ــــــــــــــــــــــــــــــــــــــ	[
Gas Outlet Flow (in. H20)	1	; •	4 - >	1	!		200	1 300	
10 20 1 2220 A COST ( 00 00)		470	420	1 6	250	250	200	1 400	1

steam is off of drum

				-/		-
Date:	8/3	<b>8/</b> 3	8/4	8/4	8/4	8/4
Time:	1740	2000			1	
<del>-</del> '	,	. 2000	1050	1220	1502	1543
Plant	!		BASE LOAD	BASE LOAD		
Mw. Gross (CHARTS)	335	345	355	355	 	
YS 0-	3.6	3.5 35%	4.5	3.9	3-8	355
ALL Steam Flow (lbs/hr) x/003	2700	23 <b>0</b> 0	2400	2400	: 3-0 : 12350	, 4 Z 2400
Steam Pressure (psig) <pre>_</pre>	23	23.5	23.	23.5	23.5	23,0
Steam Temperature (°F) Opacity, Bypass (%) 44 /45	990	970	985	990	970	טרף
Opacity, Scrubber (%)	26/10 26	27/20	<i>'35/13</i>	35/12	30/10	30/8
COAL FLOW (T/HR)	~6 _	3 <b>4</b> 140 (	2 150	24	- 32	ــــــــــــــــــــــــــــــــــــــ
ESP		•	CAAL	145		
Section A1; AC Voltage (v)	250t20	250±20	250±20	250120	.240 ±20	530 2 30
AC Current (a) DC Voltage (Kv)	100 = 20 OUT	100± 40	120 = 20	120=20	3	_ 75 = 20
· DC_Current. (ma)	550 - 50	0u⊤ 550±50	0UT 100 550	650 50	-10014	으스
Spark Rate (spm)	90	90	80	85	_:510±40	480 = 20
Continue 82 o 80 Voltago (u)				SAME	70 Same	ડિવબાઇ
Section A2; AC Voltage (v) AC Current (a)	SAME	SAM R	SAME	JAME -		ડવના છ
DC Voltage (Kv)			•			•
DC Current (ma)						· · · · · · · · · · · · · · · · · · ·
Spark Rate (spm)			-	; ;	_	1
Section A3; AC Voltage (v)	200±30	200±40	200±40	220 ±40	210=20	200±25
AC Current (a)	75t5	75± 5	80±5	,80º 10	. છડ્- <i>s</i>	75±5
DC Voltage (Kv)	ouT	OUT	047	OUT	Out	, out
DC_Current (ma)	360 I 30	380.E30	380 £ 20	400120	400 20	380-70
Spark Rate (spm)	out	out	OUT	OUT	المرى	out
Section A4; AC Voltage (v)	SAME	SAME	SAME	SAME .	SPAL	Sauc
_ AC Current (a)	7.7.7.	)	37	:	3116	٥,
DC Voltage (Kv)						
DC Current (ma) Spark Rate (spm)				•		
Spark (and (spin)	270	٠		-		
Section B1; AC Voltage (v)	43	_280 <sup>T</sup> 16	280±10	280520	270 -ان	. 270+10
AC Current (a)	125	130°t 5	130±5	130±10		. 120-5
DC Voltage (Kv)DC Current (ma)	but	OUT	- OUT	: out	· out	ant
Spark Rate (spm)	185	600 ± 20	600±20	620 = 20	575-20	540 ± 20
	03	185	. 185	185	182	185 _
Section B2; AC Voltage (v)	_SAME .	SAME.	SAME	. SAME	same	SAME
AC Current (a) DC Voltage (Kv)			•	_		! .
DC Current (ma)		-				•
Spark Rate (spm)					-	
Continuo Da AC Valence (v)	23. t 20	and a	44.534	230±20		١ ــ
Section B3; AC Voltage (v) AC Current (a)	230± 20 70±5	230± 20	230-20			552 <u>,</u> 52
DC Voltage (Kv)	70±5 31 <b>₽</b>	-70±5	70± 10	- 75±10	•	75
DC Current (ma)	300	30 300 ±10	30 300 ±10	3/		30.0.
Spark Rate (spm)	60	55 -10	45	· 320 I 20	. 45	70
Section B4; AC Voltage (v)	0.4			45		
AC Current (a)	.same .	SAME	SAME	SAME	.Same	, same
DC Voltage (Kv)			<u>.</u> -	•		• - • •
DC Current (ma) Spark Pate (spm)						
Spark Rate (spm)		3 - 1 · · · · · · · · · · · · · · · · · ·	· ·	1!		
UBSERVER - R. ID	A					

## CHEROKEE STATION UNIT 4 DATA SHEET

A-34		(continued)	<del></del> -			
n-34	-1/2	- 8/3 .	8/4	8/4	8/5	8/5
Camikhan	8/3	. 012	1000	1225	- '	· , - i
Scrubber Fans: ID Discharge Pressure (in. H2O)	-0.75	-0.8	-0.5	-0.5		i
Fan B Outlet Pressure (in. H2O)	OUT	OUT	OUT	OUT	700	_ <i>OU</i> I
Fan <u>C</u> Outlet Pressure (in. H2O)	11	$\mu$	, 14	15	/5	15
Fan D Outlet Pressure (in. H20)	10	. 10	12	12	12.5.	12 .
Fan ──B Amps メ (O Fan ──C Amps × (O	20	21	2/ ' 23	21	<u>2</u> 2	-22 -1
Fan D Amps x 10	23	· · · · · · · 23.	24	24	23	23.5
_ Stack Damper A, Pos. (% Opn)	0	., 0	0	0	0	່ ວ້ '
Stack Damper B, Pos. (% Opn)	0	0	0		0	. 0
Pumps; Recirc. Pump $\mathcal{B}($ (amps)	13.	. 13	13	. 13	. 13	13 :
Recirc. Pump 82 (amps)	. 16	16	16	16_	.16	16 .
Recirc. Pump <u>83</u> (amps)	14	.14	13	! /3	12.5	12.5
Recirc. Pump <u>こ</u> (amps) Recirc. Pump <u>こ</u> (amps)	- 11	11	//	12	11.5	. 11.5
Recirc. Pump C3 (amps)	1.4. 14	!2. 13	11 ;	12 :	- 11.5	, 11.5 ·
Recirc. Pump o: (amps)	1.2	12	14.	13	-13-	. 13.5
Recirc. Pump D2 (amps)	12	12	13	73		1.3
Recirc. Pump D3 (amps)	!2	12	12	12	13	. 14
Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F)	4-5	45	` <i>55</i> ,	55	33 _	. 55 L
Steam Pressure (psig) Buse	15		-		15	 .5
Section B; Presat. Water Flow (gpm)	185		15 . ! 185	185	188	182 , 12
Outlet Gas Temp. (°F)	_ 145	190 .	140	140	140	140
Bed Diff. (in. H <sub>2</sub> 0)	5	0	. 6.5	7.6	8,0	7.6
Demister Diff. (in. H <sub>2</sub> 0)	. 0.8	0,9	0.9	0.9	- 0.9	0.9
RH Diff. (in. H <sub>2</sub> 0) G <del>as Outlet_Fl</del> ow_(inH <sub>2</sub> 0}	0	• 0	0 <del>2=4</del>	10	0	O
REHELT SIM PLESS PSIG	400	. <i>380</i>	390	380	380	380
Section C; Presat Water Flow (gpm)	155	. 155	180	. 180_	175	. 185
Outlet Gas Temp. (°F)	150	150	130	150	120	155 .
Bed Diff. (in. H <sub>2</sub> 0) Demister Diff. (in. H <sub>2</sub> 0)	/4.8	·· ·-· · · /4,8 ·· · · · · · · · · · · · · · · · ·	14.8	14.8	. 14,6	14.7
RH Diff. (in. H <sub>2</sub> 0) – (	2.83	1.9	0.4 0.5	0.4	0,3 0,3	0,3 0,3
GAS-Outlet-Flow-(in-H20) REHEAT STA Parss PSA	350	3 <i>50</i>	. 390	380		3 <i>8</i> 3
Section D; Presat. Water Flow (gpm)	190		- 180	185	<b>ී</b> වීට	, 185
Outlet Gas Temp. (°F)	155	160	- 180 155	1850	. 185°	, 185 , 185
Bed Diff. (in. H <sub>2</sub> O)	5.4	5,4	6.8		76	7.6
Demister Diff. (in. H2O) RH Diff. (in. H2O)	2,2	2,4	2.0 .	2.1	2.3	2.6
Gas Outlet Flow (in. H2O)	0 -	- 0	0	-   -0 -	. <i>O</i> .	0;
REHEAT STM RESS PSIG	1_200.	200	350	350	350	320
Comments OBSELVER - R. IDA		•		-		

	1							<del>,-</del> -		
Date:	8/8/77	8/8/77	8/9/77	2/9/27	ľ				i i	
Time:	10/8	1054	1415	, , , ,						
<del></del>	-			1517						
Plant				i	1					
	<del> </del>		<del></del>	<del></del> }						
Mw, Gross	350	357	357	354	1	1		'	1	
XS 02	4.3	4.3	4.3	4.6						
Steam Flow (lbs/hr) x103	33.50	2300	2350	235C			· · · · · · · · · · · · · · · · · · ·			
Steam Pressure (psig)	2300	2350	2350	2350						
<u>Steam Temperature (°F)</u>	965	470	470	970						
Opacity, Bypass (%)	24/7	27/7	26/8	18/6		i				
Opacity, Scrubber (%)	22	23	20	18						
	1									
ESP	İ								į į	1
Section Al; AC Voltage (v)	240+30	250-70	250-20	250-20						
AC Current (a)	! <i>!!\$</i> 20.	100-25	untico!	120 - 10						1
DC Voltage (Kv)	- Dut	OUT	out.	0					<u> </u>	L
OC_Current (ma)			354 =00							
Spark Rate (spm)	85	83	90	90					t	
1	SUME	]							į .	i
Section A2: AC Voltage (v)	\$100 PD		Same	SOME					<del> </del>	<b> </b>
AC_Current (a)	1 45.10		ļ				<del></del>		1	<u> </u>
OC_Voltage (Kv)	1	!		!					<u> </u>	
DC Current (ma)	<u> 940</u>	L	<b></b>							
Spark Rate (spm)	. <del></del>	<u> </u>	<u> </u>						L	<u> </u>
	1			ι.	.					
Section A3; AC Voltage (v)			190-10						<del> </del>	
AC Current (a)			(55/0		<b></b>				· 	
DC_Voltage (Kv)		0.4		_ابم					<b></b>	<del></del>
DC Current (ma)			310 × 10		<u> </u>			ļ <u>.</u>	1	<u> </u>
Spark Rate (spm)	1001	1004	DU	40℃					j	! ;
0 11 10 10 11 1 1 1 1 1 1 1 1 1 1 1 1 1	16	1 .								1
Section A4; AC Voltage (v)	1 5 9 Mc	Save	2-46	Same				<del> </del>	<del></del> -	<del> </del>
AC Current (a)	<del></del>	<u> </u>	<del> </del>		<b></b>	<del></del>		<del> </del>	<del> </del>	<del>}</del>
DC Voltage (Kv)		<del></del>	<del> </del>		<del>  </del>			<del> </del> -	<del>!</del>	
DC_Current (ma)	<del></del>	1		<b> </b>	<del> </del>				<del> </del>	<del> </del>
Spark Rate (spm)	!	1						ļ	·	<u> </u>
Continu Di. AC Valtage (v)	204	290	280	290-10				ł	i	;
_Section_Bl;_AC_Voltage (v) AC Current (a)	290				<del></del>	<del></del>	<u> </u>	<del> </del> -	<del> </del>	<del>                                     </del>
DC Voltage (Kv)	185	135=10	130-15	130 £10	<del> </del>		<del></del>	<del> </del>	<del> </del>	<del> </del>
DC Current (ma)	100+	7767	Dut	04	<del> }</del>		<del> </del>	<del>├─</del>	<del> </del>	<del> </del>
Spark Rate (spm)	680	-640	صري الم		┝╼━┿		<del> </del>	<del> </del>	<del> </del>	<del> </del> -i
Shark vare (shiii)	183	185	185	185			<u> </u>		·	<u> </u>
_Section_B2; AC Voltage (v)	Ctures	Saine	SAMO	Same	ı 7			! .		
AC Current (a)	<del>-&gt;</del> 3₩₩₽	·	3-17/1	700.15	;──}		<del> </del>	<del></del>	<del> </del> -	<del> </del> '
DC Voltage (Kv)		ļ	·}	<del> </del>	<del>                                     </del>		<del> </del>	<del></del>	<u> </u>	<del> </del>
DC Current (ma)	· ———	<del> </del>	<del>i</del>	<del> </del>	<del>'</del>		<del> </del>	<del> </del>	<del>:</del>	<del>(                                    </del>
Spark Rate (spm)		<del> </del>	<del> </del>	<del> </del>	<del></del> }		<del> </del>	<del>:</del>	<u> </u>	<del></del>
	<del></del>	<del> </del>	ļ	ļ			ļ	!	·	<b></b>
Section_B3;_AC_Voltage (v)	215 10	210-10	22052	210+20	• 1			1	:	!
AC_Current (a)	_ GO	Co		55±5				1	,	
DC Voltage (Kv)	31	31	33	33	<del></del> †				<del></del>	<b> </b>
DC Current (ma)	240			340=10			<del>                                     </del>	1	•	†
Spark Rate (spm)	45	240	55	530	<del>:</del> -		T	1		
	<del></del>	<del> </del>	<del> </del>	<del>  -</del>	<del></del> ∤		<del> </del>	<del> </del>	i	<del> </del>
Section B4; AC Voltage (v)	Same	Same	Since	Same	, '		L	<u> </u>	1	
AC Current (a)		1	T	I			<u> </u>			
DC Voltage (Kv)		-, <del></del>	1		·					
DC Current (ma)			1	1				!		
Spark Rate (spm)		1	i	ı	}		1	1	1	i
,			┸	4	اـــــــــــــــــــــــــــــــــــــ			<del></del>	<del></del>	<del></del>

## CHEROKEE STATION UNIT 4 DAIA SHEET (continued)

A-36

	<del></del>		<del></del>				<del>,</del>	<del></del>		<del></del> .
Scrubber	•	}	1	1	Ì	]	1	1		İ
Fans: ID_Discharge Pressure (in. H2O)	-46	~.5	- 6	- 5						
Fan R Outlet Pressure (in. H2O)	OUT	Out	OUT.	0.4						
Fan ← Outlet Pressure (in. H2O)	13.5	13.0	16.5	u						
Ean Outlet Pressure (in. H20)	12.5	115	12.12	11.5						
FanAmps	250	1 240	245	240			ļ	<u> </u>	!	<u> </u>
Fan Amps	225	220	225	22.5	· .			<u> </u>	<u> </u>	
Fan D Amps	24 C	230	235	2.25	ļ			<del> </del>	ļ	<del> </del>
Stack Damper A, Pos. (% Opn)	0	0	8-	<del>  0</del>		ļ	<del> </del>	ļ		
Stack Damper B, Pos. (% Opn)	0	0					1		į	
Pumps: Recirc, Pump R (amps)	12.5	12.5	12.5	12.5	1			ĺ		ī
Recirc. Pump B2 (amps)		16	16	160	<del></del>	<del>                                     </del>		1		
Recirc. Pump 83 (amps)	12.5	13.5	12.5	12.5	!	<u> </u>	†	<del>                                     </del>	<del>                                     </del>	†
Recirc. Pump c. (amps)	13.5	12.5	13	13	<del>                                     </del>	<del>                                     </del>	<del>                                     </del>	1	<del>                                     </del>	<del> </del>
Recirc. Pump (2 (amps)	0	(2)	13	13		<del> </del>	1	<del></del>	<u> </u>	<del> </del>
Recirc. Pump c3 (amps)	12.5	12.5	13	13	<u> </u>				<del></del>	<del> </del>
Recirc. Pump (amps)		13.5	_/3.5	135		<b>†</b> -	·	<del>                                     </del>	<del></del> -	;
Recirc. Pump Oz (amps)	13.5	. 13.	12.5	13	<u> </u>		1		<del>                                     </del>	
Recirc. Pump 53 (amps)	1 14	14	13.0	135		1	1			<del>                                     </del>
		<del></del>			,	<del> </del>	<del> </del>	<del> </del> -	<u> </u>	<del>†</del>
Reheater: Steam Flow (M lbs/hr)	60	60	50	50	<u>'</u>	<u> </u>	<del></del>	<del> </del>	<del> </del>	<del></del>
Steam Temp. (°F)		!			<u> </u>	ļ	<del> </del>	<del></del>	<del> </del> -	
Steam Pressure (psig)	1462	1480	1480	1480	<u> </u>	l				1
Section B; Presat. Water Flow (gpm)	185	125	185	185						
Outlet Gas Temp (°F)	135	125	/23	125						
Bed Diff. (in. H <sub>2</sub> O)	. 0	0.7	14.3	1/.0				T		
Demister Diff. (in. H <sub>2</sub> 0)	0.4	0.9	0.9	0.9						Ĭ
RḤ Diff. (in. H <sub>2</sub> 0)	<u></u> 0	_0_	0_	0						Γ
Gas Outlet Flow (in H20)	820	320	270	270	ļ	1	1	i		}
Section C: Presat Water Flow (gpm)		175	175	175	1	i	T	•	Ţ	
Outlet Gas Temp. (°F)	155	1/55	150	150	1	<del></del> -			1	} !
Bed_Diff. (in. H <sub>2</sub> 0)	14.6	14.6.	116	11.0		i	1	1	1	<del>                                     </del>
Demister Diff. (in. H <sub>2</sub> 0)	0.3	0.3	0.25	0.25						
RH_Diff. (in. H <sub>2</sub> 0)	0.4	0.4	6.2	0.1						
Gas-Outlet-Flow-(inHoO)	500	470	i .	470					Ī	Ī
Supin Piers (boid)			460		<del></del>	<del> </del>	<del> </del>	<del> </del>	<del> </del>	┼
Section_ <u>D</u> : Presat. Water Flow (gpm)_	_نع،	185	125	185		ļ		<del>!</del>	<del> </del>	<del></del>
Outlet Gas Temp. (°F)	155	140	155	160	<u> </u>	<del> </del>		- <del></del>		·
Bed Diff. (in. H20)	74	6.6	7.3	7.3	<del>:</del>	<del> </del>	<del></del>		<del> </del>	<del> </del>
Demister Diff (in. H20)	2.4	2.1	2.7	2.2	<del> </del>	<del> </del>	<del></del>	<del>'</del>	-}	<del> </del>
RH. Diff. (in. H20)	<del>; -c-</del>	<del>                                     </del>		0	<del>                                     </del>	<del> </del>	<del> </del>	<del></del>	<del> </del>	<del></del>
Ras Outlet Flow (in, H2O)	160	430	460	460	1	i		1	1	1

Date:	8/6/17	8/6/17	\$1911	8/10/17	
Time:	1335	1500	2215	2350	
Plant					
Mw, Gross	345	350	295	295	
	3 4.5	4.4	3.7	4,0	
Steam_Flow_(lbs/hr) &	2300	2300	1800	1550	
Steam_Pressure_(psig)	2300 <del>2300</del>	2300		23.50	
Steam_Temperature (°F)_	N/45 30/6	25/15	2012	1000	
Opacity, By <del>pas</del> s (%) <u>4</u> Opacity, S <del>crubber (</del> %)				12/8	<del></del> -
COAL FLOW (T/HR	22	23	120		
ESP CONC FLOW (1/MZ			1	120	
Section Al; AC_Voltage (v	) 270 E 30	270 30	270±30	270339	!L
AC Current (a	) 130 E/U	135±10	140 = 10	140±20	
DC Voltage (K	v) ou I _	out	OUT	OUT	
DC_Current (m	a) <u>650 = 50</u>	650550	700=50 ·	100±54	
Spark_Rate (s	pm) <u>85</u>	35	60	45	
Section A2: AC_Voltage_(v	)SAME	SAME	SAME	SAME	
AC_Current (a	)				
DC Voltage (K	v) . ;				
DC Current (m	a)				
Spark Rate_(s	pm)	i	<del></del>		
0 IN IN U-3 I.	1 2047 201	240E30	220 = 2U	220 = 20	i I
Section A3; AC Voltage (V	240±30	190±20	75=14	80 1/0	! <b></b>
AC Current (a		047	13-19	euT	
DC_Voltage (K DC Current (π		500 F 20 1	350±40	40V F 50	
Spark Rate (s		OUT		OUT	;
	04.		SIME	SAME	
Section A4; AC Voltage (V	SAME	SAME	Simo	SAMO	<del></del>
AC Current (a DC Voltage (k			<del>  </del>	<del></del>	<del></del>
DC Current (n	na)	<del></del>	<del></del>	<del></del>	
Spark Rate (S	spm)		- I - I		
_Section_Bl;_AC_Voltage_(	1) 300 <sup>I</sup> 10	3 26±20	320-20	320±20	1
AC Current (a	·	SC TAO	150 T20	150 Edo	
DC Voltage (	(v)   out	OUT	OUT	OUT	<del> </del>
DC Current (	na) 800 F20	8NJ 50	750±30	750±401	
Spark Rate (	spm) 185	185	185	185	
_Section_B2; AC Voltage (	1	SAME	SAME	SAME	:
AC Current (a	a) ' <u>i</u>				1
DC Voltage (	(v)				
DC Current (			,		
Spark Rate (	·				
_Section_B3; AC Voltage (	v) 250±30	250 = 40	250± 40	250=40	
AC Current (	a) <i>&amp;v_C20</i>	80120	70±10	70±10	<u></u>
DC Voltage (	KV)30=1	30±/	34±2	38±2	
DC Current (		400 £ 40	300 £ 20	300±20	
Spark Rate (	spm) 40	45	35	35 '	
Section B4; AC Voltage (	V) SAME	SAME	SAME	SAME	<del> </del>
AC Current (					<del> </del>
DC Voltage (		<del></del>	<del></del>		<del> </del> -
DC Current (				<del> </del>	<del> </del>
Spark Rate (	July [				1

Scrubber	8/6/77	8/6/77	8/14/77	8/10/77	
Fans; ID_Discharge Pressure (in. H2O)	7-0.5		-0.5	-0.5	
Fan B Outlet Pressure (in. H20)	OUT	047	OUT	OUT	
Fan C Outlet Pressure (in. H20)	13	12,5	11	11 kB	
Fan Outlet Pressure (in. H20)	12	/2	23	9 1	
Fan B Amps ×10	24	24	23	1,73	1
Fan	22	22	22	22	
Fan // Amps // Con Stack Damper A, Pos. (% Opn)	23	23	24	24	<del></del>
Stack Damper A, Pos. (% Opn)		0			
Stack Damper B, Pos. (% Opn)	0	0	0	0	<u>i</u>
Pumps: Recirc. Pump <u>Bl</u> (amps)	/3	13	/3	13	
Recirc. Pump BL (amps)	16	16	16	16	
Recirc. Pump #3 (amps)	. 13:	13	13	/3	
Recirc. Pumpcimi (amps)	ル	12	/3	1/3	
Recirc. Pump C2 (amps)			12	( )	-
Recirc. Pump <u>c3</u> (amps)	13 .	13	/3	/3	
Recirc. Pump <u>D'</u> (amps)	13	13	/3		
Recirc. Pump 02 (amps)	/3	/3	_/3	/3	
Recirc. Pump 03 (amps)	· 13 ,	13	1.13	/3	
Reheater; Steam Flow (M lbs/hr) Steam Temp. (°F)	55	55	1.50	50	
Steam Pressure (psig) x/0 <sup>2</sup>	15				—- <del> -</del>
5 Ceam F1 E3541 E (p319) A/0	/5 ,	15	12	12.5	!
Section B : Presat. Water Flow (gpm)	185	185	190	190	
Outlet_Gas_Temp,_(°F)	145	145	130	130	
Bed Diff. (in. H <sub>2</sub> 0)		1/	6.8	6.8	
Demister Diff. (in. H <sub>2</sub> 0)	0.9	0.9	0,9	0.8	1
RH Diff. (in. H <sub>2</sub> 0)	OUT	out	DYTI	our:	
Gas Outlet Flow (in. H20)	4.5	4.5	1.4	1.4	! !
_Section_C; Presat Water Flow (gpm)	175	175	1 175	175	
Outlet Gas Temp. (°F)	155	155	150	150	
Bed Diff. (in. H <sub>2</sub> 0)	148	14.8	7.8	8.0	
Demister Diff. (in. H <sub>2</sub> 0)	0.3	0,3	10,2	1.9	
RH Diff. (in. H <sub>2</sub> 0)	0. 2	0.1	1,9	1.9	
Gas_Outlet_Flow (in. H2O)	400	4,0	1 4.6	4.6	
_Section_D_; Presat. Water Flow (gpm)	185	190	1 190	190	
Outlet Gas Temp. (°F)	160	160	160	160	
Bed Diff. (in. H <sub>2</sub> O)	6.8	6:4	5.6	5.6	
Demister_Diff(in. H2O)_	2.0	2.1	118	1.8	
RH Diff. (in. H2O)	OUT	OUT	OUT	out.	
Ras Outlet Flow (in. 1120)	4.0	4.0	24	2.4	1 1

Date:	8/2/11	8/12/11	8/13/17	8/13/17	
Date: 4"	1400	1530	0450	1/20	<del></del>
Plant		1330		.,	
Mw. Gross	350	350	DOWN	DOWN	
	6.9	7.0	FOR		
Steam_Flow_(lbs/hr) <i>/0</i> /	2350	2350	STEAM		
Steam_Pressure (DS19)	2350	2350	LEAK		
Steam Temperature (°F) Opacity, Bypass (%) Lim Suylu	990	980 34	<del></del>	╼╂╼╂╼╌┧╼╼┈	
Opacity, Scrubber (%) 4N/4S	22/15	22/17	BUILER	<del> - </del>	<del>-</del>
CONC PLW THE	140	145	BUILDE	<del></del>	!
ESP	1 1	1 · i		1 1 1	i
Section Al: AC Voltage (v)	250230	250-30		!	
AC Current (a)	125-20	115=20			
DC Voltage (Kv)	out	550ESO			
DC_Current (ma)	600 T 50	550E50	<del></del>		
Spark_Rate (spm)	90	90	!\_		
Continue AC AC Voltage (u)	SAME	SAME			
Section A2: AC_Voltage_(v) AC_Current (a)		277MR	<del></del>	<del></del>	<del></del>
DC Voltage (Kv)	<del></del>			<del></del>	<del></del>
DC Current (ma)	<del></del>	<del></del>	<del></del>	<del></del>	
Spark Rate (spm)	. ;	i	<del>-                                     </del>		
		<del></del>			
Section A3; AC Voltage (v)	200740	200 140			1
AC Current (a)	70+10	70±20			
DC_Voltage (Kv)	OUT:	OUT			
DC Current (ma)	3:0T40	1 200=40			
Spark Rate (spm)	· out	OUT			
			<del></del>	<del></del>	
Section A4; AC Voltage (v)	SAME	SAME			
AC Current (a)				<del>  </del> i	
DC_Voltage (Kv)	<u></u>				
DC Current (ma)					
Spark Rate (spm)			!		
Continu DI AC Voltago (v)	280-20	280520			
AC_Voltage (v) AC_Current (a)	130±20	125±20	<del></del>	<del></del>	
DC Voltage (Kv)		047		<del></del>	<del></del>
DC Current (ma)	660±20	640±40		<b></b>	
Spark Rate (spm)				<del></del>	
	7200	7200			
Section B2; AC Voltage (v)	SAME	SAME		1 • 1	
AC Current (a)		,			
DC Voltage (Kv)					
DC Current (ma)		:			
Spark Rate (spm)	•				1
		20.+30		<del></del>	
_Section_B3;_AC Voltage (v)	220 ±40	220=30			<del></del> }
AC_Current (a)	60±5	60=5			
DC Voltage (Kv)	35	34+2		;	
DC Current (ma)	250120	230±20			
Spark Rate (spm)	50	65			
Section B4; AC Voltage (v)	60-6	SAME		1 1	]
AC Current (a)	SAME	1 DKME		<del></del>	
DC Voltage (KV)		<del></del>	<del></del>	<del></del>	
DC Current (ma)	<del>  </del>	<del></del>	<del></del>	<del>!</del> -	<del></del>
DO OUTTOILE \mu/.					

8/12/77. REHEAS STEAM OFF DUE TO BOKER LEAK TRYING TO CONSERVE STM.

_Scrubber_	<i>बार्या</i>	8/47	8/13/77	8/13/77	
_Fans: ID Discharge Pressure (in. H20)	-0.5	-0.5			
Fan D Outlet Pressure (in. H2O)	OUT	047	ROLN	DOWN	<del></del>
Fan B Outlet Pressure (in. H20) Fan C Outlet Pressure (in. H20)	16	16.5	140470	// Juliu	<del></del>
Fan D Outlet Pressure (in. H2O)	12.5	12.3			
Fan B Amps	23	22	— <i> </i>		
Fan C Amps	22	22			
Fan D Amps	24	24			
Stack Damper A, Pos. (% Opn)	U	0			
Stack Damper B, Pos. (% Opn)	: 0	0	1 7 1	7	
Pumps; Recirc. Pump B, (amps)	13	/3			
Recirc. Pump By (amps)	16	16			
Recirc. Pump 🔏 (amps)	12	12			
Recirc. Pump & (amps)	/3	/3			
Recirc. Pump C1 (amps)	7/	//			-
Recirc. Pump C2 (amps)	13:	13			
Recirc. Pump D. (amps)	/3	/3			
Recirc. Pump Dr (amps)		/3			
Recirc. Pump (amps)	12.	12			
Reheater; Steam Flow (M lbs/hr)		0			ļ. ļ.
Steam Temp. (°F)		1 - 1			1
Steam Pressure (psig)	<b>–</b> ,	7			
Section 🌶 ; Presat. Water Flow (gpm)	180	185			
Outlet_Gas Temp(°F)	120	1 120		17	
Bed Diff. (in. H <sub>2</sub> 0)	10	10.4			
Demister Diff. (in. H <sub>2</sub> 0)	40 0.9	0.9			
RH Diff. (in. H <sub>2</sub> O)		_			
Gas Outlet Flow (in. H2O)					_
Section C ; Presat Water Flow (gpm)	175	175			
Outlet Gas Temp. (°F)	120	120			
Bed Diff. (in. H <sub>2</sub> O)		11.4			
Demister Diff. (in. H <sub>2</sub> 0)	6.2	0,2			
RH Diff. (in. H <sub>2</sub> 0)					
Gas Outlet Flow (in. H <sub>2</sub> 0)	_	-			
Section <u>.D_;</u> Presat. Water Flow (gpm)	190	190			
Outlet Gas Temp. (°F)	125	125			
Bed Diff. (in. H2O)	7.9	7.2			
Demister Diff(in. H2O)	2.2	2.3			
RH Diff. (in. H2O)				Ĺ į	
has Dutlet flow (in. H20)		-	,		T i

		<del></del>			~ <del></del>
Date:	8/15/17	8/15/17	8/22/17	13/22/77	
Time:	1005	1/30	0950	1115	
Plant			10730		
					<del> </del>
Mw. Gross	270	270	365	350	
XS <sup>O</sup> 2 Steam_Flow (lbs/hr) <u>×0<sup>3</sup></u>	4.270	1650	2450	3.5%	
Steam Pressure (psig)	2350	2350	3400	2350	
Steam Temperature (°F)	1000	1000	1000	1000	
Opacity, Bypass (%) CAL SIEGL		24	31	30,	<u> </u>
Opacity, Scrubber (%) 4N/45  COAL FLOW T/HR	15/5	18/8	43/26	35/17	
ESP FOR 1742	110	110	150	145	
Section A1; AC Voltage (v)	260TIO	260±20	2/0-10	210 10	
AC Current (a)	70 t 20	70 = 20	70 = 20	7UIZU	
DC Voltage (Kv) _	ouT	0UT 400	CUT	OUT	
DC_Current (ma) Spark Rate (spm)	400	165	7200 ·	400 I20	<del></del>
abai v uace '(abm\ _	1			7200	<del></del>
_Section_A2:_AC_Voltage_(v)	SAME	SAME	SAME	SAMO	
AC_Current (a)				<u> </u>	<u> </u>
DC Voltage (Kv) DC Current (ma)	<del></del>		<del></del>	<del></del>	
Spark Rate (spm)			<del></del>		
•					
Section A3; AC Voltage (v)	230±20	230 - 20	220-10	200±20	
AC Current (a)	90±6	SOLIO	100 E 10	80±10	
DC_Voltage (Kv) DC_Current (ma)	950 - 20	1 04T	SUF	04 T 400 ± 20	
Spark Rate (spm)			OUT		
<del>,</del>	OUT	OUT		OUT -	
<u>Section A4; AC Voltage (v)</u>	SAME	SAME	SAME	SAME	
AC_Current (a)					
DC Voltage (Kv) DC Current (ma)		<del></del>			<del></del>
Spark Rate (spm)	<del> </del>				<del> </del>
				112.T20	
_Section_Bl:_AC Voltage_(v)	320	310	270-20	270-20	
AC Current (a) DC Voltage (Kv)	DOT 5	WOIS OUT	130 120	150±30	
DC Current (ma)	560=20	580120	580±3c	560=20	
Spark Rate (spm)	15	20-10	185	185	
Section_B2; AC Voltage (v)	SAME	SAME	SAME	SAME	
AC Current (a) DC Voltage (Kv)					
DC Current (ma)	'· · · · · · · ·				
Spark Rate (spm)		<del></del>	i		<del></del>
	o/atan	060±60	2/1	200	
_Section_B3: AC Voltage (v)	260±20	260±40	250	250	<del></del>
AC_Current (a) DC Voltage (Kv)	_110±201 	100±20 31±2	110	140 33±2	
DC Current (ma)	460-20	460±40	520	620±20	
Spark Rate (spm)	45	45	105	105	
			!		
Section B4; AC Voltage (v)	SAME	SAME	SAME	SAME	<del></del>
DC Voltage (Kv)	,				<del></del>
DC Current (ma)					
Spark Rate (spm)			,		

				1115	
Scrubber	8/15/77	8/15/77	8/22/11	8/22/77	
Fans: ID Discharge Pressure (in. H2O)	-0,2	-0.5	-04	-0.4	
Fan & Outlet Pressure (in. H20)	OUT	OUT	out	OUT	- <del> </del>
Fan C Outlet Pressure (in. H20)	10.5	10.5	11.5	12	
Ean_D Outlet Pressure (in. H20)	7.0	7,5	12	12	
Fan & Amps × 10	22	22	20	21	
Fan C Amps XIO	12	22	22	221	<del></del>
Fan D Amps - 210	23	23	24	24	<del></del>
Stack Damper A, Pos. (% Opn)	0	0	U	0	
Stack Damper B, Pos. (% Opn)	0	0	0	0	
	<del>,</del>				<del></del>
Pumps; Recirc. Pump 👸 (amps)	13	13			+
Recirc. Pump Br (amps)	16	16		16	
Recirc. Pump 🚜 (amps)	13	/3 i	1_3	<u> </u>	
Recirc. Pump 🕰 (amps)		/3		12:	
Recirc. Pump C. (amps)	14	14	13	13	
Recirc. Pump <u>C</u> ₄ (amps)	14!	14	14	14	
Recirc. Pump <u>D</u> , (amps)	13	13	13	13	1
Recirc. Pump 🕖 (amps)	13	/3	13	13	
Recirc. Pump 1)3 (amps)	12	12	13	13	-
Reheater; Steam Flow (M lbs/hr)	40	40	1 45	45	
Steam Temp. (°F)	72	7			
Steam Pressure (psig)	600	600	1000	1000	
Carlo R Daniel Haber Electron	100	185	190	185	1
Section $B$ ; Presat. Water Flow (gpm)	185				
Outlet Gas Temp(°F)	135	120	140	125	
Bed Diff. (in. H <sub>2</sub> 0)	3.8	3.8		8.0	
Demister Diff. (in. H <sub>2</sub> 0)	0.7	0.7	0.7	0.9	
RH Diff. (in. H <sub>2</sub> O)	0.4	0,4			
Gas Outlet Flow (in. H20)	250	300	300	310	_
_Section_C : Presat Water Flow (qpm)	175	175	! /75 !	175	
Outlet Gas Temp. (°F)	160	160	155	155	<del></del>
Bed Diff. (in. H <sub>2</sub> 0)	7.2	7.2	1 8.0	10.4	
Demister Diff. (in. H <sub>2</sub> 0)	0.2	0.2	0,2	0.3	
RH Diff. (in. H <sub>2</sub> O)	047	OUT	0.4	0.4	
Gas Outlet Flow (in. Hat)	3/0		380	380	
RH STM-PR (PL)		320 185	190	190	<del></del>
_Section_D; Presat. Water Flow (qpm)	185	160			
Outlet Gas Temp. (°F)	/k0		150	150	
Bed Diff. (in. H20)	420	4.0	11.8	9.8	
Demister Diff. (in. H20)		1.3	2.0	200	<b></b>
RH. Diff. (in. H20)	OUT	- OLAT	OUT	0471	
Ras Outlet Flow (in. 1120)	100	100	100	100	l1.
Comments	<del></del>				

Date:	8/23/71	15/23/77	8/24/11	8/14/11	8/24/11	8/14/77
Time:	0945	1045	0410	1/30	1305	1500
Plant FUEL	100% COAL	100% COAL	100% COM	L/00%	100% COM	100%, con
Mw, Gross	365	360	365	360	355	355
XS 02 3	318	3.8	4/10	4%	3.8%	3.8%
Steam Flow (lbs/hr) //O	2400	2400	2450	2400	2400	2400
Steam Pressure (psig) Steam Temperature (°F)	2250 990	9350	970	2350	2350	ن رور
_Opacity, Bypass (%) LEAR 5	IEALER 30%	¥ 50%	28%	990	24%	990
Opacity, Scrubber (%)44//	15 31/18%	35/16	24/16	30/25	25/18	30%
COAL FLOW THE	150	150	150	150	145 +==	40/30
ESP	1 1	1 1				1
Section Al; AC_Voltage (v)	220-10	220	230	230	230	220
AC Current (a) DC Voltage (Kv)	75 I 5	75	60	7055	105	. 75
	047	10 380 E20	- QUI	out	our	DUT
Spark Rate (spm)		7200	x# 5	300±20	280	340
The state of the s	1	700	<del>- /</del>	FFE 1307	F50 10 '	7200
<u> Section_A2;_AC_Voltage_(v)_</u>	SAME.	SAME	SAME	SAME	SAME	SAME
AC Current (a)						
DC Voltage (Kv)						
DC Current (ma) Spark_Rate_(spm)	\ : <i>(</i> :	<del></del>	<del></del>			
Shark_kate_(shii)	<i>'</i>				<del></del>	<del></del> :
Section_A3; AC Voltage (v)	20U - 3U	20UI.20	200=20	acutad	220 = 30	20UI 3
AC Current (a)	40 E 10	80-10	90±10		100 = 20	8523
DC_Voltage (Kv)	OUT	vur	OUT		cut	047
DC Current (ma)	420 = 36	40U = 201	400 52	O 4UUEZO	480±40	400 = 40
Spark Rate (spm)	out	OUT	our!	OUT	OUT:	OUT
Section A4; AC Voltage (v)	49.CT 30	1.01.5			SAME	
AC Current (a)	SAME	SAME	SAME	SAME	SAME	SAME
DC Voltage (Kv)	7///=					
DC Current (ma)						
Spark Rate (spm)	)					
		070 T (	on t			<del></del>
Section_Bl;_AC_Voltage_(v)	250-10	170-10		270-20	270520,	270-20
AC Current (a)	./30 <u>~</u> 1.0°	140=10	130 = 130		150±40	125-120
DC Voltage (Kv) DC Current (ma)	650 E40	OUT	OUT	our	OUT!	OUT
Spark Rate (spm		600130	<u>640 I 20</u>		1720 I 20:	<u>550±</u> 5
	185	180	180	185	185	180
Section_B2:_AC Voltage (v)	SAME	SAME	SAME	D-7-	SAME	SAME
AC Current (a)			;	SAME		
DC Voltage (Kv)						
DC Current (ma)			··	<u> </u>		
Spark Rate (spm	, , , , ,		l		,	
Section_B3;_AC Voltage (v)	260	250	260	240	20	260
AC_Current (a)	115	115	1115	130510	115	
DC Voltage (Kv)	33	34	. 33	33	33	720
DC Current (ma)	100	620120	580 = 20	580-26	560±20	620 = 2
Spark Rate (spm	105	105	105	105	105	100
Section B4; AC Voltage (v)		1 ' 1	-	!		1
SPETION KA! AL VOITAGE (V)	SAME	517ME	SAME	SAME	SAME	SAME
	• :	1 1				
AC Current (a)			<del></del>			

\* REHEATER STA OFF

\*\* I'L OF A SECTION HAS BEEN GROUNDED DUE TO EXERSSIVE

SPARKING

	<del></del>	<del> </del>		<del></del>
Scrubber	8/23/17	8/23/71	8/24/71 8/24/71	8/24/77 8/24/77
Fans: ID Discharge Pressure (in. H2O)	-0.5	-0.5	-0.5 -0.5	-0.4 -0,4
Fan 6 Outlet Pressure (in. H2O)	OUT	OUT	64T 04T	OUT OUT
Fan C Outlet Pressure (in. H2O)	13	12	12 12	113, 125
Ean D Outlet Pressure (in. H2O)		12,5	12 12.5	12.5 12
Fan Amps X10	2/	21	21 21	21:21
Fan C Amps X /O	23	23	22 22	122 22
Fan // Amps // /OStack Damper A, Pos. (% Opn)	+ 3	0	24 29	24 24
Stack Damper B, Pos. (% Opn)	· · · · · · · · · · · · · · · · · · ·			. 0 _ 0
Stack bumper by 103. (a opiny	. 0	U	0 0	0 0
Pumps; Recirc. Pump $\beta_{I}$ (amps)	/3	13	/3   /3	13 13
RecircPump 🙃 (amps)	16	/(2	16 16	16 16
Recirc. Pump 3 (amps)	/3	13.	/3 /3	/3 / /3
Recirc. Pump C (amps)	121	12	12 /2	12 8912
Recirc. Pump C2 (amps)	13	131	1/2 /2	13 13
Recirc. Pump(amps)	/3;	13	14 14	15 14
Recirc. Pump D, (amps)		131	13 13	/3 /3
Recirc. Pump Da (amps) Recirc. Pump D3 (amps)	/ 3	13	13 13	13 13
Recirc. Pump 03 (amps)	12.	121	13 12	12 12
Reheater; Steam Flow (M lbs/hr)	20	OFF	55 55	55 55
Steam Temp. (°F)		1 -	-	
Steam Pressure (psig)	450:	-	500 1700 1700	1700 1700
Section $\underline{\mathcal{B}}$ ; Presat. Water Flow (gpm)	185	190	190 190	185 185!
Outlet_Gas Temp(°F)	125	125	125 125	125 125
Bed Diff. (in. H <sub>2</sub> O)	9,0	8,4	8,4 8,4	8,8 8,6
Demister Diff. (in. H <sub>2</sub> 0)	0.9	0.9	0.9 0.9	0,9 6.9
RH Diff. (in. H <sub>2</sub> 0)	OUT	ouT	OUT OUT	04T 04T
Gas Outlet Flow (in. H20)  RIF 5779	120	OFF	260 280	300 ! 3 00
Section Presat Water Flow (qpm)	175	175	1825 180	175 155
Outlet Gas Temp. (°F)	135	120	145 155	155 155
Bed Diff. (in. H <sub>2</sub> O)	7.9:	4.0	4.2 12.4	14.4 13.8
Demister Diff. (in. H <sub>2</sub> O)	0.4	0.4	0.2 0.4	0.4 0.4
RH Diff. (in. H <sub>2</sub> 0)	0.3	0.8	1,0 0.5	0.4 0.9
Gas Outlet Flow (in. H20)	100	OFF	1 440 440	440 440
Section_D : Presat. Water Flow (qpm)	190	190	190 190	190 185
Outlet Gas Temp. (°F)	145	125	155 155	155 160
Bed Diff. (in. H2O)	2.0	9,6	· OUT 125 7.4	9.4 6.6
Demister_Diff(in. H2O)_	2.0	2.0	12,0 2.0	1 2.5 2.5
RH Diff. (in. H20)	047	OUT	OUT OUT	1 out out
-Ras Outlet Flow (in, H2O)  RH SIM PR (P>G)	50	OFF	420 420	420 420
Commonte	<del></del>			

	101/	11				
Date:	¥25/17	8/25/11				
Time:	0640	0845				
Plant FUEL	DUZ CONL	VOUPS COAL				
Mw. Gross XS 0 <sub>2</sub>	365	360			<del>-  </del>	
Steam_ <u>Flow_(lbs/hr)</u>	2400	2400	<del></del>	·	<del></del>	
Steam_Pressure (psig)	2350	2350			-i	
Steam_Temperature (°F)	980	980				
Opacity, Bypass (%) L CAA	5164. 30%	26%	_			•
Opacity, Scrubber (%) 4	N/45 28/28	30/18				
ESP	į			1	1 1	1 1
Section Al; AC Voltage (	1) 230	240				
AC Current (a		110 = 20				
DC Voltage (I		047		~		
Spark Rate (		520 = 2U		·	<del>  </del>	<del></del>
		7200		<del></del>	<del></del>	
_Section_A2;_AC_Voltage_(	v) <u>SAME</u>	SAME				i
AC_Current (	a),'	<del>-</del>				
DC Voltage (1	KV				<u> </u>	!
Spark_Rate_(		<del></del>			<del></del>	
Section A3; AC Voltage (	v) 200 [20]	230				:
AC Current (		110				
DC_Voltage (DC_Current (	Kv) <u>. OuT</u> ma) <u>380 ± 20:</u>	60 I ZU				
Spark Rate (					<del></del>	<del></del>
,	Dat	out			<del></del>	
Section A4: AC Voltage (	v) ( 270-10	SAME				
AC_Current (						<del></del>
DC Voltage (DC Current (		<del></del>			- <del></del>	
Spark Rate (			<del> </del>		1	<del></del>
<del></del>	1100		<del></del>		~ <del> </del>	
_Section_Bl:_AC_Voltage_(	v)	270-10				
AC Current ( DC Voltage (	a)	125=10		<del></del>		
DC Current (	ma)	600±20				
Spark Rate (	spm)	180			<del></del>	
Section_B2;_AC Voltage (	V) SIME 260 /	SAME				
AC Current ( DC Voltage (	KV) V/3/				<del>-!</del>	
DC Current (			<del> </del>	<del></del>	<del></del>	
Spark Rate (	spm) //oX		<del></del>			<del></del> -
		<del></del>				
_Section_B3:_AC_Voltage (	v)	260	1		<del></del>	<del></del>
AC_Current ( DC Voltage (		132	<del></del>		<del></del>	
DC Current (		1.01 20		<del></del>		
Spark Rate (	spm) /05	105				
		i i				
Section B4; AC Voltage (	v) sane	SAME			<del>-                                    </del>	<del></del>
AC Current ( DC Voltage (	(Ky)		ı	<del></del>	-	<del></del> -
DC Current (			<del></del>	<del></del>	<del></del>	<del></del>
Spark Rate (						
·		<del></del>	<del></del>		<del></del>	<del></del>

Scrubber	8/25/11	7/25/22					-
Fans: ID_Discharge Pressure (in, H2O)	1-0,4.	-0.4					Τ -
Fan B Outlet Pressure (in. H2O)	OUT	DUT				7	-
Fan C Outlet Pressure (in. H20)	14	14				:	!
Ean D Outlet Pressure (in. H2O)	12	12				1	1
Fan Amps	21	21					Τ -
Fan C Amps	22	22				l	<b>—</b> ·
Fan	24	23					
Stack Damper A, Pos. (% Opn)	-0	0					1
Stack Damper B, Pos. (% Opn)	0	0					
Pumps; Recirc. Pump <u>b</u> (amps)	13	13				<u> </u>	i
Recirc. Pump(amps)	16	16					<u> </u>
Recirc. Pump B; (amps)		12				<u> </u>	L
Recirc. Pump (amps)	12	12	i_		i	<u> </u>	
Recirc. Pump C. (amps)	14	14				<u> </u>	
Recirc. Pump C, (amps)	14	14				<u> </u>	1
Recirc. Pump Os (amps)	14	14			L		<u> </u>
Recirc. Pump Dr (amps)	/3	13			l		1
Recirc. Pump $\overline{D_4}$ (amps)	<u>' 13</u>	! /3					1
Reheater; Steam Flow (M lbs/hr)	60	60	•				
Steam Temp. (°F)		_   _   _				<u> </u>	!
Steam Pressure (psig)	1700:	1700					
Section_B ; Presat. Water Flow (gpm)	185'	190					1
Outlet_Gas_Temp,_(°F)	130	1/25					
Bed Diff. (in. H <sub>2</sub> 0)	9	18,8					
Demister Diff. (in. H <sub>2</sub> 0)	0.9	0,9					
RH Diff. (in. H <sub>2</sub> 0)	OUT :	UYT				Ī	
Gas Outlet Flow (in. H20)	360	360					
Section_C ; Presat Water Flow (qpm) _	175 '	175		Į.	•	1	
Outlet Gas Temp. (°F)	155	155			,	T	
Bed Diff. (inH <sub>2</sub> 0)	4.4	6.4				<b> </b>	
Demister_Diff. (in. H <sub>2</sub> 0)	0,4	0,4			<del></del>		;
RH Diff, (in. H <sub>2</sub> 0)	1.6	1.0				·	
Ges Outlet Flow (in. H.O)  RH SIM PR (PSILY)	480	480	i			1	1
Section_D; Presat. Water Flow (gpm)_	190	190		<del></del>		<del> </del> -	<del></del>
Outlet Gas Temp. (°F)	160	160					<del></del> -
Bed Diff. (in. H20)	OUT	UUT	<del></del>	<del></del> }		-	• -
Demister Diff. (in. H20)	2,2	2.7					·
RH Diff. (in. H20)	OUT	OUT			- <del></del>	-	·
Gas Outlet Flow (in. H20)					<del>-i</del>	<del> </del>	
RIL STM PR 126	1280	arn		Ĭ	1	1	i

<del></del>								·	
Date:	0/04	3/19/7	ا ـ 42 ك دا	191	36. *	2-1/2	-6-1		1
Time:	103%			,			8/28/77		
	7030	· -1	1739	1843	142,5	217	0971	1048	
Plant									
Mw. Gross	356	355	366	360	250	367	253	27/	
XS 02 (%)	4:0_	-2-1-	3.9	5.9	3.5	4:	_{1}P	3.8	
Steam Flow (1bs/hr) >103	2350	22201	2350	2400	3±0C_	2400	1450	1600	
Steam Temperature (°F)	2350	970	980	2350 975	3.57	2240 G/90	2350	2350	<del></del>
Opacity, Bypass (%)	25/20		35/15	32/16	40/27	13/5		1000	<del></del>
Opacity, Scrubber (%)	24	25	45	48	40	31	28/22_ 26	33/23	
			73	<del>'</del> A		31	-66	Ž6	<del></del>
ESP	1						1	1	İ
Section Al; AC_Voltage (v)	_ 530	230	230	530	336	225	335	235	
AC Current (a)	125-10		75	70	17.	135	105	100±10	<u> </u>
DC Voltage (Kv) DC Current (ma)		-20:0	001	201		_Qu:	-eut.	cut	
Spark Rate (spm)	>200		<u>300                                   </u>	290	(84.)		150	450-201	
opai k_rase_Yop/	7200	2200	30_	10	>200	7,200	200	7200	
Section A2: AC Voltage (v)	- Solution	Same	Same	غصود	انمنا	23.50	Same	Same '	
AC_Current (a)								,	
DC_Voltage (Kv)						<u> </u>			<del></del>
DC Current (ma)		<u> </u>						l	
Spark Rate_(spm)		!- <del></del> -				<b></b>		·	
Section A3; AC Voltage (v)		_ + .					_	1	
AC Current (a)	230	20-20		200 ± 20	-555	220_	190	200	
DC Voltage (Kv)	-22.55	50.75 50.72		85=10	9.5	90	90	100	<del></del> -}
DC Current (ma)		325-7	0,1			340	Azo	400	
Spark Rate (spm)	200 L	- <u>27.</u>	402F	100-10	UU.	204	OUT.	Out	
	<u> </u>	, - <del>,</del> -		ļ —		ļ	<u> </u>	-	<del>!</del>
Section A4; AC Voltage (v)	عنيمك'	1.50133	Same	Same	50 m;	- nee	Same	SEME	
		·	I						
DC_Voltage (Kv)			<u></u>			<u> </u>	<u> </u>	<u> </u>	
DC Current (ma)			<b>!</b>	ļ		<u> </u>	<u> </u>		
Spark Rate (spm)	!	<u> </u>	<u> </u>	1			<u> </u>	<u> </u>	
_Section_Bl; AC_Voltage_(v)	່ະລາ	285	270	270	75	240	305	305	1
AC Current (a)	140		135	135 70	10-	1.0=1.2	140=10	165=10	
DC Voltage (Kv)	1345	+ + + + + + + + + + + + + + + + + + + +	Out	DON.	35	ماد	out	D.A.	<del></del>
DC Current (ma)	780_	250	600	640±10	~22	540	850±50		
Spark Rate (spm)	. 134	180	185	185	185	152	185	185	
	<del></del>	<del>i</del> -	<del></del>	<del> </del>	! • .	<del>                                     </del>	<del>  _                                   </del>	<del>!                                    </del>	
_Section_B2;_AC Voltage (v)	Someth		<u>sam.</u>	SOME	رومه بعث	Same	Save	So≝6	
AC Current (a) DC Voltage (Kv)	·		l	ļ	<u></u>	ļ	<del> </del>		
DC Current (ma)	·	}	<del>}</del>	<del> </del>	<u></u>	<del> </del>	<del>}</del>	<del></del>	
Spark Rate (spm)		j	<del> </del>	<del> </del>	<del> </del>	<del> </del>	<del> </del>		<del></del>
	-j	<del> </del>	<u> </u>	<u> </u>	<u> </u>	<del> </del>	<u> </u>	<del>'</del>	
_Section_B3:_AC Voltage (v)	252:12	14:51	260	260	220	735	230	230	
AC Current (a)	95	3,57	120	130	. יים	100	90	<i>8</i> 0	
DC Voltage (Kv)	7 7	7:	33	33	37	.,,,	29	29	
DC Current (ma)	+70	1 410	610	600	535	250	400	410	
Spark Rate (spm)	35	93	103	105	135	105	105	103	l·
Section B4; AC Voltage (v)		-		Same		1	30	1	
	S. 250 4	Same	Some	1 ZaMe	Cash	30 Mr	30 00	<del></del>	
		!	i	1		•			
AC Current (a)			<b>}</b>	<u> </u>	<del>,</del>	•	<del> </del>	<del> </del>	
			) 	<del></del>		· · · · · · · · · · · · · · · · · · ·		!	

<sup>+</sup> Cobinet doors on At-12, 12-14, 83-24, open by portale that on very lot

\_ \* A-1 section is grounded are no-

Scrubber							<u> </u>		
Fans: ID Discharge Pressure (in. H2O)	5	5	-1.2	5	- 6	7.5	~.5	7.5	
' Fan B Outlet Pressure (in. H2O)	00+	سرو	Out	00+		201	OUT	at	_
Fan C Outlet Pressure (in. H2O)	21	27	125	12	150	12.5	85	9.5	_
Ean D Outlet Pressure (in. H2O)		: 70.5	/2	11.5		, - <del>,</del> -	7.0	7.5	
Fan <u>a</u> Amps	215	122-	202	205	1.1 "	7.10	230	230	
Fan C. Amps	235	237	230	220		3.7	220	23()	_
Fan	235	25.7	235	235	230	21. )	225	225	-
Stack Damper A, Pos. (% Opn)	0	ا ي ا	0	0	Q	()	- 8	0	
Stack Damper B, Pos. (% Opn)	0	ာ	U	0	O	ن	0	0	
Pumps: Recirc. Pump & (amps)	13	13	13	12.5	13	17	14	<i>1</i> 4	
Recirc. Pump 82 (amps)	14	11/	16-	ı.h	11	12	16	16	
Recirc. Pump 33 (amps)	13	125	125	12.5	12	12_	12.5	12.5	
Recirc. Pump CI (amps)	13.5	13.5	4.5	17.5	12	- 12	-LZ-E_	12.5	_
Recirc. Pump 62 (amps)	14	12	12.5	125	/4	14	13.5	13.5	-
Recirc. Pump cs (amps)	14.:5	4.5	14	14	14	14.5	13.5	/3.5	
Recirc. Pump Di (amps)	13	13	13	/3	14	14	14	13.5	
Recirc. Pump ਹੁੰਦ (amps)	13	. 13	,3	12.5	12	13	4	13.5.	
Recirc. Pump 03 (amps)	12	15	12.5	125	13	13	13.5	/3 0	-
Reheater; Steam Flow (M lbs/hr)	40	40	0	0	155	55	60	60	
Steam Temp. (°F)						_	-	-	
Steam Pressure (psig)	1,000	1050	0	0	T/ZO	1730	1730	730	
Section_B; Presat. Water Flow (gpm)	185	182	185	185	100	1947	185	18'5	. <u>.</u>
Outlet_Gas Temp(°F)	125	12.	115	120	140	140	120	130	
Bed Diff. (in. H <sub>2</sub> 0)	6.2	712	8.7	۶.7	6.5	8:6	5.4	3,9	_
Demister Diff. (in. H <sub>2</sub> O)	-0.8	( ( )	08	0.8	O(3)	0,3	0.6	0.7	-
RH Diff. (in. H <sub>2</sub> 0)	<u></u>	2-1	<u></u>		0_	1-a -	0.	ု ပ	
Steam Press Gas Outlet Flow (in. H20)	270	2.20	30	30	3,90	2:0	300	300	
Section_C; Presat Water Flow (gpm)	175_	175	_175_	175	17:-	175	175	;75	
Uutlet Gas Temp. (°F)	1.55	151	117	117	14: 3	150	160	160	
Bed Diff. (in. H <sub>2</sub> 0)	j4.6	14 .	13.2	91	14.5	14.	0	0	
Demister_Diff. (in. H <sub>2</sub> 0)	_2يه	0, 2_	0.3	0,3	ے در	2-	0.2	0.2	
RH Diff. (in. H <sub>2</sub> 0)	33C	_ /.3,	0.7	1.4	• :	7,3	43	16	
Gas Outlet Flow (in. H <sub>2</sub> 0)	33C	330	30	30	435	410	410	410	
Section_D_: Presat. Water Flow (gpm)	185	175	185	185	, Re	,o.t	1995	185	• •-
Outlet Gas Temp. (°F)	145	145	125	135	150		L75	125	
Bed Diff. (in. H20)	.6.4	á.T.	5.7	4.0	14.7	11.2	43	4.6	
Demister_Diff(in. H2O)	2.3	2.	7,6	2.7	3.1	2,7	_1.3	1.4	
RH.Diff. (in. H2O)	0.0	<u>ಲ.</u> ೨	0.0	S		7	O	0	
Gas Outlet Flow (in. H2O)	80	90	0	0	4:0	440	420	420	

## APPENDIX B

SUMMARY OF VISIBLE EMISSION OBSERVATIONS

CHEROKEE STATION

PUBLIC SERVICE COMPANY OF COLORADO

July - August, 1977

B-1

APPENDIX B.

Summary of Visible Emission Observations

Unit 1 and 2 Stack - Cherokee Station

Public Service Co. of Colorado

(continued)

Date (1977)	Time of Observation	Average Opacity (%)	Opacity Range (%)	Meter <sup>†</sup> Reading	Unit <sup>†</sup> Load (MH)	Comments <sup>†</sup>
8/25	0734-0740	29	20-40	18/16	111/100	coal/25% gas and 1Al recirc. pump out
8/28	1000-1006	30	20-45	24/12	105/83	coal/25% gas and
8/28	1014-1012	31	25-40	24/12	105/83	1Al recirc, pump out coal/25% qas and 1Al recirc, pump out
10/4 10/7	1410-1416 1000-1009	19 31	15-30 25-40	12/8 12/7	106/100 113/115	
10/11	1438-1447	34	30-50	12/12	115/110	
10/12	1330-1340	39	35-40	22/10	117/115	
10/13	1045-1054	29	20-40	13/11	107/78	40% bypass
10/14 10/18	1000-1009 0910-0919	11 29	10-15 25-35	14/7 12/12	105/100 108/108	-

<sup>†</sup> Unit 1/Unit 2.

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APPENDIX B

Summary of Visible Emission Observations
Unit 1 and 2 Stack - Cherokee Station
Public Service Co. of Colorado

Date (1977)	Time of Observation	Average Opacity (%)	Opacity Range (%)	Meter <sup>†</sup> Reading	Unit <sup>†</sup> Load (MY)	Comments <sup>†</sup>
7/27	1400-1406	10	10	0/15	116/113	gas/25% gas
7/29	1033-1039	22	10-35	0/17	116/111	gas/25% gas
7/30	1918-1924	5	5-10	5/13	109/91	coal/25% gas
8/1	1803-1809	26	15-45	7/12	118/104	coal 25% gas
8/4	1030-1036	27	20-30	10/10	113/111	coal/25% gas
8/4	1131-1137	32	25-40	10/10	113/111	coal/25% gas
8/4	1517-1523	20	20	13/13	113/111	coal/25% gas
8/6	1406-1412	21	15-25	10/0	99/75	coal/25% gas
8/8	1015-1021	23	20-30	7/12	118/108	coal/25% gas
8/8 8/9	1027-1033	23	15-35	7/12	118/108	coal/25% gas
8/9	1445-1451	16	5-30	8/10	117/108	coal/25% gas
8/12	1503-1509 1449-1455	22 14	15-40	8/10	117/108	coal/25% gas
8/12	1503-1509	11	5-30 5-40	9/5 0/5	116/102	gas/25% gas
8/13	1010-1016	29	5-40 20-40	9/5 9/5	116/102 112/99	
8/13	1025-1031	29	20-40	9/5	112/99	coal/25% gas coal/25% gas
8/15	1010-1017	14	10-20	15/13	116/112	coa1/25% gas
8/15	1025-1031	18	15-20	15/13	116/112	coal/25% gas
8/19	1030-1036	15	10-20	10/9	94/113	coa1/25% gas
8/19	1048-1054	6.	5-10	10/9	94/113	coal/25% gas
8/22	1000-1006	31	30-35	22/19	118/114	coal/25% gas and
				,	,	1Al recirc. pump out
€/22	1012-1018	3℃	30-35	22/19	118/114	coal/25% gas and
						1Al recirc, pump out
8/23	1000-1006	29	25-40	17/13	113/109	coal/25% gas and
0.400	1010 1010					1A1 recirc. pump out
8/23	1012-1018	32	30-40	17/13	113/109	coal/25% gas and
0/22	1002 1000	0.1	15 20	30/34	11-11-	IAI recirc. pump out
8/23	1803-1809	21	15-30	19/14	117/113	coal/25% gas and
8/23	1809-1815	20	15-30	19/14	117/113	IAI recirc. pump out
0/ 23	1003-1013	20	13-30	19/14	117/113	coal/25% gas and lAl recirc, pump out
8/24	1020-1026	30	25-35	19/15	112/99	coal/25% gas and
-,	1000 1000	•	23 ,3	13713	112/33	IAl recirc, pump out
8/24	1037-1043	36	30-45	19/15	112/99	coal/25% gas and
		-		,		1Al recirc. pump out
8/24	1509-1515	32	25-40	21/15	116/115	coal/25% gas and
					,	1A1 recirc. pump out
8/24	1550-1556	22	15-30	21/15	116/115	coal/25% gas and
					-	IAl recirc. pump out
8/25	0720-0726	30	20-40	18/16	111/100	coal/25% gas and
						1Al recirc, pump out

<sup>†</sup> Unit 1/Unit 2.

B-3 APPENDIX B Summary of Visible Emission Observations Unit 3 Stack - Cherokee Station Public Service Co. of Colorado

Date (1977)	Time of Observation	Average Opacity (%)	Opacity Range (%)	Meter Reading	Unit Load (MV)	Comments
8/9	1451-1457	17	10-25	0	136	Scrubber Out -
8/9	1509-1515	14	5-20	0	136	Scrubber Out - 75% Gas
8/12	1442-1448	46	40-70	24	130	Scrubber Out - 50% Gas
8/12	1510-1516	42	30-60	24	130	Scrubber Out - 50% Gas
8/13	1018-1024	46	40-60	19	129	Scrubber Out - 50% Gas
8/13	1033-1039	48	40-60	19	129	Scrubber Out -
8/15	1018-1024	40	35-45	22	132	Scrubber Out - 50% Gas
8/15	1032-1038	42	40-45	22	132	Scrubber Out -
8/19	1036-1042	24	15-35	15	140	50% Gas Scrubber Out - 50% Gas
8/19	1054-1100	21	10-30	15	140	Scrubber Out - 50% Gas
10/4	1016-1022	25	20-30	4	108	100% Coal 3B3 Recirc.
10/11	1029-1038	92	70-100	23	165	Pump Out 75% Coal 3B Booster Fan Out
10/12	1342-1351	57	55-60	10	103	50% Bypass 50% Coal
10/13	1054-1103	45	30-75	11	164	100% Bypass 50% Coal
10/14	1010-1019	13	10-15	5	155	50% Bypass 50% Coal
10/18	1009-1018	25	20-30	1	163	50% Bypass

B-4

APPENDIX B

Summary of Visible Emission Observations
Unit 4 Stack - Cherokee Station
Public Service Co. of Colorado

Date (1977)	Time of Observation	Average Opacity (%)	Opacity Range (%)	Meter Reading	Unit Load (MW)	Comments
7/27	1406-1411	15	15	20	356	
7/29	1020-1026	17	10-30	16	358	
7/30	1924-1930	8	5-20	21	343	
8/1	1809-1815	30	29-40	28	314	
8/4	1037-1043	19	15-20	26	355	
8/4	1140-1146	24	20-30	26	355	
8/4	1523-1528	20	20	22	355	
8/6	1414-1420	18	10-25	22	348	
8/8	1021-1027	16	15-25	22	353	
8/8	1033-1039	17	15-20	22	353	
8/9	1457-1503	9	5-20	19	355	
8/9	1515-1521	11	5-20	19	355	
8/15	1010-1017	6	5-10	22	270	Low reheat stm flow
8/19	1042-1048	6	5-10	24	355	Low reheat stm flow
8/19	1100-1106	5	5-10	24	355	Low reheat stm flow
8/22	1006-1012	24	20-30	30	358	
8/22	1018-1024	23	20-30	30	358	
8/23	1006-1012	18	15-25	40	360	No reheat stm
8/23	1018-1024	22	20-25	40	360	No rehea. stm
8/24	1028-1034	11	5-15	31	362	1 ESP section out
8/24	1044-1050	8	5-10	31	362	1 ESP section out
8/24	1502-1508	8 5 5	5	27	355	1 ESP section out
8/24	1516-1522	5	5-10	27	355	1 ESP section out
8/25	0727-0733	27	20-35	28	362	1 ESP section out
8/25	0741-0747	26	20-35	28	362	1 ESP section out
8/28	1007-1013	17	10-25	26	262	
8/28	1021-1027	19	10-25	26	262	
10/4	1022-1028	10	5-15	18	190	
10/11	1420-1429	24	20-35	26	250	
10/12	1352-1401	25	25	24	241	
10/13	1103-1112	11	10-15	24	246	
10/14	1019-1027	13	10-20	21	230	11
10/18	1028-1037	36	0-60	-	-	Unit start-up

#### APPENDIX C

ELECTROSTATIC PRECIPITATOR DATA
AND
CALCULATIONS
CHEROKEE STATION
PUBLIC SERVICE COMPANY OF COLORADO
July - August, 1977

DATE	T-R SET	PRIMARY VULTAGE	PRIMARY CURRENT	POWER	SECOMPARY VOLTAÇE	SECONDARY CURCEUT	POWER	EPAZK Ratū	POWER EFF.C.ENEY		COMMENTS
(1977)	1111	(v)	(A)	(RW)	( k V )	(MA)	(hw)	(SPM)	2 1 2 2 2	MA/FE mk/m2	╺┨╶╅╼┦╴┞╌╏╼┞┄┦╶╏╼┨╼╁╍╋╼╞╾╧╼╴│
7/27	A1-A2 B +B2	160	53	4.5 4.5	33	180	5.9	0	62	7.54 081	10273 15115
	_  _	-   _   _   _									
729	M-AZ BJ-62	155	50	9.0	33	1/60	53	0	59	6 49 .059	الانتصال كالكاكا كالكاكات كالمان المراجد بغراها ا
				281.5		14/9	<u> -  -  -  -  -  -  -  -  -  -  -  -  -  </u>				
7/30	BV-82	150	60	12.2	<u> </u>	170	15.15	7500	76	7.20 .00:	
	OV OF									<u> </u>	
8/1	B1-82	180	35	9, 7 5.8	37.4	115	1.7	325 280	14	3 99 015	LOW SEC. CLICENT
							-   <del>-   -   -   -   -   -   -   -   -</del>				
8/3	81-82	180	35	9,0		1.25	4,5	250	<b>5</b> 0	4.29 04 3.77 M	
							-				
8/2	A1-A2 B1-82	175	50	9.5	35	140	4.3	390	51	9,80 05:	
	-   -   -   -					Ĭ Ĭ Ĭ	9.2	- - -			
8/6_	A1-12 B1-82	190	25	5.3	38	10	7. 2	300	29	3.77 .04/	LOW SEC. CHARENT
	: <u> </u> - -	11! T						╏╌┧╌╏╼┟╼┨╶			
8/8	A1.A2 B1.62	155	40	7.1	32	95	3 0	195	4.2	3 26 030	
	;_ <del> </del> _ _						29		<del>- - - -</del>  -		
8/9	A1-A2 B1-62	178	35	5.5	36	9 4	2,9	230	47	3,26,03	
8/10		200		6.6	<b>F</b> o		14.8	╟┼╁╁	<b> - </b>	2.12 04	
	81-62	170	33	61		120	<u>                                     </u>	335		3 43 1 1 037	<b>/</b>
1 2 3 4 3	7 10 0 10 1	112 12 14 13 14	17 16 18 46 21 22 Gra 1888 0 - 179-545	23 24 75 26 27	28   28   30   31   32   33	74 35 16 37 14 34	40 41 42 43 44	-5 40 47 40 45	50 21 52 52 4	25 56 57 26 25 20 81 62 07 64 6	5 fee fer fee fee 176 fee 172 fee 72

SUMMARY OF UNIT I ESP OPERATING DATA - JULY AUGUST 1977 CHEROKEE STATION PUBLIC SEEVICE CO. OF COLURADO

1/2

DATE	T-R SET	PRIMARY VOLTAGE	PRIMARY CURRENT	POWER	SECONDARY VOLTAGE	SECONOARY	POWER	SPARK RATE	POWER EFFICIENCY		177	COMMENTS
(1977)	1	(v)	(A)	(RW)	(kv)	(~^)	(80)	(SPM)	(70)	NA/FLZ	m4/n.2	والمراقع والمراق المراقع والمراقع والمراقع المراقع والمراقع والمراقع والمراقع والمراقع والمراقع والمراقع والمراقع
	1 7 1 7 16	· <del>┃─╽┈</del> ┟╺ <del>┃</del> ─┦──╂┈	10 17 10 19 20 21 7	2 23 24 25 26 27	_	·····································	╌╽╴╬╾╽╶┝╼╬╼┆	╌╊═┪╶┤╌┊╶╬╼╴	39 51 52 53 44	╸╏╼┼╼╌┟═┝╼┼╼┼═┤	<del>┩┈</del> ┦╌╶ <del>╏┈╏┈</del> ┦╌╌╎	<b>╶┟┈╂┈╡╌┼┈╁╌┠┈┟╶┼┈┼┈┼┈┼┈┼╌┼╌┼╌</b>
8/12	191-92	11/95	28	155	4/	80	33	2/0	69	2.74	1,930	<del></del>
1444	81-32	1/70-	30	15.1		<u> </u>	-\- <del>-</del>	150	<b></b>	12.23	1,029	1-
	<del></del>		<b>-┃-</b> ┃-┃-┃-	1-1-1-1-1	11111	1-1-1-1-1-		1-1-1-1	1-1-1-1-	· <b>[- - - </b> -	<del></del>	1-1-1-1-1-1-1-1
8/13	41-12	190	1 28	13.3	40	75	1 3,0	200	1 57	7.57	1028	-!
14:14	81.52	11/70	1 30	1-5-1-	1	65		175	<u> </u>	23	1021	
	<b>!</b>		┧╁╁╁┼	1111	1-1-1-1-1		1414	1-1-1-1-			+++++	PNXS
8/15	AI-AZ	1 1 17 54	35	4.1	38	10	3.0	295	50	2.74	1.030	
	81-82	1/69	1 35	5.8	<del>┃</del> ┧ <del>╽</del> ╪╅┼	170-	1-1-1-1-1	200	-	2.40	1.026	4
1-1-1-1-1				-	- - - - - -	- -}- - -	-1-1-1-1-		.   -   -   -   -   -		1111	
8/19	A1-A2	200	40	8.0	40	85	1.1.1/_	275	143	2.91	1.03/	1
	41-82	178	35	412		11/29	<u>-</u> ┃ _┣=┃	395	·1-1-1-	4.12	.099	1111111111
			<u> </u>		11111	1111-	1111	1111.	1-1-1-1-		1111	1-
8/22	41-42	180	35	6.3		190	372	270		4.80	052	
	81-02	160		1.6		_ _ _  3    .	1-1-1-		┨┼┼╪┼	1.03	1.01/	
	<u> </u>			1-1-1-1-				.	1 111	1-1-1-1-1-	11111	
8/23	ALAZ	180	40	7.2	37	105	3.9	720	59	3.60	-039	6.) - 4.—— <del>——————————————————————————————————</del>
(AM)	31-82	160		2.8		20	. _ _	1/20	<b></b>	0.69	1.007	2 LOW CURRENT
					_		11111	11111		┨┹╁╁┼	111+1	
8/23	A1-A2	108	45	7.6	35	3P.D 45	38	7500	37	2.77	1030	
(PM)	B1-B2	160	425	3,2		75		180		1.57	.012	
							.   _   _   _   _		1+++	-1-1-1-1-	11111	<u> </u>
8/24	AI-AZ	180	45	81	36	1/35	4,9	450	60	4.63	1050	-
	81-82	160	23	3.7		50		280	11111	1.71	9/8	<b>?</b>
							11111		-   -   -   -   -   -   -   -   -   -	-1.1-1-1-1-	1111:	1:11-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-
8/25	AI-AZ	175	70	70	34	140	1 58	400	68	4.80	1 .052	
	81-82	120	10	111.17		35		120	11+11	1,20		3 SPHIRIKING
											11111	
8/26	AL-AZ	108	40 225 11 17 11 12 18 11 11	6.7	33	85	2.8	400	42	2.91	1 031	9 NO SPARK
	BI BZ	185	225	B 3		11115				0.86	1009	INO SIPARK
1/2/3/4/3	2 7 0 0 16		16 17 19 10 20 11 7	/2 23 24 25 26 2	.7 28 29 30 31 32 3	23 24 35 2L 37 34 36	35 60 41 42 43 44	. 3 0 0 0 0	18 50 51 32 53 54	s 55 50 57 50 59 60	0 61 62 67 64 65	25 00 07 00 20 70 71 72 72 74 72 74
8/28	2. 2.		500 1001 0-111 101	7,2					30	2.05	,022	2 , []
1/28	A1-A2		40		36	60	2.2	410				2/2
	81-82	165	25	4.1	_	40	-	350		2.05	.022	D-

	1								1 1	CUZRENT	
PATE	T-R	PRIMARY VOLTAGE	PRIMARY CURRENT	POWER	SECONMRY VOLTAGE	SECONDARY	POWER	SPARK Rate	POWER EFFICIENCY	_	COMMENTS
(1477)	SET	(v)	(A)	(RW)	(kV)	(mn)	(hw)	(SPM)	(%)	MA/fer ma/	,, >
1,,,,	. ,		17 16 19 20 21 .		70 [9 30 31 32 35	14 * 36 ,* ,8 39	40 41 4 41 44	45 45 47 49 49	الديا حطينا	1 16 12 1 29 60 91 97 97 67	C 61 12 62 63 64 76 71 71 73 74 75 75 71 71
7/27	Ar h	200	20	*0	22	25	1/1/	0/150	28	2.25	24 75% COAL
	1         9	245	50	10.3	76	300	15.3	45	1 1-4		62
	13-44 13/62	1776		7.3	27	Jas	2.3	4		6. 38 .00	
	83-8-1	£ 20	75	21.8	32	250	20.4	9/0			05
	01-62	1996	27	19. 2	007			06	1 + + 1		
	03-04	253	92	2₹. 3	38	2:50	الا ارد	12	90	·	55
2/29	Al-AZ	178	18	٤.2	23	39	0.9	7201/20	28	1.90	20
	A1-A2 A3-14	290	60	17.9	23	1 20 1 20	12.2	0	70	14.5	56
	31.62	193	22	7.2	200	35	1.4	00	53	14.5	15
	E3-EX	25.7	193	26.5	37	13,6	10.0		73 25		32
	01-12	238	37	1017				0/0	<b> </b>	9.0	
	03-CA	248	92	2.2.8	36		18. 7	14	इंड	39.4 .4	29
											<u> </u>
7/30	41-42	325	=8	18.9	36	178	7/	0/5	38	9,9 1	
	A3 14	278	1 4/	182	43	205	13.1		22	15.3	6 P
	61-02	TRIPPE	اطياه			<u> </u>	<u> </u>				<u> </u>
	33-67	1300	52	27.6	39	198	4/2	0 0	27		40
	41.05	3/2	56	125	-  -  -	198				12.9	50 1 1 1
	C3-C4	258	1921	125 23.7	39	1 525	20.5	1/0/	1 26		729
	!								1 : i		<u> </u>
8/1	AIFAIA	273	18	3.1/	29	35	7.2	7201/10	26	175 0	19 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	A3-A4	190	48	17.5		1/4/4.	7.2	1 1 4 9	63 - <b>5</b> 8	9.51	2
	31 - 52	265	10	2.6	36	. 43	1,5	0/0	58		35 - 1 - 1 - 1 - 1 - 1
	53-BV	220	64	15.0		325.	11.0	0	73	25.9	79
	Ç1- C	1/8/18/		11.1		1/20	<u> -  </u>	0	+		9.7
	C3-C4	255	1 1 4 2 [	23.5	38	550	20.9	10	89		25
				<u> </u>							
						-			<u> </u>		
1::1,   •   •	6 12 8 8 1			23 24 25 16 27	26 26 30 31 42 34	24 25 26 27 18 16	45 41 42 43 44	45 46 47 48 44	30 51 32 53 54	15 56 57 4 53 60 81 62 87 9	cles   e   es   e   c   20   20   22   22   24   2   24   2   22   2
88 6 131		•	CFy 1808 0 110 345								•

SUMMARY OF UNIT 2 ESP OPERATING DAYA JULY - AUGUST 1977 CHEROKEE STATION PUBLIC SERVICE CO. OF COLUNADO

DATE (1977)	T-R SET	PRIMARY VOLTAGE (V)	PRIMARY CURRENT (A)	POWER (RW)	SECONIMEV VOLTAGE (kV)	SECONOMRY CURRENT (MA)	POWER	SPARK RATE (SPM)	POWER EFFICIENCY (70)	CURRENT DENSITY MA/ft <sup>2</sup> MA/m <sup>2</sup>	Comments
أأمان		11 12 13 14 15 16	17 11 19 20 21 7	13 24 25 24 27	28 29 30 31 32 34	<del></del>	10 41 42 43 44	15 10 17 10 10	20 31 32 53 34	2 -9 57 55 39 60 61 62 62 64	
<del>┣</del> <del>┤</del> ╼┼╼╁╼┼╌	┡ ┡ <del>╌</del> ┡╌╏╼╏╼╏	350	╏╶┝╍┠╺┦╺╄╍┦┄	<del>   </del>	┨┪╢┪┪╏	┃╸┃╸┃╺┝╍┢╌┢╸	╏╌╆╾╏╼╞╼╬╴┆	10/8	37	╽╺┞╼╌ <del>╞╌</del> ╽ <del>╍</del> ┥╍╅╼╅╌╌┇╴ <del>┪╼╬╸</del> ┩	╶┨╌┝═┪╌┽╼╋╾┞╌╅╺╅╼╁╼╂╌╁╼╁╾┿═╴│
187	14.15		56	18.8		150		╽╼┼╌┼╧┞╧╌┞╼	1-3/	▎▗▕ <del>▃</del> ▗▎ <del>▃▍▞</del> ▍ <del>▃</del> ▎ <del>▃</del> ▎ <del>▀</del> ▍▀▍▀▍▀▍▀▍▀	
<b></b>	12-61	250		1818	-  ===	1.70	9.5		69	12.5 1/3	<del>┋</del> ╂┼┥┼╂╂╂╂╂╂╂╫┈
<b> </b>	B1-87	2/4	47.	14.8		1.1 大学中1.	5.6	1-4-	38	▎ <del>᠃</del> ╏ <del>╸╚╸┖╞═</del> ┟ <del>╺╁╼┆═</del> ┝	<del>╶┨┼┼┼┼┼┼┼┼┼┼</del>
	B= -84	_ 2 20	55 47 23 23 93	229	37 37 35	420 150 545	14.7		66	31.5 .34	<del>}</del>
	C1 - C2 C3 - C4	310	43	13.3	┨ <del>┇</del> ┇╬╬	[_] <b>/</b> _\$_\$	<del>┇</del> ┩	9/0	88		<del>/ - - - - - - -</del> -
	C3-64	<u>  273  </u>	93	1 5 7			22 3	1/9/	88	43.9 99	
			<u> </u>	<u> </u>		<u> </u>		<u> </u>			
8/4	A1-41-	340	30	10.2	42	90	3.8	35/9	37	450 ,04	<u> </u>
	A3-A7	260	5 5 7-7	14,3	71	235	4.6	0	67	11 8 12	<u> </u>
	01-82	340	72	15.0		1 130	4.9	0	33	9.76 10	<b>;                                    </b>
	83-64	340	73	12.5	38	270	10.5	0	60	9.76	s Till Till Till Till Till Till Till Til
	83-64 C1-C3 C3-C4	340	57	19.1	]   [ <del>  -</del>	240 /90 540		0		25 5 7	2
<del>                                    </del>	C3 - C4	1 2 70	93	123,1		500	21.6	10	86	10.5 .43	2
			<b>!</b>		1- - - -						
8/6	Al-AZ	350		1115	12	1/24	6.9	25/20	92	8 26 .09	7
<b>                                      </b>	A1 A2		58	16.0	40	165	6.9	a	7/	8 26 .09	
	31-02	3/5	48	5			4	0/0	37	11.3	,
- <del> - -</del>  - -		390		151	36	150 520	20.0		25	39.01 . 22	
	03 04 C/ - C2	320	91 52	18 2	1-+	-00				15.0 .16	2
1::11	c3 · c4	255	1 92	182	37	5 30	19.6	10/0	83	39.0 . 92	
<del>                                      </del>	1577	- - - - -	<b> </b> <del>                                    </del>		<b>[- + * </b> *		1-14-6	1-1-1-1			
8/8		1	27	7,2	37		4	35/10	33	3.25 0.03	el
-9/6	AIAZ	265	49	10.3	37	45 160	54		33	8.00 0.08	71
1-	1/: 5  - -	210	28	10,13		[+-[]]	1 27.7		1 20	6.00 .07	
<b> -++</b>	81-83	485		10,3 8.0 16.4	36	- - - - -	5,9		- 70		<b>5  + +   + + + + + + + + + + + + + + + +</b>
- <del> - - - -</del>	83-87	225	73	199	<u> </u>	350		1 <del>-:</del> <del></del> 1	60	14.6 .15	<b>3 </b>
<b> </b>	01- 12 C3- C4		<u>_</u>	181	<b> - - - - - -</b>  -	- -  <i>/</i>  7 <del>.</del> 2-	1+15+		40 68 7/		
1+1-1-	1631 CA	1 2 10		15.8	- -  <del> 3</del>  2 -	350	1/1-2	- -  <i> 1</i>   -	- - 4/- -	26.3	<b>╸</b>
1-1-1-	╏┤╾┾┼┥┄	<b>╏</b> ╼┧╼╁╌┼╌┼╌	╏ <del>╶</del> ┼╌┤╌┼╌┤╴	╏╍╂╾┼╍╂╌┞╌	┨╼┼╼┼╼╂╼┼═├╴	┋═┤╺╣═┼═┤╶╏╸┆	-	<b>1</b> -	<del>┠</del> ╶┟╍┢╍┠╼┨╶╏	┆ <del>╎╸</del> ╎╸├ <del>╸</del> ┟╼╂╼╂╸┨╸╏	╸┫╶╂╌╂╼┠╌╆╼┠╸╠╌╣╾╌╌┆╌╂╸╬╌┩╺┈│
<b>  </b>	╏-┼╌┼╌╎╴	- - - -	╏╌┼╌┤╶┤╌┞╼┼╴	┨╌┠╌┟═╂═┞═	<del>┨</del> ╾╁╌╂╼╂╾┼╴┞╌	╏╼╞╼┥╸┽╸╁╶╽╺╽	<b>│</b> -├-├- ├-┽-	<b> </b>	<b>{┼┼- ├-├-</b> ┆	- - <del> - - - - - </del> - - - - -	
11:00		[][][]	[1.   10   16   20   21   41	1,3   5 -   5 3   5 6   5 7	20 25 30 31 32 31	30 30 37 30 36	10 11 12 13 14	12 46 5 40 10	30 51 32 53 34	35 38 57 58 59 60 61 67 43 64	15 14 67 69 64 70 71 72 77 74 77 6

2/5

F40 (ME0-11) 414

PATE (1977)	T-R SET	PRIMARY VULTAGE (V)	PRIMARY CURRENT (A)	POWER (RW)	SECONDARV VOLTAGE (KV)	SECONOMRY CURRENT (MM)	Power (hw)	SPARK RATE (SPM)	POWER EFFICIENCY (70)	CURRENT DENSITY MA/ft2 ma	COMMENTS
1 1 1 4 5	1 7 1 1 10	11 12 13 14 13 16	17 18 19 20 21 2	23 24 25 26 27	26 26 70 31 32 3.	14 5 36 12 48 16	40 4 42 43 44	45 49 47 48 49	,	15 50 57 59 59 80 61 62 63	64 04 62 67 62 69 70 71 71 73 74 75 76 77
8/9	ALAZ	270	38	8.9	37	100	\$2	35/10	72	5,0 -0	54
	A3-A9		sta	12.8		174	3.7		125	8.76	19-2
	11-64	293	36	10.5	34	1/20	4.2	0	75	1 <del>- 7 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 </del>	97
	33-04	220	70	15.9	3/	370	11.5	0	79		99
	61-62	330	37	18.8	++1/+-	220					62
	C3 - C4	205	73	15.0	133	360	11.9	0	79	127.0	29/
	<del>- [ ] - [ ] - [</del>	+////	╏┼┼╀┦┼╎				<del>                                    </del>	<del>╒╒┋</del>			<u> </u>
2/10	A1-A2	7 7 7 2 1 2	40		++++++	205	7.6	2/3-	1 22	17013 11111	
8/10	A1. A2	333 250	59	13.5	37		7.7	0	38		62
╏═╪═╃╒╌╻┼═╏		280	47	13. 2	134	300 /50	-   -   -   -	0	33		2/1
┝┼╌┼┼┤╢	83 87	220	75	16.5	32	- -  <u>- - - </u>	12,0		73	22.2	03
<del>▎</del> ▎ ▎	61.62	250	36	15.7	++ <b>*</b> *	375 200 540		0		1 7	16 2
┠╌┼╌┼╌╂╌╏	C3 - V4	260	92	39	1	540	30,5	16	86		4.37
╏╅╁╅╂╏	7771	+177+	<b> - - -1-5- </b> -	1-1-1-1-I	-				92		
8/12	AI-AL	340	55	18-1		175	6.8	12/10	36	876	94
	73-11	190	50	9.5	1 39		5.9	0		9.0 .0	97
	41 82			186	1	180	8.7	a	57		154
<del>╏╸┼┈┆╸┤╸╏</del>	· !	320	73	15.3			10.8	0	70		3,7
┠┼┼┼┼	33-84 C1-K2	300	56	17:17		385				17.6	150
<del>╒┋┋</del>	C1-K2	200	75	16 8	31	195	22.1	0	8/		3/5
<del>┠┋</del> ┼┼┼┼╏	7-7-1		╽┼┼╀╩┼╎	150	┤┼╬┼╎╴	329	144	- - <del> </del> '4+-		#### <del>                                </del>	<del>[</del>
1 7/3		220	23	5.1	+	+ + -   -		7200/50	22	2.75	23
8/13	A1-A1	185	) <del></del>	8.9	30 35	43 105 90	- 7,4	3 _ i _ i _ i _ i _ i _ i _ i _ i		/ <del></del>	89
<del>╏┋┋</del> ┼┼┼╏	1 1 1 1 1	200	48	8.4	++11+		5.0	0	55	2.76	273
<del>╒</del> <del>╒</del> <del>╒</del>	81-82	300 2/0 335		100	27	10	10.8		68		2 3
<del>Ĭ</del>	B3 -84	1 2 2 -	75	15.8		190	-   -   -			14,3	1512
	C1-E2 C3-C4	200	++11-	150	- <del> -  - - - - -       -</del>	395		10	82	1 2 3 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	2,9
<del></del> <del> </del> -	03-64		<u>- - - </u> - -	+4344-1		-   -   5-  7-  -	12.2	<u> </u>		29.7	<b>?'                                    </b>
┝┽╌╬╌╂	╶┼╾┼╾┼╌┨		╏╍┽╾┤╾╎╸┼╌┼╌		<del>╶</del> ┼╌┼╶┼╌┦╶	<del>╶</del> ┾╌┤╌╏	╎╌┼╌┼╌┤╾┼╌	┃ <del>╶</del> ┠╾┦ <b>╼</b> ┠╸┃	<del>╎</del> ┼┼┼┼	<b>╿╶╎╴├</b> ╌┼╼ <del>├</del> ╌┼╌	╂╌┠ <b>╏</b> ╶╂╌╏╌╏
- <del> </del> - - - -	.  -  -  -  -	, , , , , , , , , , , , , , , , , , , ,	17 18 19 76 21 7.	2) 14 25 26 77	78 79 30 31 32 43	3 3 3 3 3 3 3 4 4 6	40 41 42 43 44	15 46 47 48 49	50 51 52 53 54	54 50 57 58 59 60 01 62 43	00 10 10 10 10 10 10 10 10 10 10 10 10 1

4-151 500 10019-111

3/5

DATE (1977)	T-R SET	PRIMARY VOLTAGÉ (V)	PR.MARY CURRENT (A)	POWER (RW)	SECONDARY VOLTAGE (RV)	SECONDARY CURRENT (MA)	POWER (hw)	SPARK RATE (SPM)	POWER EFFICIENCY (70)	CURRENT DENSITY MA/ft m4/nit	Commentso
11111	1 7 1 1 15	11 12 13 14 13 14	17 18 19 20 21 2	2 23 24 25 26 27				45 46 47 48 49	20 21 22 74	5 '6 57 56 59 60 6 62 63 64 6-	- 61 67 63 64 70 7 71 13 14 11 15 17 71
8/15	A1-A2	300	58	17.4	34	195 300 155 540	4.6	0/5	38	7.76 -105	
		233		13.5	T AE	1 300	10.2	13	745	1 /51.6       1   1 /6 2	
	19. A4 181-82	233	55	13.8	27		12.2		1 20		4
	83-54	250	91	228	33	590	10.8	0	78	1937	,
	c/-C2	270	56	13.4	11171	11/1/21		Ha/aT	1 + 1 - 1 - 1	14.8 155	
	C3-C4	230	92	12/2	34	520	122	172	83	89.0 1.120	
8/19	ALAZ	1 305	50	153			56	0/5	1 3 7	8.0 .086	
	A3-A7	305	50	15.4	1 79	300	11カブ		76	150 1/62	
		200	177	17.1	25 29 25	11/46	5.6	0/0	3/	15 0 .162 10 5 .113 39.4 .21 14.5 .156	
<u> </u>	1 - 82	253	9/	130	11133	420	17.3	6	75	10 5 1/3	
	63-82 C1 C2 C3-C4	190	136	134		140 525 193		0/0		14.5	
<u> </u>	171-17	-1,5	92	19.8	32	520	16.6	10	87	39.0 126	
17111	1771	1	1-1-1-1-T-1-								
8/22	Al-AL	333		19.0	34	185	7.9	9/5	39	9.76 105	
	الممادادنا	333	\$8	18.7	1	300			1 75	9.76 105	
<u>                                     </u>	1-64	255	50	12.8	27	150	4.1	1161	1 35	1 <del>-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-</del>	
<u> </u>	e3-09	275	1	125.0	36	150	18.7	1161	1 76	39.4 924	
	C1-E2	290	·   -   -   -   -   -   -   -   -   -	13.9	1	195		0	32 76	14.6 .158	
<u> </u>		220	54	20.0	32	175	16.6	40	23	39.0 ,420	
<del>┣┋</del> ╬	43-67	1-1-1-1-	1-1-7-1-1				1-1-1-1-1-1	1-14-1-			
8/23	1-1-1-1	370	58	1	1-1-1-1-1	200		3/5	4	10.0	
9/67	A1-A2			12.6	43	160	5.8		40 51 35	1 - 6 1 1 1 1 1 1	
\ <del>\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\</del>	A3 - A4	360	48	1-17	33	190			1 2	12.3	
<del> </del>	A3.A4 B1-U2 B3-U4	230	70	16.1		1-1-3-6	9.0	Ila	56	8 0 086 14.3 159 21.8 .339	
┠╬╌┼╌┼╌┼	01-Cz			140	1++#1+	190	1-1-2-1-	1 d	56	14.3 13	
<del></del>	C3 - C4	250	1 56	29.2	· [	530	20.1	10	53	39.8	
<u> </u>	(S) 7(4)	260	<del>                                      </del>	7 3		1-1-1-1-1-1-1		1-1-1-	- - - - -	1 77-1-1-1 7-7-90	<u>                                     </u>
<b></b> -	<b>\  \-\-\-\-\-\-\-\-\-\-\-\-\-\-\-\-\-\-\</b>	1-1-1-1-1-	·IIIII	1-1-1-1-	╽╼┼╍┾╼╞╼┊╌┤╌	1-1-1-1-1-	· <b>!</b> · <del>  -   -   -  </del>	┨╾┤╌┤╾┼╌┼╌	╢╌┼╌┼╌┞╌┼╌	┨ <del>┤</del> ┼┼┼┼	·┨┤╌┧╌┠╌╬╌╏╌╏╌╏╌┇╌╏╌┪╌┋╌╗
	1.+,+,+,-,-,-	1 12 13 14 15 16	6 17 16 19 20 21 7.	. 23 24 25 26 27	7 18 23 30 31 32 33	3 34 35 36 37 36 39	0 40 41 42 43 44		50 51 52 53 54	55 56 57 48 59 60 61 62 63 64 65	5 00 67 60 60 79 71 72 72 72 74 74 76 7
0.00	1.1.1.1.1		Cr0 1960 0 - 121 161	<u> </u>	<del>                                      </del>	1-1-1-1-		1.1.1.1		<u> </u>	<del></del>

PATE (1977)	T-R SET	PRIMARY VILTAGE (V)	PRIMARY CURRENT (A)	POWER (RW)	SECONDARY VOLTAGE (KV)	SECONDARY CURRENT (MA)	POWER	SPAZK RATE (SPM)	POWER EFFICIENCY (70)	CURRENT DENSITY MA/ft mA/n	COMMIENTS
11111	4 7 4 9 75	17 12 13 14 15 16	17 18 19 70 21 7	<del></del>	28 29 10 31 32 3.	14 4 25 17 .8 35	40 41 42 43 44	45 46 47 48 49	<del></del>	. *6 57 29 35 60 6 67 63 64	
8/24	A1-A2	3.45	157	19.7	38		17.5	0/-	37	951 .10	
	43.44		57	13.0	39	1 2 2	7.2	66	64.	13.3	
	81-52	280	58	16.5	1 3/	265	1.5		35	13.9	
	83.69	270	86	23, 2	35			6	1 33	30, 2 38	
<u> </u>	C1-C2	14/9	56	23,2	<del></del>	470	4.5		1+14+	-#- <del> - - - - - - </del>	9
	C3-C4	235	9/	21.4	35	190	18.0	T <sub>i</sub> a T	89		6
		<del>    43 2</del>  -	┠ <del>┤┤╇</del> ┦┼┤	<del>                                      </del>	<del>                                     </del>	<u> </u>	1-19-14	╽┼╂┸┼	1-1-7-1-	17944	
8/25	41-42	365	56	20.4	40	193	7.7	4/5		9.64	<del>╽</del> ╌╸┫╌╍┋╌┈┠╌╌╂┈╌╂╌╌╏╌╌╏╌╌╏╌╌╏╌╌
	A1-A2 AB-A4	295	35	16.2	43	260	1.2	la	38	130 11	
<u> </u>	81-82	300	59	12.7	32	185	5.9	0	3:3	13.9 . 15	•
	63-87	275	85	23.4		775	18.1		77	35.7	
		72	56	13 7	30	195				35.7 35	
	C1-C2 C3-C4	245	924	ورد	36	052	1191	10	87	34.8	8
		+									
8/26	AV-AR	7/5		23 2	95	200	20	2/5	39	10.0 .10	8
	A3-A0	240	56	/2.0	38	190	7.2	6	39	4.51 -10	
		320	59 70 55	23 2 /2.0 /8.9	36	190	6.8	0	1 3 1	14.3	
	81-81 81-81	235		16.5	122	320	12.2		62	25, 0 , 25 14, 3 , 15 39, 8 , 4	9
	7 52	260	विद्वा	16.5		320 190			17-11	14.3	7
	C1 - C2	265	93	29.6	39	530	20.7	10	84	14.3 15	8
8/28	ALAZ	270	3/	8.4	34	90	3,1	0/5	36	9.5 .04	8
	A1 A2 A3 A9 B1 B2 B3 B9	278	56	15.6	34 40 29	310	12.9		79	13,5 16	2
	B1 82	274	47	12,9	29	150	10.9	o	34	1/3 1/2	
	83 89	275	93	26.8	38	550	20.9	0	78	41.3 44	5
	C/ C2	230	57	/3,/		195			1 1 1 1 1	14.6 15	8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	c3 C4	235	924	21.6	35	550 795 530	18.6	12	84	39.8 42	8
		1   1									
	<u> </u>	<u> </u>									
111111			17 18 19 26 21 22	23 24 25 26 27	28 29 36 31 32 3,	14 25 36 27 20 23	40 41 42 43 44	.5 44 47 48 49	JO 51 52 5J 54	15 56 57 58 58 83 61 62 63 64	es   e   e   e   e   e   e   e   e   e

6 131 CP6 1M1 0 - M7

LATE T-R PRIMARY PRIMARY SECONDARY SECONDARY POWER RATE EFFICIENTY DENSITY  (1917) SET VOLTAGE CURRENT POWER (RV) (MA) (AW) (SPM) (76) MA/FL MA/N."	COMMENTS
7/27 41-42 50 90 4.5 3 3 5/0 -4.1 0 9/2 3/3 3/3 3/3 3/3 3/3 3/3 3/3 3/3 3/3 3	2, cone
7/29 41-42 50 92 11 8 5/0 1 90	
7/30 81-82 50 92 8 7/57	
B1-5-1-10-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1	
1-1-1-1-61-82 - 120-1-120-1-1-1-1-1-1-40-1-1-600-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-	
8/5   4/142   59   95   1   1   1   1   1   1   1   1   1	
81.62 V20 110 1111 91 120 111 P 111 P 111 P	645
8/9 A/1 A2 220 AS /A/3 38 280 10.6 0 74	2 COAL
5/10 A1-A2 235 130 30.6 40 710 28 4 0 93 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	

SUMMARY OF UNIT 3 ESP CHEROKEE STATION PUBLIC CRUICE CO, OF COLORADO

DATE (1977)	T-R SET	PRIMARY VOLTAGE (V)	PRIMARY CURKENT (A)	POWER (RW)	SECONARV VOLTAGE (KV)	SECONDARY CURRENT (MA)	POWER (hw)	SPARK RATE (SPM)	POWER EFFICIENTY (70)	CURROUT DENSITY MA/ft mA/n.2	Commerts
11/1/1	4 7 & 8 16	11 12 13 14 13 16	17 18 18 20 21 42	23 24 25 26 27	78 25 10 31 32 33	14 4 36 17 .8 31	40 41 42 43 44	45 46 47 48 49	9 79 31 32 33 34		C+ 67 69 62 C 71 77 73 74 75 76 77
8/12	41-A2	235	125	20.17	38	490	25.4		88		50% CUME
	41-A2 B1-B2	230	/33		51	7 3 5	1-1-1-1-1	5	<del>┨</del> ┼╄┲┪╌		
			<del>                                     </del>		<del> -</del> -  <del>-2 /- </del> -	╏╌┤╶┩╶╀╧╅╌┤╌	╢╌┼┼┼┼	<del>                                     </del>	╂┼┼┼┼	<del>┨</del> ┼ <del>╏╏╏</del> ┼┼┼┼	<del>╏╃╏┼╏╂┧╏╏╏</del>
8/13		23.5	125	25 8	37		34.2			<del>┠┧┼╏╏╏╏</del>	50% CUAL
19/17	71-42	230	125	<del>                                     </del>	50	70	1-149-1-	100	╂┼╌┼╌	<del>╏╌╎╶┦╶╏╸╏╸╏╸╏</del>	3070 6074
<del>                                      </del>	31-62	1230	<del>                                     </del>	<del>- - - </del> -	- <del>                                   </del>	740	╢╌┼╌┼╌┼		- <del>╿╼</del> ┼╾┼╌	<b>┃</b> ┈┤╾ <del>╏</del> ┈╏┈┤	<del>┨╏╏╏</del>
8/15	<del>                                     </del>	<del></del>	<del>╒┋</del>	1	<del>┠╒┋┋</del>	<del>╒┋</del>	┨╌┼╌┤╌┤╌┤	<del>╎┤</del> ┋	┨┼┼┼┼	┠┼┼┼┼┼┼┼┼	
8/15	A1-A2	230	125	28,8	36	7/0	25,A	280	1 89	<del>┃</del> ╾╽╾┪╼┟╌╽ <del>╌╏╸╏</del> ╌┼ <del>╌</del> ┼╌┼╌┼	50% COAC
<b>                                      </b>	9/+44	<del>                                     </del>	<u>  /₽</u> ₿	$\left  - \right  - \left  - \right  - \left  - \right $	<del>- - -\$</del> 4- -	- -	1-1-1-1-1		╢┼┼┼┼╌	<del>╿╴╎╴┤╺├┈┟╺╃╺┩╶┦┈╏</del> ┯╇	<del>┩╌┊╌</del> ┞╌ <del>┞╌┆╌┊╸</del> ╁╌┋╌╌╌
┞ <del>┤</del> ╌┤╌╏	│- ├ <b>- ├                             </b>	╺┾╅╍┾╶╄╼╧╸╽	╽ <del>┋</del> ┼┩┋╇	╎┼╁┤┼	┨┽╛┼┼┤╴	<b> </b>	┨┿╢╃	╎╌┤╌┤╌	┨╌┞╼╂═╂╌	┨╌┼╼╁┄┾╌╂╌╂╼╏╌╂╾╂╾╂╾	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
8/19	A1-42	235	123	28.7	37	615	.72. 8 36. 8	115	79	<b>╽</b> ┽╃╃┪	50% CONC
<del> - - - </del>	31-82	230	132	309	50	735	36.8		1-1-1-	<u> </u>	1-
		4-1-1-1				<u> </u>	1-1-1-1		1- - - -		<u> </u>
	- - -				- - -	<u> </u>	<u>.                                     </u>		1 - 1 - 1 -	<u> </u>	
		_ <u>                                     </u>	.			<u>                                     </u>	<u> </u>				
	<u> </u>				<u> </u>	<u>                                     </u>		<u>.                                     </u>			
						<u> </u>					
		<u> </u>						+   -			\ <del>```</del> \ <del>`</del> \``\`\`\`\`\`\`\
		7-1-1-1-1	-  <del>- </del> - - -	-   -   -   -	<b> </b> - - <del> - - -</del>  - -			-  -  -			
	-   <del>-</del>   -		- <del>  -   -   -   -   -  </del>	-   -   -   -	- - - - -		† †   † †		I i i i i i i i i		1 · -
	<del></del>	-   -   -   -   -	- - - - -		- -  - - -	- - - -	- <del> - - - -</del>	-  -  -  -  -	<del>                                     </del>	<b>┊</b>	
1++++1	<u> </u>	· - - - - -	┊ <del>╌┼┈╏╸</del> ┟╸┢╴┩	-+ +- -	<b> - - - </b> -	-  -	<b> </b> -	┊╬╬┼	┧ <del>┝</del> ╅┼┪╴	<del>┃┤╌┟╼┠═┝</del> ╼┾═┧╌┧╼┞╌┼ <del>╸</del> ┆╸	<b>\</b>
╽╌┇╌╏╌╂╌┠	╎╌┧╾╅╾┧╴┨	╁╁┼	┆ <del>╌</del> ┫╼╂═┨╴┠╌╂╌┆	<del>- - - - </del> -	<u> -    </u>	- -	<b> </b>	╽┪┿┿╌	1-1-+-+-	<b>╏</b> ╼┤═┥═╁╼┾═┧═┟═╏═┽╌╞═┊╌	
<u> </u>	<del>╶╎╌┼╌┤╌</del> ┨	┝┪╢		<del>╎╸┟┈┤═</del> ┼╌	<del>┃</del> ╌ <del>┃</del> ┼┼┼┼┤┤	<b>┊</b>	<b>╽</b> ┝╅┼┼	╽╁┽┼┽	<del>                                    </del>	<b>┨</b> ╌┤╾┝╾╅╼╄╼┥╼╂╼┆╌┼╴╟╴	[·   -   -   -   -   -   -   -   -   -
┠╁┼┼	╶ <del>╎╸</del> ┧╼┼╌ <b>╢</b>	╌╢╾╂╌╂╼╏╸	<del>┈</del> ┤╌┼╾┥╾┼╾╏╴┆	<del>│</del> <del>─</del> ┼╌├ <del>─</del> ┼╌┼╌┆	<b>├</b> ┤ <del>-</del> ├-├-┼-┤-	╽╼╁╌╏╌	<b>⋒</b> -┼-┼-┼-╏	<del>╎╸</del> ┤╶╂╾┼╼┤╴	1-1-1-1	┨╼┾═╅╍╂╍╄═╄╍┞╍╏╍┽╼╄	<u> </u>
<u> </u>	+++	<del>┈╎╸┝╸┞╸╏</del>	╼╁╼╁╼╁╼╂═	<del>╶┤╌┤╶</del> ┤╌	<b>[-</b> [- <del> - -</del>  ]	┃╌┟╼┨╼┨╼╏╺╡╼	<b>┃</b> ╌├─┼─┤─┤	- <del> - </del> - -	╽ <del>┼</del> ┼┼┼	╏╌┼╌┞╌┼╌┞╌┼╴┼╾┼╌╎	<u> </u>
<del>┃┤┤┤</del>			╎╌┤╼╀╼┤╼┼╌┼╌┼	<del>╶┤</del> ╌┤╌┤╌┤	<del>┋</del>	<b> </b>	<b>┃</b> ╶├╴├╌├╾╎━╏	<del>╶</del> ┼-┼-┤╴	1-+	┨╌┟╼┼╼┼╍┼╌┤╴	┨╬╶┼┼┼┼┼
<del>                                      </del>			17 16 19 20 21 72	23 24 25 26 27		- - - -			- - - -	25 56 57 58 59 60 61 62 63 64 65	<b>Ĭ</b> -┤ ┾╾┤╾┾╌┤ ┤╾┆╍┰╌╁╌┆╺┾╴┤╌
11 1-13	-1, 1, 1, 1, 1,		17   14   19   20   21   7"						30 31 32 33 54	35 56 57 58 53 60 61 62 63 64 65	06 67 66 69 70 71 72 73 74 75 76 77 1

UNIT 3 ESP

PATE (1977)	T-R SET	PRIMARY VILTAGE (V)	PRIMARY CURRENT (A)	PowER (Rン)	SECONDARV VOLTAÇË (KV)	SECONDARY CURRENT (MA)	POWER	SPAZK RATÜ (SPM)	Power' Efficienty (70)	CURRENT DENSITY MA/ft ma/m+	COMMENTS
1000	.,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,	0 11 11 11 11	77.01.01.01.01.7	23 24 25 26 77	20 20 70 77 32 32	4 13 3 4 4	10 11 12 13 11	15111111111			C 47 68 6. 20 71 -2 -3 -4 -2 7 72
7/27	A1-A2	270	140	378	. +   _	700	╽┾ <u>┟</u> ┼┼	20		15,0 161	
<del>                                      </del>	43-44	250	Va	27 5			- - <u> -</u>  -	1 17-1-		15,0 ,16/	
			150	45 8			┟┼╁┼┼	>200		17 1 187	
	131-52 133-34	250		250	160	200 200	12.7	125	59	10.5	
<del></del>			1-1991	<del>-  2 3  4</del> -		! !					1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
7/29	41-42	180	1+0	392	-+++	100 100 140 400		35	1 + 1	15.0 161	
	2 <del></del>	250	120	392		200				12.8 1/38	
	13-14	320	1245	76.9	++	740		185	+11	しよ。     しけっち	
	B3 C7	250	Ma	25.0	30	400	14. 9	35	58	10.3 110	
7/30	41-92	265	125	33.1		415	<b>│</b> <del>  - </del>	76		19, 2 , 15 3	
1/11/	1917-19	225	1 23	127		715	<u> </u>	<u> </u>	<u> </u>	1 8 9 1 1 2095	
	41-82	300	148	94.9		750		200		1600 125	
	83-87	235	85	20.0		375	11/6	25	158	8.0 .086	· <b>1</b> - <del>4-4-4-4-4-</del>
							- - -	<b> </b>		<b>                                     </b>	1-1-1
2/1	ALAL	230	128	79.12		670	<del>┃</del> ╌╟═╣╌┤╶	375	<u> </u>	19.3 159	1
	43-49	200	83	39.9		- 95 690 335	- -+- -	<b>╿╌┼╌┼╌</b> ┞╌┼╴┆	<b> - - -</b>	8.9 .091	
		285	1/10	39.9	<u> </u>	690		185		14.7 . 159	
- <del> -</del>  -	03-04	225	78	12.6	30	335	10	50	52	7.2 027	
	<b> </b>	+	<b> </b>	<b>╶┤</b> ╾╁╼┟╼┟╶╽			╁┼┼	<b>!</b>	<b></b>	┨╌╎╌┼╾╅╌┼╌┼╌┼╌┼	<u> </u>
8/3	AIAZ	250	75 75 728 70	27.5 15.0		350	- -  <b>-</b>  -	90	<del>       </del>	117 .126	
<del></del>	A3 A4	-200-	1			170	┃ <del>┆</del> ╏╇╢╸	┋ <del>┞╌</del> ┼╌┤╌╎	<b>}-}-∓-</b> }- -		
+	31-52	1 1212151	1-1-1-18-1	35,2	- -†- - -	300	- - - -	1 / 55 5	56	42 8 1 1 1 138	1.1.1.
	83-84	230	79	16.4	1 20	300	9.0	58	156	6.4 .069	'
	i <del>-      </del>	+	$\begin{bmatrix} + + \end{bmatrix}$	╌┼╸┼╾┤		╽╎┤├┤╌	<b>│-├-┤-┤-</b> ┃-	[-+++++]	-	<del>╿╒┋</del> ╇╇╇	<u></u> ┛┡┾╢╌┼┼╬┿┿╬╗
8/4	ALAL	250	80	30, 9		475	<b>│</b> -┤-│ <del>-</del> ┤-┤-	83	- -	14.4 155	·┃·┤·;-┆·;-┆·;-┤·┼·;-·┤·┼·;-
╽┼╎┼┼	13-44	12/01	1-180		-	975		185	- <del> -T_+- </del> -	813	
<u> </u>	11-32	280	130	36 9	1 30		9.3	1777	+= -	13 2 143	<del>、</del> ┫╌┼╌╎╴┼╌╌╌┼┼ <del>╌</del> ╌┤
	13-P7		73	14.8	20 20 30 31 32 33	3/0		. 47 4 4	55	55 30 37 30 53 00 61 02 C 0 62 C	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
<b>el 6</b> -101	L []	<u>.                                     </u>	SO 1900 8 - 3,1 300				<u> </u>	<u> </u>	<u> </u>	<u> </u>	1/4
				0.4			CORLIT	4 ES	P		//4

SUMMINARY OF GNIT 4 ESP

OPERATING DATA JULY-AUGUST 1917

CHEROKEE STATION PAULIC SERVICE OF COLORADO

D-11

P475 (1977)	T-R SET	PRIMARY VOLTAGE (V)	PRIMARY CURRENT (A)	POWER (RW)	SECONDARY VOLTAGE ( k V)	SECONOMRY CURRENT (MA)	POWER (hW)	SPARK RATE (SPM)	POWER EFFICIENCY (70)	CURRENT DENSITY MA/ft ma/n	COMMENTS
11111	1.7.1.		6 17 18 19 20 21 7	-	<del></del>	5 14 5 26 17 18 35		╌╉╼╼┰╌╌┰	•	2 - 0 52 50 50 60 61 62 65 64	<del></del>
3/6	41-12	270	1 133	35.9		6-0	<u> </u>	85	1141	13.9 114	5
3/6	1-1-4-4 -1-4	270	95				1	1+95	1+++-	1 1 1 1 1 1 1 1 1 1	<del>[                                    </del>
<del></del>	A3-A4	1-1-1-1-1	150	22.8	1-+-+ <del></del>	500 200	╌┼╌┼╌		1-1-1-1-		L
╟╫╫╫	1-1-1-1-1	3/0	- 1 1 1 1 1		1-1-1-1-1-	<del> - -  - - - - - - - - - - - - - - - - </del>	- - <del> - - -</del>	- - <del> - - - -</del>	1++++	·····································	<del>[             </del>
<del></del>	63-84	1-1-50	80	20,0			-    <u>/  2   -  </u> -	- ++	40	1 5-5 1 - pp	<del></del>
8/8	A1-A-	29:	108	26.5		505	<b> </b>	85	11111	10.8	<u> </u>
	A2 - A4	1 1 1 1 1 1		13.0	1-	280					
	A3 -A9 01-82 03-09	202	13.5	13.0 37.2 12.8		470	·┃ <del>┤</del> ╌┼╌┆	185	<u> </u>	6.0 .06	
h++++	03-09	1-13/2-	60	12.8	3/	240	7.4	45	58	3.1 05	
<b> </b> -+-+- -	17719"	1+17/1-				1-+- <del>4</del> 7-4-	1+1'   1+1	- <del> - - -</del>			
8/9	191-02	250	1/15	28.8		1111		90		10.1 -10	
	MBTAG	250	65	12.4		1 3/4			-	16.6 07	
	81-62		180	34.9		61.5		18-		1/3//     ///8/	
	63-64	260	55	1008	33	230	7.6	55	69	4.9 .05	3
8/10	A1-A2 A3-A4 B1-52	270	140	37,8		700		60		15.0 +16	2 1111111
	A3-A4	720	1   ->x	17.2	. . - -	375		1 1-		8.0 08	6
	81-52	320	150	48.0		750	1   1 - 7   1	185			2
	83-84 83-84	1 350	70	37, 8 17, 2 48, 0	36	300	10.8	35	62	6.4 106	2
	1111'			_	.   _   _   _   _   .		1-1-1-1				
8/12	A1-4-	250	120	30.0	<u>                                     </u>	675		90		19.4 .15	5   1   1   1   1   1   1   1   1   1
	A3:A4	ممر	70		111	300				6.4 06	9 : 1 1 1 1 1 1 1 1
	81-82	280	128	35.8		1 1 1 650		7200	,[ ]	13/8/1/9	49
	B1-B2 B3-B4	220	60	13, 2	34	2-70	8.2	60	62	5,1 05	
	1 1 1										
5/15	41-42	وعاتها		18.2		420	<u>                                     </u>	165		5,5 0; 9,6 ,10 12,4 ,13	2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
7,	A3-A4	225	70	19,1		450	I H L	<u> </u>		96 10	13
	A3-A4 B1-B2	225	140	37.8		450 580 460		20	+	12 4 13	3
	133-64	260	/05	27.3	3/	460	14.3	95	52	1   20   24       1   1/0	6
12 1 0 5		11 12 12 14 15 14	4 17 14 19 0 21 7/	25 24 25 24 27	70 25 10 31 14 17	3 24 25 26 37 73 46	6 40 41 42 43 44	1 13 14 17 18 18		5   57   58   19   66   61   61   61   64	65 66 67 66 69 76 71 73 77 74 77 79 71
4a 4-1G1			CPO 1860-111 300	^							2/4

SUMMARY OF UNIT 4 ESP OPERATING DATA JULY- AUGUST 1977 2/4

DATE	T-R	PRIMARY	PRIMARY		SECONDARY	SECONDA AY		SPARK	POWER	CURRE		ConnENTS
(1977)	SET	VULTAGE (V)	CURKENT	POWER (RW)	VOLTAGE (kv)	CURRENT	POWER	RATE (SPM)	EFFICIENCY		, ,	C 0 mm = 2 . 3
	17111	- (V)	(A)	13 24 25 26 27	2 13 16 11 17 13	(mA)	(hw)	<del></del>	(70)	MA/762	m4/n2	(1 67 65 64 76 71 72 71 74 75 76 77 74
8/19	A1-A2	230	125			.580		7200		12. 2		
	A3-AA		ão l	17.2		720	-		1-1-1-1		093	<del>╎╂┧</del> ┦ <del>╃┧</del> ╏┼┼┼┼
	61-67	280	135	37.8	· <del> - -<u> - </u>- -</del>  -	722	<del>▕▕</del> <del>▐</del>	137		15.0	16/	
	3:-37	Dak	88	21.6		710	13.4	-	63	7	.01	
			1-1-90-1				124		1		1111	
8/22	41-42	210	70	14.7		110		7200		9.6	.094	1 SECTION SECUNDED &.
	A3-H4	1 151.4	90	18.9		760	T   <u>-</u>   T			9.8	1005	
	61-62	270	140	18.9		1780	1111	18			133	
	63-62	270 270 250	125	18.9 38.8 31.3	33	570	13.8	105	60	12.2	•/3/	
		- - - - -	<u> </u>					_ _ _ _		_ _ _	_ _ _	
5/23	A1-4-	230	73 1	14.8		775		20		4.3		1 SENT. GROUNDED OUT
	93-99	200	125	17.0	<u> </u>	705			<u>                                     </u>	8-7	1090	
1	81-02	270	1.80	35./		620	<u>                                   </u>	185	┇╌┼╼┼╌	4.2	473	<del> - - - - - - - - - - - - - - - - - - -</del>
<del>                                     </del>	83-87	1260	145	38.2	33	605	20.p	105	44	1 /m 2	132	<del> - - - - - - - - - </del>
┠╂╂	╶┧╾┠╍┽╾╁╴╏	-  - -	- <del> - - - -</del>  -	- -[- -]	- - - -		-  <del> - -</del>  -		<b> </b>	╏╌╁╌╁╌╂╌┨╼╂		
8/29	121-A.2	220	75	16.5	- - <del> - - -</del>	350		7294	<b>I</b> -F-I	81	087	1 SECT GROUNDED WIT
	A3-17	2 00 2 7 5 2 5 5	80.	/6·5 /6·0 37./	- - - <del> -</del>  - -	410	│ <del>┈</del> ┤╌┞╌┤╌┆	185	<b>!</b> - <del> </del>	13.4	097	╽┧╼╀╀┼┼
<del></del>	83-69	275	115	37./	33				1-1-5-1	17-1-1-1	150	<del>╏┩┪┩</del> ╫╢ <del>┆</del> ┈┆┼┤ <del>┆</del> ╴┃
- <del> - - - -</del>	P3-69	- ES-	- <del> - </del> 2{\$- -	29.3		- 610	20,1	45	69	- ا ا ا	190	╽ <del>╶</del> ╁╌┩╼╏╌╏
8/25			- - - -	_ _	-			7200	╽┤╾┤╾┤	╽╏┪┪┼	1087	╽╌╬╸╁╼┧╺┟╴╂╌┦╌╸╾╍╌┼╾┤╸╌╌╍╴
-912-5-	A1-A2		8.5	20 0 24.5 35.1	- + <del> - </del> - -	330	+[++		1 -11-1-	10, \$		1 SECT, GROUNDED DUT
<del> - - - - </del> -	A3-A9 B1-B2	275	100			1 3 3 3		180		ا ایرا ایرا ا	138	
<del> - - - -</del>	83-04	235	130	29.9	32	6,20	19,2	105	64	1-16	1	
	23 07	-260-	- -4-4-	2		-   <del>            </del>			97	<del></del> └ <del>┦</del> ╾ <del>╏</del> ╌╏╌╏		
8/26	A1-A2	220	125	27.5		625		7200		13,4		
	A1-A2 A3-A4	200	70	78. a		530					122	<del>                                    </del>
	81-132	755	105	26.8		520 520	17-1	185			.120	
	B3-B9	235	110	26.8	32	550	17.6	105	68	11.7	126	
112 3 4 5	A 7 6 3 1C		17 18 18 20 21 22		24 29 36 31 32 34	14 35 76 37 18 10	46 41 4. 43 44	15 44 47 48 43	1 51 22 53 54	25 56 57 4 21 60		
8618 IG1			6/9 (960 A - 143 ten									3/4
								E . D		و ده س	_ 1	, ,

SUMMARY OF UNIT & ESP OPERATING DATA

JULY - AUGUST, 1977

D-13

. DATE (1977)	T-R SET	PRIMARY VOLTAGE (V)	PRIMARY CURRENT (A)	POWER VOLTAGE (RW) (RV)	SECONDARY CUREENT POR (MA)	GPARK RATE LW) (SPM)	POWER EFFICIENCY (%)	CURRENT DENSITY MA/ft ma/m2	COMMENTS 4
11111	1. 7. 1. 7	11 12 13 14 15 16	17 18 19 20 21 27 27	7 4 25 26 27 28 29 10 11 12 3	3 14 5 36 17 48 35 40 41	1: 43 44 45 45 47 48 49	99 31 32 33 34	72 10 57 58 59 63 61 62 63 64 65	CS 67 65 61 70 71 72 73 76 72 76 77 78 76 44
8/18	A1-A2	1 2 1 -	105	124.17	450	7200	1+1-+-		<del>╏╌╞╶┦╌┦╼╏╌╂╶╂┈╂┈╂╌╂┈╬┈┋╧</del> ╾╴│
- 425		235	95	1/2/				X.8 102	<del>┠╶┾╌╞┈╏┈╏┈╏╸╏┈╏┈╏┈╏┈╏┈</del> ┼┼╸│
++++	まんし はんし		95	24.7 + 18.5 + 47.7 - 19.6 29	·╽┈├┈┠ <del>╶╣</del> ╧┩╾┧╼╏╴┤╾┟	╌┼╌┞╌╿╼┼┯┟┉╁┾═╁┈	╂┼┟┼	18.5	<del>╏┼╏╏╏╏╏╏</del>
++++	B3-B4	305	163	19.6 29	1-1-1-1-1		40	87 613	<del>╏╏╏╏┋</del>
+++-	123 74	239		19.6 29	1-405-1-4	17 1/05	140		<del>╏╞┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋┋</del>
+++-	<b> </b>	<del>▎</del>	<del>╏╼╂╌╂╍╏</del> ╾┨╾┨╼	<del>┦┤┩┋</del> ┪┼┼┼┼	┨╼┾╍┠╾╅╺╊╍╂━┠╸┠╍┠	<del>┼┼</del> ╂ <del>┼</del> ┼┼	╂╁┦┼┼	<del>╏┩╌┠╌╏╌╏╌╏╌╬╌╬╸╏╸</del>	<del>╿╼╃╌┩╶┩╶┩╸┋╶┩</del> ╌┈
++++	┇┼┼┼┼		┠┪┵┼┼┼┼	┼┼┼┼┼	╂═┼═┼╌┆╌┦╶┼═╏╼╂═┤	<del>╶╎┈╎┈╏╌┟┈┟╸┟╺</del>	┨┼┼┼┼	<b>╿┽┽┼┼┼┼┼┼┼</b>	<del>┨┼┟╏╫┇╫╏╏╏╏</del>
++++	╉┤╾┤╾┤╾┤	╽╌╂╍╂╼╂╼┦╼┤	<b></b>	╌┟╌╂╌╂╌╂╌┨╌╏╌╂╶╂╌╂╌╏╌	┨╌├╴╏╌┼╶╁╌┠╺╂╺╉╼┽	-  - - - -	<b>!</b> -+-+-+	<b>Ⅰ-┤-┤-┤-┤-┤-┤-</b>	<del>╏╌╏╌╏╶╏╶╏╶╏</del> ╶╏
++	<del></del>	<del>╶┤╌┤</del> ╌┤ <del>╌</del> ┤╌┤╌┤	[ +-}-]	<del></del>	╂╾┾╼┝╼╟╼┟╌┼╸┠╴╏╼╏		┨┼┼┼┼	I	┠╫╫┼
	┇╬╌┼┼┼	- <del> - - - - </del> -	┃ <del>╶┤</del> ╾ <del>┤</del> ╾┤╾┤╾┤	┼┼┼┼╏┼┼┼┼	<del>┨</del> ═╁═╁═┠═╏═╂═╏	<del>╌</del> ┼┈┤╌╏╌ <del>╎╶┼╶╎</del> ╌┤╾	╢╌┼╌┼╌	┨╌┼╾┩╺╂╌┦╾╂╼╂╼╂╼╂╼┼	<b>                                     </b>
<del></del>		$\begin{bmatrix} + + + + + \end{bmatrix}$	<b> </b>	<del></del>	1-1-1-1-1-1-1	- - - - - - -	-	<del></del>	<del>┃ ╡╶</del> ┼┼┼┼┼┼┼┼┼
++++-			<b> - - - </b> - - -	<del></del>		1-1-1-1-1-1	1-1-1-	<b>   - -                                </b>	
1111		- - - - -	- - - - - -	<del>                                     </del>	┨┙╌┼┼┼┼┼	_ _ _ _		<del></del>	
444			<u> </u>	<u> </u>				<b>1</b>	
-				<u> </u>		<u> </u>	1-1-1-		
					<u>                                     </u>	<u> </u>		<u>                                     </u>	
111									
				<u> </u>	1	<u> </u>			
<del></del>	<b> </b>	- <del> - - - </del> -	- - - - - -	<del>                                      </del>	1-1-1-1-1-1-1-1				
		╽╶ <del>╽╼╏╍</del> ╁╼╂╼╂╾╏	<del>╏╶┦╾┩╍╂╼┞</del> ╾┩╴┨╺┤	·│ <del>─</del> ┌─┼─┼┈╏┈┼╴┼╌┼╌│╴	1-1-1-1-1-1-1-1	⁺┼ <b>╎</b> ┃┤╴┼ <del>╺┼</del> ╌	1-1-1-1-	<b>!</b>	
	<b>┃</b> -┼╾┼-┼-┽-┤	-	┠╼┼═┼═┼═┼╌╽╴╏╺┤	<del>╶</del> ┼╸┼╶┼╸┨╴┟ <del>┈</del> ╏═┾╼┼╶┤╶	┨ <del>╺</del> ┼╌┤╾╏╍┥╌┤ <mark>┃</mark> ╼╏╺┤	┿╽╢┼┿┼	1-1-1-1-	▋ <del>┤</del> ╌╏╼┾╼┼╴┆╶┼╌┨╶┼╌╏	
			i	++1	<b>                                     </b>	- -	<b> </b> -   -   -   -   -	┨╣╢╏╞╗╌╢╢	
	┨╌┼╌┼╌┼	<del>╶┤╌</del> ┧╾╂╾┠╌ <del>┋</del> ╌	<del>╏╌╏┈╏┈╏</del> ╌┤╾┼╾┤╌╏╌┤	<del>╎</del> <del>┊</del> ┼┼┼	┨╶┼╾╎╌┥╌┼╌╏╌╏╾┼	<del>╶┤</del> ╌╎ <del>╸</del> ┃ <del>╌╏╶╏</del> ╌┼╾┤╸	┨╾┠═┞╌┠╌╂╌	<del>╏┤╌┝┼┼┼┤</del> ┼┼	┠╸╞╼┟╼╁═╁═╏╌┦╼┑╼╸╸┠╾╃╼╍╼╍╌╸╴│
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SUMMARY OF UNITY ESP WERATING DATA - SULY AUGUST 1977 1/4

#### APPENDIX D

ELECTROSTATIC PRECIPITATOR

STACK TEST SUMMARIES

CHEROKEE STATION

PUBLIC SERVICE COMPANY OF COLORADO



# Public Service Company of Colorado

PO BOX 840 · DENVER, COLORADO 80201

October 26, 1977

QCT 2 6 1977

Mr. Irwin L. Dickstein Director, Enforcement Division U.S. Environmental Protection Agency Region VIII 1860 Lincoln Street Penver, Colorado 80203

Attention: Mr. Robert Gosik, NEIC

Dear Mr. Dickstein:

Subject: Request for Information, 42 U.S. 1857c-9(a)(ii)

Reference 8E-EL

Attached is our response to your request of September 29, 1977, for particulate emission information from outlet tests of the four Cherokee Station electrostatic precipitators. The attachment complies with your request to include stack test summaries identified by NEIC during the September 27 meeting.

These tests were performed using various methods to evaluate the amount of particulate emissions from the electrostatic precipitators on each of the four Cherokee units. Test results for unit number 2 represent particulate loadings entering the stack. Test results for units number 1, 3 and 4 represent inlet grain loadings to the scrubbers and do not represent stack particulate levels.

Sincerely.

Green, Manager Environmental Affairs and

Planning

1ik attachments

cc: Mr. William Auberle, Director Colorado Air Pollution Control Division Environmental Protection Agency Request for Information, 42 U.S. 1857c-9(a)(ii) October 26, 1977

Test Date	Flow Rate ACFM	Temp.	Grain 1 <u>Inlet</u> gr/1	Loading Outlet SCF
Precipitator -	Cherokee #1			
11/4/65	528,000	300	.647	.228
11/5/65	537,000	300	.617	.303
8/25/66	508,508	300	.3702	.2132
3/15/68	517,000	287		.220
3/16/68				.317
3/18/68	532,000	303		.482
11/18/68	519,800	289		.248
11/19/68	555,800	290		.386
11/20/68	550,700	295		.318
8/24/71	549,600	299	.877	.240
8/25/71	516,600	298	.943	.314
8/26/71	515,800	285	.582	.442
8/27/71	491,600	296	.529	.403
5/6,7/76		260		.345
Ħ		260		.333
H		260		.337

rironmental Protection Agency

luest for Information,
42 U.S. 1857c-9(a)(ii)

cober 26, 1977

1 -00er 20,	1977				
Test Date	Flow Rate	$\frac{\text{Temp.}}{o_{\text{F.}}}$	Grain I <u>Inlet</u> gr/S	<u>Outlet</u>	Comments
ccator - Che	rokee #2				
1/68	475,000			.0871	
6/7/68	470,000	277		.0855	Research Cottrell Test
/5/68	483,600	282		.150	Steam coil air heating leak during test.
/6/68	463,300	278		.178	
8/20/69	501,892	289	1.47	.0835	Research Cottrell Figures
20/69	507,000	289	1.28	.0840	PSCo Corrections to Above
/17/69			1.117	.266	Research Coatrell Tests*
12/19/69			1.1205	.2658	Research Cottrell Tests*
· 2/70	452,000	280	.673	.125	
5/5/71	473,000	270	.8152	.0248	Research Cottrell Tests
5/71	473,000	270	.974	.0408	Research Cottrell Tests
5/71	519,000	270	1.05	.0530	Research Cottrell Tests
5/6/71	519,000	-270	.990	.0623	Research Cottrell Tests
7/71	480,600	286	.888	.0390	
10/6/72	455,000	264	.733	.0220	
4/76	513,800	292.6		.0374	
4/76	518,800	295.8		.0280	

<sup>\*</sup> These tests are reported here for informational purposes only, since the accuracy of the test method used is questionable.

Environmental Protection Agency Request for Information 42 U.S. 1857c-9(a)(ii) October 26, 1977

			Grain	Loading
Test Date	Flow Rate	Temp. OF.	Inlet	<u>Outlet</u>
	ACFM	<b>-r.</b>	gr/	SCr
Precipitator - Cher				
10/27/65	590,800	292		.192
10/28/65	573,600	287		.106
10/29/65	275,000	285	.246	.163*
11/1/65	289,500	303	.214	.203*
11/2/65	289,500	296	.182	.107*
2/14/68	144,200	265.5		.172*
11	637,100			.230
11	637,100			.211
2/17/68	640,000			.113
11	640,000			.120
2/7/69	142,887	291.2		.712*
12/11/69	687,500	293		.212
8/12/70	751,700	294	.327	.331
8/14/70	648,800	303	.315	.330
8/26/70			.208	.211
10/22/70	615,000	287	.368	.179
11/4/70	615,000	265	.785	.385
4/21/71	590,500			. 237
5/20/71	641,300	295		.523**
5/24/71	611,100	292		.615**
5/25/71	631,700	296		.490**
5/27/71	633,000	291		.593**

 $<sup>\</sup>star$  One-half of precipitator tested.

 $<sup>\</sup>ensuremath{\mbox{\scriptsize **}}$  These tests were performed without gas conditioning.

Environmental Protection Agency Request for Information, 42 U.S. 1857c-9(a)(ii) October 26, 1977

Test Date	Flow Rate	$\frac{\text{Temp.}}{\mathbf{o_{F.}}}$	Grain I <u>Inlet</u> gr/S	Loading Outlet SCF			
Precipitator - Cherokee #4							
11/4/69	1,421,807	272		.2067			
11/5/69	1,442,966	271		.2542			
1/21/70	1,390,000		.309	.167			
11	1,490,000		.297	.196			
1/22/70	1,490,000		.235	.170			
1/23/70	1,520,000		.270	.124			
1/27/70	1,530,000		.223	.114			
1/29/70	1,500,000			.142			
1/29/70	1,510,000			.138			
1/26/71	1,340,000	275		.235			
1/27/71	1,226,000	280		.223			
1/28/71	1,254,000	277		.278			
9/29/71	1,484,000	269		.088			
10/5/71	1,570,000	279		.075			
10/6/71	1,506,000	275		.113			
10/8/71	1,519,000	277		.171			
11/30/71	1,490,000	278		.312*			
12/3/71	1,517,000	276		.405*			
12/15/71	1,557,000	277		.265*			
12/16/71	1,400,000	271		.332*			
12/14/72	1,407,000	268		.19			
12/19/72	1,436,000	271		.212			
11/13/73	1,325,000	244		.167			
11/15/73	1,334,000	267		.147			
11/16/73	1,425,000	265		.146			

<sup>\*</sup> These tests were performed to evaluate the precipitator performance without gas conditioning.

#### APPENDIX E

CALIBRATION OF BAILEY BOLOMETER
ON UNIT 2
CHEROKEE STATION
PUBLIC SERVICE COMPANY OF COLORADO

APPENDIX E

CALIBRATION OF BAILEY BOLOMETER ON UNIT No. 2

### A. Opacity (0) Measurements

Screen - %	Bailey Meter - %
-	6
20	24-25
40	42
60	60.5
80	78
100	99
	20 40 60 80

## B. Optical Density (0.D.)

0.D. = 
$$-\log_{10}(1-0)$$

Reading	Screens	Bailey Meter		
3	-	.027		
2	.097	.119125		
3	.222	.237		
4	.398	.403		
5	.699	.658		
6	<b>xx</b>	2.000		
15				

C. See Figure 16 for plot of data